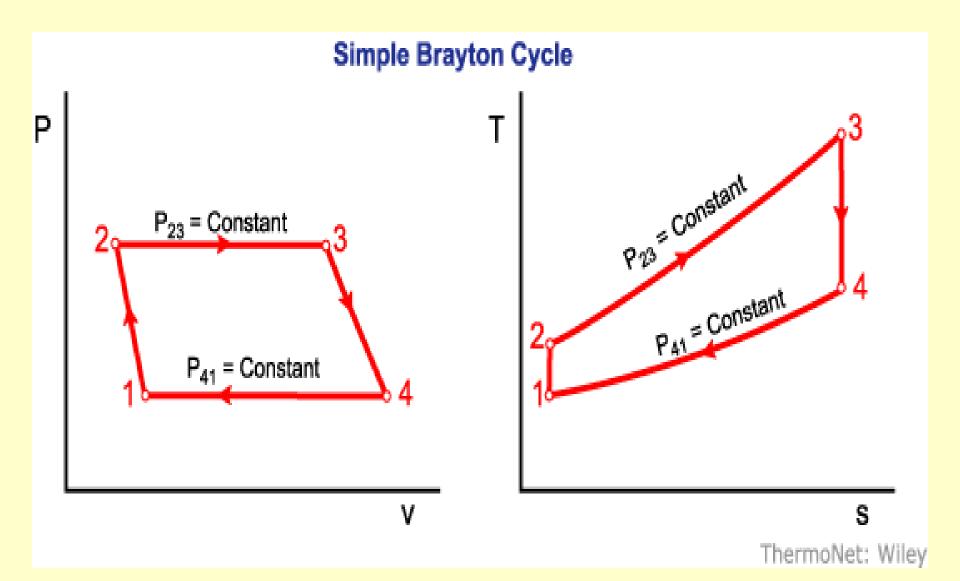
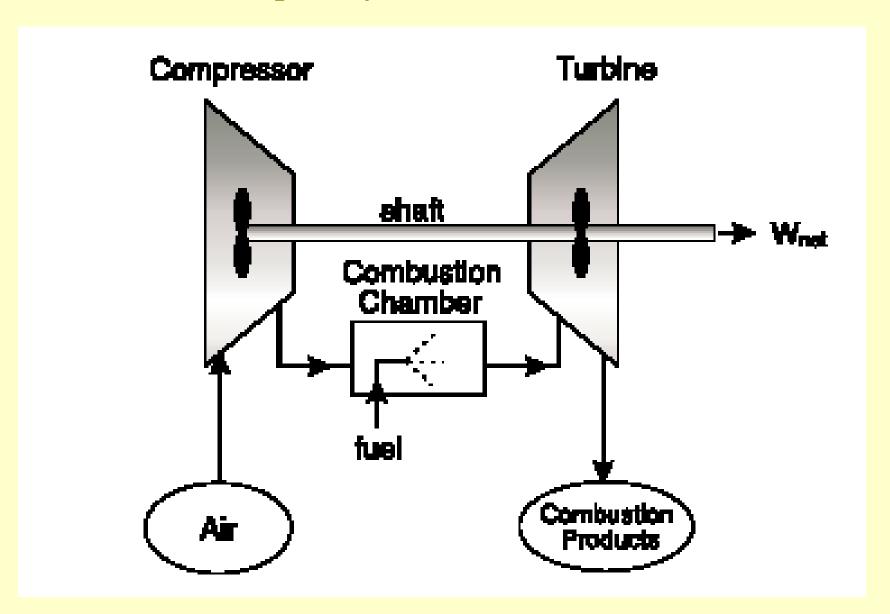


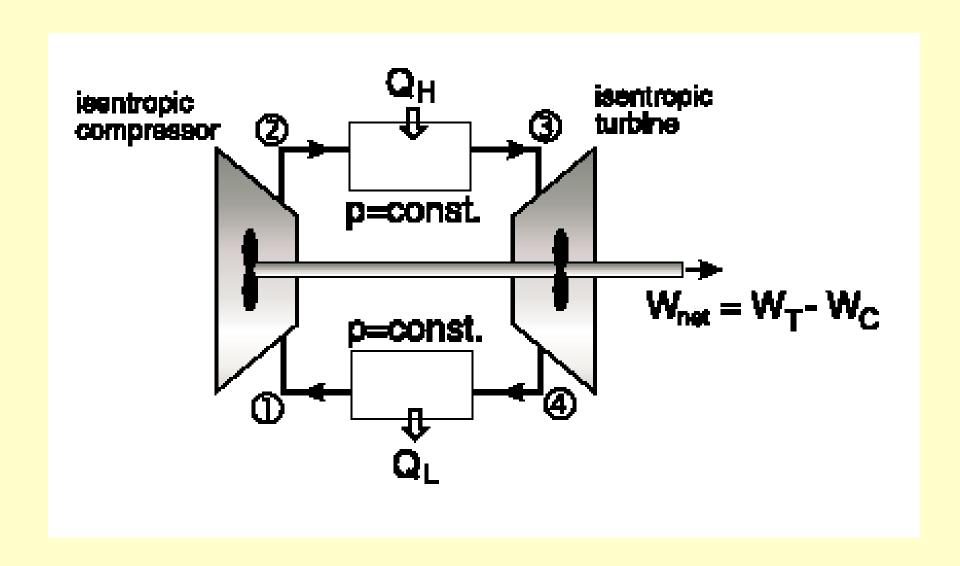
Brayton Cycle



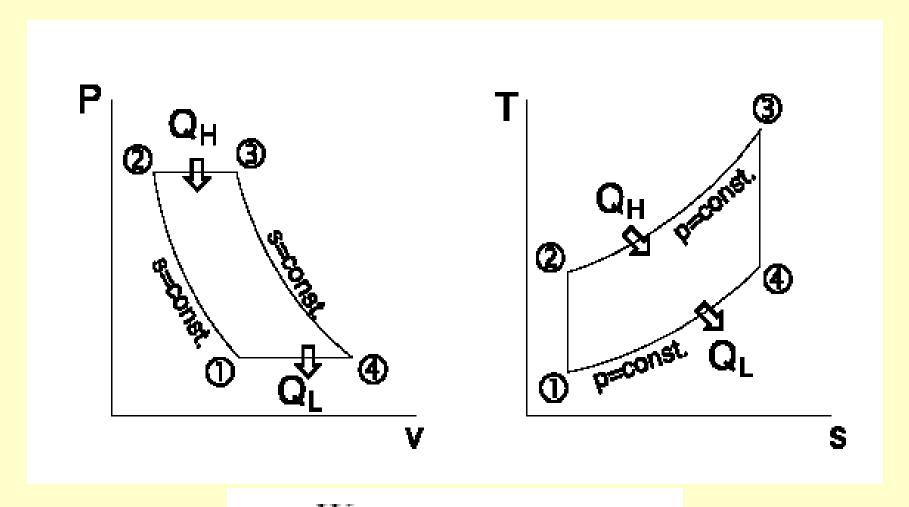
Open Cycle Gas Turbine



Idealized air standard Brayton cycle



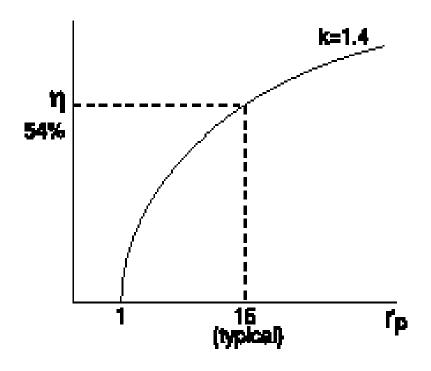
Idealized air standard Brayton cycle



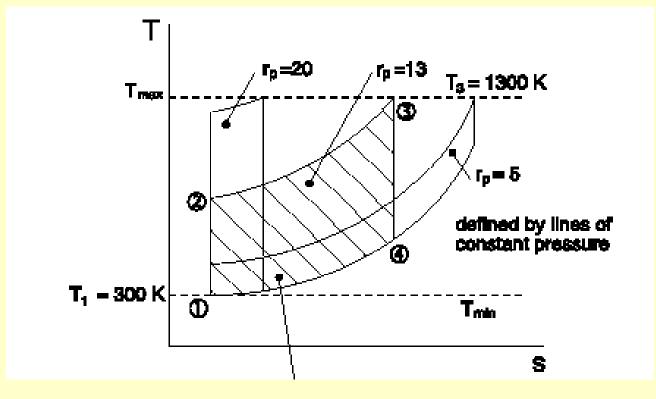
Efficiency,
$$\eta = \frac{W_{net}}{Q_H} = 1 - (r_p)^{(1-k)/k}$$

Brayton cycle efficiency

$$\eta = \frac{W_{net}}{Q_H} = 1 - (r_p)^{(1-k)/k}$$

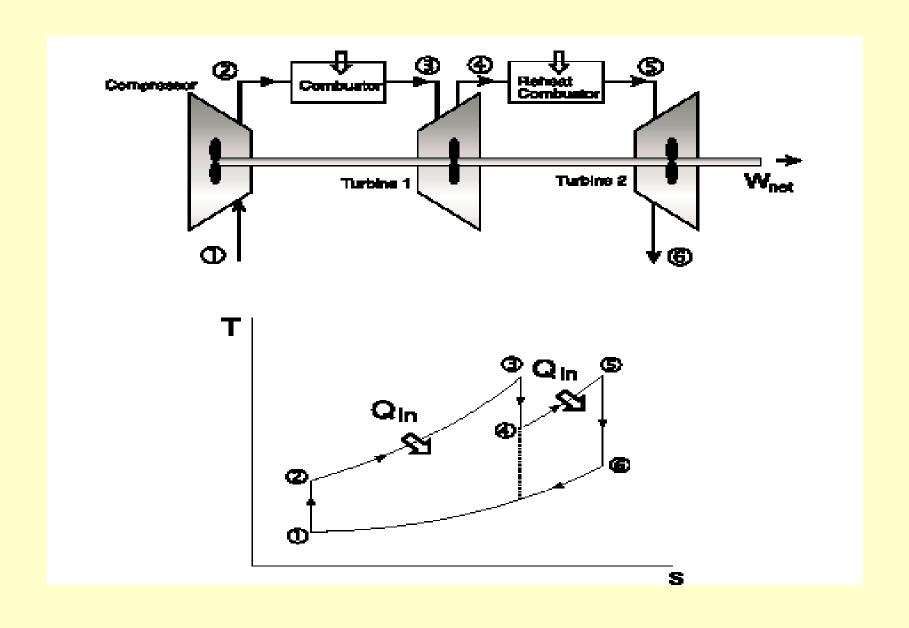


Effect of Maximum Pressure on Cycle output

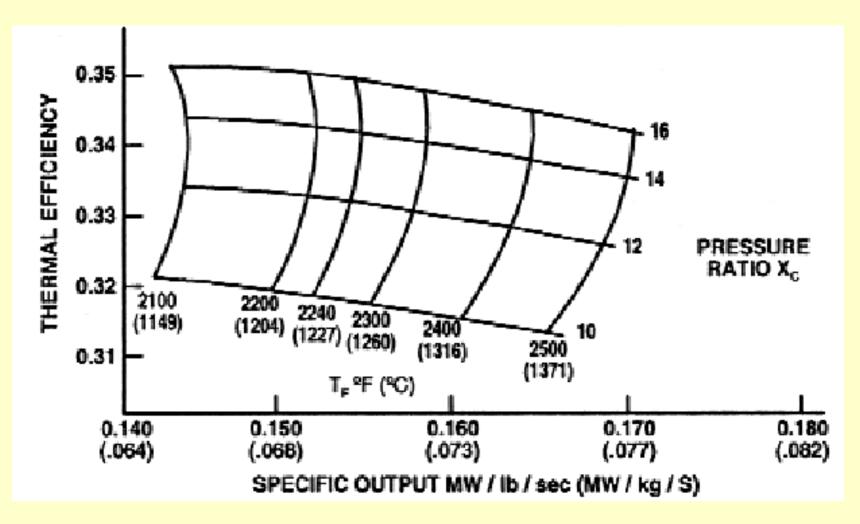


Work Net =

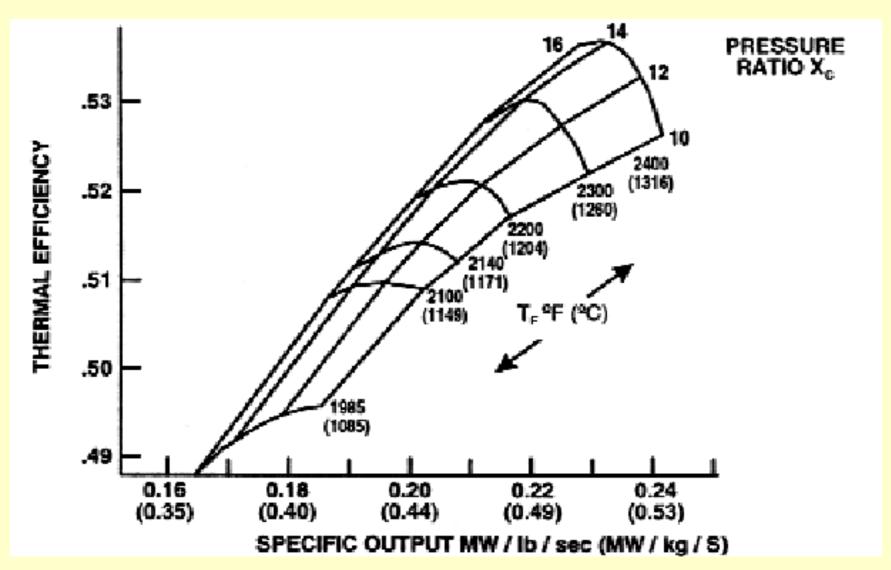
Brayton cycle with reheating



Gas turbine thermodynamics -Simple cycle



Gas turbine thermodynamics -Combined cycle



Gas turbines

- Ranges in size from ½ MW to 300 MW
- 1 MW powers 1000 homes
- Two basic types –Industrial and aero derivatives
- Can fire gas as well as liquid fuels
- Technology well proven
- Peaking resource in simple cycle

Simple cycle plants-Features

- Peaking ,load following and base load capabilities
- Typically 30+ years design life
- Capacity factor –30-85% in peaking mode
- Availability>90%
- Typical efficiency ranges 35-42 %
- Capital cost :\$360-\$440 /kw
- Operating cost based on 85 % capacity factor -\$34-\$44/kw
- Equipment lead time –6-12 months
- Project completion time –12-18 months

Simple Cycle Pros/Cons

Pros

- Clean
- Quick start-(10-12 minutes)
- Well proven technology
- Reliable
- Fast construction
- Good peaking resource
- Low o&m cost
- − Can use small site −1-10 acres
- Lower \$/kw compared to CCPP

• Cons

- Gas/fuel price
- Lower efficiencies

Gas turbine rating

- ISO rating
- ISO conditions
 - Ambient DBT –59 Deg F(15 Deg C)
 - Relative humidity –60%
 - Ambient barometric pressure 15 psi (1.03kg/cm2)
- Site rating
- Base rating Output that can be maintained for 6000 hours or more per year
- Peak rating Output that can be maintained for 2000 hours or less per year

Brayton Cycle

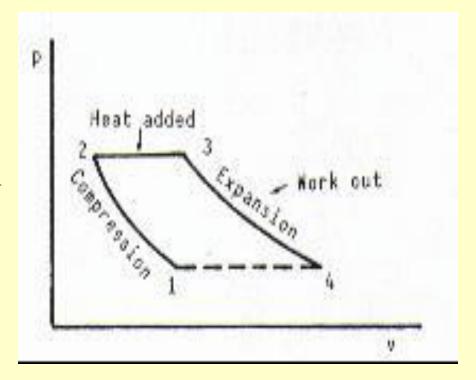
- Unlike diesels, operate on STEADY-FLOW cycle
- Open cycle, unheated engine

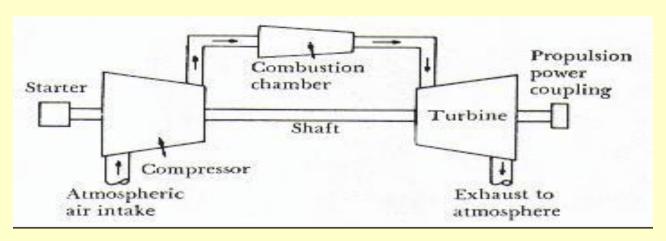
1-2: Compression

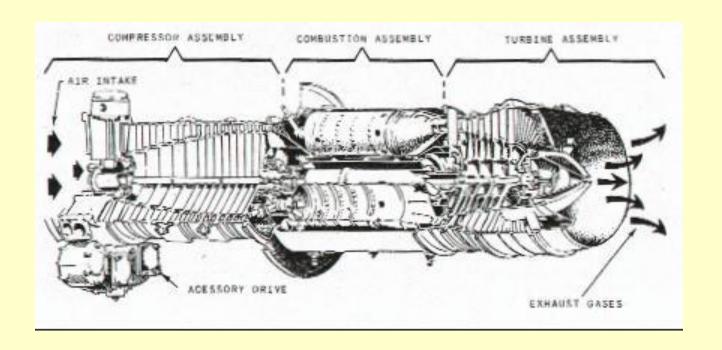
2-3: Combustion

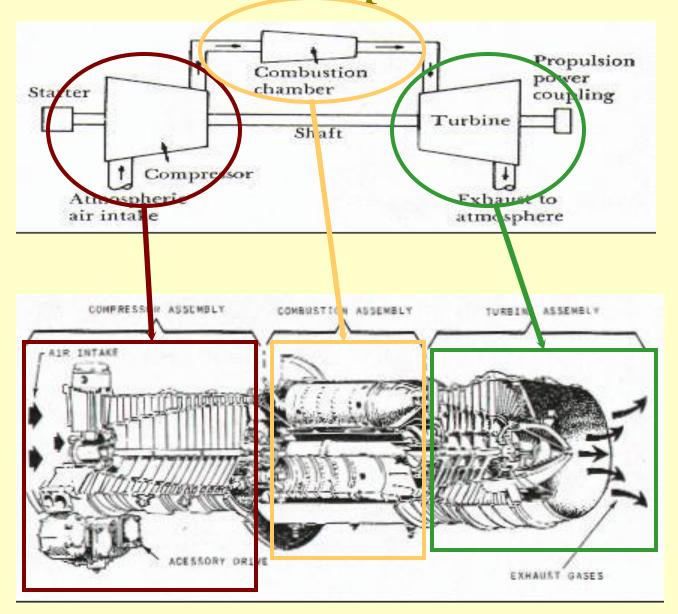
3-4: Expansion through Turbine and Exhaust Nozzle

(4-1: Atmospheric Pressure)

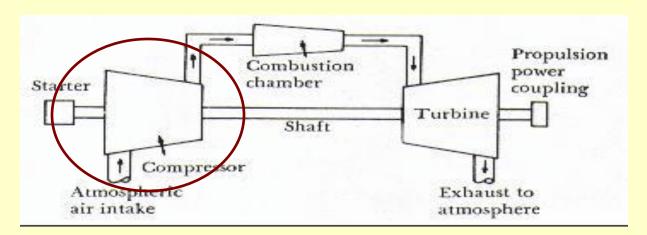




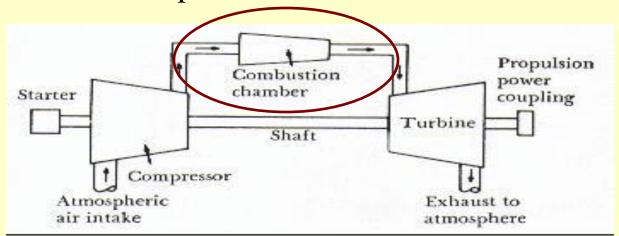




- Draws in air & compresses it
- Combustion Chamber
 - Fuel pumped in and ignited to burn with compressed air
- Turbine
 - Hot gases converted to work
 - Can drive compressor & external load



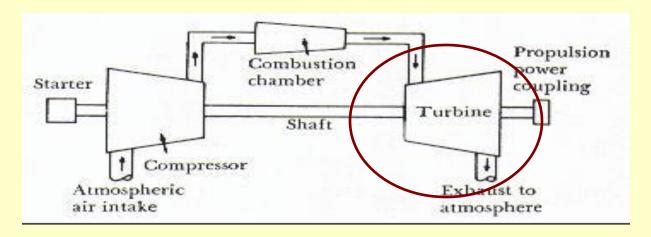
- Compressor
 - Draws in air & compresses it
- Combustion Chamber
 - Fuel pumped in and ignited to burn with compressed air
- Turbine
 - Hot gases converted to work
 - Can drive compressor & external load



- Compressor
 - Draws in air & compresses it
- Combustion Chamber
 - Fuel pumped in and ignited to burn with compressed air

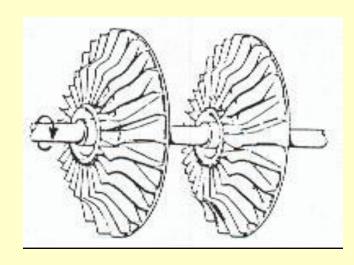
Turbine

- Hot gases converted to work
- Can drive compressor & external load

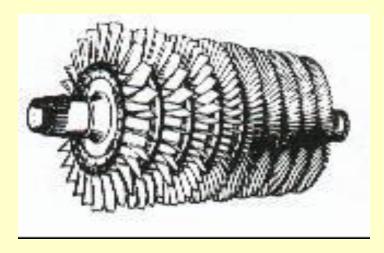


- Supplies high pressure air for combustion process
- Compressor types
 - Radial/centrifugal flow compressor
 - Axial flow compressor

- Radial/centrifugal flow
 - Adv: simple design, good for low compression ratios(5:1)
 - Disadv: Difficult to stage,
 less efficient



- Axial flow
 - Good for high compression ratios (20:1)
 - Most commonly used



- Controlling Load on Compressor
 - To ensure maximum efficiency and allow for flexibility, compressor can be split into HP & LP sections
 - Vane control: inlet vanes/nozzle angles can be varied to control air flow
- Compressor Stall
 - Interruption of air flow due to turbulence

Use of Compressed Air

- Primary Air (30%)
 - Passes directly to combustor for combustion process
- Secondary Air (65%)
 - Passes through holes in perforated inner shell & mixes with combustion gases
- Film Cooling Air (5%)
 - Insulates/cools turbine blades

Combustion Chambers

- Where air & fuel are mixed, ignited, and burned
- Spark plugs used to ignite fuel
- Types
 - Can: for small, centrifugal compressors
 - Annular: for larger, axial compressors
 - Can-annular: for really large turbines

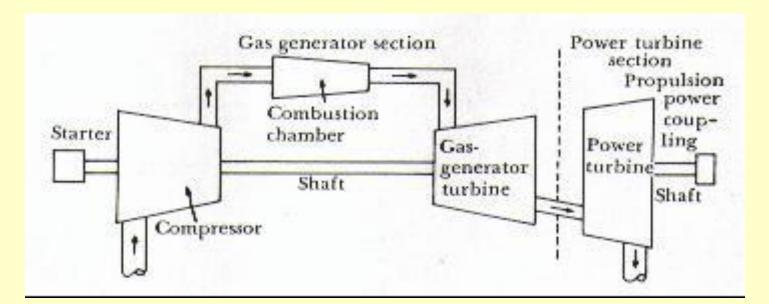
Turbines

- Consists of one or more stages designed to develop rotational energy
- Uses sets of nozzles & blades
- Single shaft
 - Power coupling on same shaft as turbine
 - Same shaft drives rotor of compressor and power components

Turbines

• Split Shaft

- Gas generator turbine drives compressor
- Power turbine separate from gas generator turbine
- Power turbine driven by exhaust from gas generator turbine
- Power turbine drives power coupling



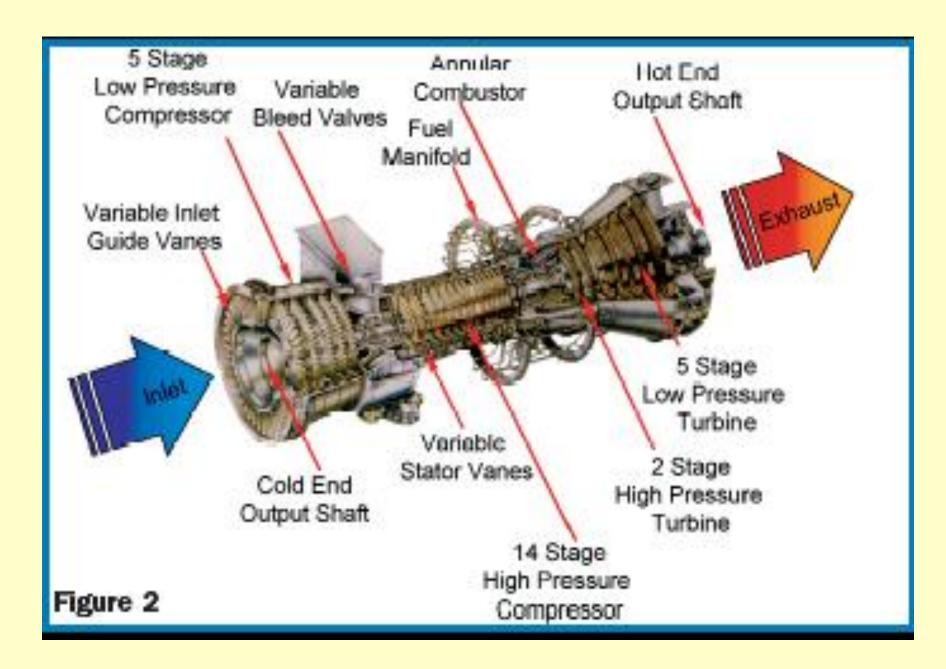
Gas Turbine Systems

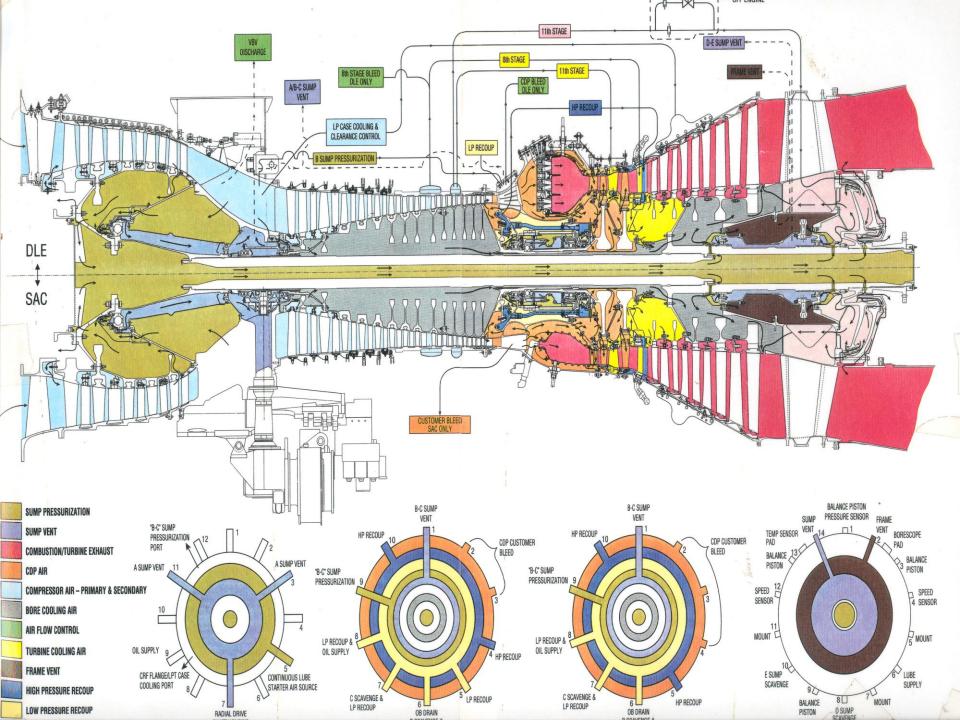
- Air System
 - Air intakes are located high up & multiple filters
 - Exhaust discharged out stacks
- Fuel System
 - Uses either DFM or JP-5
- Lubrication System
 - Supply bearings and gears with oil

Gas Turbine Systems

- Starting System
 - To get compressor initially rotated, HP air used (can use electrical also)
 - Once at certain RPM, fuel injected and spark ignited
- Power Transmission System
 - Reduction gears used to transfer torque
 - With split shaft, turbines can run @ different speeds

Aero derivative Gas turbine





GE Aero derivative gas turbine

Aeroderivative Gas Turbines													
		Output (kW)	Heat (Btu/kWh)	Rate (kJ/kWh)	Pressure Ratio	Turbine Speed (rpm)	Exhaus	st Flow (kg/sec)	Exhaus (°F)	t Temp.			
LM6000PC Sprint*		49,500	8,471	8,935	30.0:1	3,627	297	135	821	438			
50 Hz Prover Gen	LM6000PC	43,000	8,538	9,006	30.0:1	3,627	285	129	787	419			
	LM6000 PD Sprint	46,900	8,262	8,715	30.0:1	3,627	292	132	834	446			
	LM6000PD Sprint	41,700	8,366	8,824	30.0.1	3,627	279	127	838	44.8			
		-											
	LM6000PD (liquid fuel)	40,417	8,443	8,721	28.5:1	3,627	272	123	853	456			
	LM2500 PK	29,300	9,625	10,152	22.8:1	3,000	197	89	911	488			
	LM2500PV	33,500	8,920	9,409	23.0:1	6,100	197	89	907	486			
	LM2500PH	26,500	8,673	9,148	18.2:1	3,000	167	76	927	497			
	LM2500PE	22,800	10,008	10,556	18.9:1	3,000	158	72	956	513			
	LM2000	17,600	9,600	10,129	15.6:1	3,000	1.39	63	885	474			
	LM 1600 PA	13,700	9,760	10,295	20.1:1	7,900	110	50	894	478			
60 Hz Power Gen	LM6000PC Sprint*	49,500	8,451	8,914	30.0:1	3,627	300	136	826	441			
	LM6000PC	43,500	8,485	8,950	30.0:1	3,627	284	129	792	422			
	LM6000PD Sprint	46,800	8,235	8,686	30.0:1	3,627	290	132	837	447			
	LM6000PD	42,300	8,308	8,763	30.0:1	3,627	278	126	846	452			
	LM6000PD (liquid fuel)	40,212	8,415	8,878	28.1:1	3,627	268	122	857	458			
	LM2500PK	30,900	9,288	9,797	22.9:1	3,600	197	89	906	486			
	LM2500PH	27,700	8,402	8,962	17.8:1	3,600	167	76	922	494			
	LM2500PE	24,000	9,716	10,248	19.0:1	3,600	157	71	955	513			
	LM2000	17,600	9,715	10,247	15.4:1	3,600	136	62	917	492			
	LM 1600 PA	13,700	9,760	10,295	20.1:1	7,900	110	50	894	478			
Mechanical Drive		Output	Heat Rate		Pressure	Turbine Speed		st Flow	Exhaus				
		(hp)	(Btu/shp-h)		Ratio	(rpm)	(lb/sec)	(kg/sec)	(°F)	(°C)			
	LM6000PC	60,000	5,938		29.1:1	3,600	281.9	127.8	825	440			
	LM2500PK	40,044	6,702	_	22.6:1	3,000	193.5	87.7	967	519			
	LM2500PE	30,400	7,017		18.1:1	3,000	153.3	69.5	994	534			
	LM2000PA	24,300	7,117		16.0:1	3,600	141.9	64.4	894	479			
	LM 1600 PA	19,100	7,016	_	20.2:1	7,900	104.3	47.3	915	491			

GE Frame machine ratings

Gas Turbine Type	MW	Cycle	Hz	Heat Rate KJ/KWH	Efficien cy	Air Inlet Mass Flow(K G/sec	Exhaust Temperat ure °C	Pressure Ratio
9H	480	CC	50	6000	59.91			
7H	400	CC	60	6000	59.91			
9FB	412.9	CC	50	6205	57.93			
7FB	280	CC	60	6280	57.24			
9FA	390.8	CC	50	6350	56.61	624	609	15.4:1
	255.6	SC	50	9757	36.84			
7FA	262.6	CC	60	6425	55.95	432	602	15.5:1
	171.7	SC	60	9936	36.18			
9E	193.2	CC	50	6930	51.87	418	543	12.6:1
	126.1	SC	50	10653	33.74			
7EA	130.2	CC	60	7175	50.10	292	537	12.6:1
	85.4	SC	60	10991	32.71			
6FA	117.7	CC	50	6580	54.63			
	118.3	CC	60	6590	54.55			
	75.9	SC	50	10300	34.90	203	605	15.6:1
	75.9	SC	60	10330	34.80	204	604	15.7:1
6B	64.3	CC	50/60	7340	48.98	141.1	548	12.2:1
	42.1	SC	50/60	11227	32.02			
6C	62.8	CC	50	6667	53.92	117	574	19.0:1
	62.8	CC	60	6667	53.92			
	42.3	SC	50	9930	36.20			
	42.3	SC	60	9930	36.20			

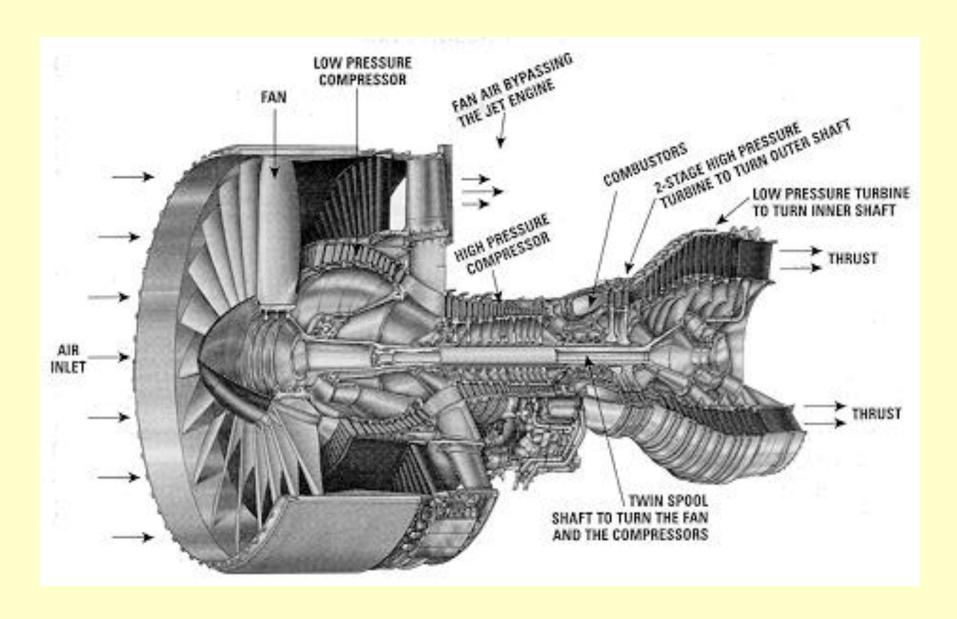
Siemens – Machine rating

Turbine Type	W501D5A	W501F	W501G	V64.3A	V94.2	V94.2A	V94.3A
Combined cycle	1+1 multi	1+1 multi	single	single	1+1 multi	single	single
Grid frequency (Hz)	60	60	60	50/60	50	50	50
Net power output (MW)	172	274	365	100	233	294	390
Net efficiency (%)	50.2	55.5	58.0	52.4	51.5	55.2	57.3
Net heat rate (KJ/KWh)	7174	6489	6204	6870	6990	6522	6283
Net heat rate (Btu/kWh)	6800	6150	5880	6512	6626	6181	5955
Combined Cycle	2+1 multi						
Combined Cycle Net power output (MW)	2+1 multi 346	2+1 multi 550	2+1 multi 730	2+1 multi 202	2+1 multi 468	2+1 multi 585	2+1 multi 780
Net power output (MW)	346	550	730	202	468	585	780
Net power output (MW) Net efficiency (%)	346 50.4	550 55.8	730 58.0	202 52.8	468 51.7	585 55.0	780 57.3

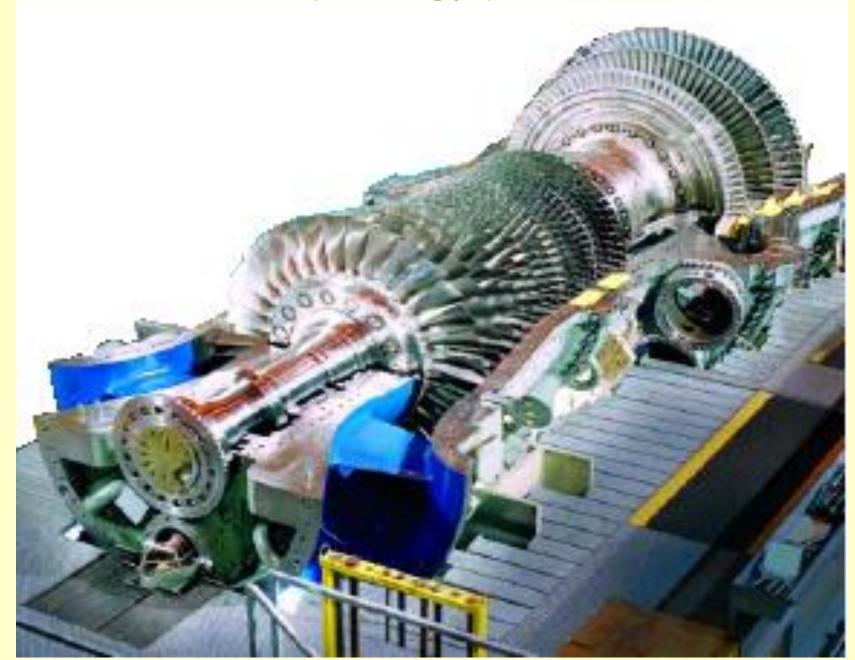
Mitsubishi Machine ratings

MODEL	Cycle	50 100 150 200 250 300 350
MF-61	50/60 Hz	6MW
MF-111	50/60 Hz	15 M W
MF-221	50/60 Hz	30MW
M251	50/60 Hz	37MW
M501D	60 Hz	114 M W
M701D	50 Hz	144M W
M501F	60 Hz	185M W
M501G	60 Hz	254MW
M701F	50 Hz	270MW
M701G	50 Hz	334MW

Jet Engine



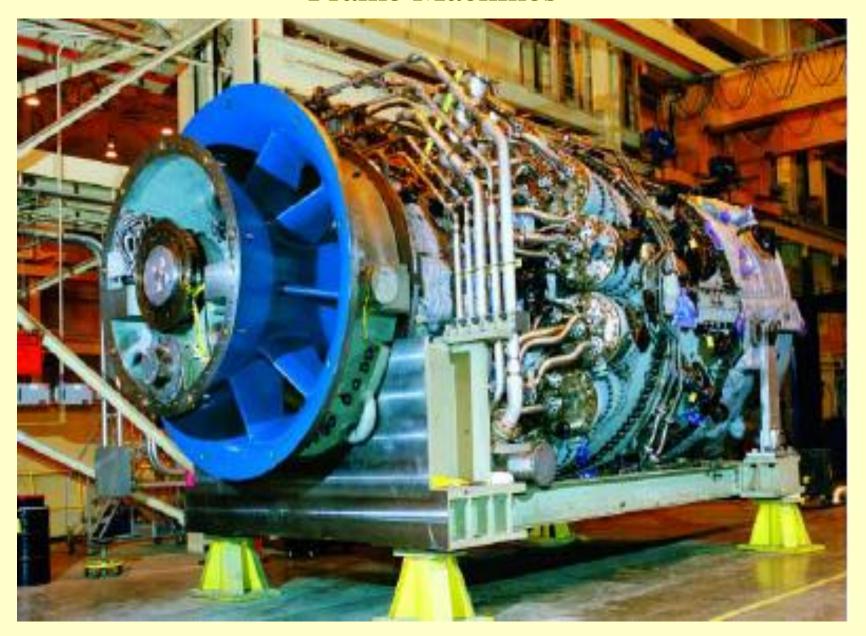
Gas Turbine(Frame type) - Cross Section



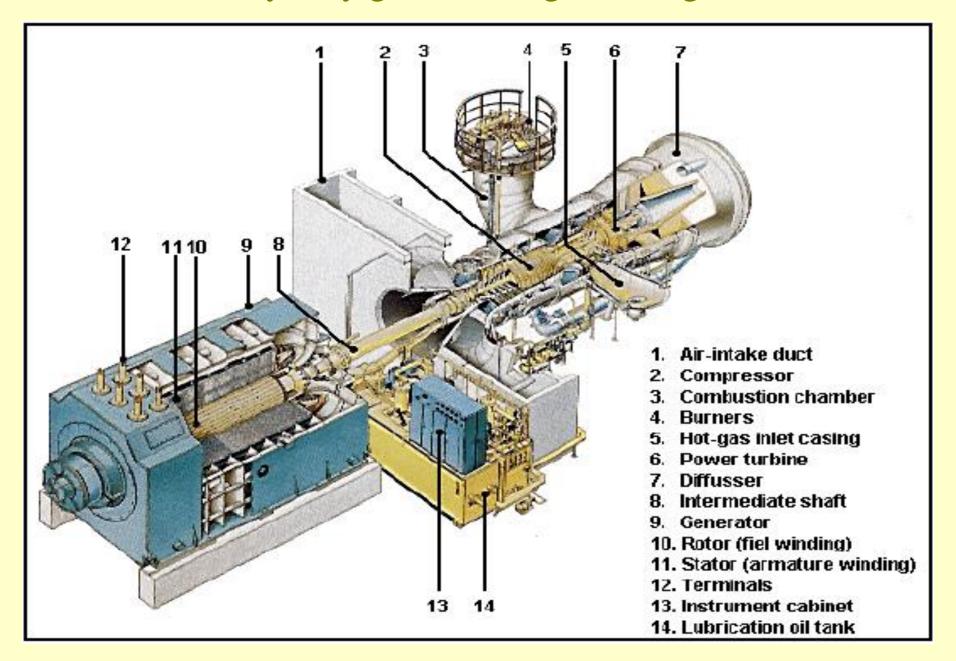
Gas Turbine Cross section



Frame Machines



Heavy duty gas turbine generating unit

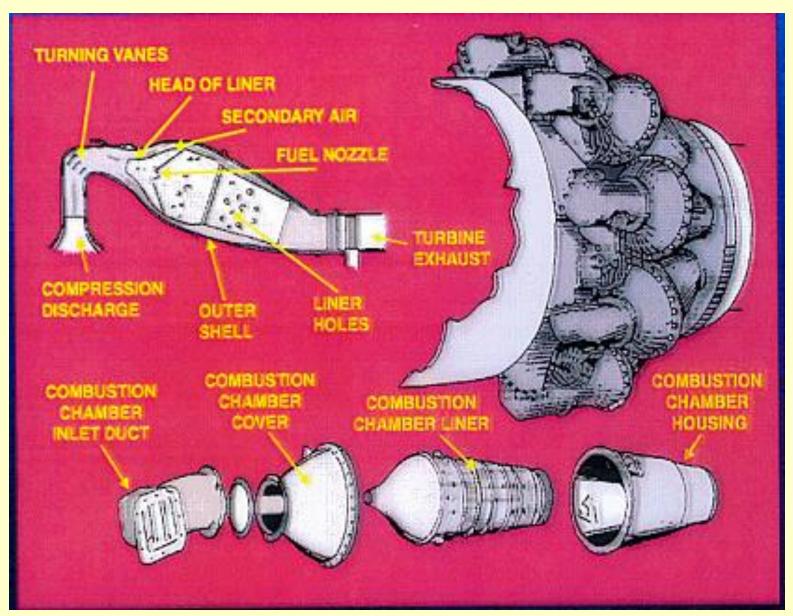


Combustion Chambers

The Basic Elements of Combustion Chamber are

- Casing
- Perforated inlet Liner
- Fuel Injection System
- Initial Ignition System
- Fuel drainage System to Drain off Unburned Fuel after Engine Shutdown

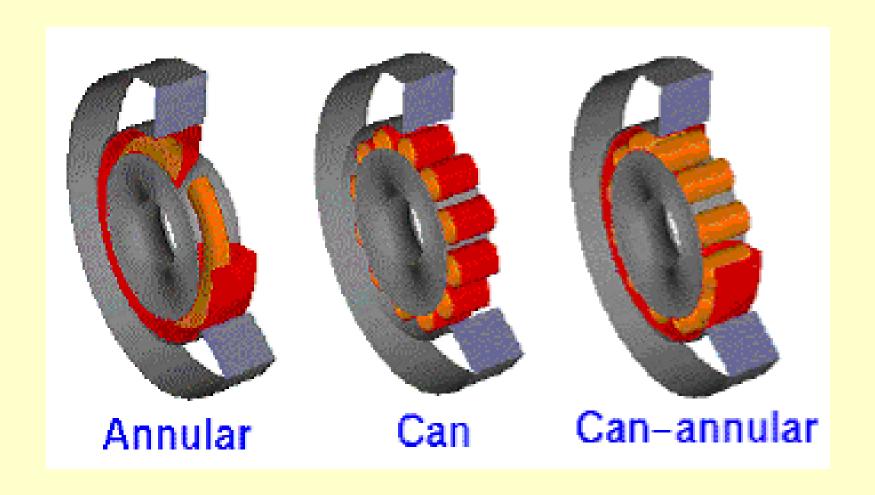
Combustion chamber



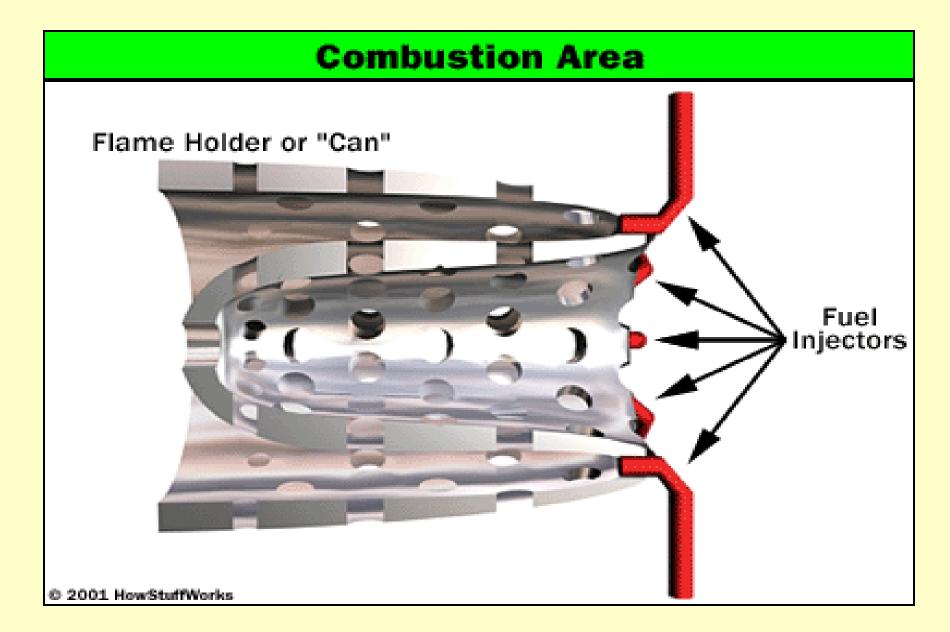
Types of Combustion Chamber

- Multiple Chamber or Can Type
- Annular or Basket Type
- Can Annular Type

Gas turbine combustor-Types



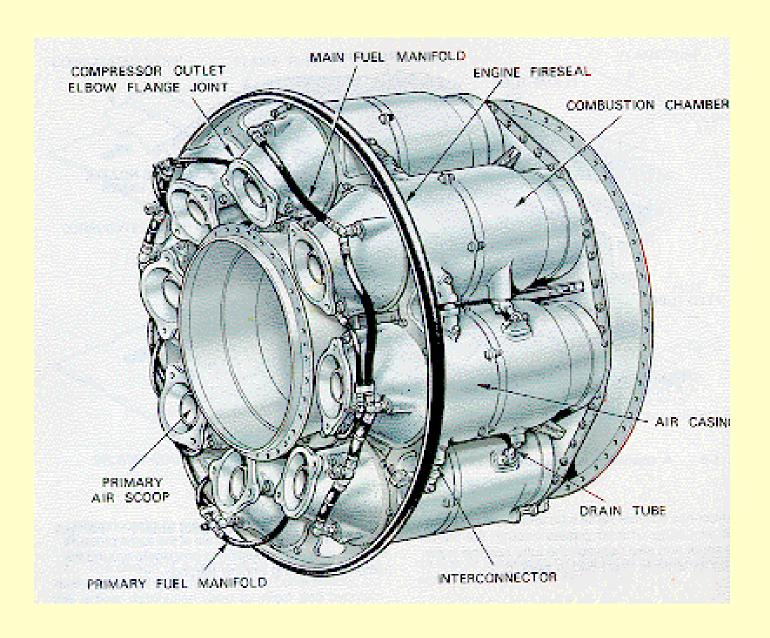
Combustion area-Can



Can Type

- Used in both axial and centrifugal flow Engines
- It is most suited for Centrifugal compressor since air gets divided equally by diffuser vanes which can be used in respective combustion chambers arranged radially around the axis of the engine.
- Usually 8 to 10 combustion chambers are provided.
- It has outer case with perforated stainless steel combustion chamber liner.
- Inter connector tubes are necessary for Flame propagation in Can type combustors
- Flame is started by two spark igniters at two of the lower chambers, it passes through tubes and ignites combustible mixture in the adjacent chamber.

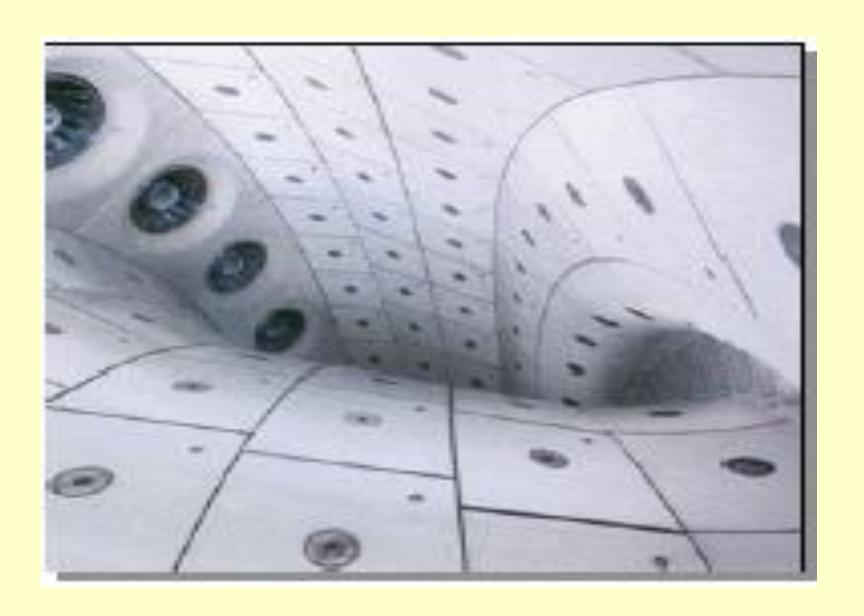
GT Combustor-Multiple Chamber



Combustor Can



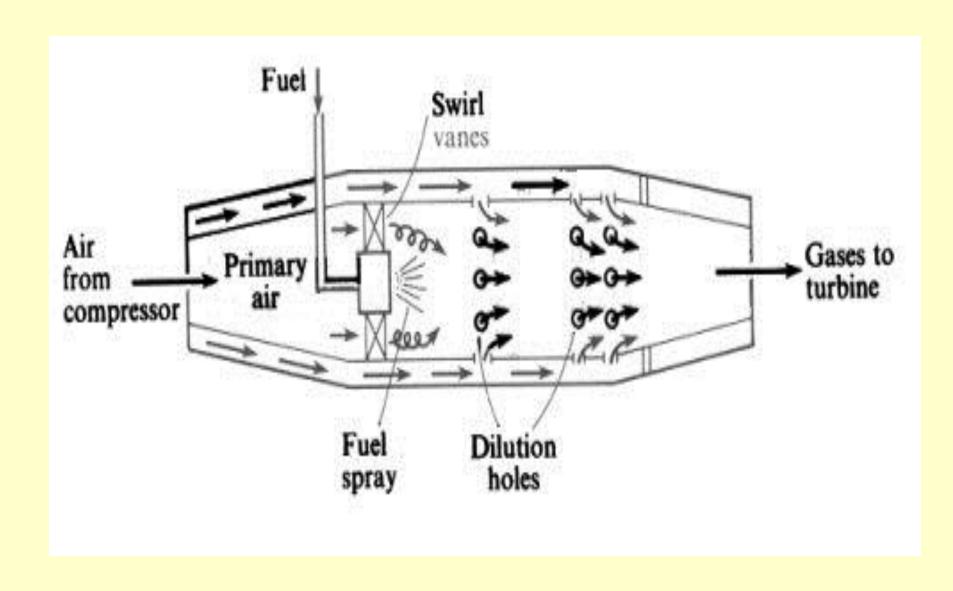
Annular combustor



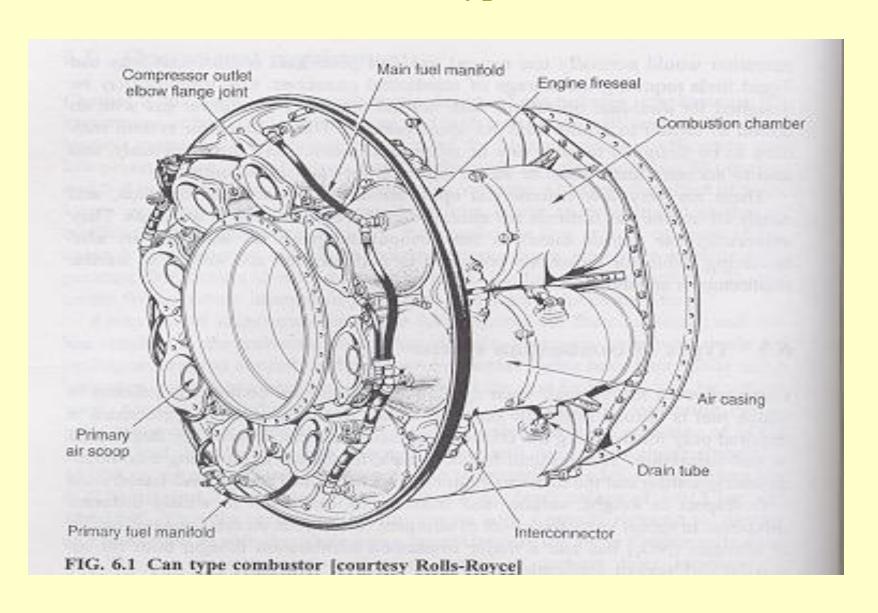
Can annular Combustor



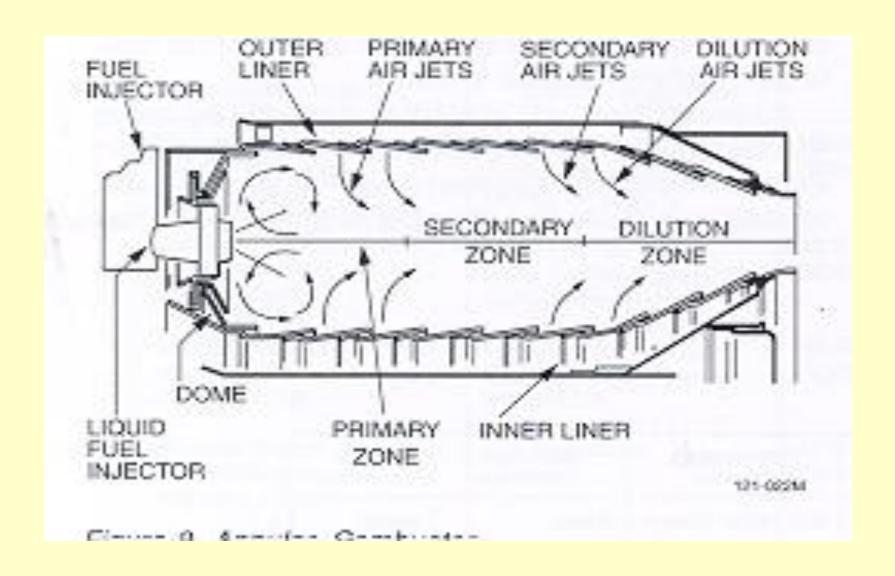
Combustor Types-Can



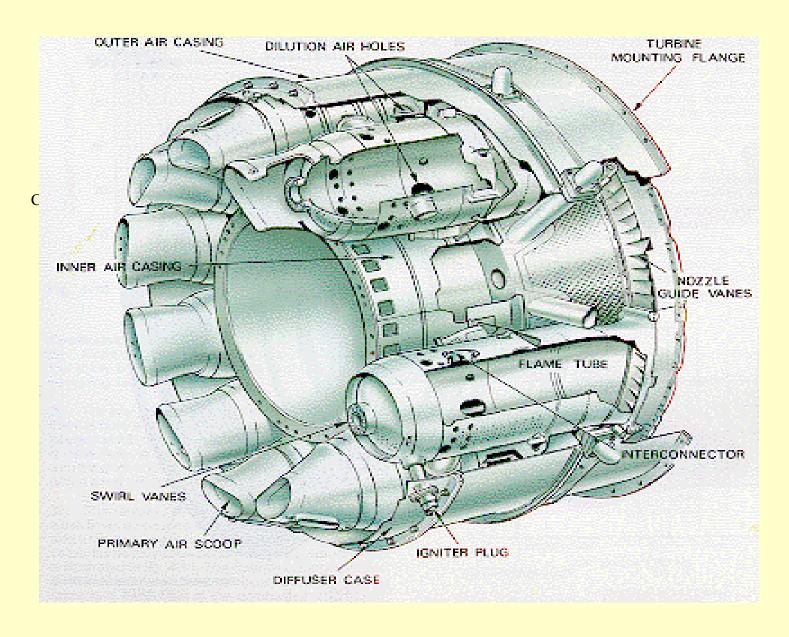
Combustor Type-Can



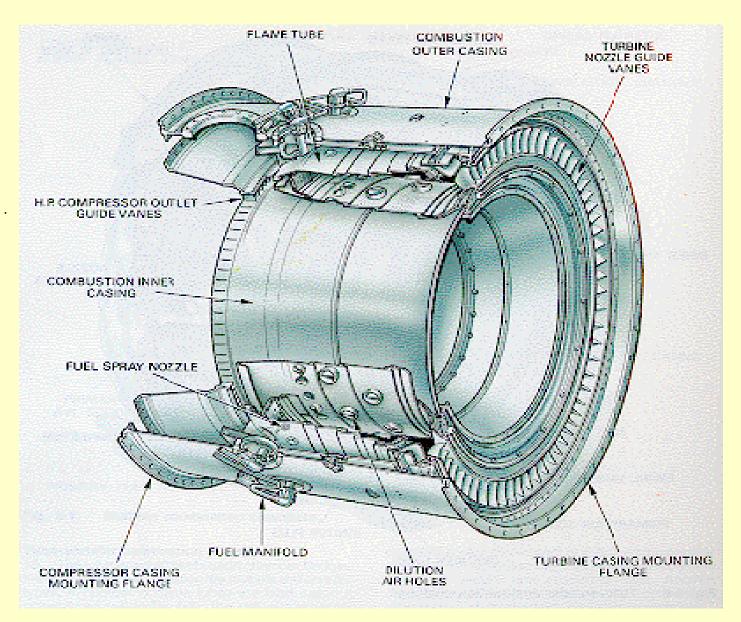
Combustor Type -Annular



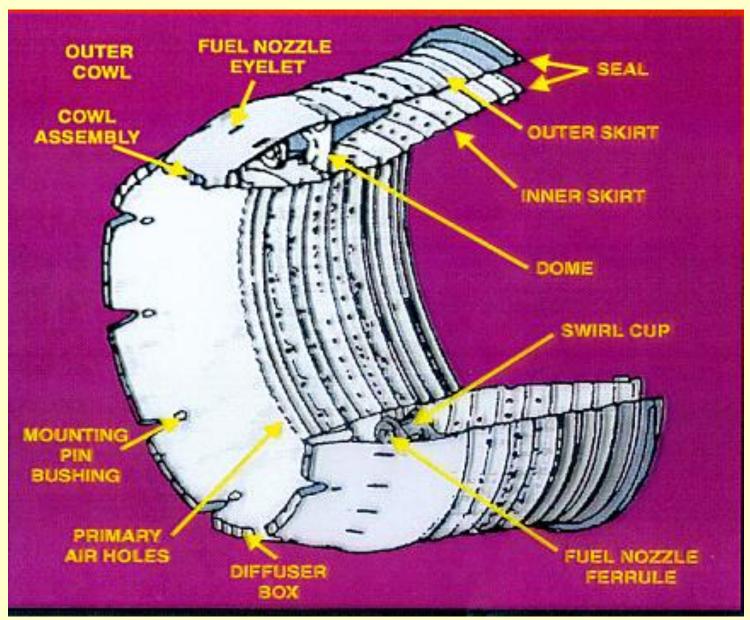
GT Combustors-Can annular



GT Combustors-Annular



Annular combustor



Annular or Basket Combustor

- Annular combustor consists of Housing and Liner
- The liner consists of an undivided circular shroud extending all the way around the outside of turbine shaft housing
- The chamber has two or more Baskets, If two or more chambers are used one is placed outside the other in the same radial plane called Double Annular combustor.
- Annular combustor is used in engines designed for axial flow

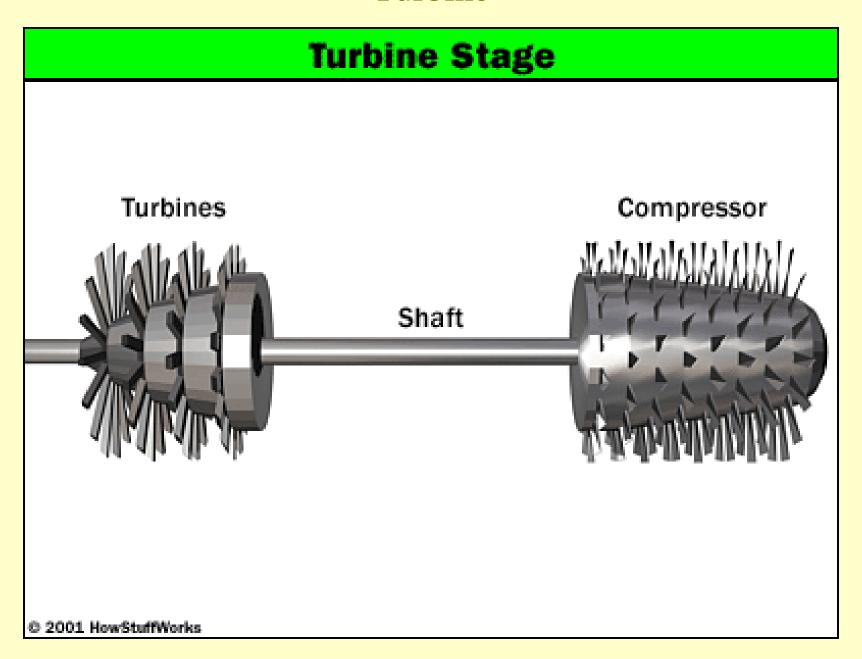
Combustor-Design requirements

- 1. High combustion efficiency at all operating conditions.
- 2. Low levels of unburned hydrocarbons and carbon monoxide, low oxides of nitrogen at high power and no visible smoke for land-based systems. (Minimized pollutants and emissions.)
- 3. Low pressure drop. Three to four percent is common.
- 4. Combustion must be stable under all operating conditions.
- 5. Consistently reliable ignition must be attained at very low temperatures, and at high altitudes (for aircraft).
- 6. Smooth combustion, with no pulsations or rough burning.

Combustor-Design requirements

- 7. A low temperature variation for good turbine life requirements.
- 8. Useful life (thousands of hours), particularly for industrial use.
- 9. Multi-fuel use. Characteristically natural gas and diesel fuel are used for industrial applications and kerosene for aircraft.
- 10. Length and diameter compatible with engine envelope (outside dimensions).
- 11. Designed for minimum cost, repair and maintenance.
- 12. Minimum weight (for aircraft applications).

Turbine



Gas turbine Blade Assembly



Gas turbine Blade Assembly



Gas Turbine Nozzle Assembly

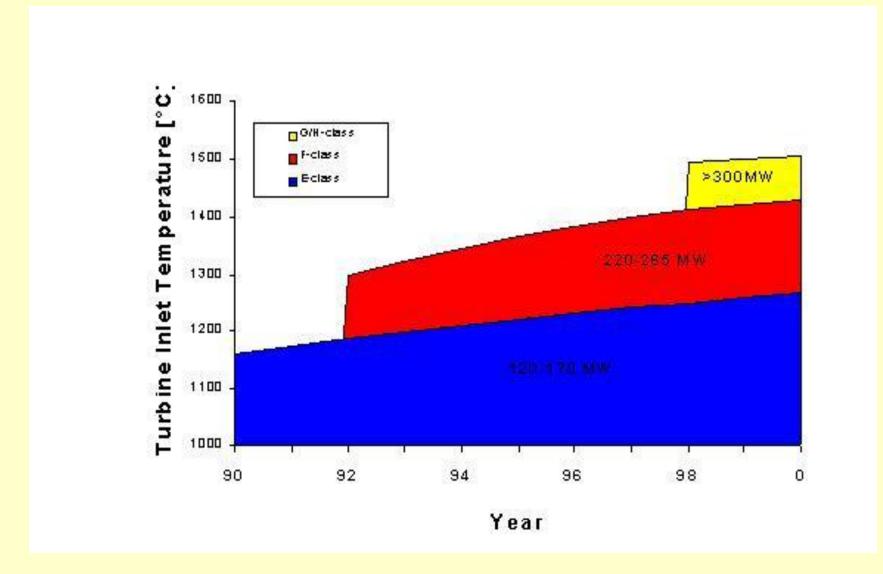


Gas turbine Blade



HPT Rotor Assembly





•

Gas turbine Blade Assembly



Gas turbine Blade Assembly



Gas Turbine Nozzle Assembly



Gas turbine Blade



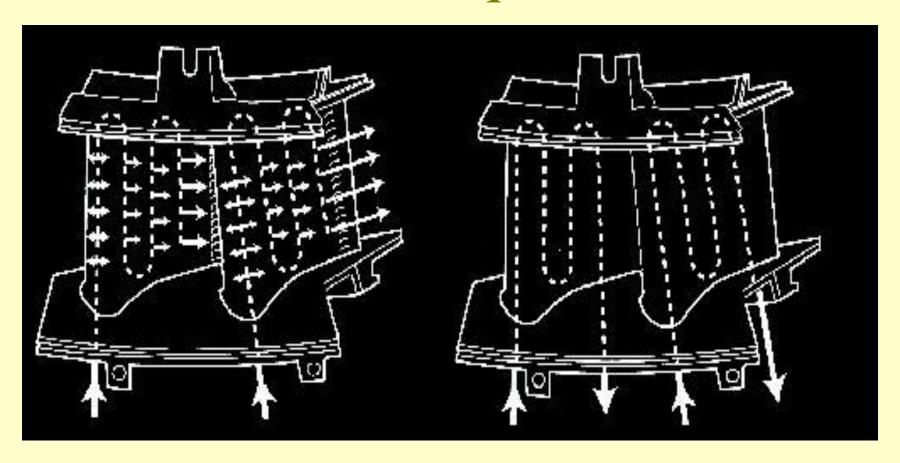
HPT Rotor Assembly



Turbine Blade Cooling



Gas turbine vane cooling techniques



Open loop air cooled

Vane 1 dT gas = 155° C

Closed loop steam cooled

Vane 1 dTgas = 44° C

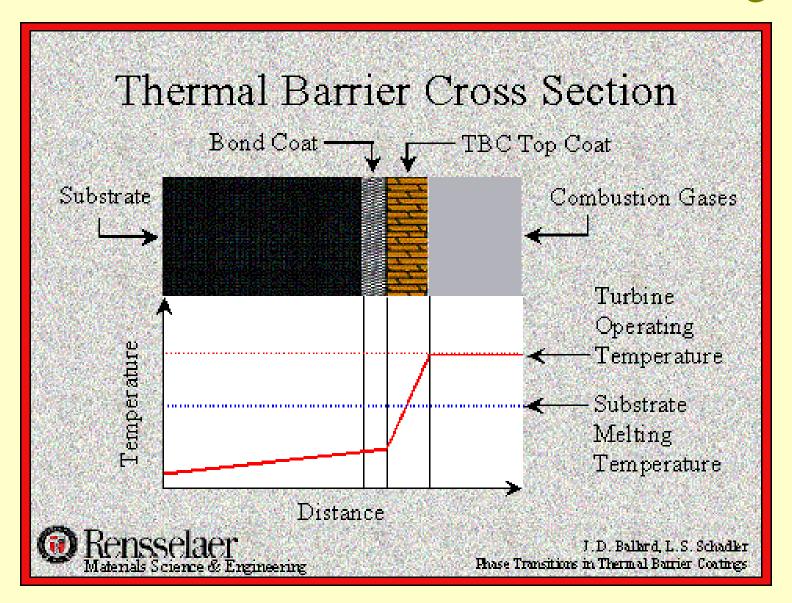
Gas turbine blading materials

- Turbine blading is subjected to high temperature /stress
- Materials –Predominantly Nickel-Cobalt alloys
- Actual metal temperatures in excess of 80% of melting temperatures
- Lowest metal melting temperature-1300 Deg c
- Nickel alloys –Highest combination of stress/temperature durability
- Cobalt –Higher temperature capability,improved hot corrosion ,thermal fatigue resistance

Turbine Blade Coatings

- To combat hot corrosion and oxidation
- Principle coating groups
 - 1.Aluminium
 - 2. Aluminium with additives such as Chromium or Silicone
 - 3. Platinum Aluminides
 - 4.MCrAlY –M –Nickel/Cobalt/Iron
 - 5.MCrAlY+Thermal Barrier Ceramic
 - 6.Ceramic -Zirconia

Illustration of a thermal barrier coating



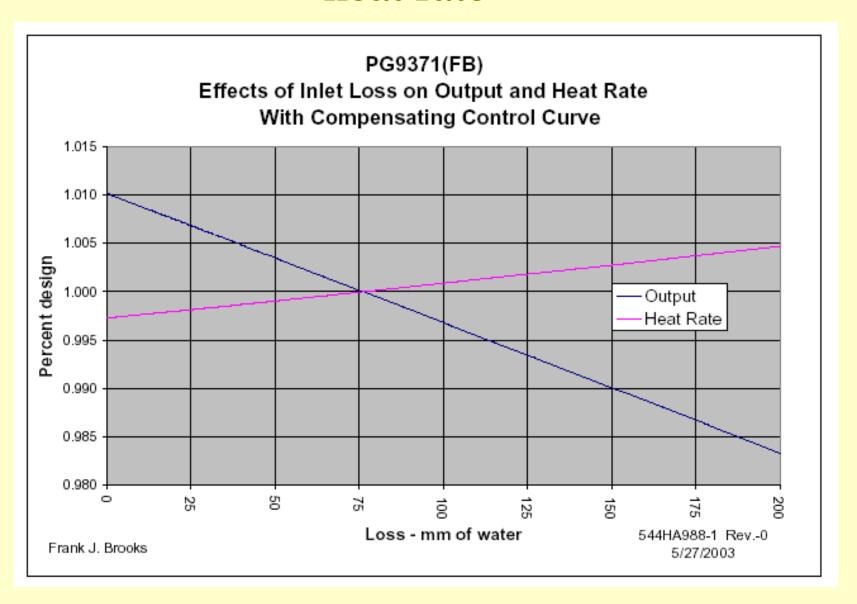
Factors affecting gas turbine output

- The energy used by the air compressor if less energy is used to compress the air, more energy is available at the output shaft;
- The temperature of the hot gas leaving the combustors increased temperature generally results in increased power output;
- The temperature of the exhaust gas reduced temperature generally results in increased power output;

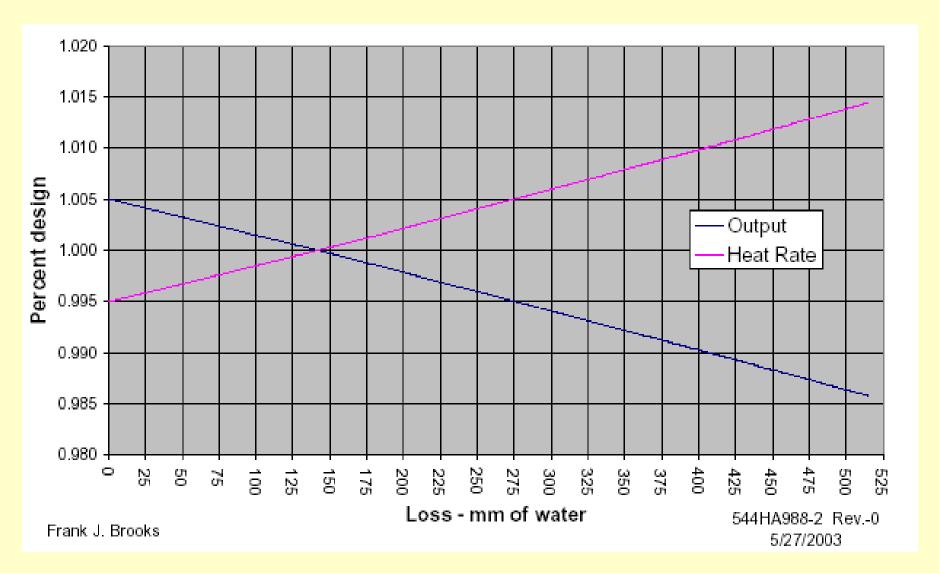
Factors affecting gas turbine output

- The mass flow through the gas turbine in general, higher mass flows result in higher power output;
- The drop in pressure across the inlet air filters, silencers and ducts a decrease in pressure loss increases power output;
- The drop in pressure across the exhaust gas silencers, ducts and stack a decrease in pressure loss increases power output; and
- Increasing the pressure of the air entering or leaving the compressor an increase in pressure increases power output.

Effect of inlet losses on GT output and heat rate



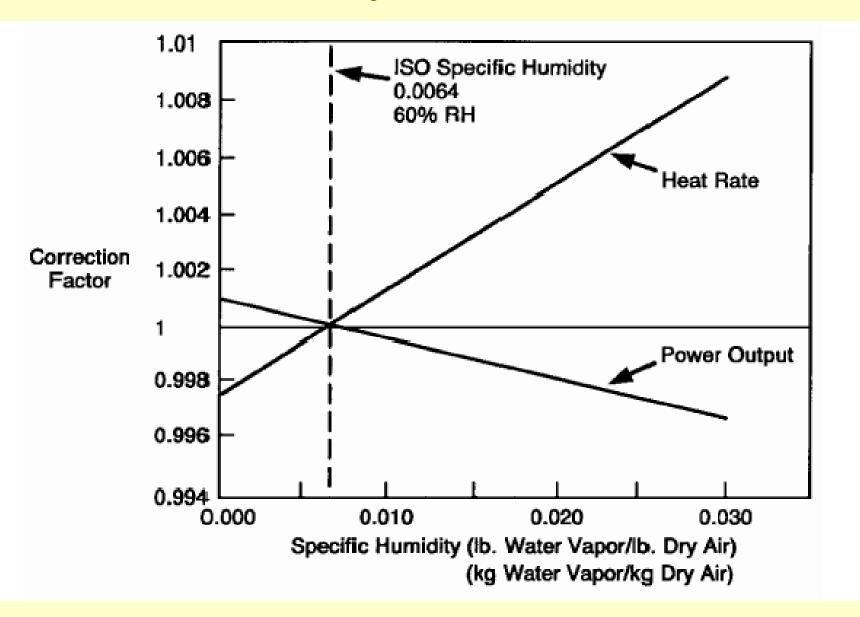
Effect of exhaust losses on GT output and heat rate



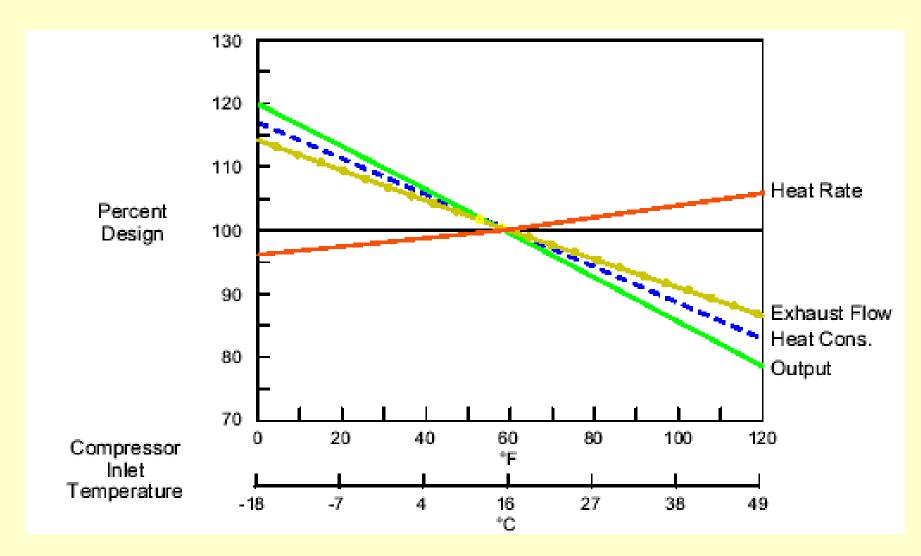
Pressure drop effect on GT

- 4 Inches (10 mbar) H2O Inlet Drop Produces:
 - 1.42% Power Output Loss
 - 0.45% Heat Rate Increase
 - 1.9 F (1.1 C) Exhaust Temperature Increase
- 4 Inches (10 mbar) H2O Exhaust Drop Produces:
 - 0.42% Power Output Loss
 - 0.42% Heat Rate Increase
 - 1.9 F (1.1 C) Exhaust Temperature Increase

Humidity effect curves

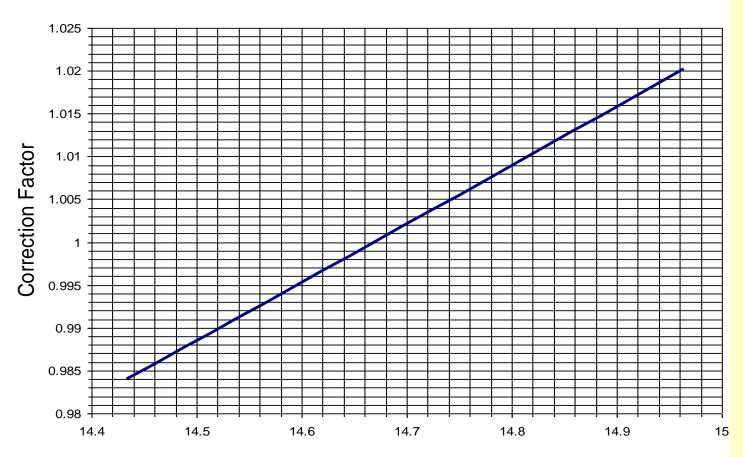


Effect of ambient temperature on GT Performance



Ambient Pressure Vs GT Output





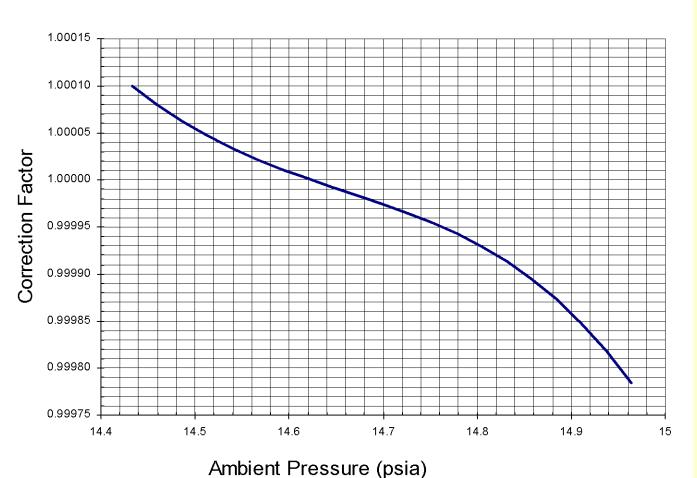
Ambient Pressure (psia)

Basis of guar 14.67 psia

Do not use for other site elevations - corrections valid for named site only.

Ambient Pressure Vs GT Heat rate





Basis of guar 14.67 psia

Do not use for other site elevations - corrections valid for named site only.

Factors influencing Gas turbine thermal efficiency

- The energy used by the air compressor if less energy is used to compress the air, more energy is available at the output shaft;
- The temperature of the hot gas leaving the combustors increased temperature generally results in increased efficiency;
- The temperature of the exhaust gas reduced temperature generally results in increased efficiency;

Factors influencing Gas turbine thermal efficiency

- The mass flow through the gas turbine in general, higher mass flows result in higher efficiencies;
- The drop in pressure across the inlet air filters, silencers and ducts a decrease in pressure loss increases efficiency;
- The drop in pressure across the exhaust gas silencers, ducts and stack a decrease in pressure loss increases efficiency.

Factors influencing gas turbine reliability

- Design of the components
- Selection of materials used in critical components
- Operational factors such as the cleanliness of the fuel and inlet air
- The way gas turbine is operates
- Quality of maintenance factors

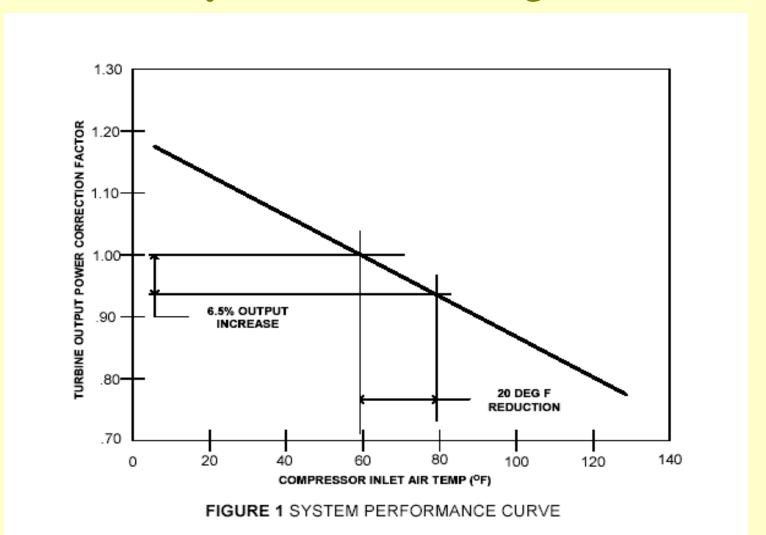
Augmenting GT output-Options

- Using the exhaust gas to heat the air from the compressor (mainly used in cold weather conditions);
- Divide the compressor into two parts and cool the air between the two parts;
- Divide the turbine into two parts and reheat the gas between the two parts by passing the gas through additional burners and combustors located between the two parts;
- Cooling the inlet air mainly used in hot weather conditions;

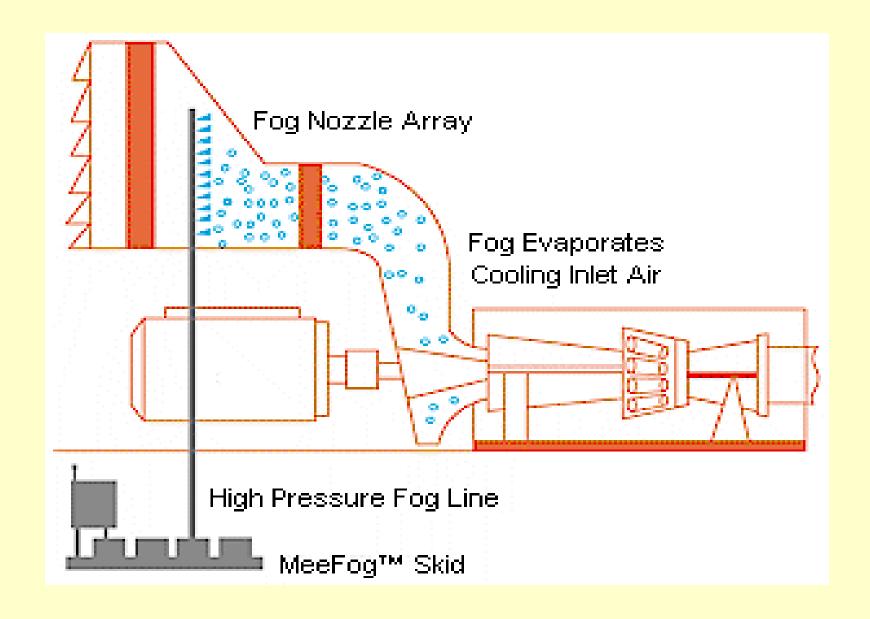
Augmenting GT output-Options

- Reducing the humidity of the inlet air;
- Increasing the pressure of the air at the discharge of the air compressor;
- Inject steam or water into the combustors or turbine;
- Wash or otherwise clean the fouling from the blades of the air compressor and turbine at regular intervals; and
- Combinations of the above methods.

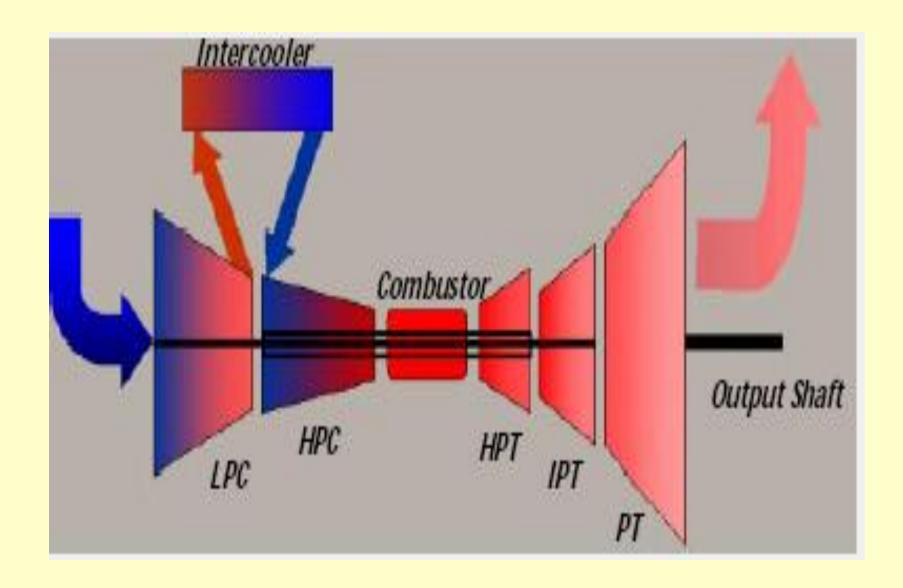
Gas turbine Performance Enhancement by Inlet air cooling



GT Output enhancement –Inlet Fogging



LMS 100



LMS 100 ISO Performance Data

Simple Cycle Gas Turbine 50Hz Applications

Model	Output	Heat Rate	Efficiency
	(MWe)	(KJ/KWH)	%
DLE	99	7921	45
SAC (w/Water)	102.5	8247	44
SAC (w/Steam)	102.2	7603	47
STIG	110.8	7263	50

Conditions:

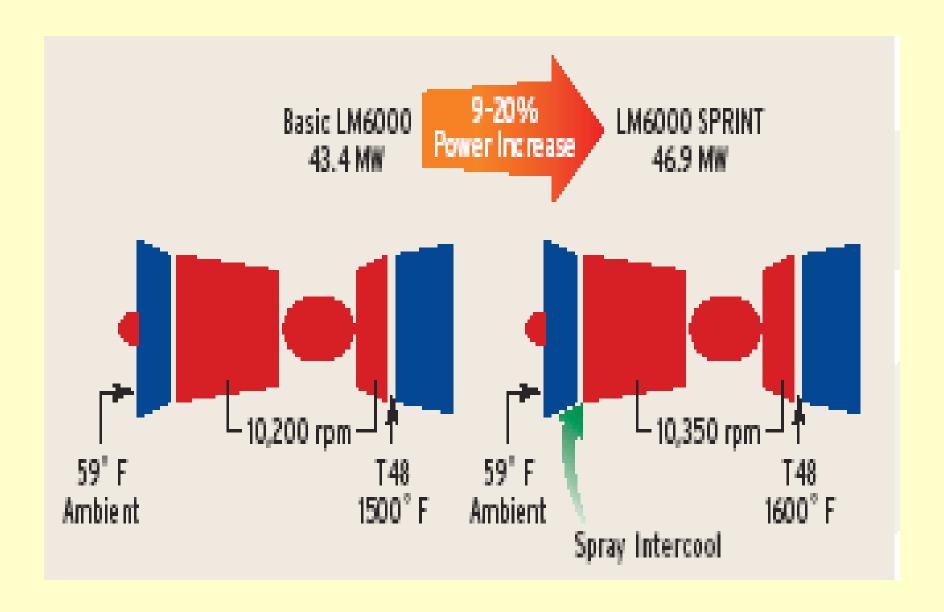
Performance at the generator terminals

NOx = 25 ppm

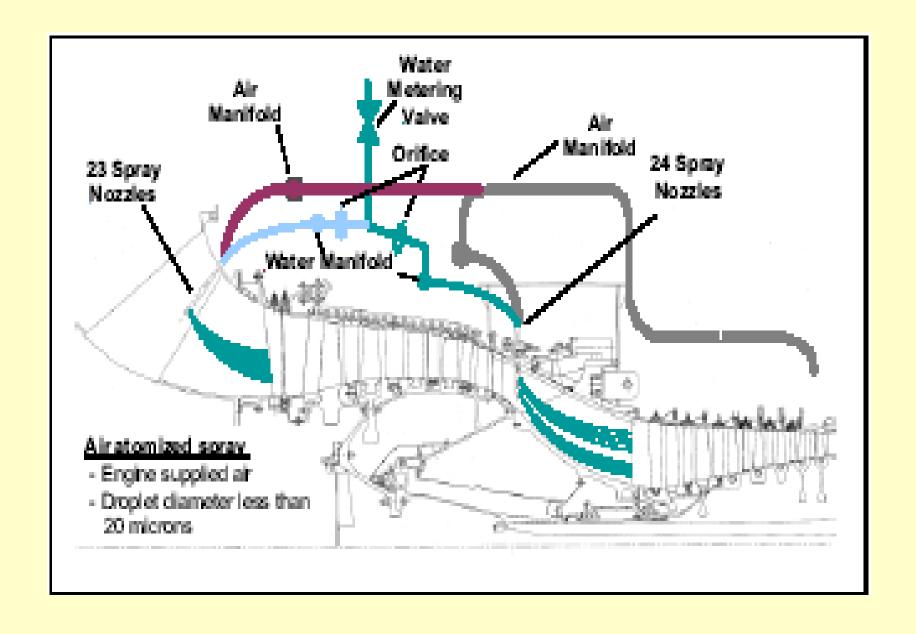
15°C, 60% Relative Humidity

Losses: 0mm/0mm inlet/exhaust Fuel: Spec Gas (LHV = 44.2MJ/KG)

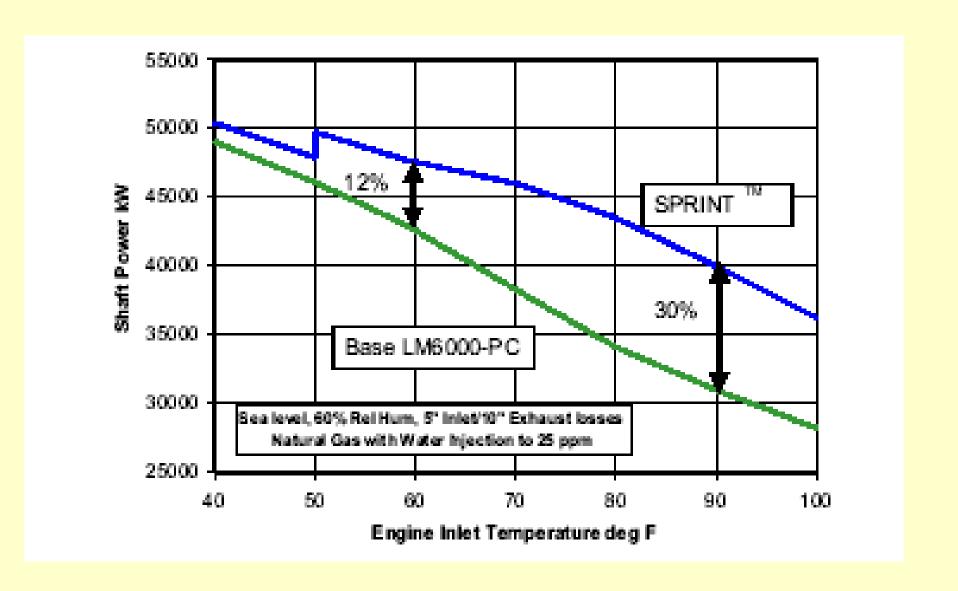
GT Output enhancement -SPRINT



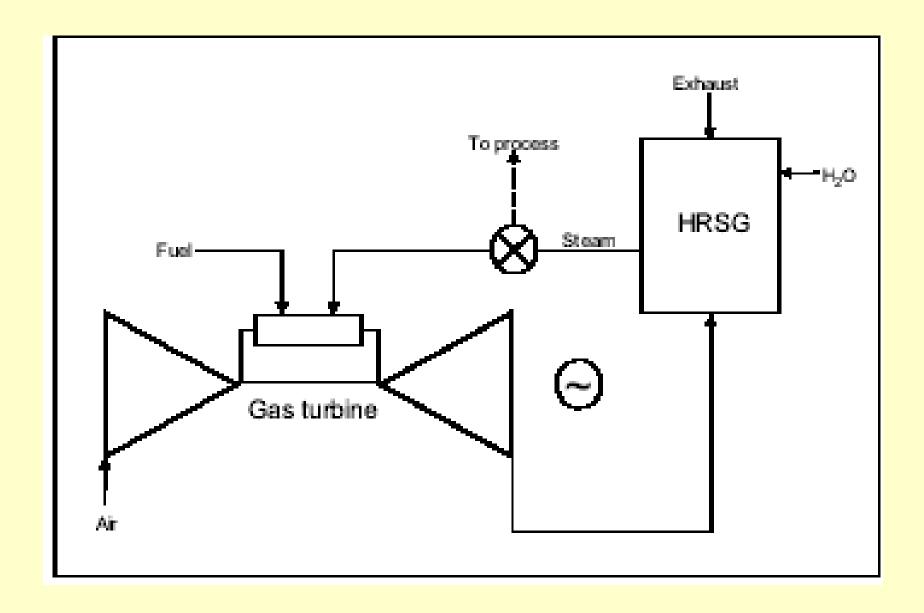
LM6000PC SPRINT Cross Section



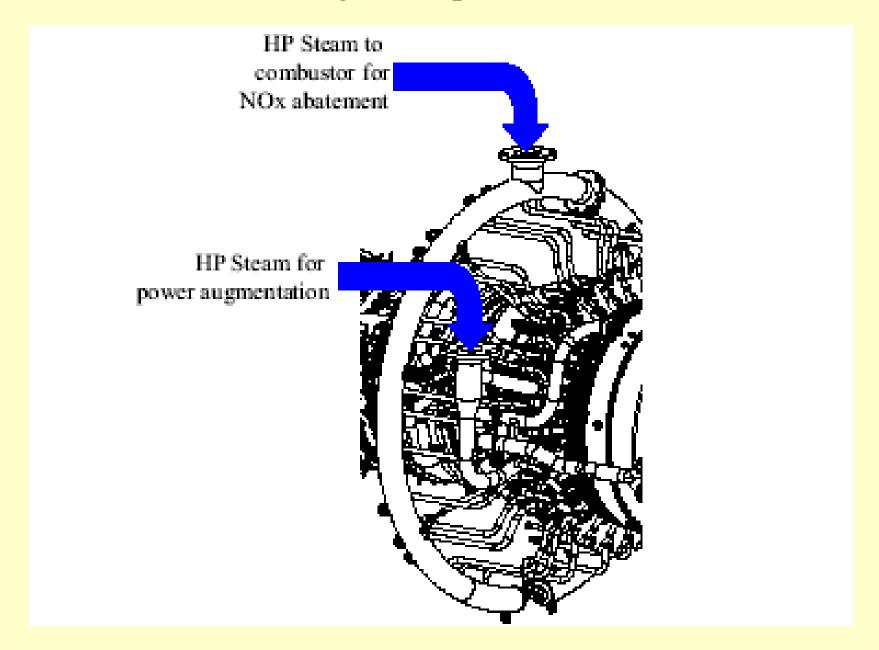
LM6000PC vs LM6000PC SPRINT



GT output enhancement -STIG



Steam injection ports -STIG



System performance enhancement- STIG

Model	Dry Rating (MWe)	%Thermal Efficiency (LHV)	STIG Rating (MWe)	%Thermal Efficiency (LHV		
LM1600	13.3	35	16	37		
LM2000	18	35	23.2	39		
LM2500	22.2	35	27.4	39		

GT Performance Degradation

- By contaminants in the air, fuel, steam and water
- Contaminants cause degradation by
 - Deposition
 - Erosion
 - Corrosion
- Degradation can be reversed by
 - Washing (partial reversal)
 - Overhauling /replacing the gas path components

Gas turbine Nozzle before and after washing



GT Performance Testing

- Performance test code –ASME PTC 22
- Testing by input /output method
- Input –Fuel flow rate
- Output –Electrical generation
- PTC 22 specifies
 - Test instrumentation
 - Data reduction
 - Correction methods
 - Test reporting requirements

GT Performance Testing-Parameters to be established

- Ambient dry bulb temperature
- Ambient wet bulb temperature (or RH)
- Barometric pressure
- Test fuel characteristics(heating value ,analysis ,temperature etc)
- Water injection rate ,pressure and temp
- Inlet and exhaust losses

GT Performance Testing-Parameters to be established

- Point where electrical output to be measured and what aux electrical loads are to be included
- Condition of the gas turbine unit (if a new unit, a definition of new and clean unit)
- Local point or points at which the test is to be conducted

Gas turbine fuels

- Gaseous fuels
- Liquid fuels
- Non conventional fuels
 - Crude oil
 - Refinery gas
 - Propane
 - Condensates
 - Orimulsion
- Synthetic fuels
 - Synthetic natural gas (SNG)
 - Synthetic liquid fuel

Gas fuel -Specifications

Fuel properties

- Heating Values
- Modified Wobbe Index (MWI)
- Superheat Requirement
- Hydrocarbon Dew Point
- Moisture Dew Point
- Flammability Ratio
- Gas Constituent Limits
- Gas Fuel Supply Pressure

Gas fuel -Specifications

Contaminants

- Liquid
- Sulfur
- Particulates
- Trace metals

Gas fuel properties - Modified Wobbe Index

- Measure of interchangeability of fuel
- Relative measure of the energy injected into the combustor at a fixed pressure ratio
- Calculated using fuel low heating value and specific gravity
- MWI range is established to ensure that required fuel nozzle pressure ratios are maintained at all combustion /turbine modes of operation

Gas fuel properties -Superheat

- To ensure that gas supply to the turbine is 100% free of liquids
- It is the difference between gas temperature and respective dew point temperature
- Condensation of moisture must be avoided to prevent the formation of gas hydrates and collection of water in low points of the gas fuel system
- The superheat requirements take into account the gas temperature drop and the relationship of the moisture and hydrocarbon dew point lines to the gas fuel pressure.
- The superheat requirement is specified to provide enough margin to compensate for the temperature reduction as the gas expands across the gas fuel control valves.

- Density and LHV
- Reid Vapor pressure
- Bubble Point
- Cloud and pour point
- Flash point
- Distillation end points
- Aromatic content
- Contents of olefins and diolefins
- Carbon residue
- Ash content
- Water and sediment contents
- Copper strip corrosion
- Corrosive contaminants

- Ignition performance based on
 - Volatility measured by Reid vapor pressure
 - Total surface area which is proportional to viscosity
- Free carbon determines the luminosity of the flame. Heavier the fuel greater the emissivity and resulting wall temperatures
- Warm enough to be capable of flowing under gravity-Pour point
- Not too warm to exceed flash point

- Capability to corrode fuel system components –
 Copper corrosion
- Ash content –Cause fuel system erosion
- Excessive content of olefins/diolefins can cause fuel decomposition and fuel system blockage
- Carbon residue more —Causes carbon deposit in the combustion system
- Solids in the fuel —Causes clogging of the fuel system especially fine flow passages in the injection system
- Certain components like Na,Li,K,Ca,V can cause hot corrosion of the hot section

- Bubble point —Temperature at which first bubble of gas appears.
 - Bubble point has to be considered for the following reasons
 - To supply the minimum fuel pressure for a liquid system
 - Fuel pressure must be high enough to allow for fuel turn down ratios and pressure drops in injector
 - To determine the size of the metering orifices
 - If bubble temperature is less than 0 Deg C at any combustor operating condition icing around injectors are likely.

LM600PC -Liquid Fuel specs

PARAMETER	UNIT	LIMIT
SODIUM+POTASSIU M+LITHIUM	PPM	0.2
VANADIUM	PPM	0.5
CALCIUM	PPM	2
WATER+SEDIMENT	PPM	1000
ASH	%	0.01
SULPHUR	%	1

LM600PC -Liquid Fuel specs

PARAMETER	UNIT	LIMIT
LEAD	PPM	1
CARBON RESIDUE	%	0.25
PARTICULATE	Mg/gal	10
FLASH POINT	$^{0}\mathrm{F}$	200
DEMULSIFICATION	minutes	20

DM water quality for NOX Suppression

PARAMETER	UNIT	LIMIT
PH		6 to 8
CONDUCTIVITY	micro siemens /cm	<1
TDS	ppm	<3
TOTAL MATTER	ppm	<5
SODIUM	ppm	< 0.1
SILICA	ppm	< 0.1

Gas turbine Emission

Emission Products

- Nitrogen oxides
- Carbon monoxide
- Volatile Hydrocarbon Compounds(VHC)
- Sulphur oxide(Sox)
- Particulate matter

NOX Emission

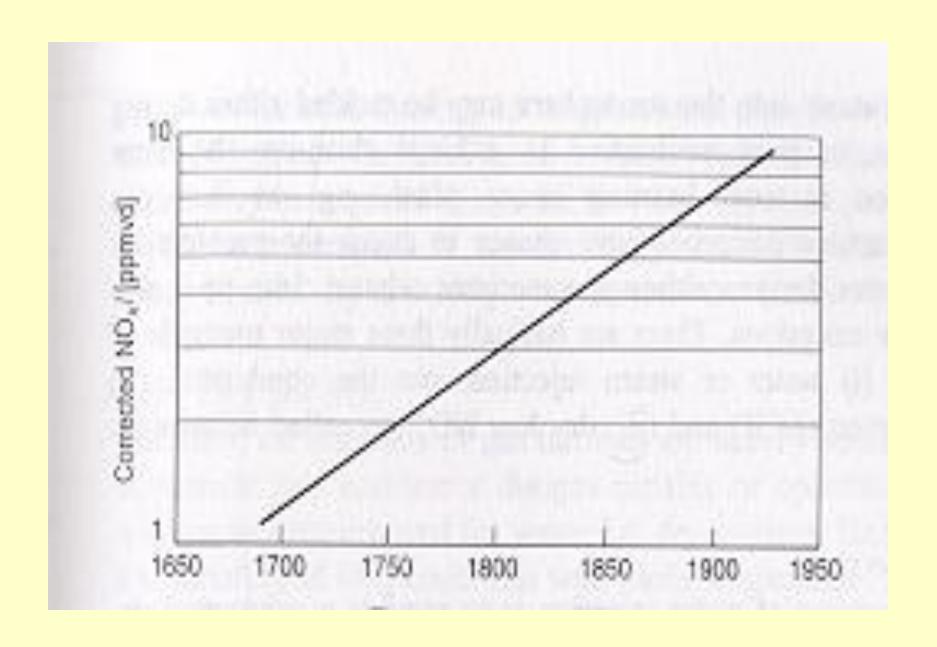
Oxides of nitrogen from gas turbine are of two types

- Fuel NOX —Oxidation of nitrogen in the fuel.Independent of combustion temperature
- Thermal NOX-High temperature reaction between Nitrogen and Oxygen in the combustion air.Dependent on combustion temperature.

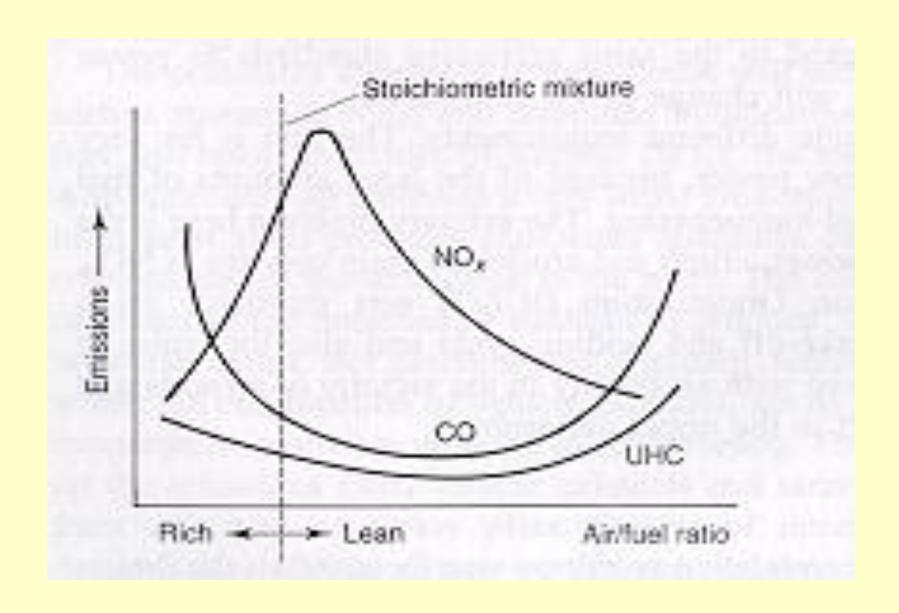
NOX Emission-Contributing factors

- Combustion chamber design
- Turbine operating parameters such as flame temperature
- Residence time
- Fuel/Air ratios in the primary combustion zone

NOX Emission Vs Combustion temperatures



NOX Emission Vs Air-fuel ratio



NOX Control

- NOX formation can be limited by lowering combustion temperatures and staging combustion.
- How to control?
 - Water or steam injection in the combustion zone
 - Improved low NOX combustors
 - Lean premixed
 - Rich/quench/lean combustion
 - Catalytic combustion
 - Selective catalytic reduction

CO and VOC control

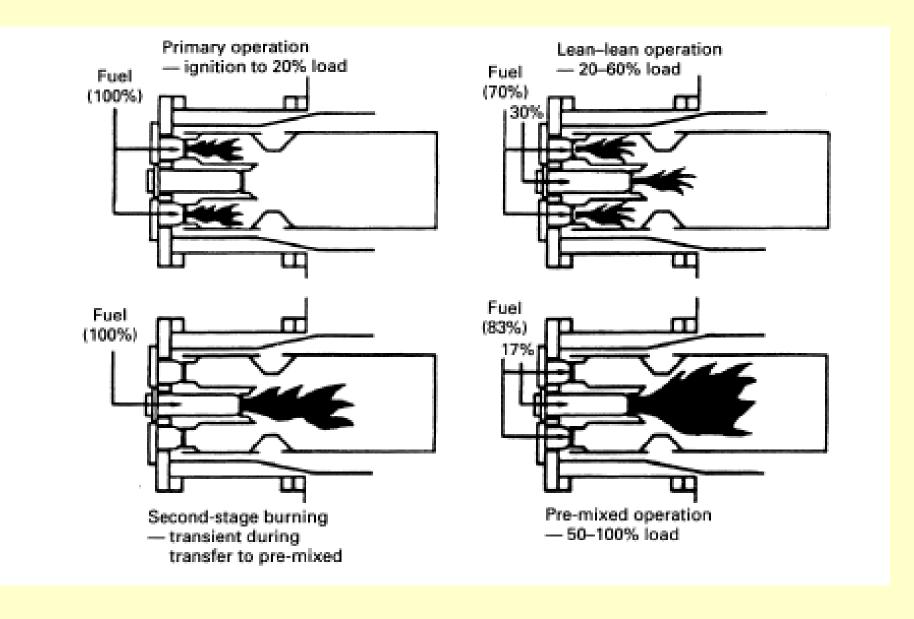
Catalytic oxidation

- CO to CO2
- VOC to CO2 and H2O

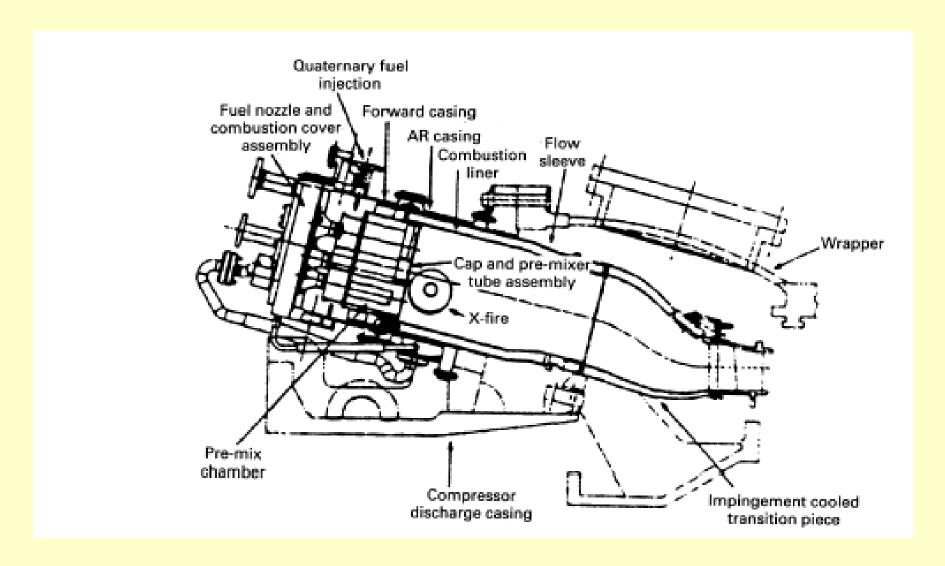
Disadvantage of catalytic oxidation

• SO3 production with sulphur bearing fuels

DLN Combustor Operating sequence



DLN Combustor



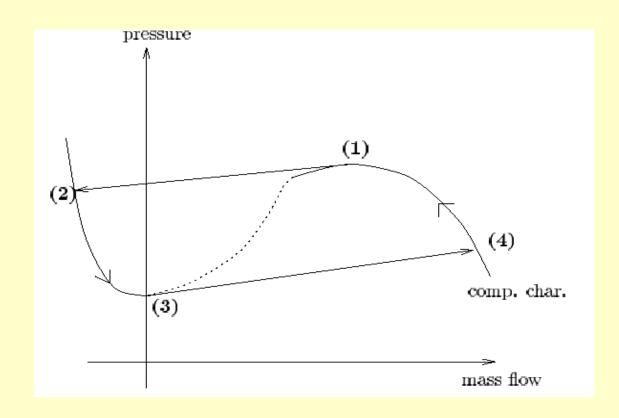
Emission control results

- Water injection reduces NOX from >200 ppm to 25 ppm
- Adding an SCR can further reduce NOX to
 5 ppm
- DLN combustors can reduce NOX to <10 ppm without SCR
- SCONOX purports to achieve <2 ppm NOX and < 1ppm CO

Gas turbine off design operation - Surge

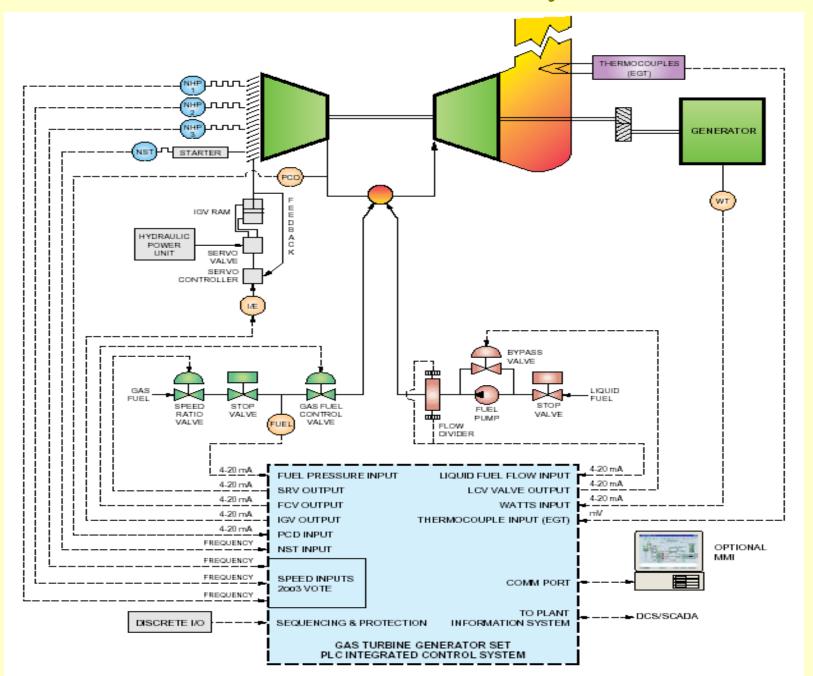
• Complete reversal of air/gas flow causing inlet temperature to rise

Compressor Off Design Operation -Surge

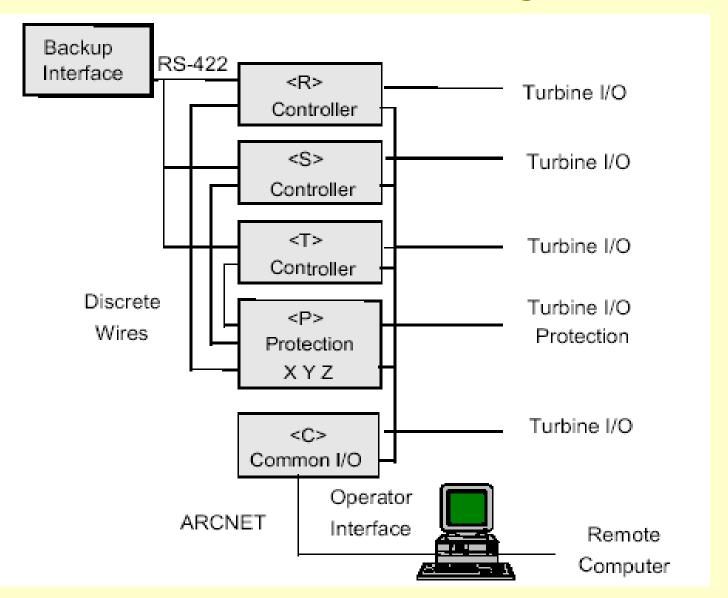


The cycle starts at (1) where the flow becomes unstable .It then jumps to the reverse flow characteristic (2) and follows this till zero flow at point (3) and then jumps to (4) where it follows the characteristic to (1)

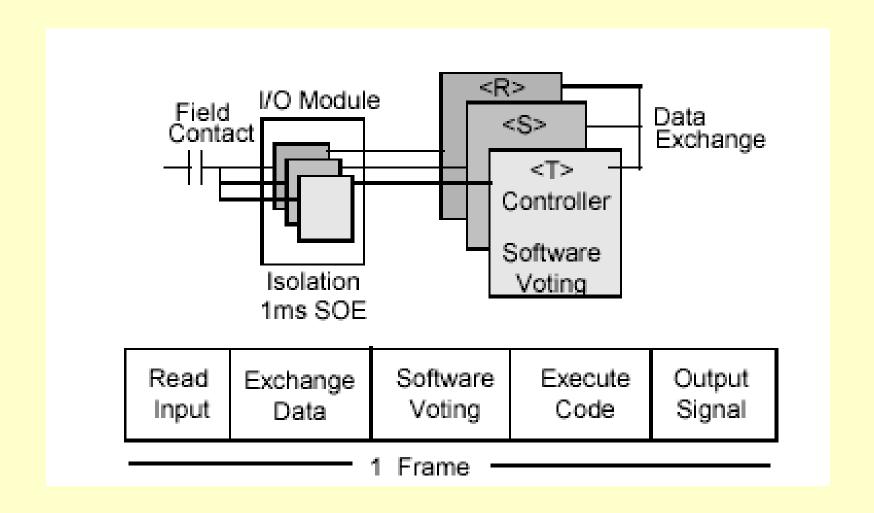
Schematic of PLC based control system for GTG



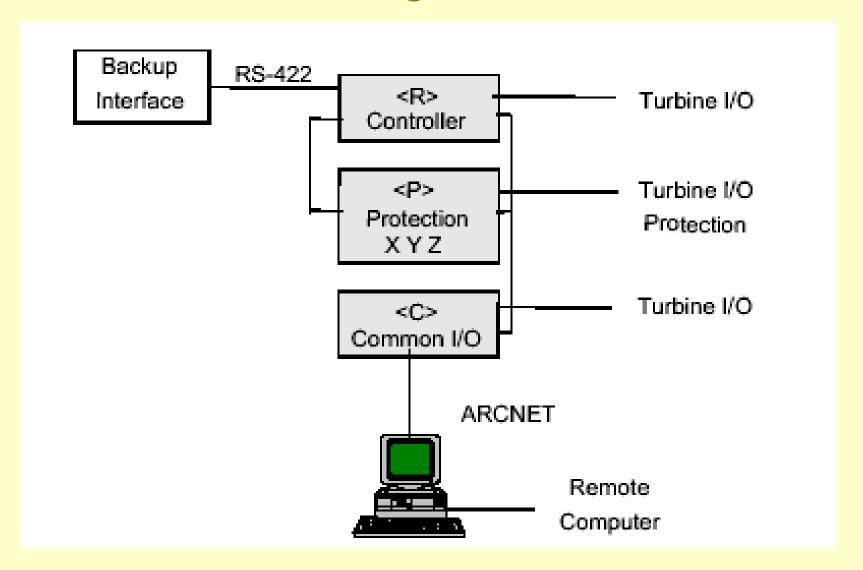
Gas Turbine control systems –Speedtronic Mark V Control TMR Configuration



Mark V TMR Control Sequence



Mark V Simplex Control Configuration



Speedtronic Mark V Hardware Inputs/Outputs

The Mark V I/O is designed for direct interface with

the following turbine and generator devices:

- magnetic speed pickups
- servos and LVDT/Rs
- vibration sensors
- thermocouples
- Resistive Temperature Devices (RTDs)

Speedtronic Mark V control-Features

- software-implemented fault tolerance
- triple modular redundant (TMR) architecture
- online maintenance
- direct sensor interface
- built-in diagnostics
- common hardware platform for gas and steam applications

Woodward- NetCon control

- A programmable control system
- Used in gas turbine, steam turbine, hydraulic turbines
- Can control
 - Speed
 - Load
 - Pressure
 - Level
 - Flow
 - Any system parameter that can be sensed either by a transducer and converted to an analog signal or by a switch or contact
 - Perform start /stop sequencing

Netcon control system

- Simplex system one CPU
- Fault tolerant system-Triple Modular Redundancy (three CPUs with own power supply)
- Dual I/O copper redundancy allowing each set of I/O boards to receive own operating power I/O signals
- Triple I/O copper redundancy

Netcon control system-Inputs

- Can receive inputs from
- Speed sensors
- Magnetic pick up/Proximity switches
- Temperature sensors
 - J,K,T,S,R,E,N type thermocouples
 - RTDS
- Cold junction sensors
- Position transducers
 - Rotary Variable Differential Transformer
 - Linear

Netcon control system-Inputs

- Pressure sensors
 - 4 to 20 ma signals
- Load sensors
 - -4-20 ma
 - -0-5 V dc
 - 0-10 V dc
- Switch or relay contacts

Netcon control system-Outputs

- Four types of output modules
 - Actuator driver module
 - Proportional –4 actuator driver channels
 - Proportional or integrating-2 actuator driver channels
 - − Discrete output module −32 or 64 channels
 - Current output module –8 outputs
 - 0-1 and 4-20 mA
 - Voltage output module –8 outputs
 - 0-5 and 0-10 Vdc

Netcon control system

Communications

- Can communicate with other devices using RS-232 ,RS-422 or RS-485 communication protocol and ModBus
- Can be part of LAN

Power supply

- Single source or redundant
- Main power supply –24 V dc
- I/O power supply-24 V dc provides 5 Vdc for CPU and I/O module

Cooling

By forced air

- **CPU** –Follows the instructions of application program control the circuits .Processor –Motorola 68030 with a math co processor or Motorola 68040
- **CPU Aux module**-Used in FT control systems to power each CPU
- FTIO module –Handle communication with I/O racks
- **SIO module** -Interfaces communication ports to the VME bus
- Ethernet module –Connects to LAN

- Actuator driver module(2 channel) —Receives digital information from CPU and generate two integrating or proportional actuator signals
- Actuator driver module (4 channel) —Receives digital information from CPU and generate four proportional actuator driver signals
- **RTD input module**-Four models of RTD input module-Each handles a different type of RTD ,converts analog data to digital data and send to CPU
- Thermocouple input module —Receives signals from 8 thermocouples and converts analog to digital and sends to CPU

- **Digital speed sensor module** —Receives signals from as many as four magnetic pick ups and sends a digital speed signal and a derivative signal for each channel to the system CPU
- **Discrete input module**-Receives signals from 48 separate switches or relay contacts and sends the data to CPU.Isolated from NetCon's balance circuitry. The module provides 24V dc for these external contacts.
- **Discrete output modules** –Receives digital data from CPU and generates either 32 or 64 relay driver signals. All discrete output modules interface with one or more Woodward relay boxes each with 16,28 or 32 relays. The contacts of these relays then connect to field wiring.

- Voltage input module —Receives eight 0-10 V signals from the controlled system ,converts analog to digital data and sends to CPU
- Voltage output module —Receives data from the CPU and generates eight 0-5 Vdc/0-10 Vdc output signals
- Current input module -Receives eight 4-20 mA signals from the controlled system ,converts analog to digital data and sends to CPU
- Current output module —Receives data from CPU and generates 0-1 / 4-20 Ma output signals

- LinkNet controller module —Communicates with the various I/O modules on the LinkNet I/O networks.Both analog and discrete information from the networks are received by this module and shared with main CPU.Information from CPU is also received by this module and then sent to appropriate I/O modules on the network
- Pressure transducer Module Receives information from Smart Pressure transducers and data is shared with main CPU
- Vibrometer digital vibration module —Six channel monitoring module. Four channels for monitoring vibration and two for speed input

Netcon control system

- Service Panel-Consists of a keyboard, a digital display .Used by operator to monitor and change system values and by the diagnostic software to display fault messages
- **Software** –Independent of hardware that is minor changes or upgrades do not require changes to the program
- · OS-
- **Application programme** –Designed using GAP and loaded to UVPROM or flash memory chip and installed on the CPU

Thermocouple ranges

T/C type	Low end	High end
J	-40 °C	1050 °C
К	-40 °C	1600 °C
E	-40 °C	800 °C
R	-40 °C	1750 °C
S	-40 °C	1750 °C
Т	-40 °C	1050 °C
N	-40 °C	1051 °C
CJ	-40 °C	150 °C

Typical Engine Instrumentation (LM6000PC)

Device	Parameter	Function*	Device Type	Qty
P0	Atmospheric Pressure	С	Press transducer	1
T2	LP Compressor Inlet Temperature	С	RTD	2
P2	LP Compressor Inlet Pressure	С	Press transducer	2
T25	HP Compressor Inlet Temperature	С	RTD	2
P25	HP Compressor Inlet Pressure	С	Press transducer	2
N25	HP Shaft Speed	C/P	Magnetic pick	2
PS3	HP Compressor Discharge Pressure	C/P	Press transducer	2
T3	HP Compressor Discharge Temperature	C/P	TC	4
PTB	Thrust Balance Pressure	C/P	Press transducer	2
P48	HP Turbine Discharge Pressure	С	Press transducer	1
T48	HP Turbine Discharge Temperature	C/P	TC	8
NSD	LP Shaft Speed	C/P	Magnetic pick	2

Typical Engine Instrumentation (LM6000PC)

Device	Parameter	Function*	Device Type	Qty
IGV	Inlet Guide Vanes	C/M	LVDT	2
		С	Servo	1
VBV	Variable Bleed Valves	C/M	LVDT	2
		С	Servo	1
VSV	Variable Stator Vanes	C/M	LVDT	2
		С	Servo	1
TBV	Thrust Balance Valve	C/M	LVDT	2
		С	Servo	1
GFMV	Gas-fuel-metering Valve	С	Servo	1
LFMV	Liquid-fuel-metering Valve	С	Servo	1
PGAS	Gas-fuel Supply Pressure	C/M	Press transducer	1
TGAS	Gas-fuel Supply Temperature	C/M	RTD	1
PLIQ	Liquid-fuel Supply Pressure	C/M	Press transducer	1
TLIQ	Liquid-fuel Supply Temperature	C/M	RTD	1
28FD	Flame Detectors	C/P	UV scanners	2

Frame Machines-Details

- Horizontally split casing
- Two walk-in combustion chambers for hotgas-path inspection without cover lift
- Combustion chambers lined with individually replaceable ceramic tiles
- 16-stage axial-flow compressor with variablepitch inlet guide vanes
- Optional fast inlet guide vanes for peak load operation and frequency stabilization

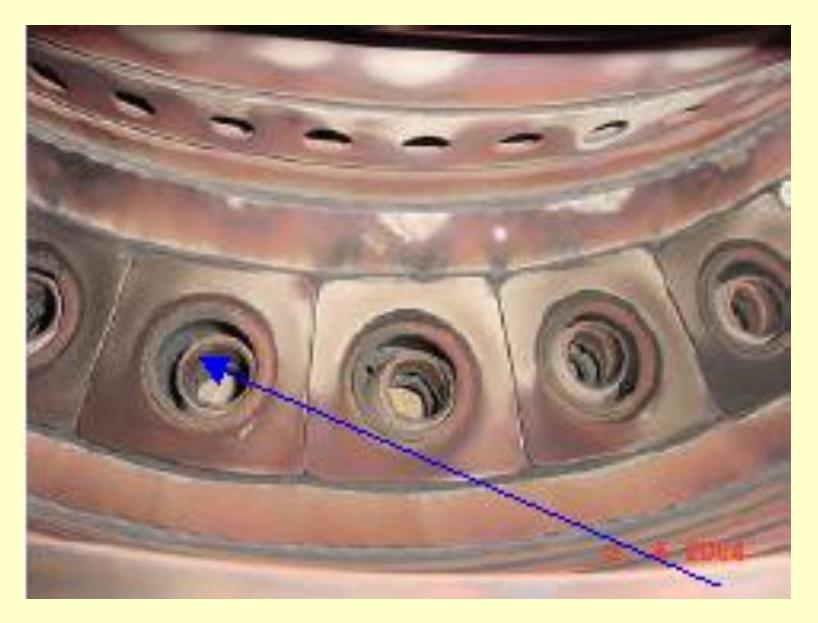
Frame Machine -Details

- Four stage turbine
- All blades removable with rotor in place
- Hybrid burners for premix and diffusion mode operation with natural gas, fuel oil and special fuels such as heavy oil and refinery residues
- Low-NOx combustion system

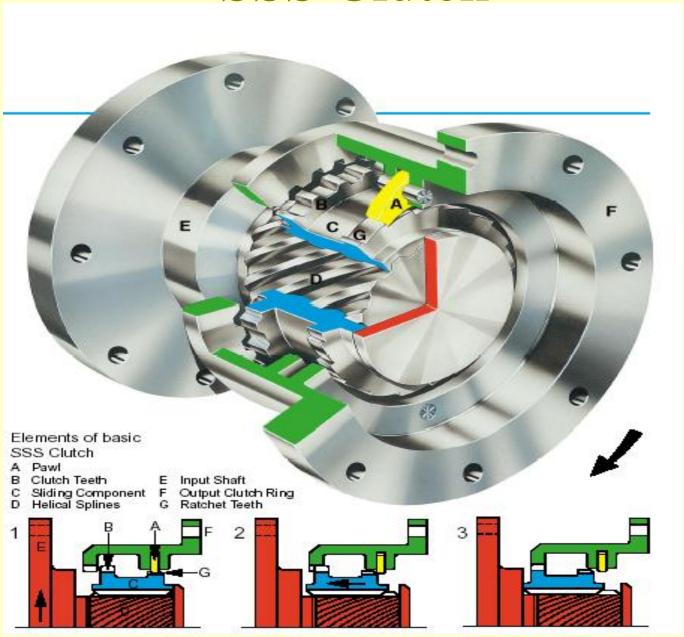
Fuel Nozzles



Combustor Swirler



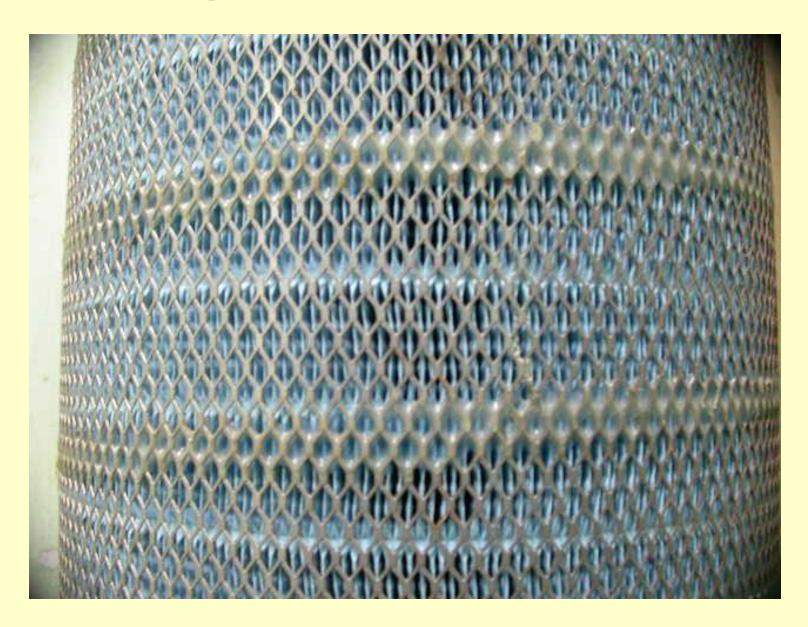
SSS Clutch



Gas turbine Filter



Gas turbine filter



Typical Alarm and trip requirements for gas turbines

		Alarm and
Indication	Alarm Only	Shutdown
Engine System		
Impending High Blade Temperature	X	
High Blade Temperature		X
Impending High Engine Temperature	X	
High Engine Temperature		X
High Gas Producer Thrust Bearing Temperature	X	X
High Power Turbine Thrust Bearing Temperature	X	X
Fail to Crank		X
Fail to Start		X
Ignition Failure		X
Starter Dropout Failure		X
Backup Overspeed Power Turbine		X
Overspeed Power Turbine		X
High Vibration – Engine	X	X
High Vibration – Accessory Gearbox	X	X

Typical Alarm and trip requirements for gas turbines

		Alarm and
Indication	Alarm Only	Shutdown
Engine System		
Lube System		
High Oil Temperature	X	X
Low Oil Pressure		X
Low Prelube Oil Pressure	X	X
Low Oil Level	X	X
High Oil Level	X	
Lube Filter High Differential Pressure		
Fuel System		
Liquid Fuel Filter High Differential Pressure	X	
Low Liquid Fuel Pressure	X	X
Gearbox Systems		
High Gearbox Output Bearing Temperature	X	X
High Vibration – Gearbox	X	X
Generator Systems		
High Generator Bearing Temperature	X	X
High Generator Winding Temperature	X	X
High Vibration – Generator	X	X
Ancillary Systems		
Low Battery Voltage	X	
Battery Charger Failure	X	
Inlet Air Filter – High Differential Pressure	X	X

