Air Conditioning Fans & VAV System

ACES Training K Y Yow 23 Jan 2015

Air Conditioning Fans

- Fundamentals
- Fan Types
- Fan Capacity Control
- Application Considerations

Air Conditioning Fans



centrifugal



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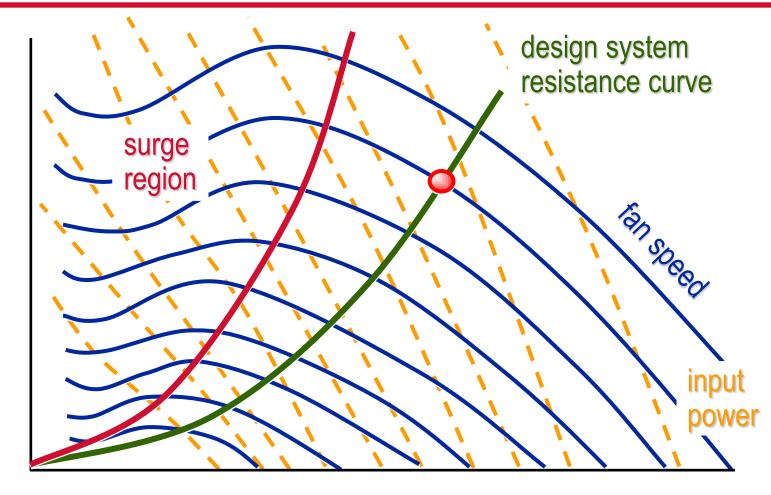
Test Setup - Model ADA 710



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Fan Curve

static pressure



Fan Law 1

Airflow 2
Airflow 1
$$= \left(\frac{\text{Fan RPM }_{2}}{\text{Fan RPM}_{1}}\right)$$
Static Pressure 2
Static Pressure 1
$$= \left(\frac{\text{Fan RPM }_{2}}{\text{Fan RPM }_{1}}\right)^{2}$$
Input Power 2
Input Power 1
$$= \left(\frac{\text{Fan RPM }_{2}}{\text{Fan RPM }_{1}}\right)^{3}$$

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Determining Fan Airflow

Velocity Pressure $(P_v) = P_t - P_s$

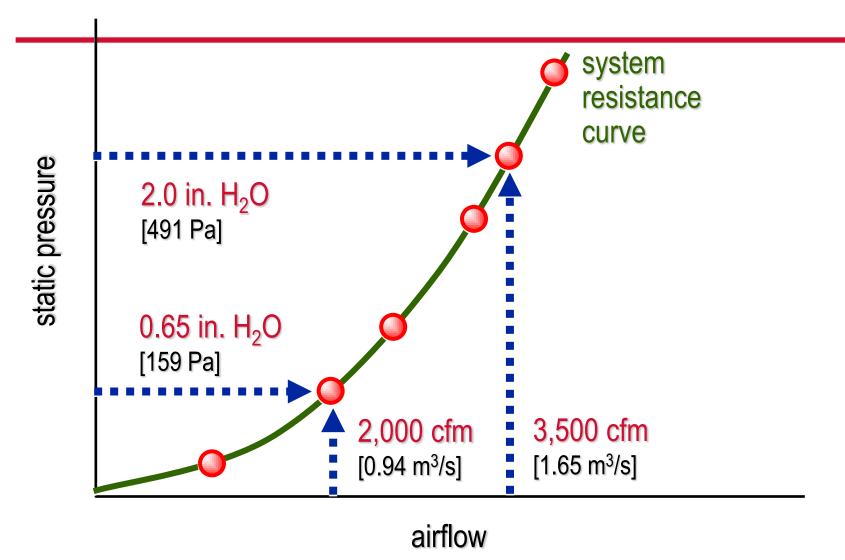
Velocity (V) = Constant
$$\times \sqrt{\frac{P_v}{\rho}}$$

Airflow = Velocity × Fan Outlet Area

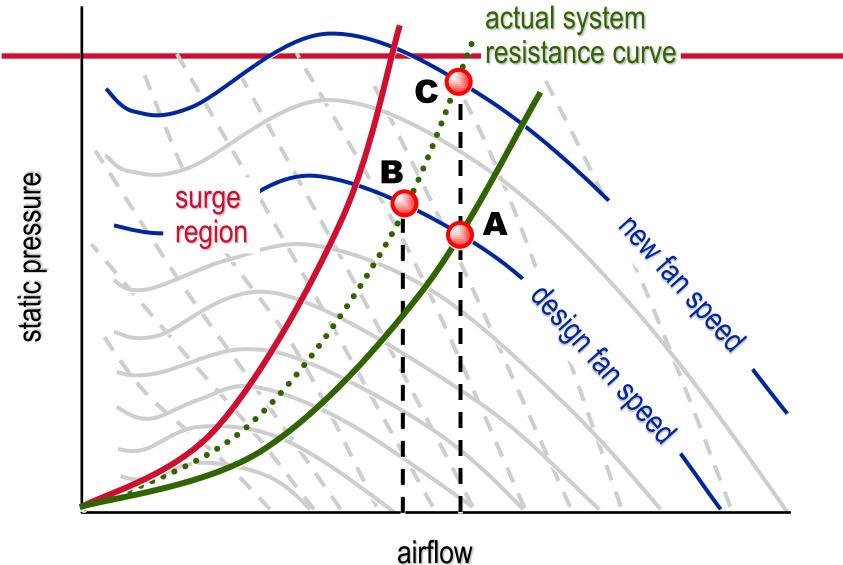
System Resistance Curve

Static Pressure
$$_{1}^{2} = \left(\frac{\text{Airflow}_{2}}{\text{Airflow}_{1}}\right)^{2}$$

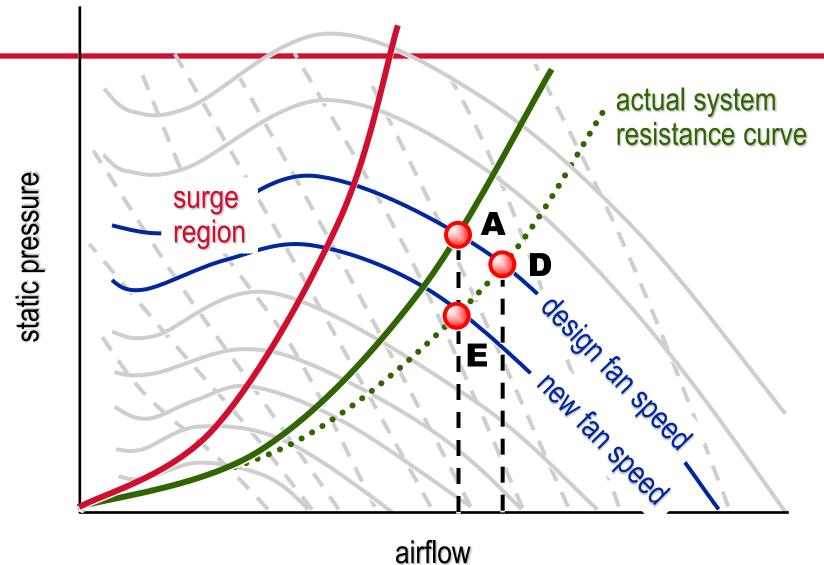
System Resistance Curve



Higher System Resistance

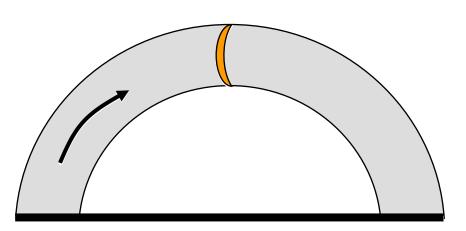


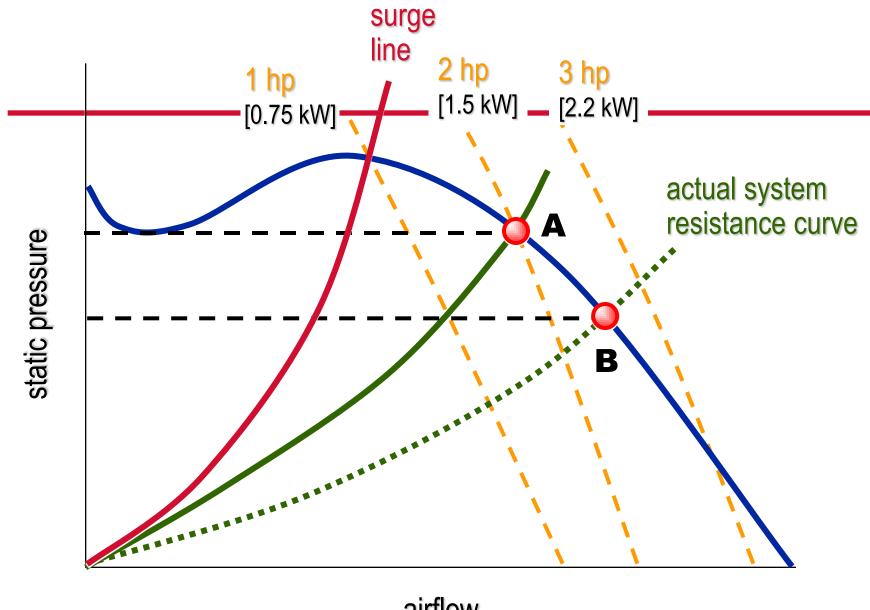
Lower System Resistance



Forward Curved Fan

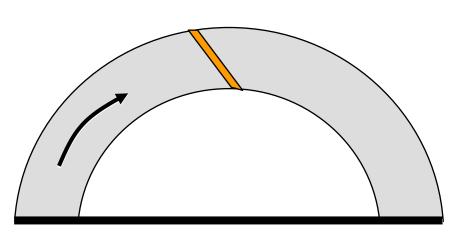




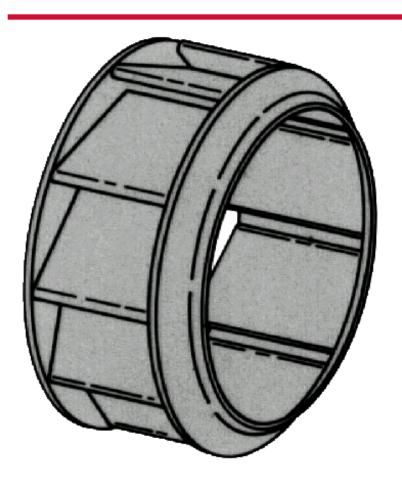


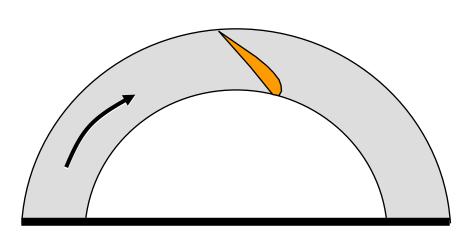
Backward Inclined Fan



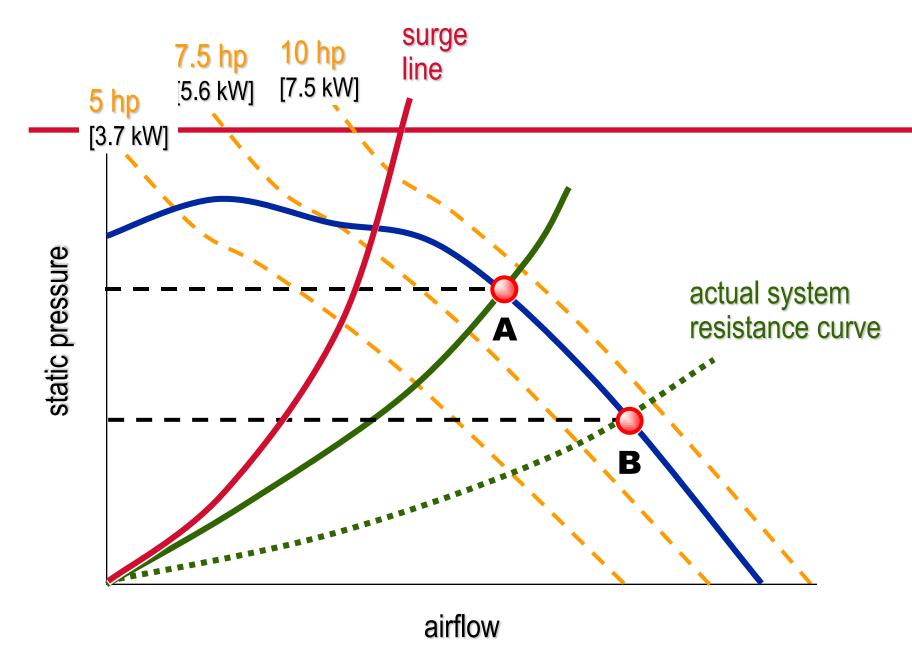


Airfoil Fan

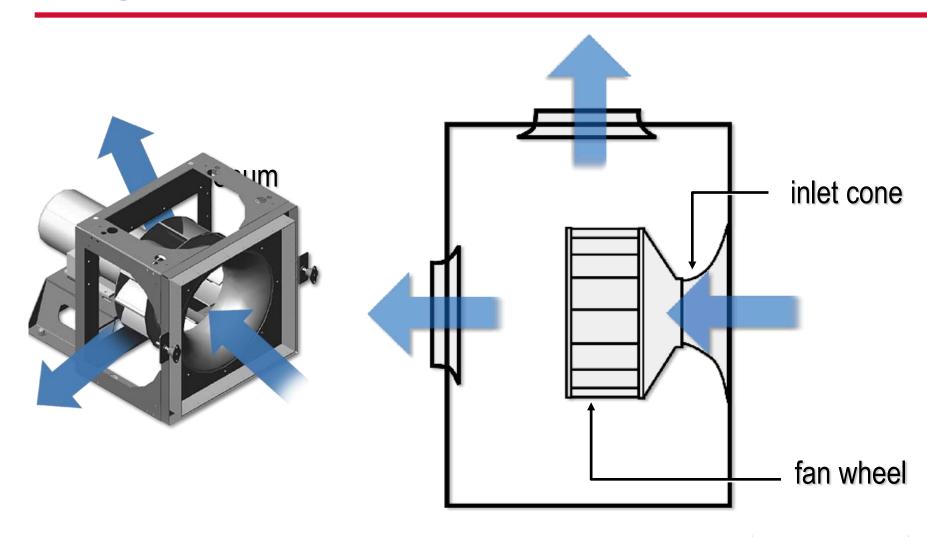




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Plug (or Plenum) Fan



Fan Selection

- Forward curved (FC)
 - Airflow with low static pressure
 - Low Speed
 - Not suitable for high static application
- Backward inclined (BI) or airfoil (AF)
 - Airflow with higher static pressure
 - Higher efficiency
 - Twice the speed of FC
 - Magnitude of surge is greater

Air Conditioning Fans

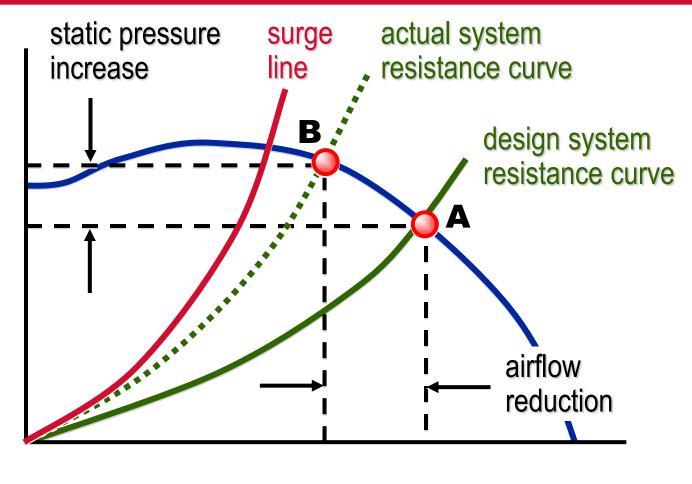
Fan Capacity Control

Fan Capacity Control

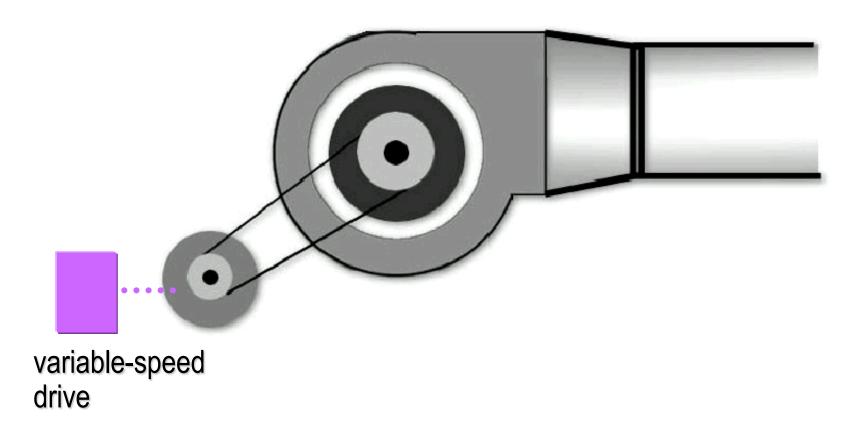
- Riding the fan curve
- Discharge dampers
- Inlet vanes
- Fan-speed control

"Riding the Fan Curve"

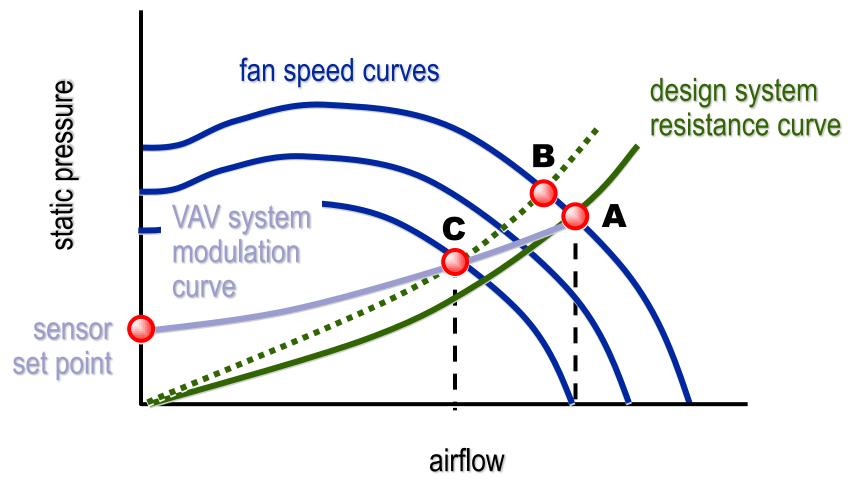
static pressure



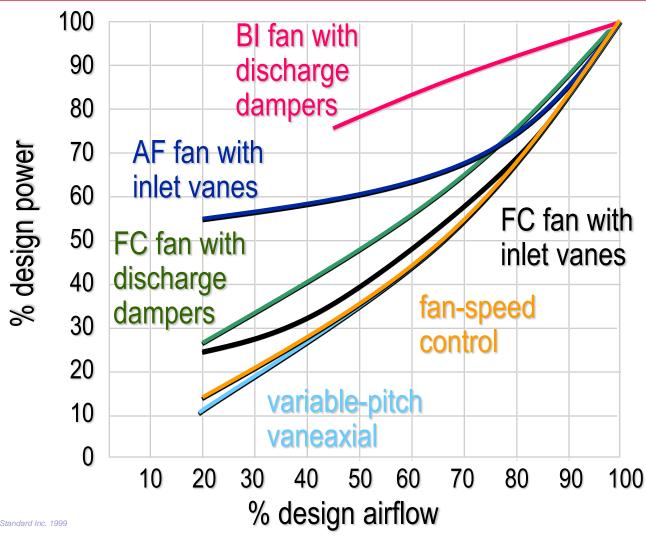
Fan-Speed Control



Fan-Speed Control



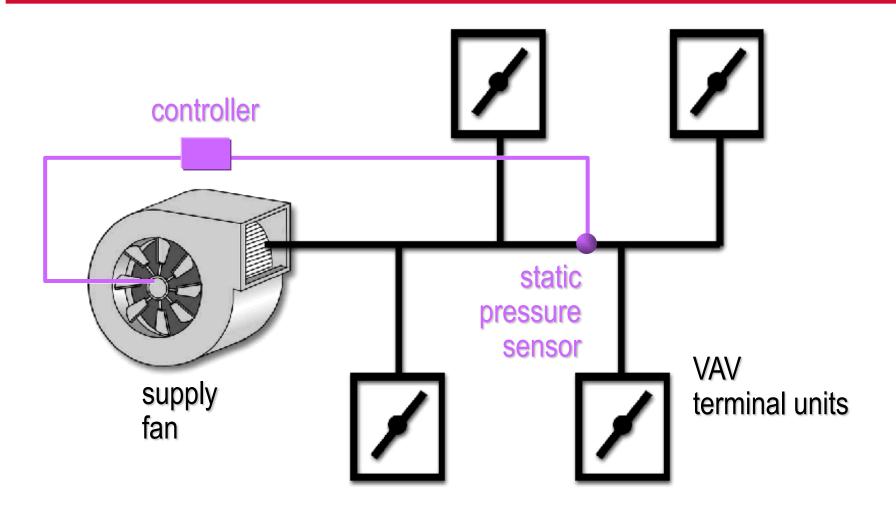
Fan Control Comparison



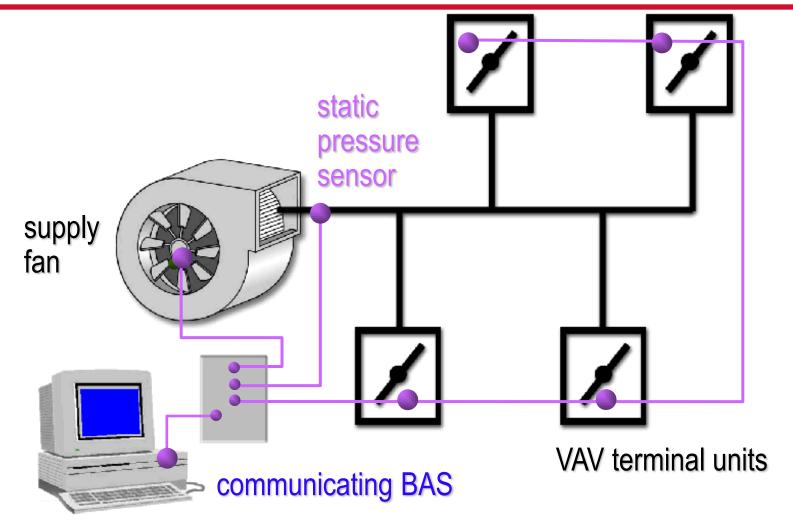
Air Conditioning Fans

Application Considerations

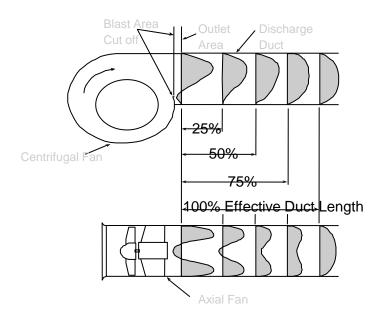
System Static-Pressure Control

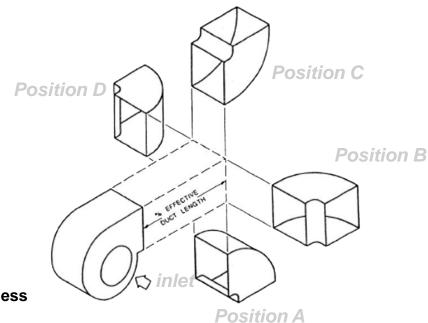


Optimized Static-Pressure Control



System Effect





Effective Duct Length = 2.5 Duct Diameters for 2,500 FPM or less

Add 1 duct diameter for each additional 1,000 FPM

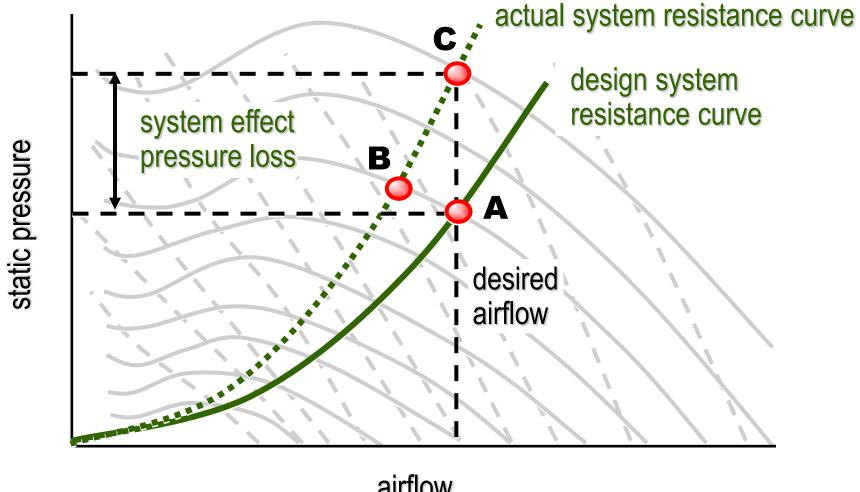
For rectangular ducts, the equivalent duct diameter is

(4 x width x length / 3.14) ^ 0.5

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System Effect



System Effect Factors are Real



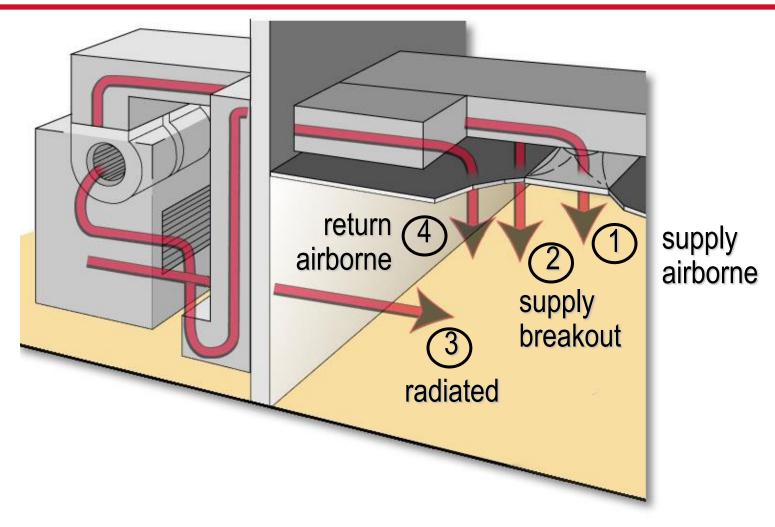
Rectangular duct elbow,

Recommendations

- Minimum 2.5 duct diameters on Outlet
- Minimum 5 to 8 duct diameters on Inlet
- Avoid inlet swirl

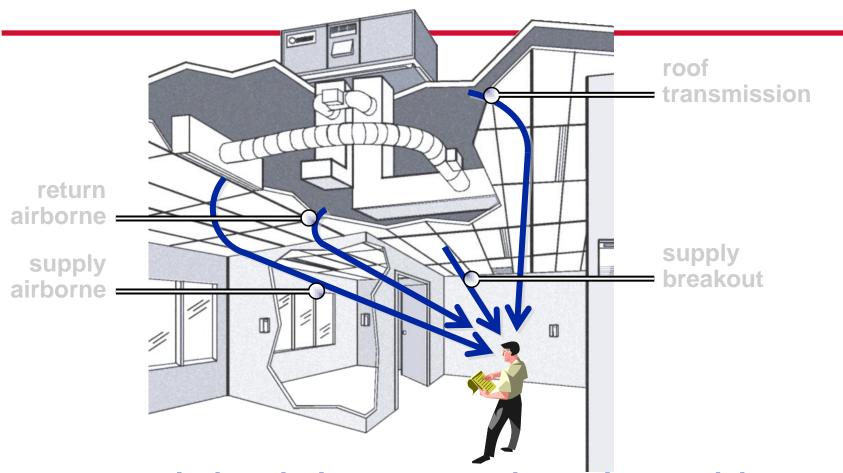
 If any of these general rules are broken, make sure to take system effect into account.

Acoustics



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Sound Transmission Paths



Acoustical analysis: Source-path-receiver model

fan application Acoustical Guidelines

- Optimize fan and air-handler selection for lowest overall sound
- Select fan to operate safely away from surge region
- Minimize system effects
- Use low-pressure-drop duct fittings (follow SMACNA recommendations)
- Avoid rectangular sound traps, if possible
- Use adequate vibration isolation

Effect of Actual (Nonstandard) Conditions

2)
$$SP_{standard} = \frac{SP_{actual}}{Air Density Ratio}$$

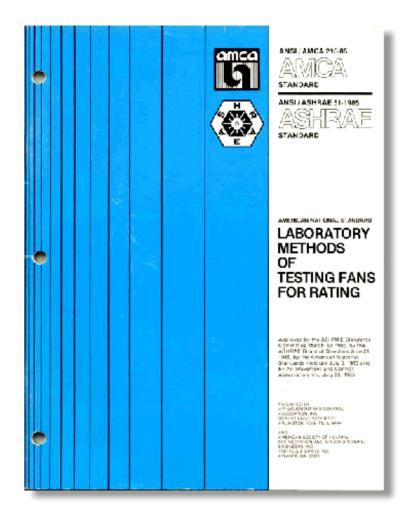
- 3) Use Airflow_{actual} and SP_{standard} to select fan
- 4) $RPM_{standard} = RPM_{actual}$
- 5) Power_{actual} = Air Density Ratio × Power_{standard}

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Equipment Certification Standards AMCA – Air Movement and Control Association

Purpose

 Establish methods for laboratory testing of air moving devices



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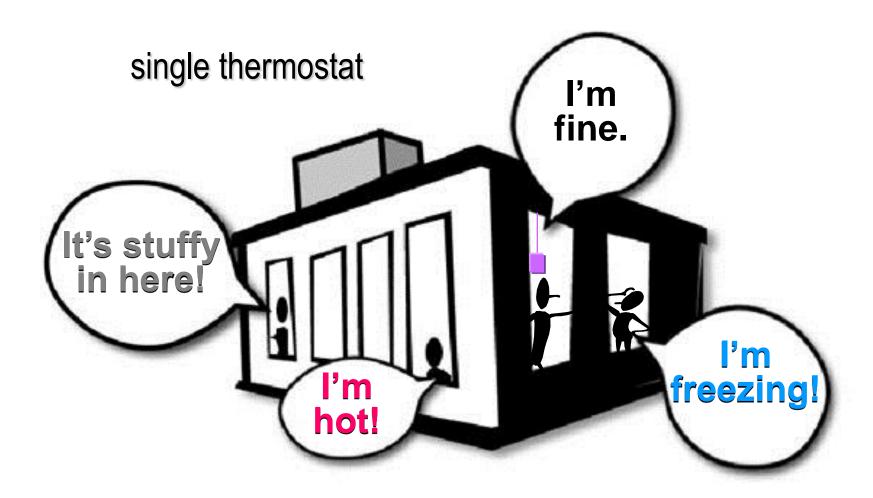
VAV Systems

A Trane Air Conditioning Clinic

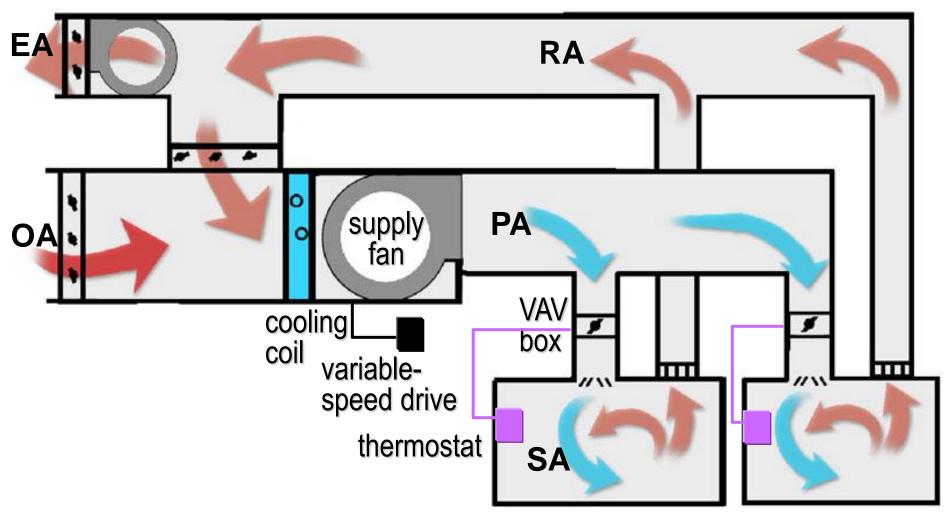
VAV Systems

period one What Is Variable Air Volume?

constant-volume, variable-temperature system ... Applied To Multiple Spaces



Variable-Air-Volume System



variable-air-volume (VAV) system ... Full Load

With these design conditions ...

space sensible heat gain = 40,000 Btu/hr

space dry-bulb temperature = 75°F

supply air dry-bulb temperature = 55°F

Supply =
$$\frac{40,000 \text{ Btu/hr}}{1.085 \times (75^{\circ}\text{F} - 55^{\circ}\text{F})} = 1,840 \text{ cfm}$$

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variable-air-volume (VAV) system ... Part Load

With these part load conditions ...

space sensible heat gain = 20,000 Btu/hr

space dry-bulb temperature = 75°F

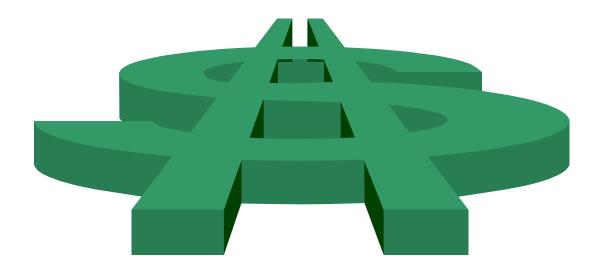
supply air dry-bulb temperature = 55°F

Supply =
$$\frac{20,000 \text{ Btu/hr}}{1.085 \times (75^{\circ}\text{F} - 55^{\circ}\text{F})} = 920 \text{ cfm}$$

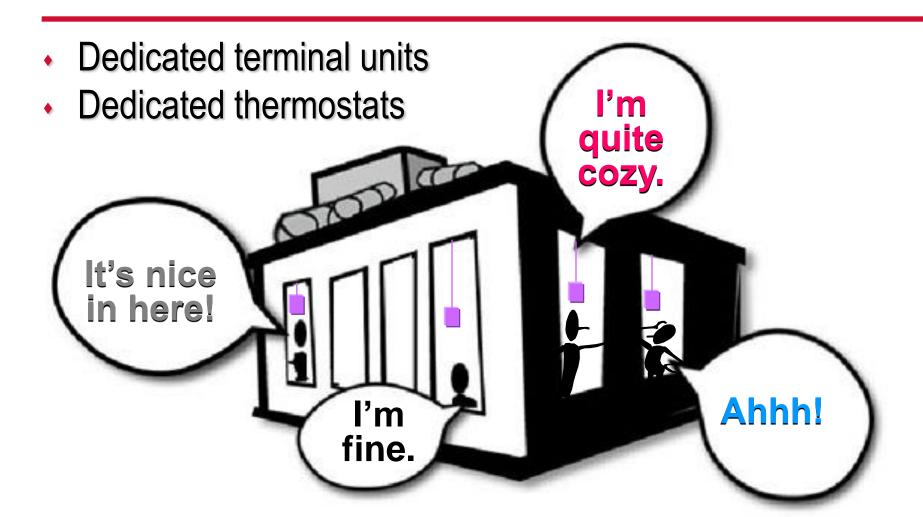
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why VAV? Part-Load Energy Savings

- Reduced fan energy
- Reduced refrigeration energy



why VAV? Comfort



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System Comparison

constant-volume, single zone

- Constant fan energy
- Refrigeration energy savings
- Delivers comfort to only one thermal zone

VAV

- Fan energy savings
- Refrigeration energy savings
- Delivers comfort to many spaces efficiently

variable-air-volume system ... **Building Characteristics**

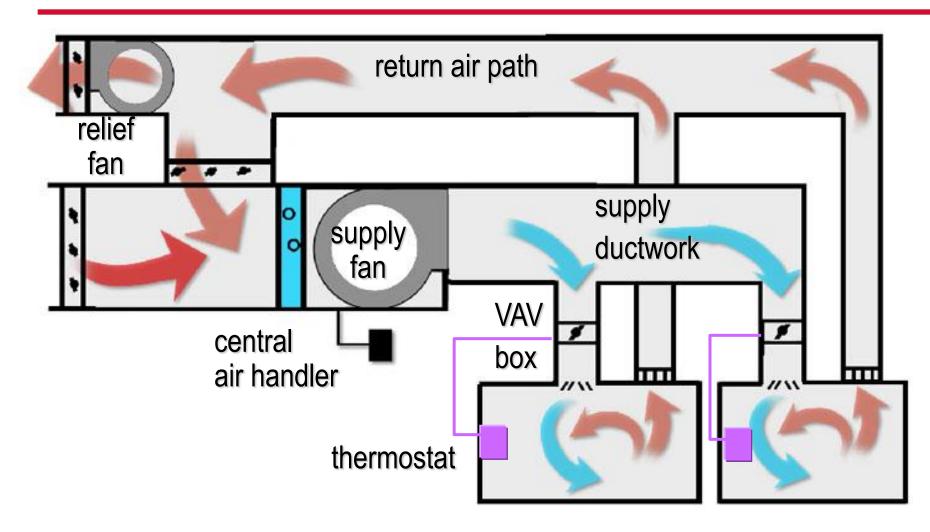
- Variable thermal load profiles
- Multiple, independently-controlled spaces
- Common return air path

VAV Systems

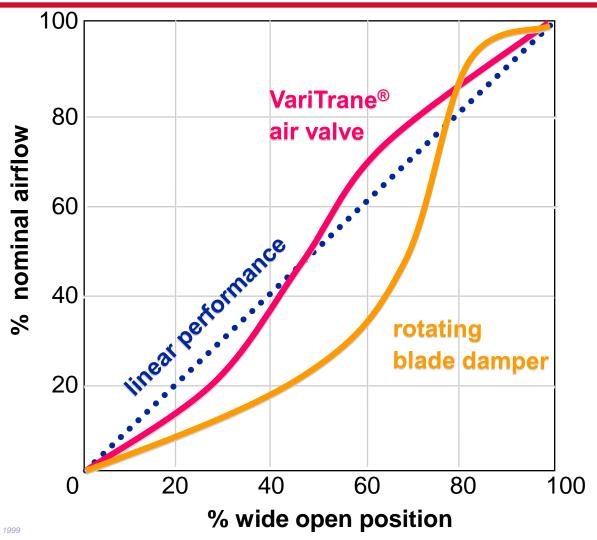
period two Components Of A VAV System

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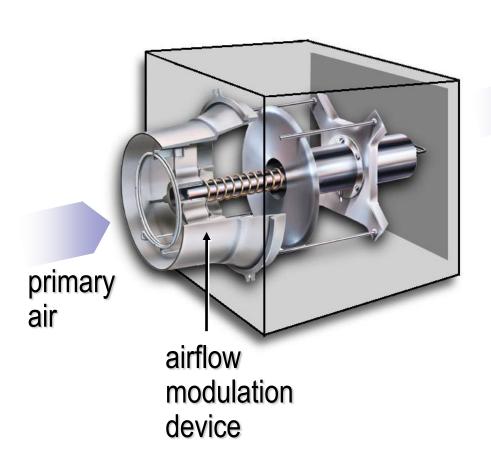
components of a VAV System



Near-Linear Performance



VAV Terminal Units

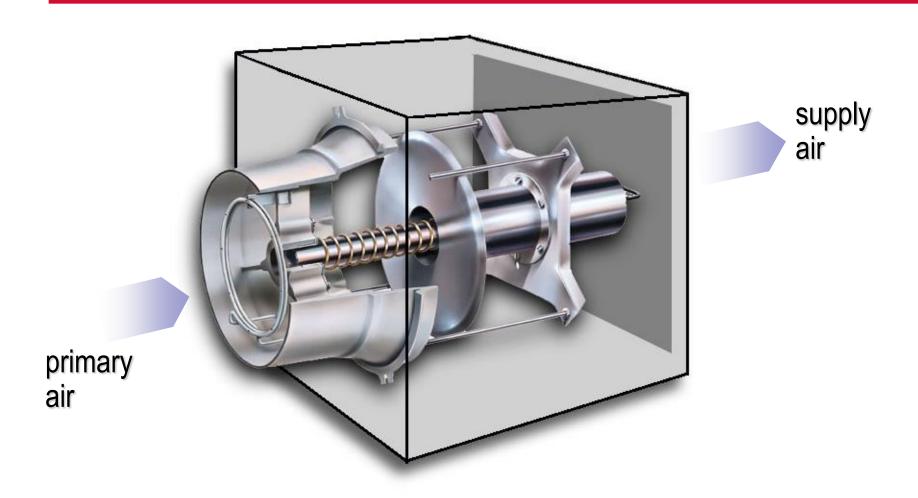




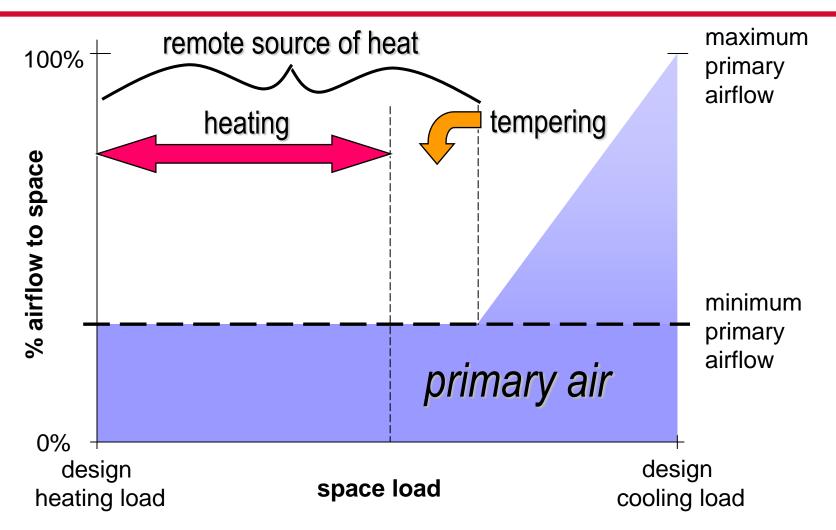
- terminal heating coil
- filter
- terminal mixing fan

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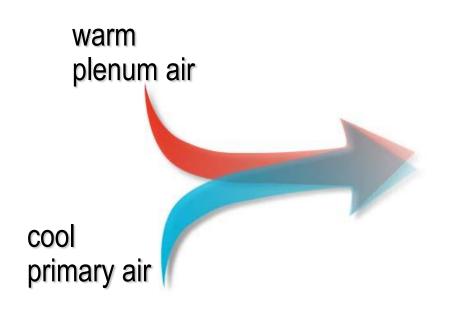
VAV terminal unit types ... Single-Duct, Cooling-Only

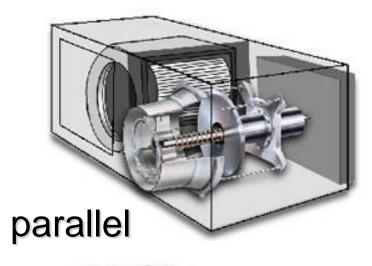


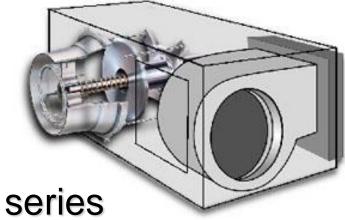
VAV terminal unit types ... Single-Duct, Cooling-Only



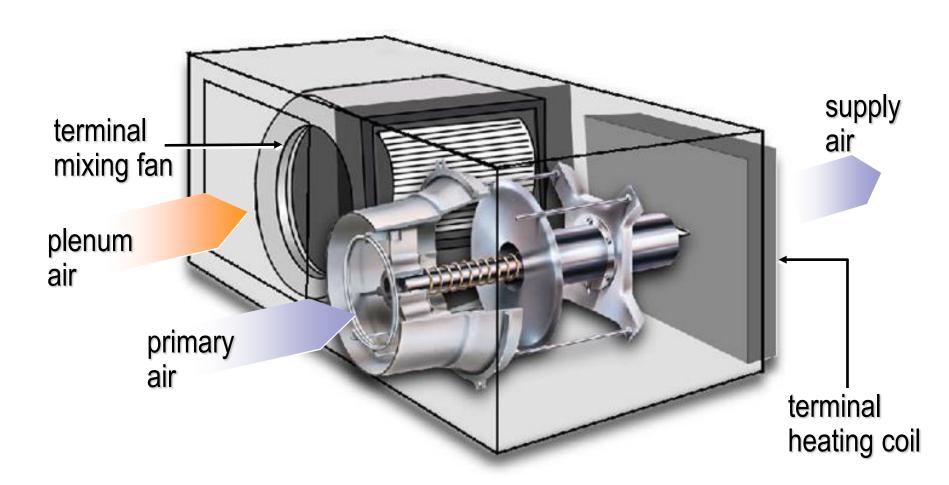
VAV terminal unit types ... Fan-Powered



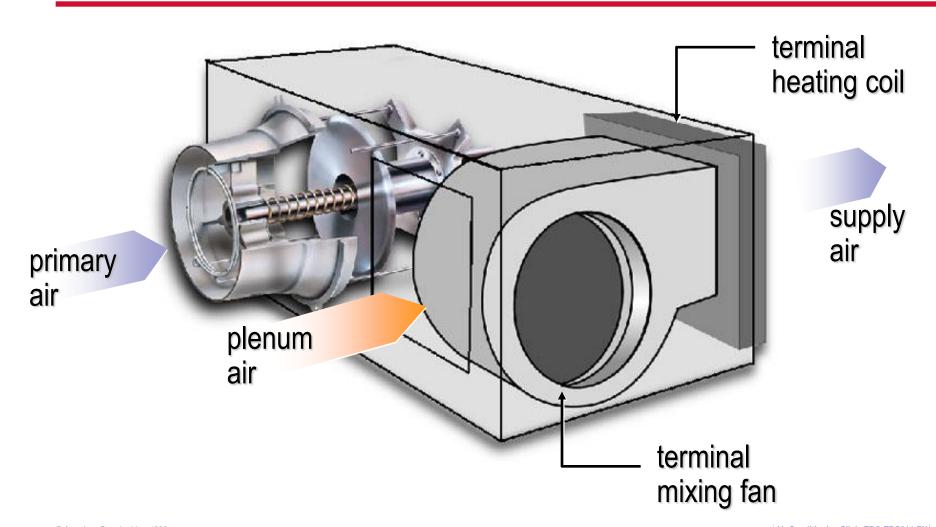




VAV terminal unit types ... Parallel, Fan-Powered



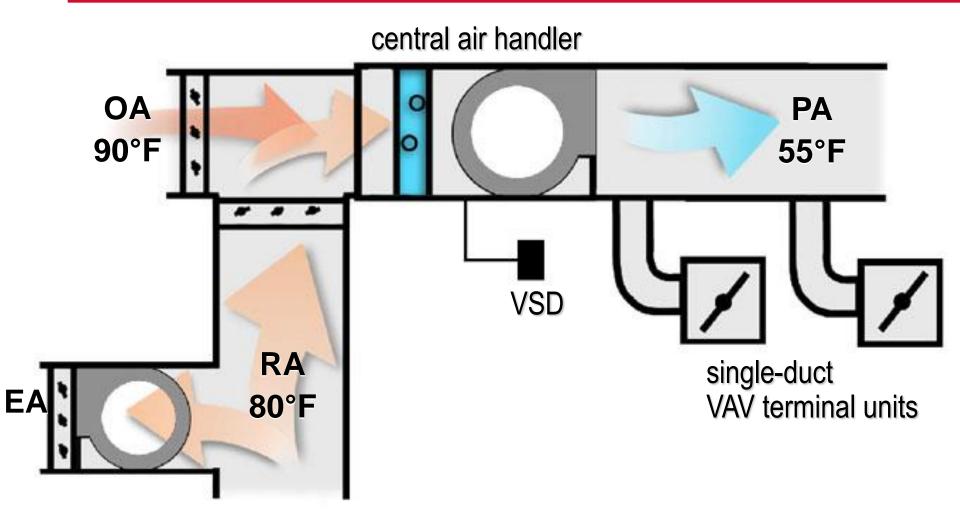
VAV terminal unit types ... Series, Fan-Powered



VAV terminal unit types ... Parallel vs. Series Fan-Powered

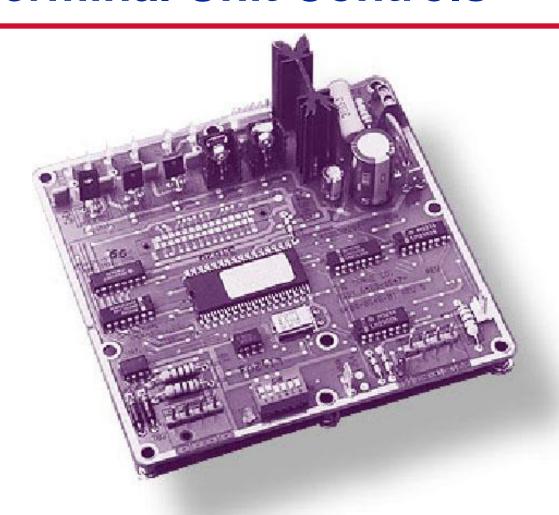
- Constant airflow to the space
- Acoustics
- Fan energy consumption

VAV terminal unit types ... Single-Duct VAV Systems



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VAV Terminal Unit Controls



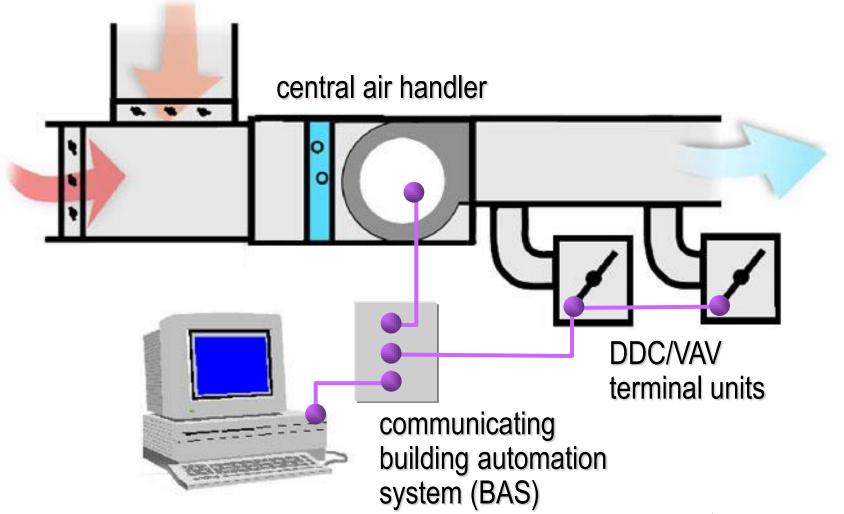
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VAV terminal unit controls ... Terminal Unit Control Technologies

- Pneumatic control
- Electronic control
- Direct digital control (DDC)

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VAV terminal unit controls ... Direct Digital Control (DDC)



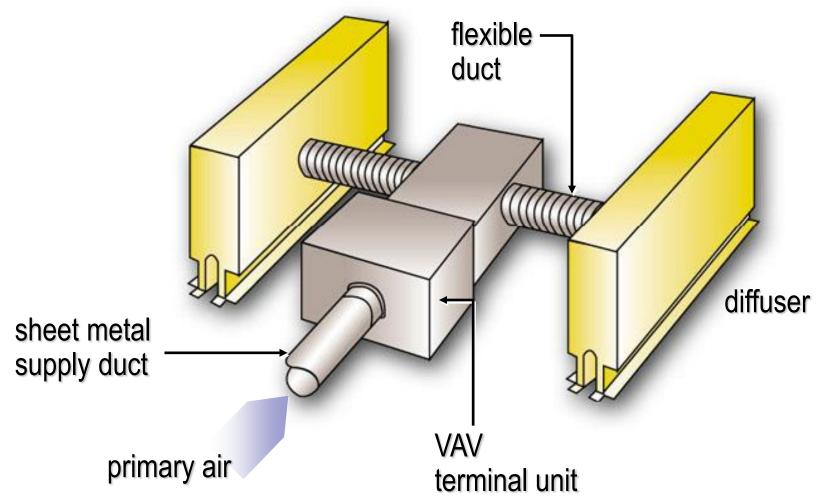
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VAV terminal unit controls ... Direct Digital Control (DDC)

- Electronic pressure transducer provides pressure-independent control
- Digital controller positions modulation device, controls terminal fan and heat source
- Minimum and maximum airflow settings adjusted through communication link
- Electric power supply operates modulation device and electronic controller

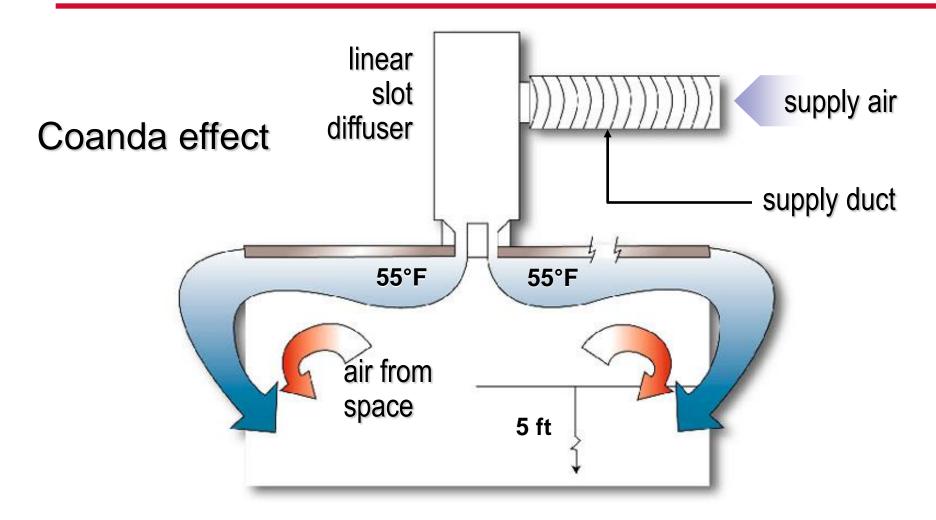
Diffusers



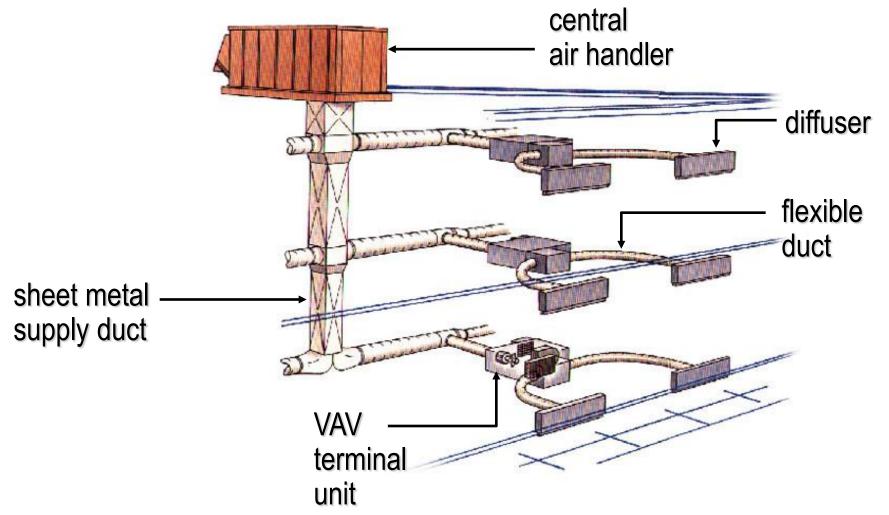
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diffusers ...

Linear Slot



Supply Duct System



Duct Design

Equal friction method

- equal static pressure drop per unit length of duct
- design can be performed by hand

Static regain method

- relatively constant static pressure throughout system
- desirable for VAV system duct design
- design often requires a computer program

Duct Design Recommendations

- Keep as simple and symmetrical as possible
- Locate main runs, branch runs, and terminal units above hallways or unoccupied spaces
- Minimize use of flexible ductwork upstream of terminal units
- Use duct lining or a duct silencer in first duct section to attenuate supply fan noise
- Place balancing dampers upstream of diffusers in all noncritical branches
- Reducing transitions should be several duct diameters upstream of terminal units

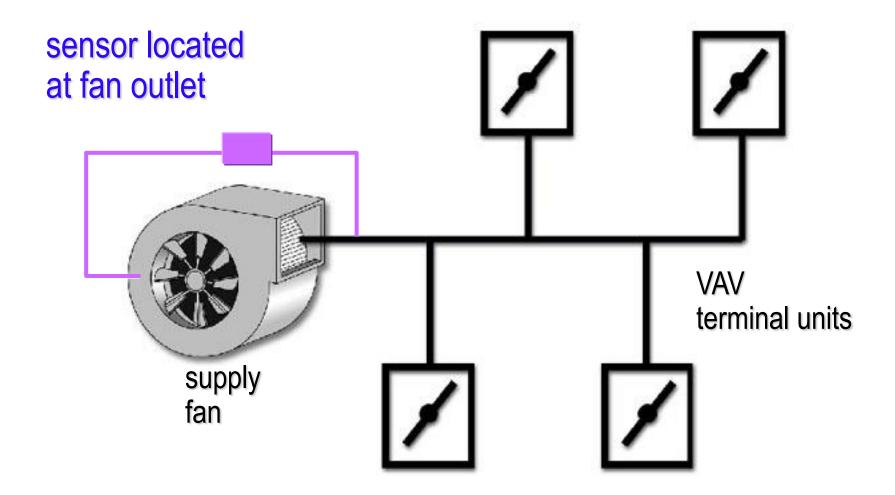
VAV Systems

period four Fan Modulation

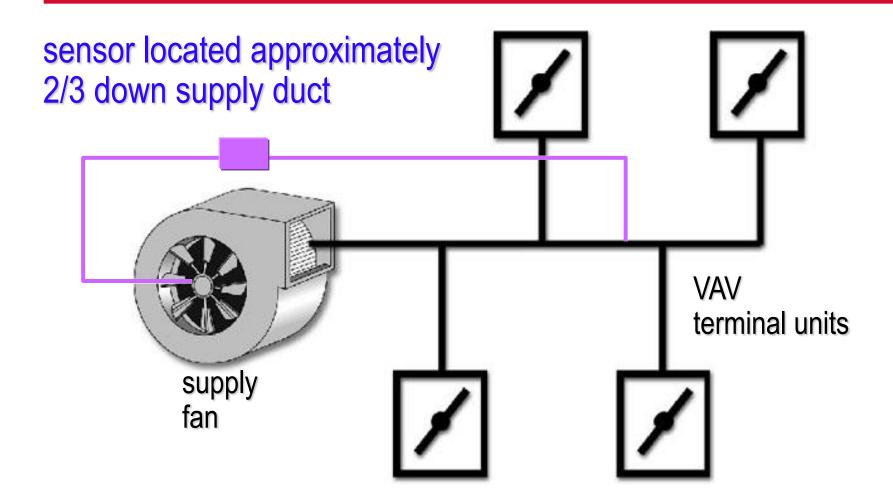
Controlling System Static Pressure

- Fan outlet static pressure control
- Supply duct static pressure control
- Optimized static pressure control

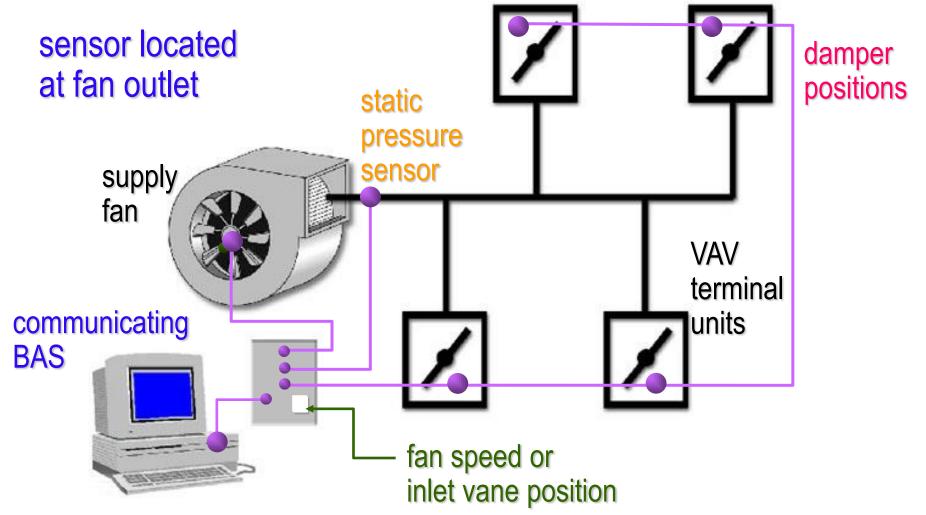
Fan Outlet Static Pressure Control



Supply Duct Static Pressure Control



Optimized Static Pressure Control



VAV Systems

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