

# Introduction to Electro-hydraulic Proportional and Servo Valves



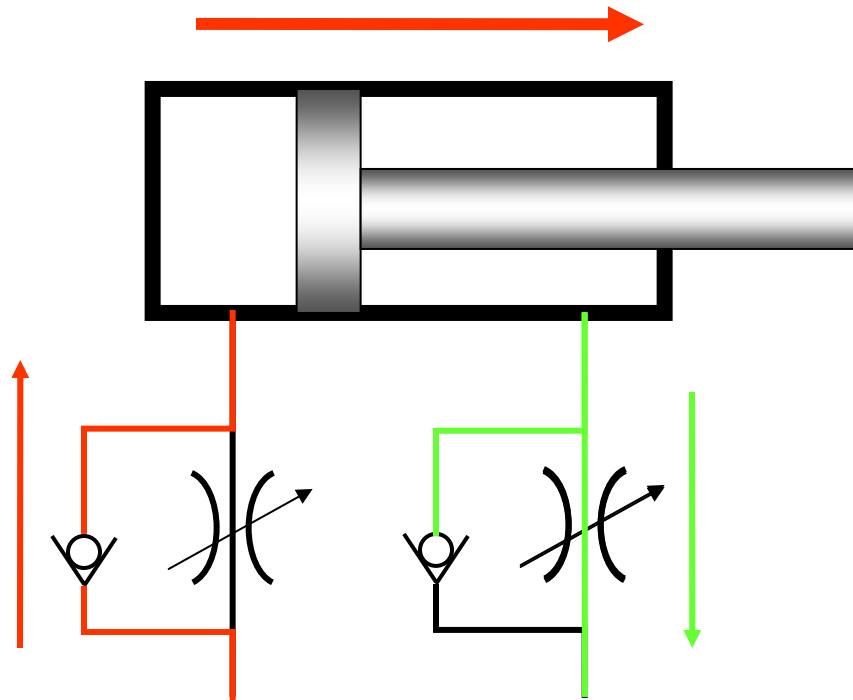
ENGINEERING YOUR SUCCESS.

# Proportional Valves

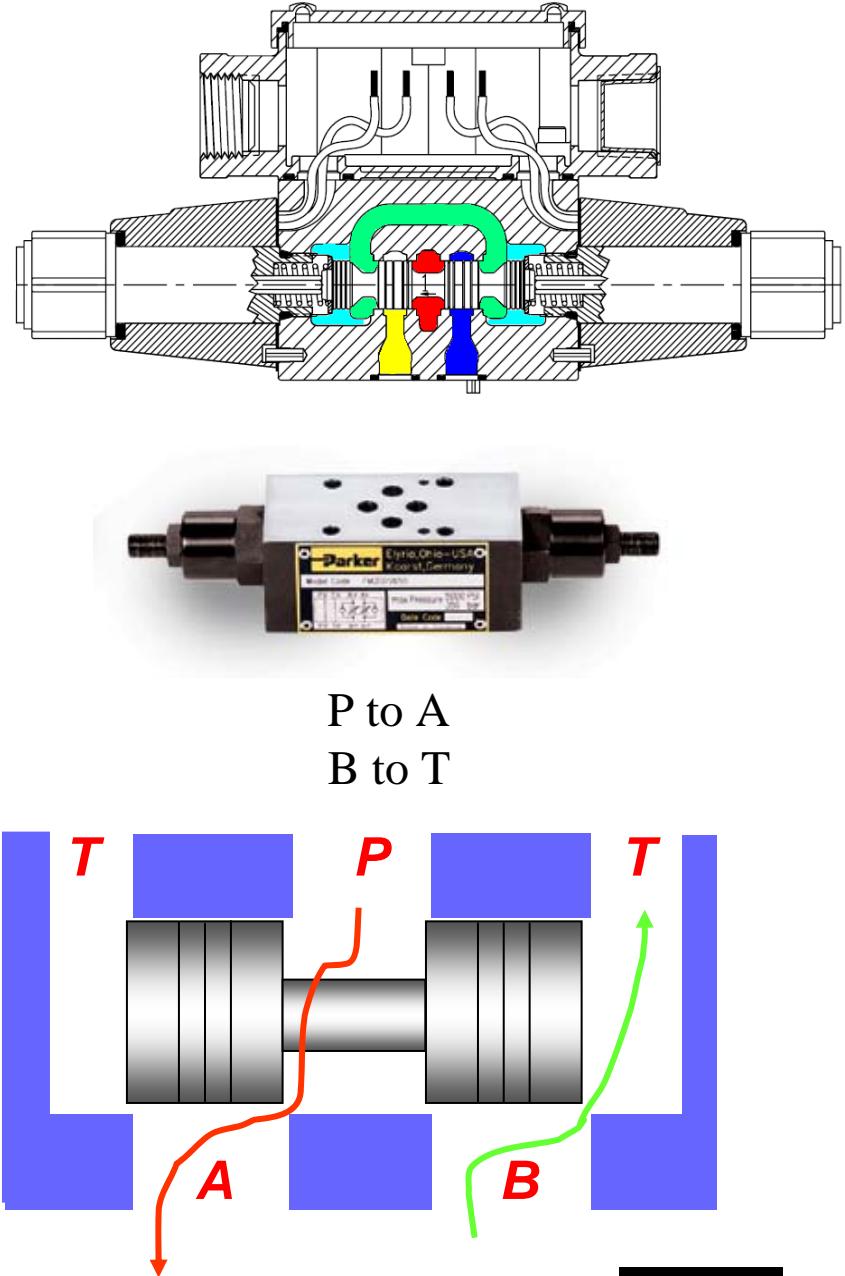
## Servo Valves

	Mobile bankable Style, Threaded Cartrdige Style	NFPA Mounting Without Spool Position Feedback	NFPA Mounting With Spool Position Feedback	Servo Performance, Closed Loop Valves with Spool Position Feedback	Servo Valves With either Mechanical or Electrical Feedback (spool position).
Parker Models	Pulsar VP, VPLs. 	D*FWs, D*FTs'	D*FXs'	D*FP D*FHs D*1FH  	BDs' DYs' SEs' 
Mechanical Construction (spool shift)	Electro-Hydraulic Pilot, Mechanical (spring) Return	Electromagnetic Force, Mechanical (spring) Return	Electromagnetic Force, Mechanical (spring) Return	Electromagnetic Force, Mechanical (spring) Return	Torque Motor Pilot Control Balance PSI spool Control
Hysteresis	3.....7%	3...7%	0.03.....1%	0.1....5%	0.1.....0.5%
Frequency Response	< 10 Hz	10...50 Hz	10....70 Hz	50...150 Hz	100...200 Hz
Center Lap Condition	Overlap 5...20%			Zero Overlap	Zero Overlap
Operating Pressures Limits (Drop across metering edge).	1000...2000 PSI			1000...3000 PSI	None
Applications	Open Loop Control		Low End Closed Loop Position	Closed Loop Position & Force	Closed Position 

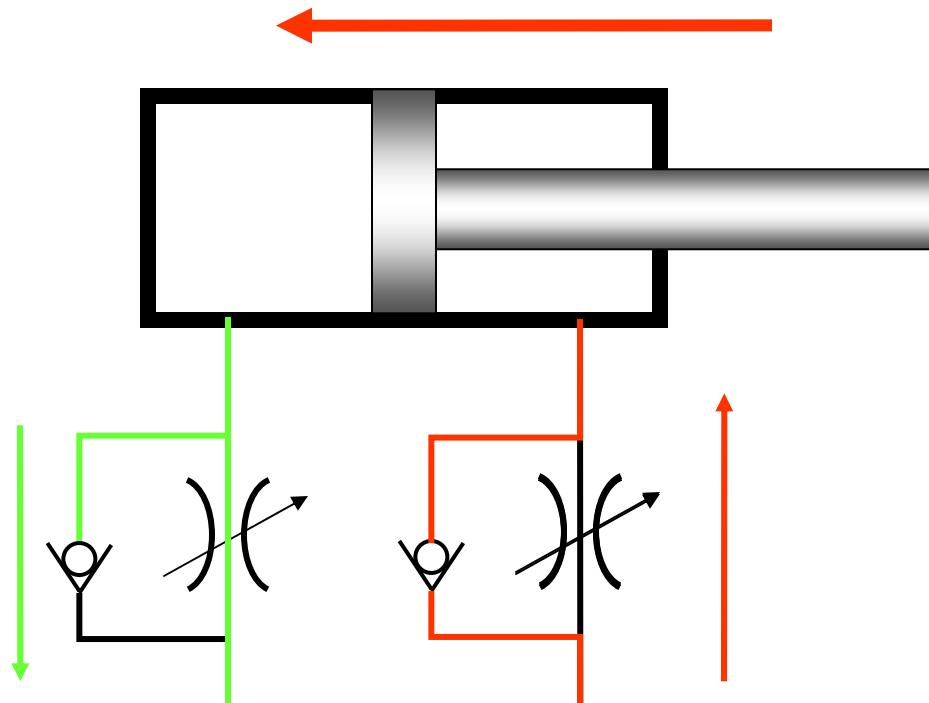
# Meter Out Circuit



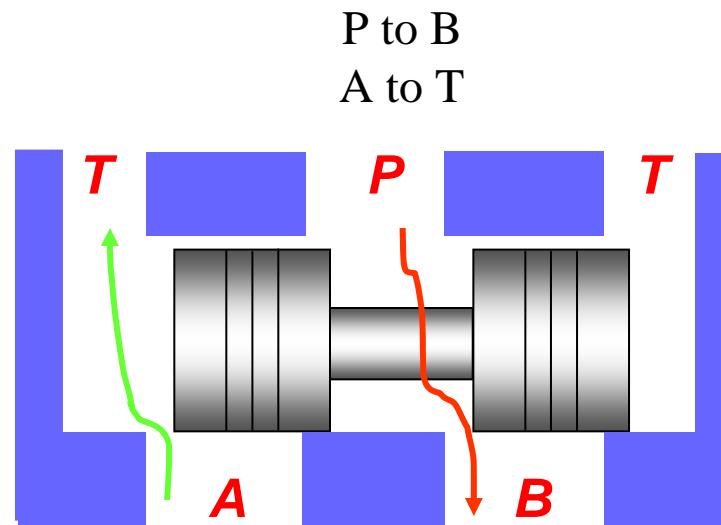
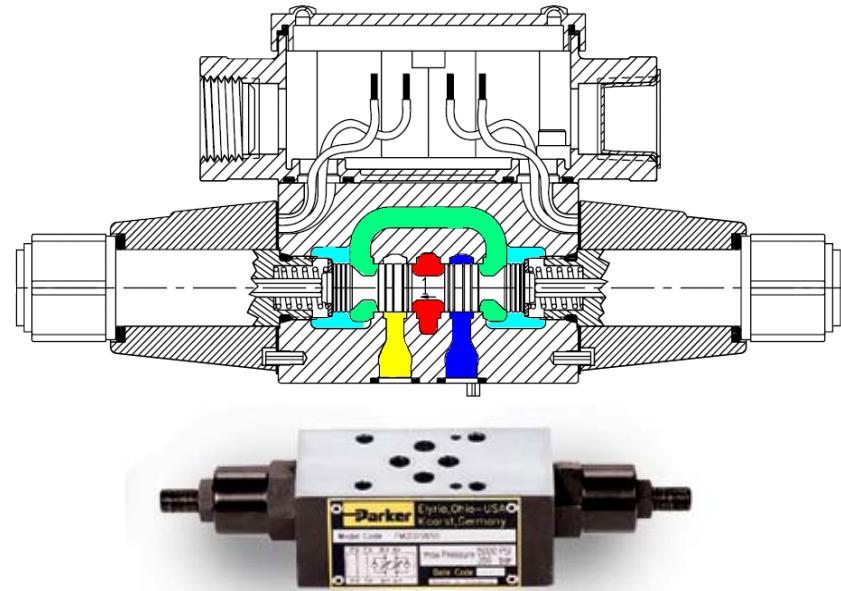
*Free flow into the “cap” end  
Metered Flow out the “head”  
end.*



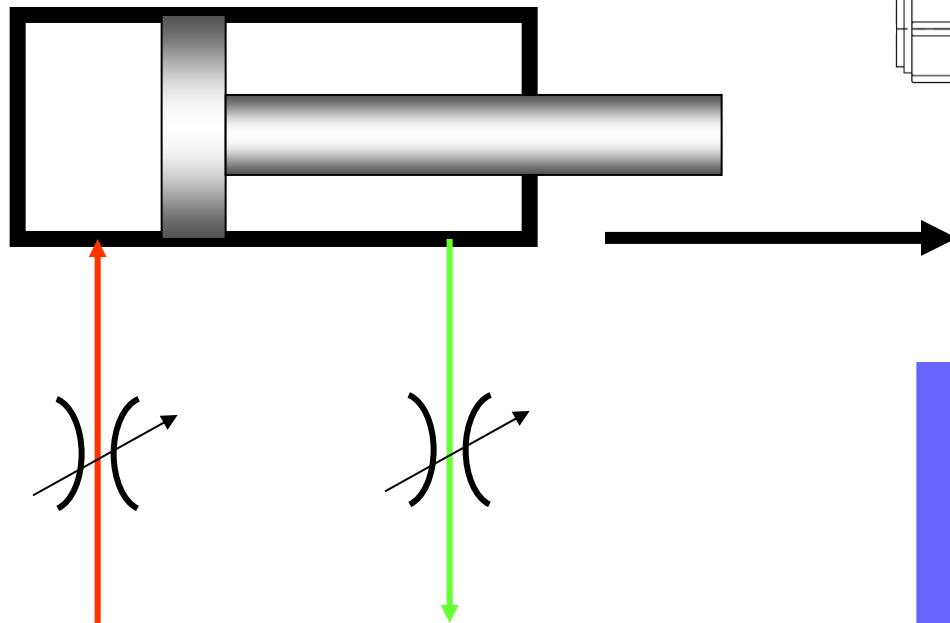
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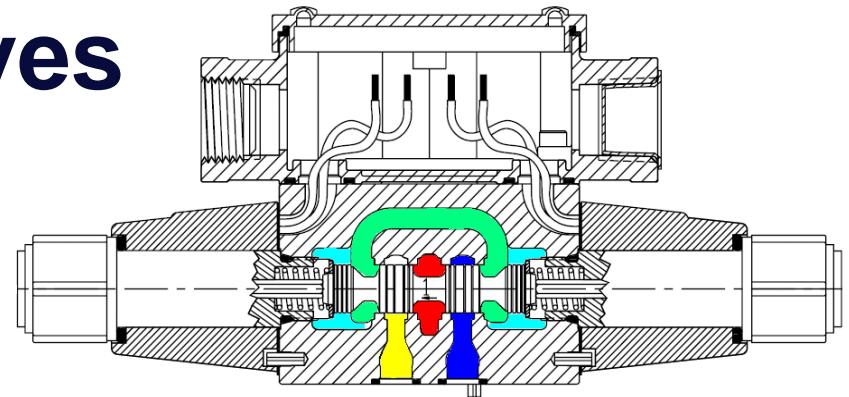
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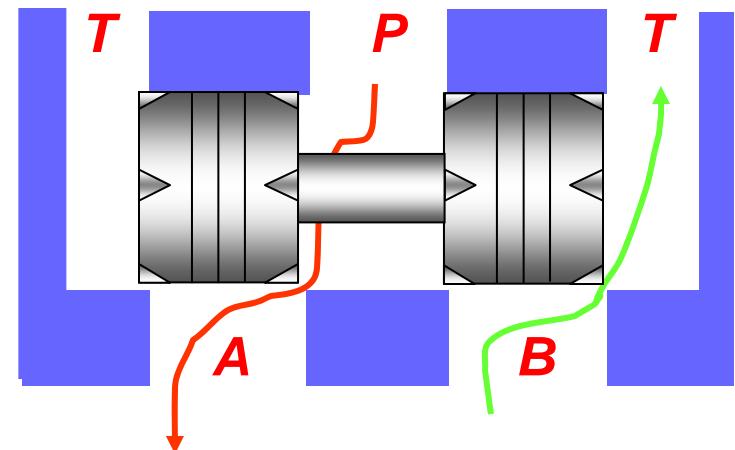
# Electrohydraulic Valves



Shift Spool slightly to create  
metering Orifice

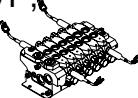


P to A  
B to T



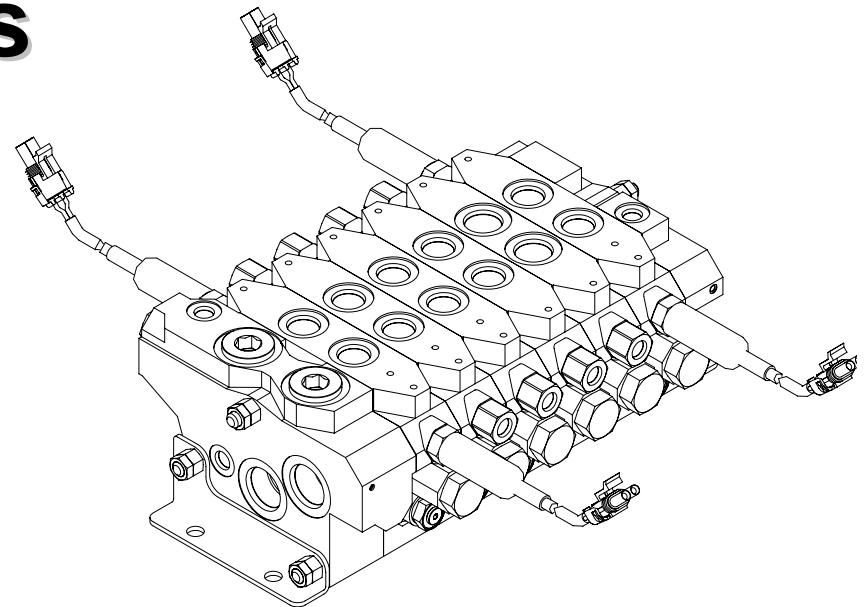
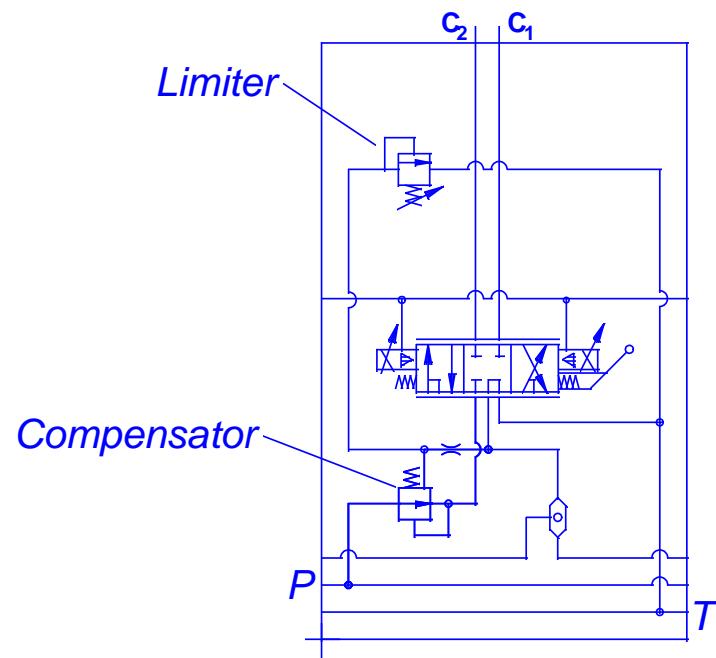
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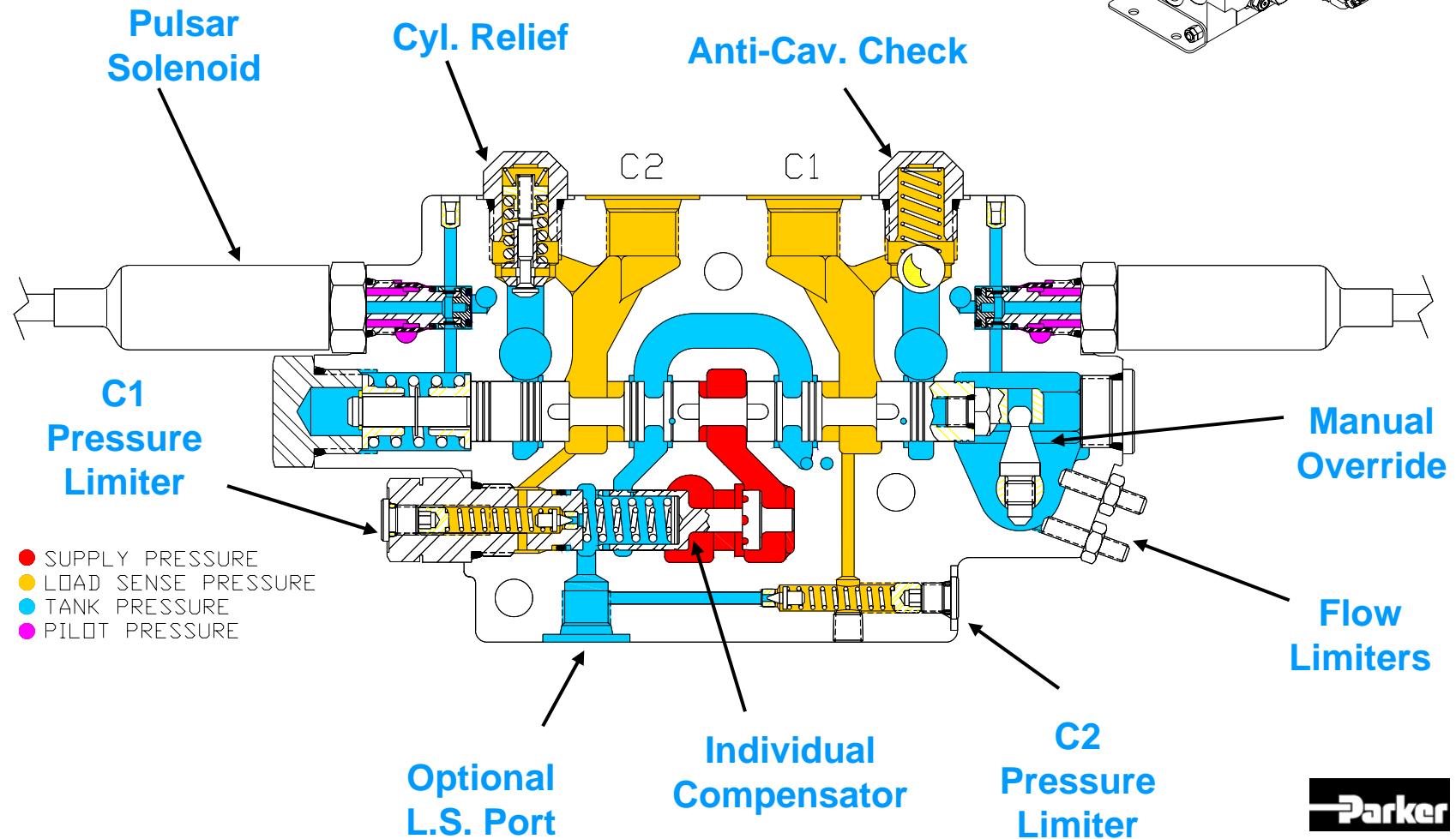
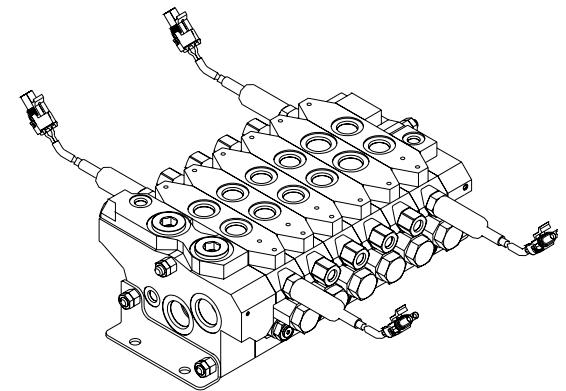
# Sectional Valves

## VPL Series



# Sectional Valves

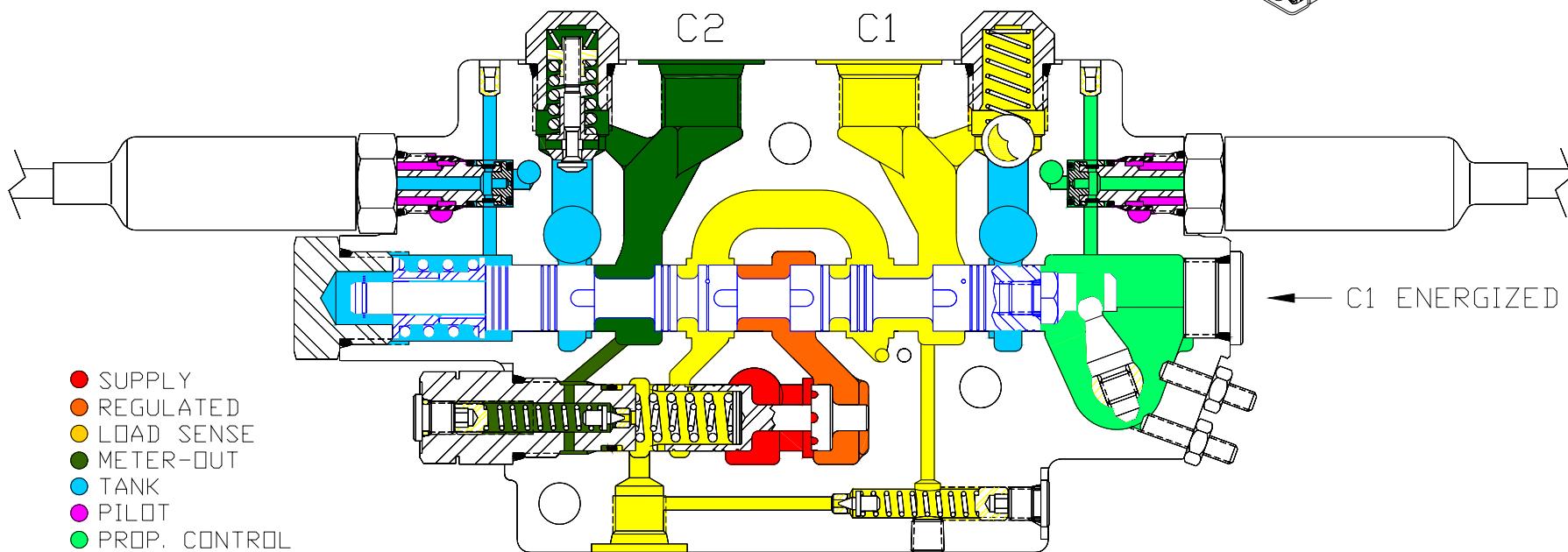
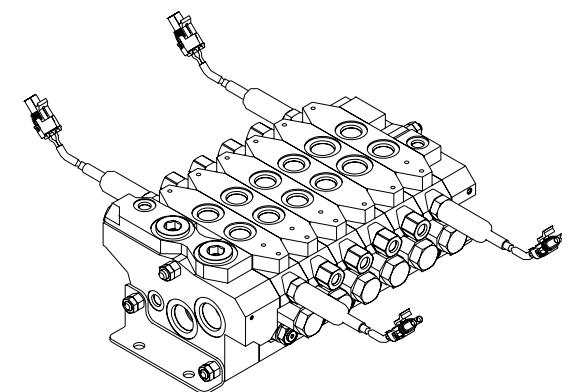
## VPL Series



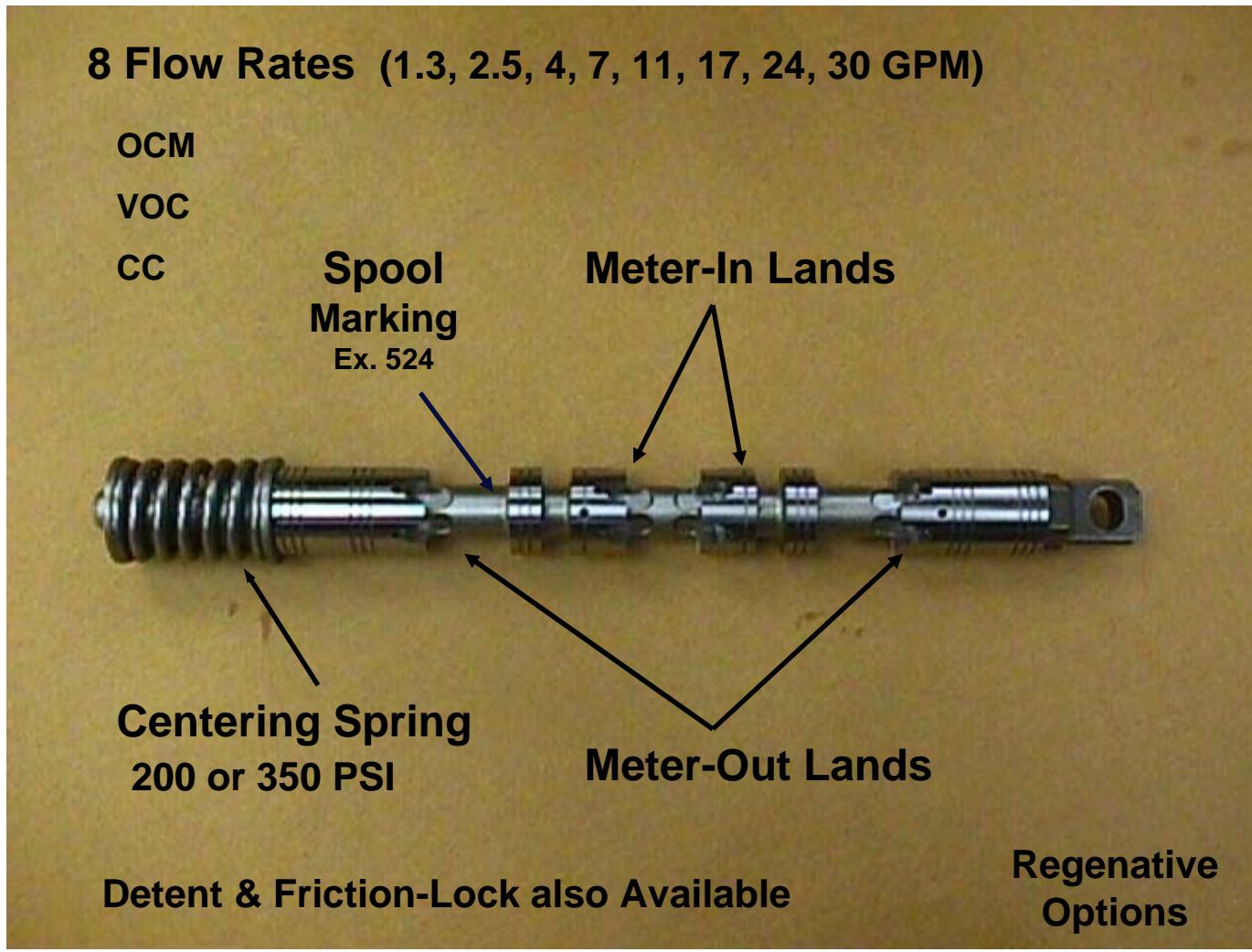
**Parker**

# Sectional Valves

## VPL Series (C1 Energized)

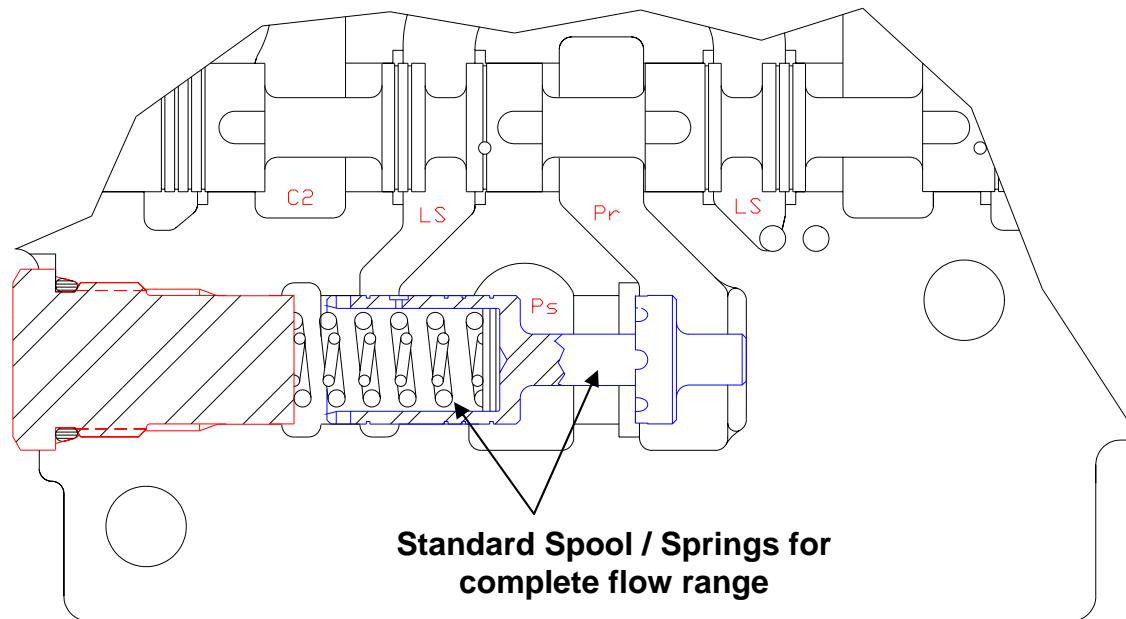


# VPL Main Spool

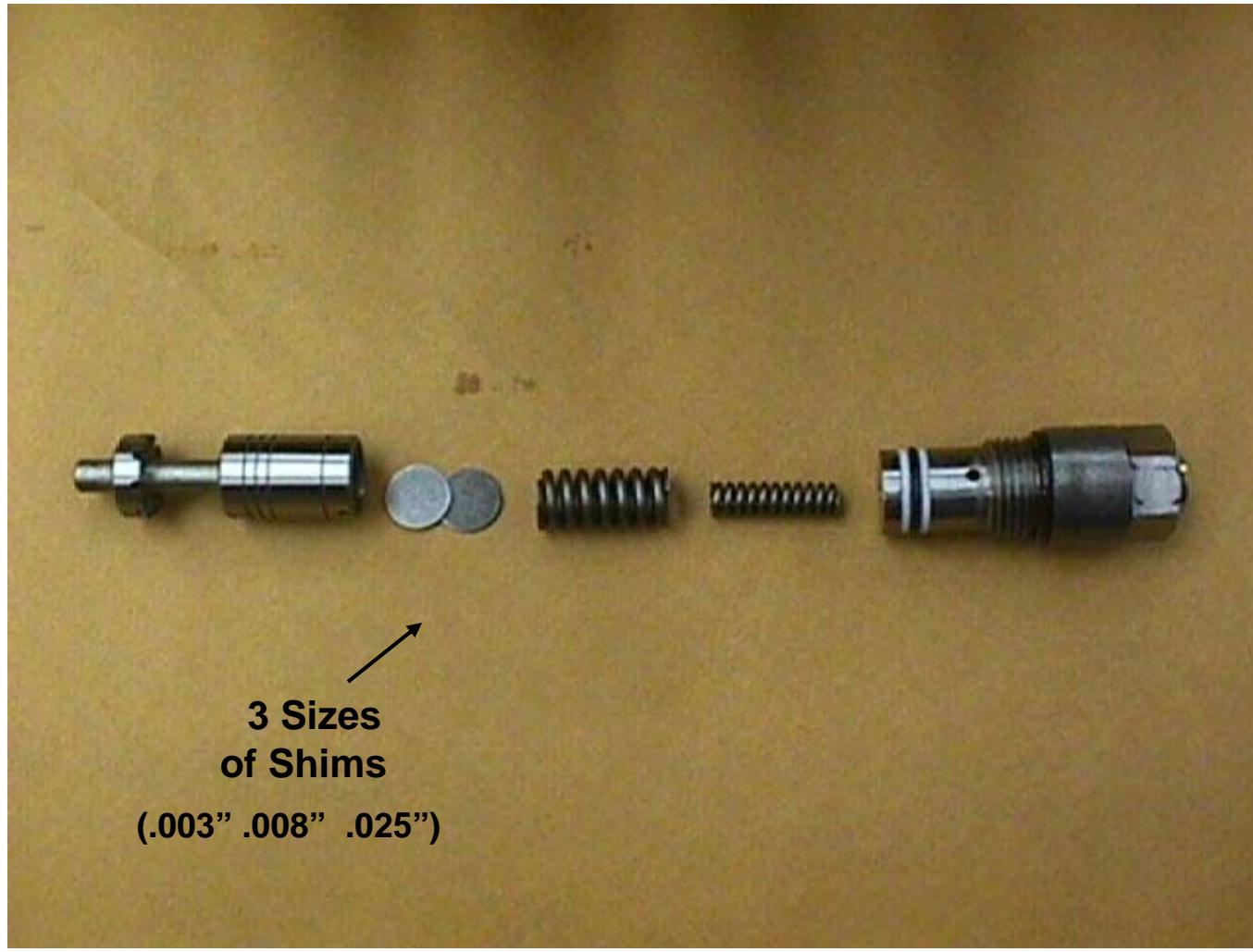


# Compensation, VPL Series

- Pressure compensation maintains a constant flow regardless of pump pressure, load pressure, or any other load in the system
- This means when running multiple sections at the same time, there will be no change in speed



# VPL Compensator Components

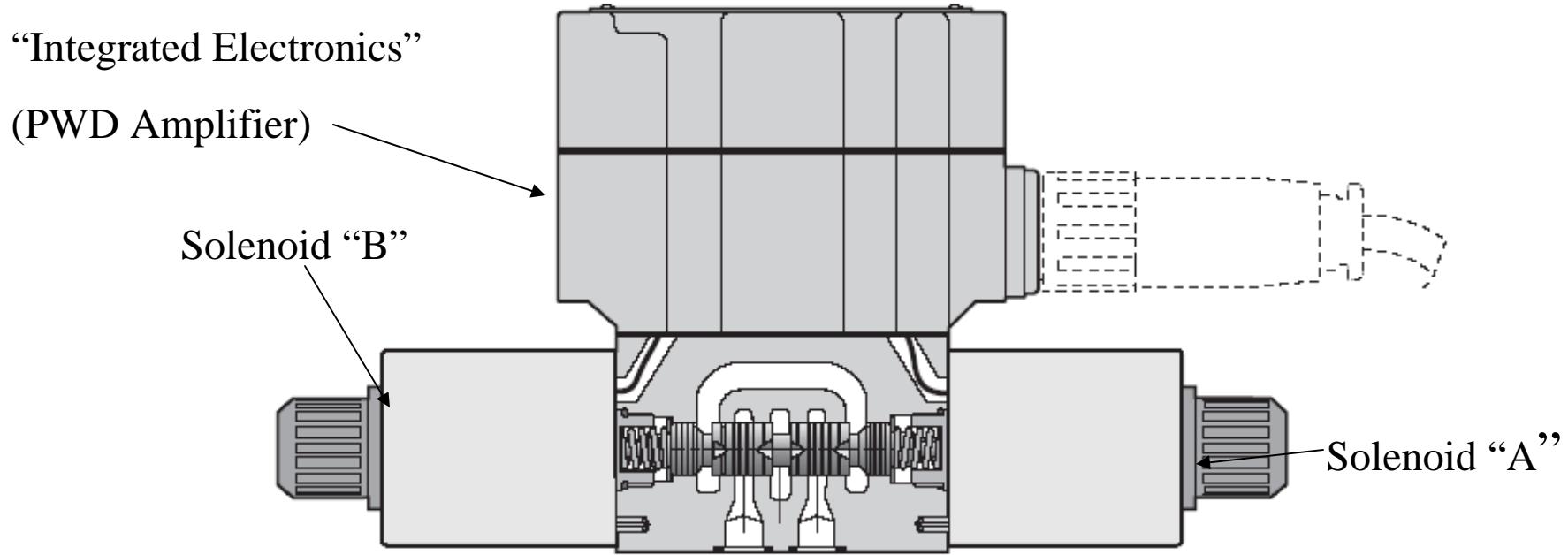


# Proportional Valves

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# How does it Work? (No Spool Feedback)



"Command Signal" based upon a % of Maximum. Typically 0 to +/- 10 VDC.

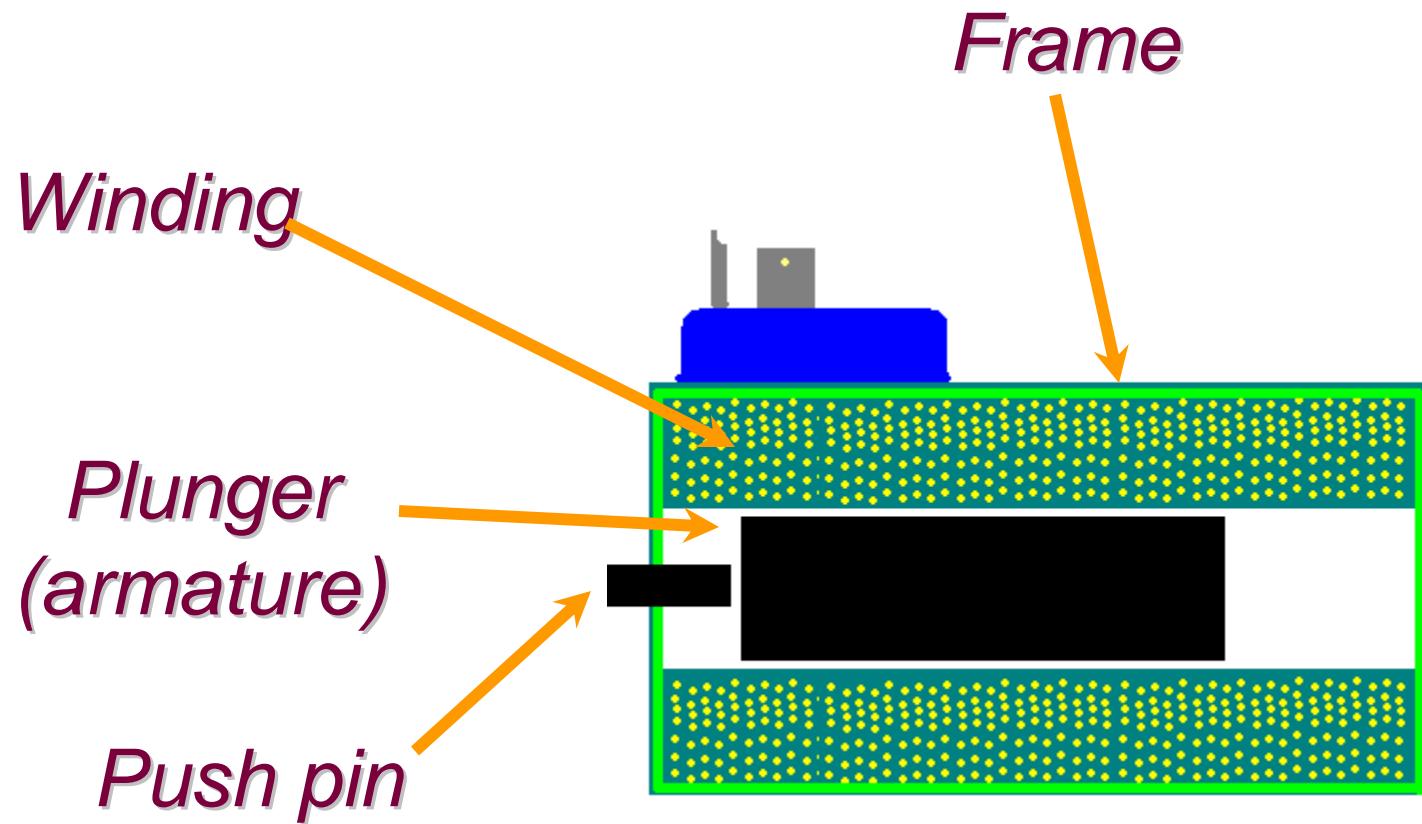
Amplifier converts Voltage (Command) into proportional Current (Typically 0..2.1 Amps).

Variable DC current into solenoid assembly produces Electromagnetic Force, proportional to current applied.

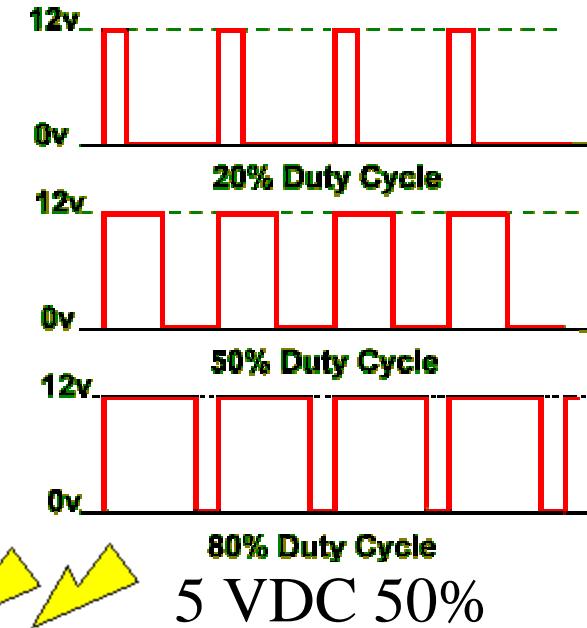
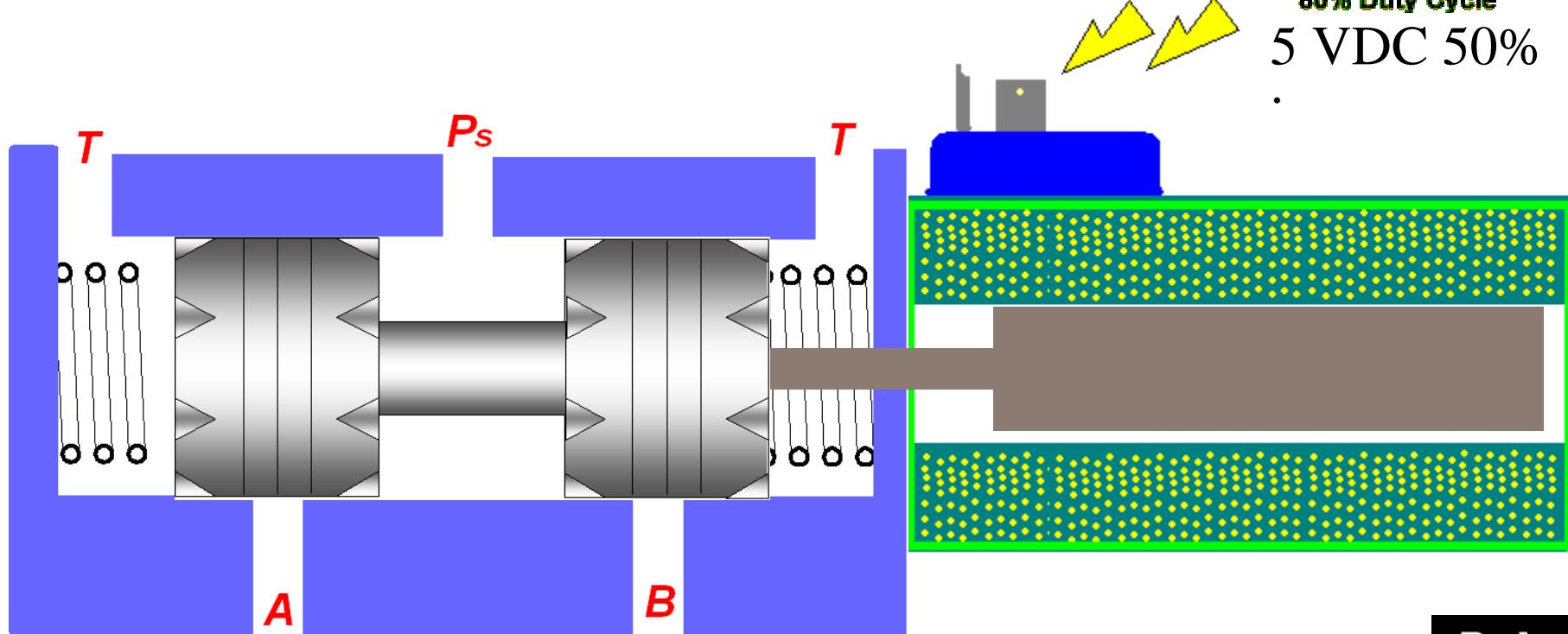
By matching Opposition Spring Force to Solenoid Force, Proportional Spool Movement is obtained.



# How does it Work? Proportional Solenoid Construction



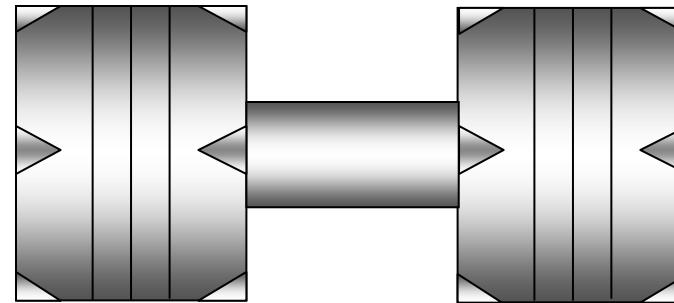
# How does it Work? Solenoid Operation



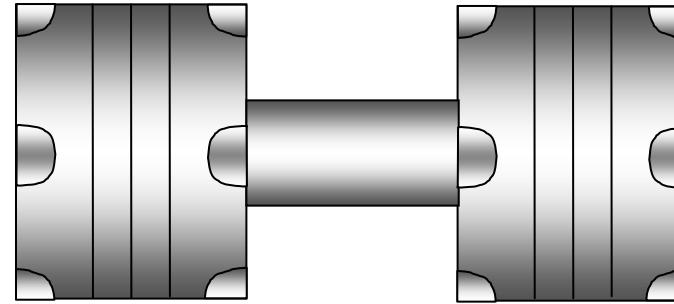
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# Proportional Valve Spool Designs

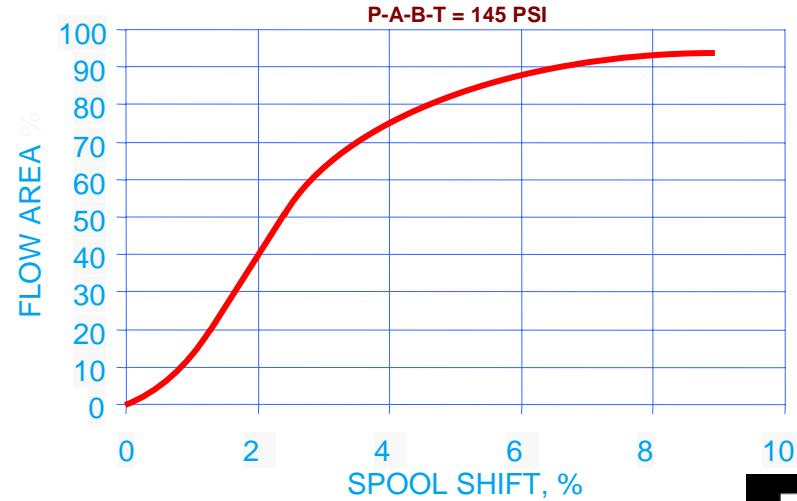
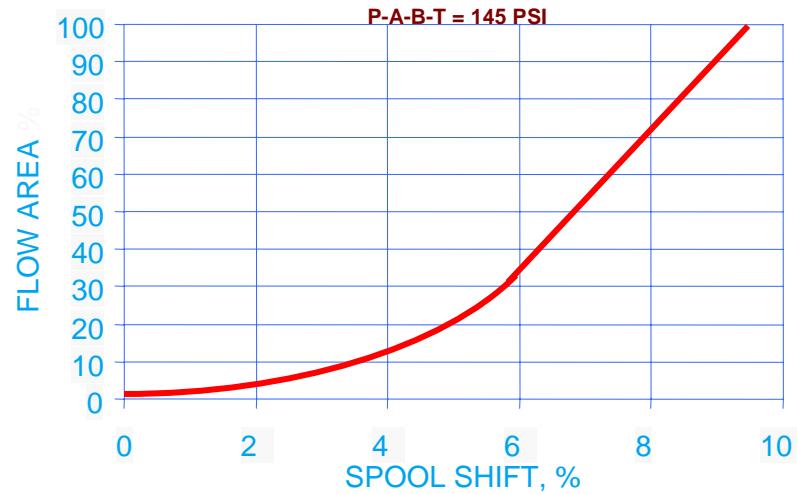
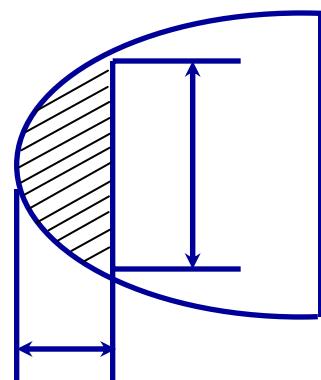
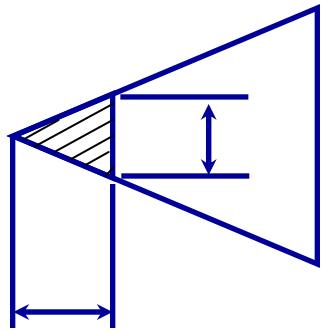
*V-Notch Spool*



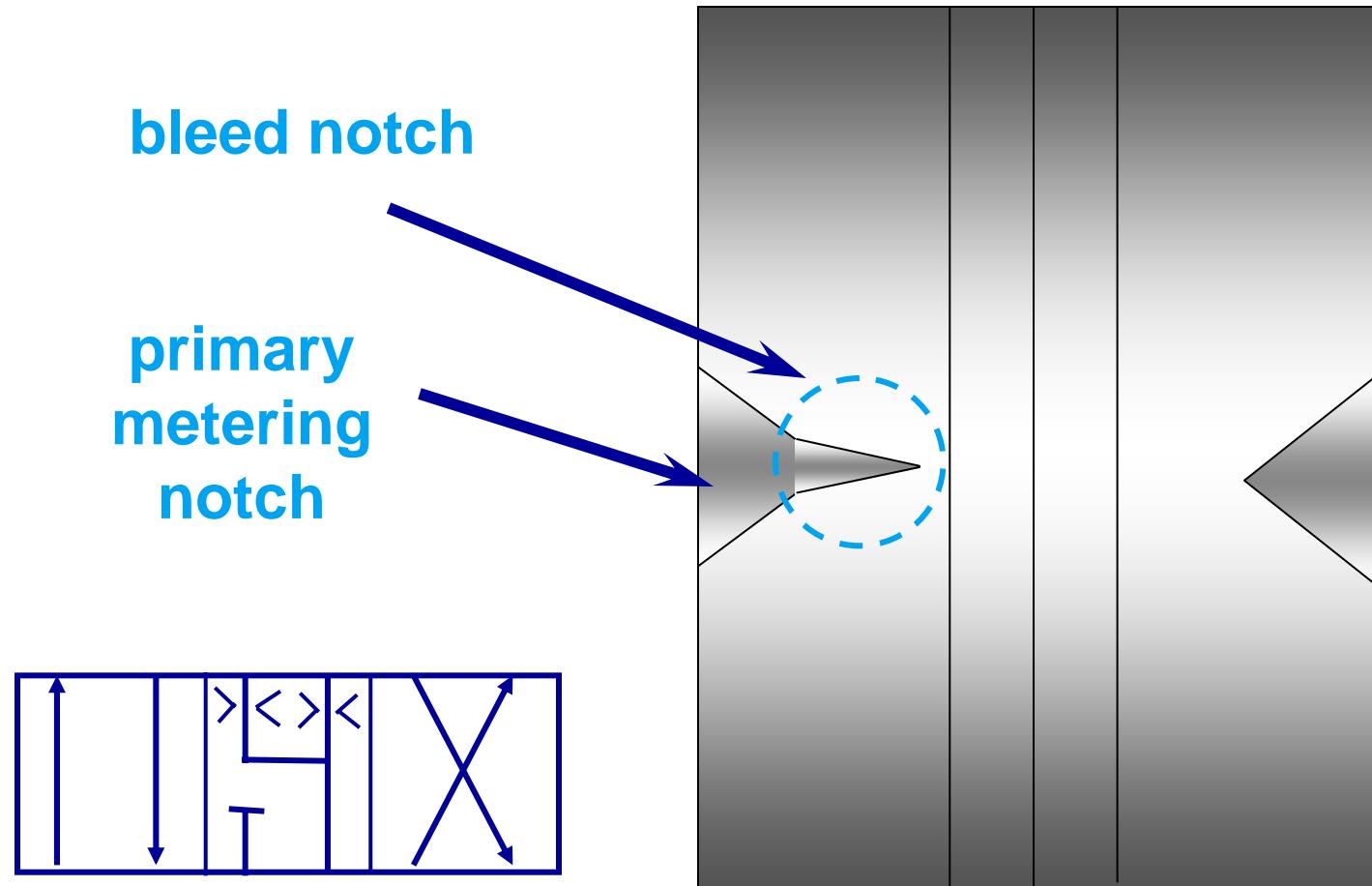
*C-Notch Spool*



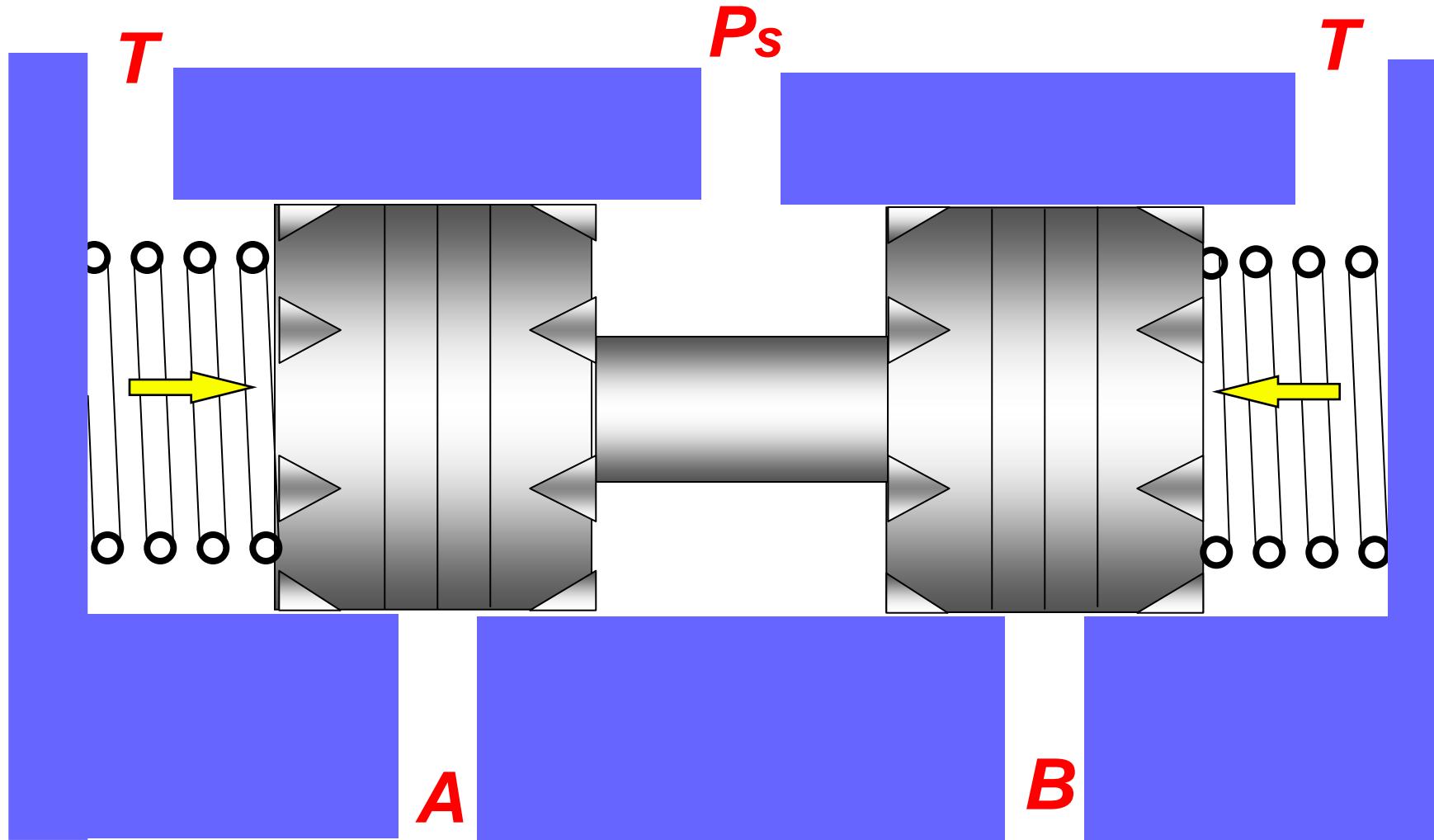
# Proportional Valve Spool Designs



# V-Notch - Bleed Center

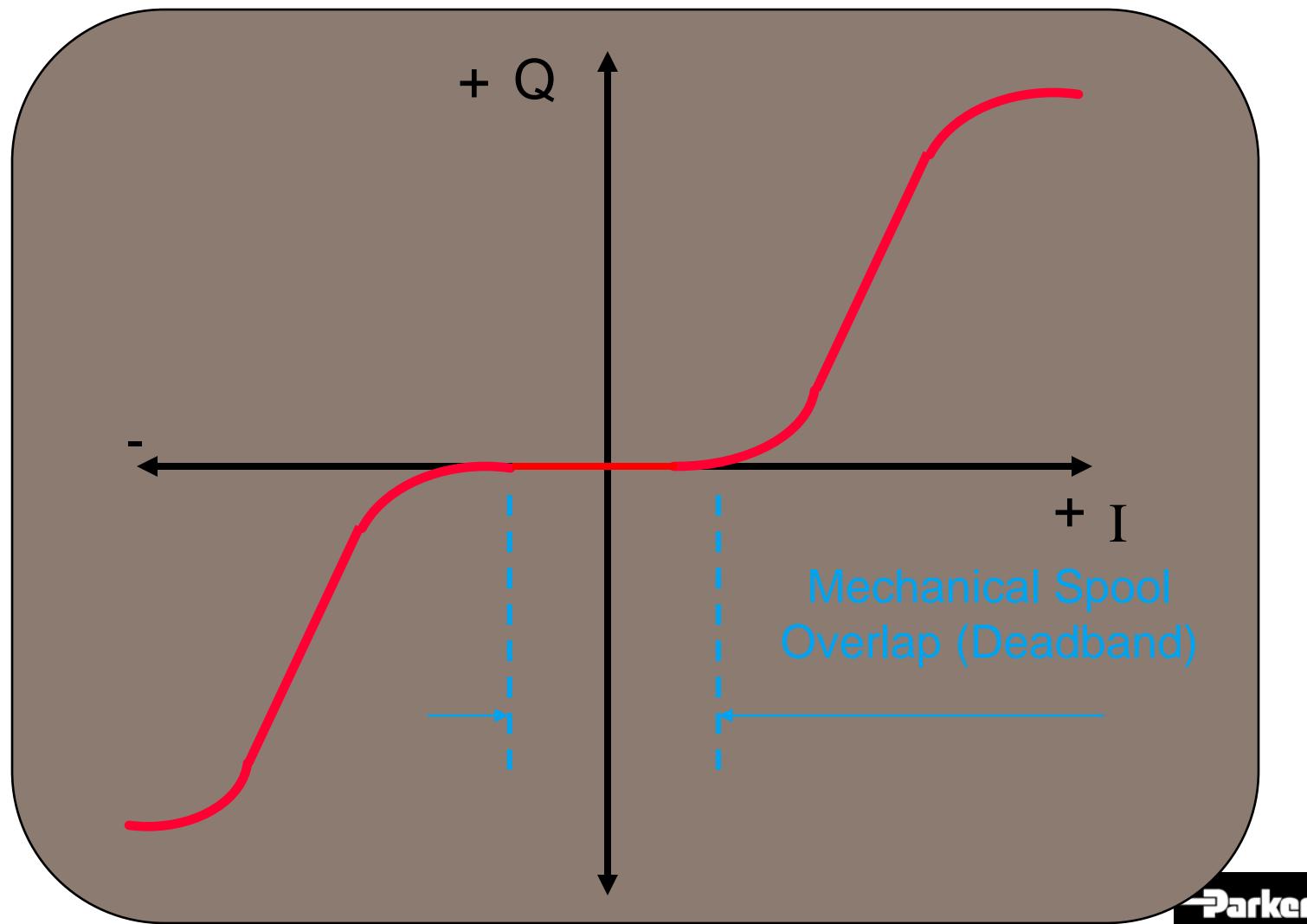


# *Proportional Valve Deadband*

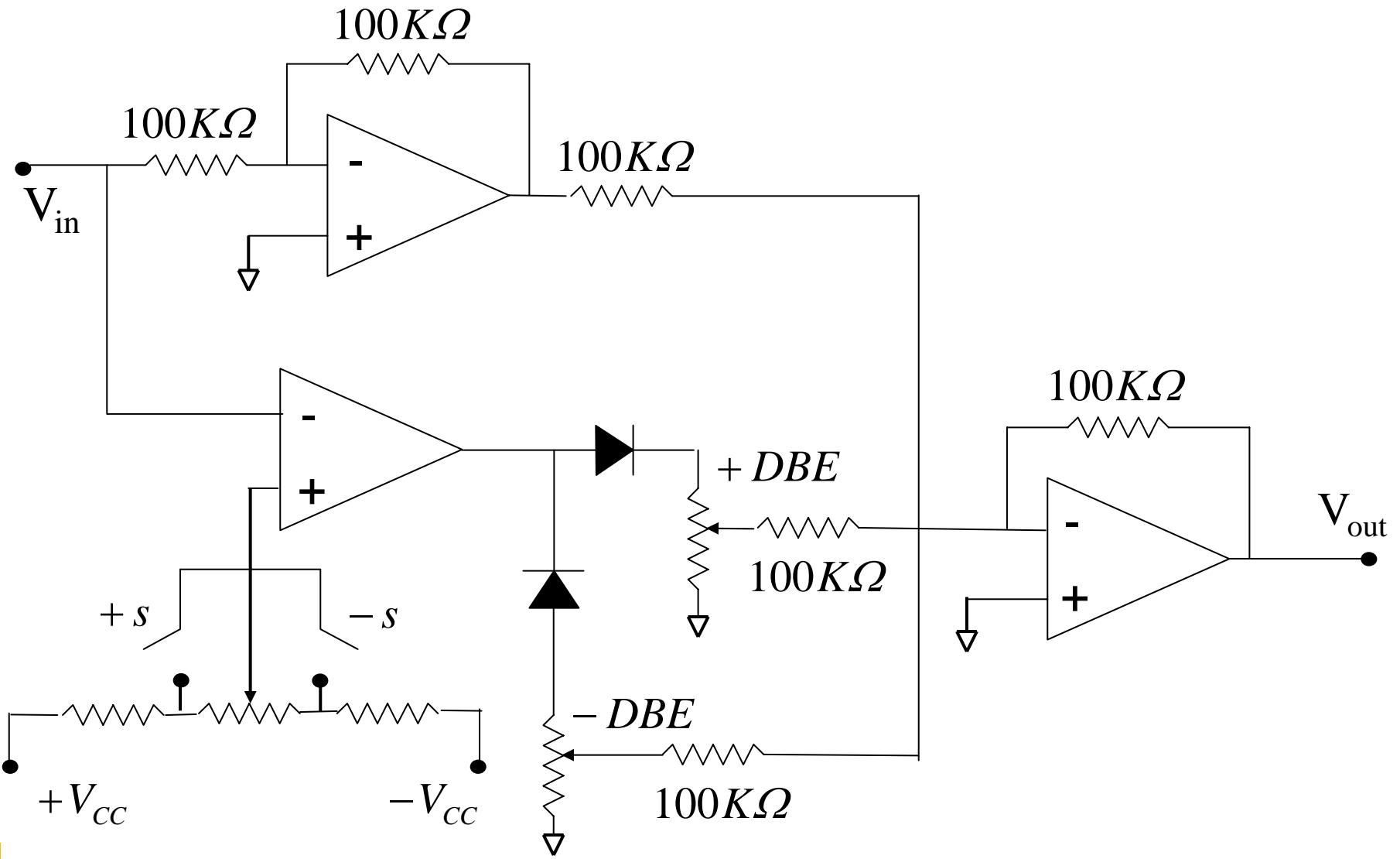


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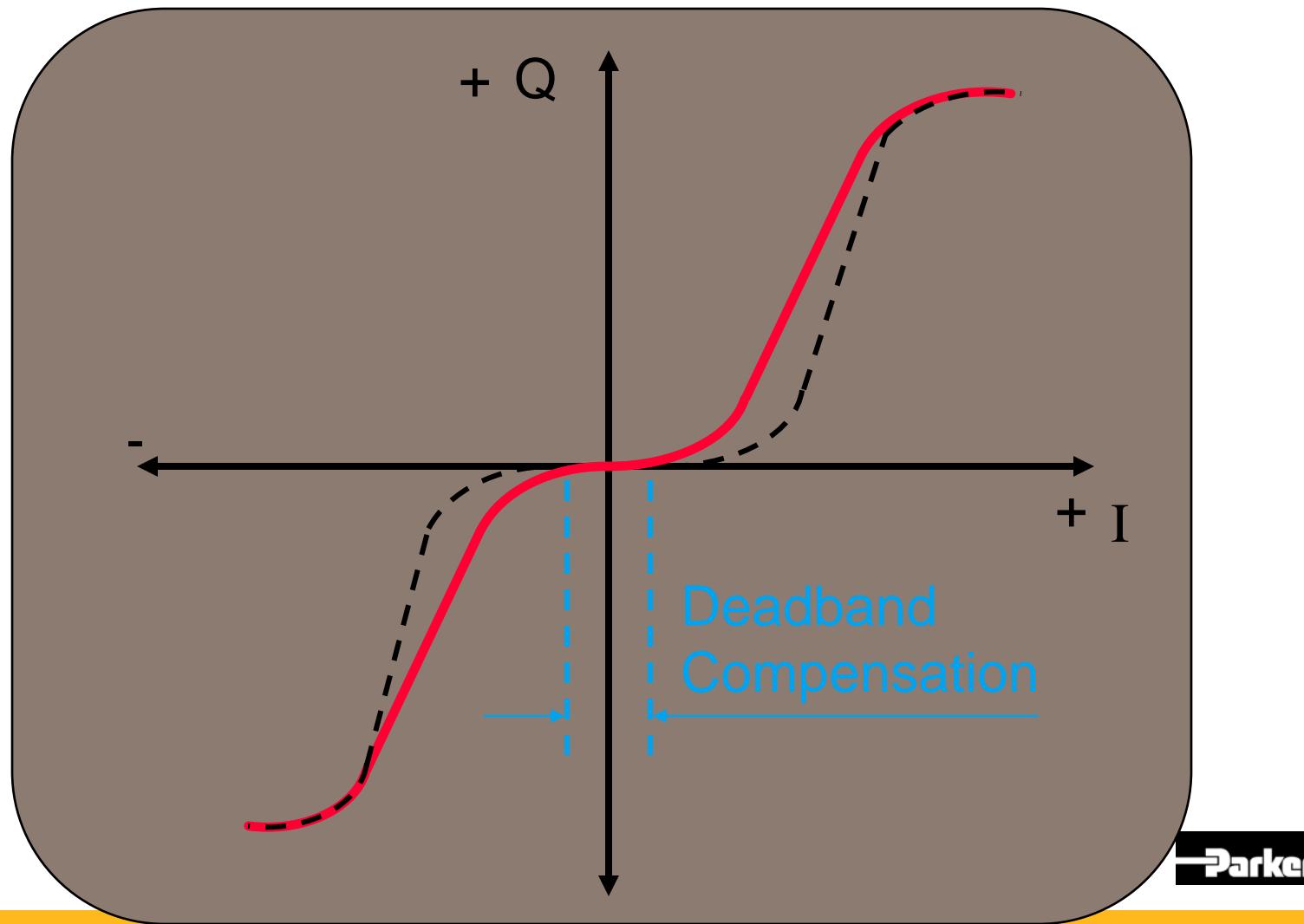
# Positive Overlap + Spring Force



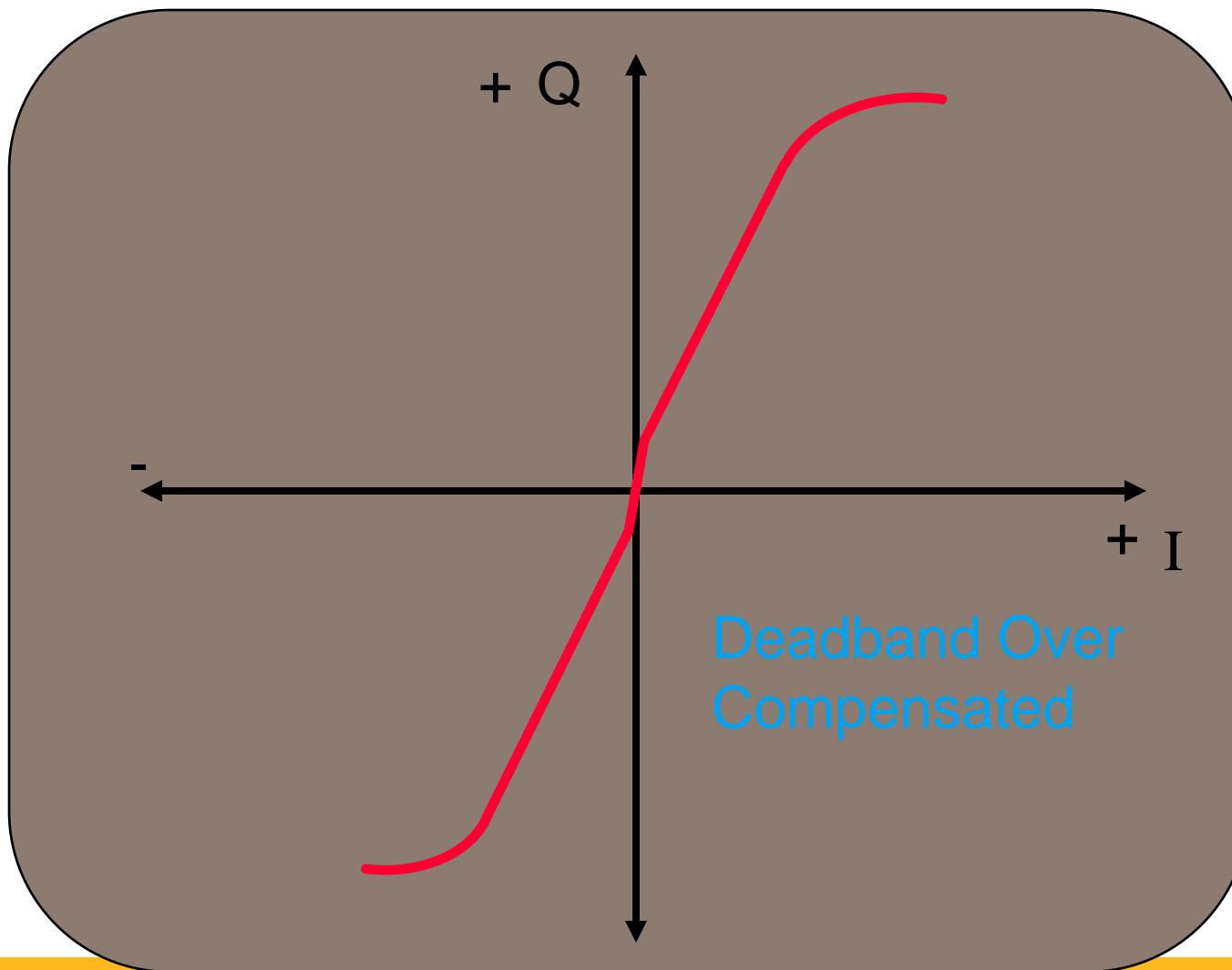
# Deadband Eliminator



# Deadband Compensation

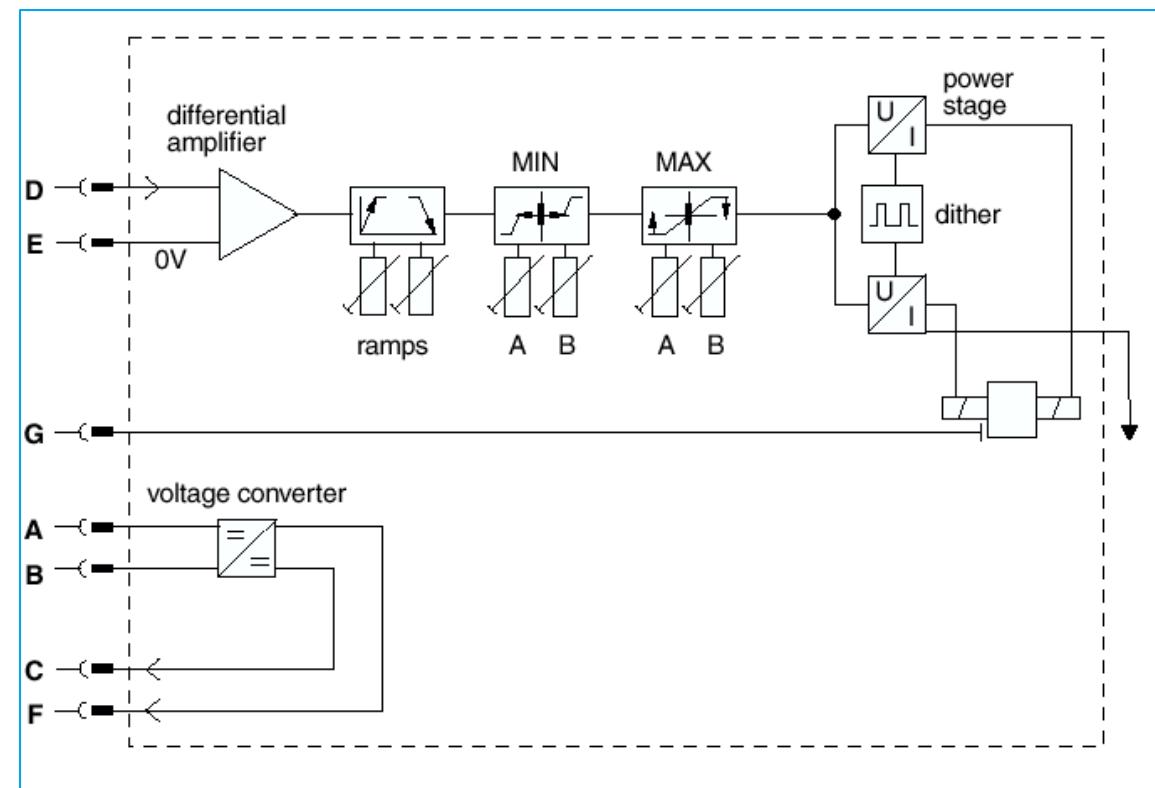
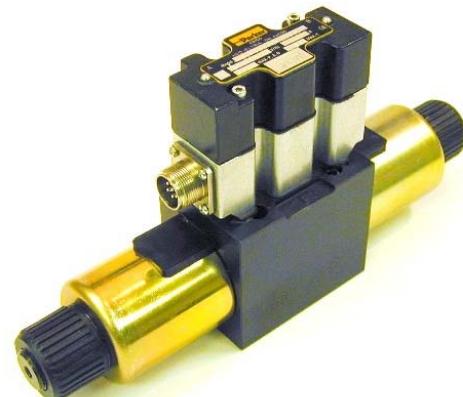
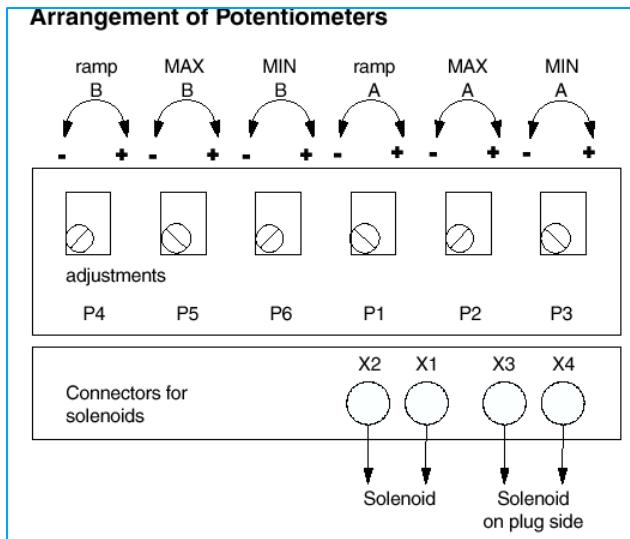


# Deadband Compensation

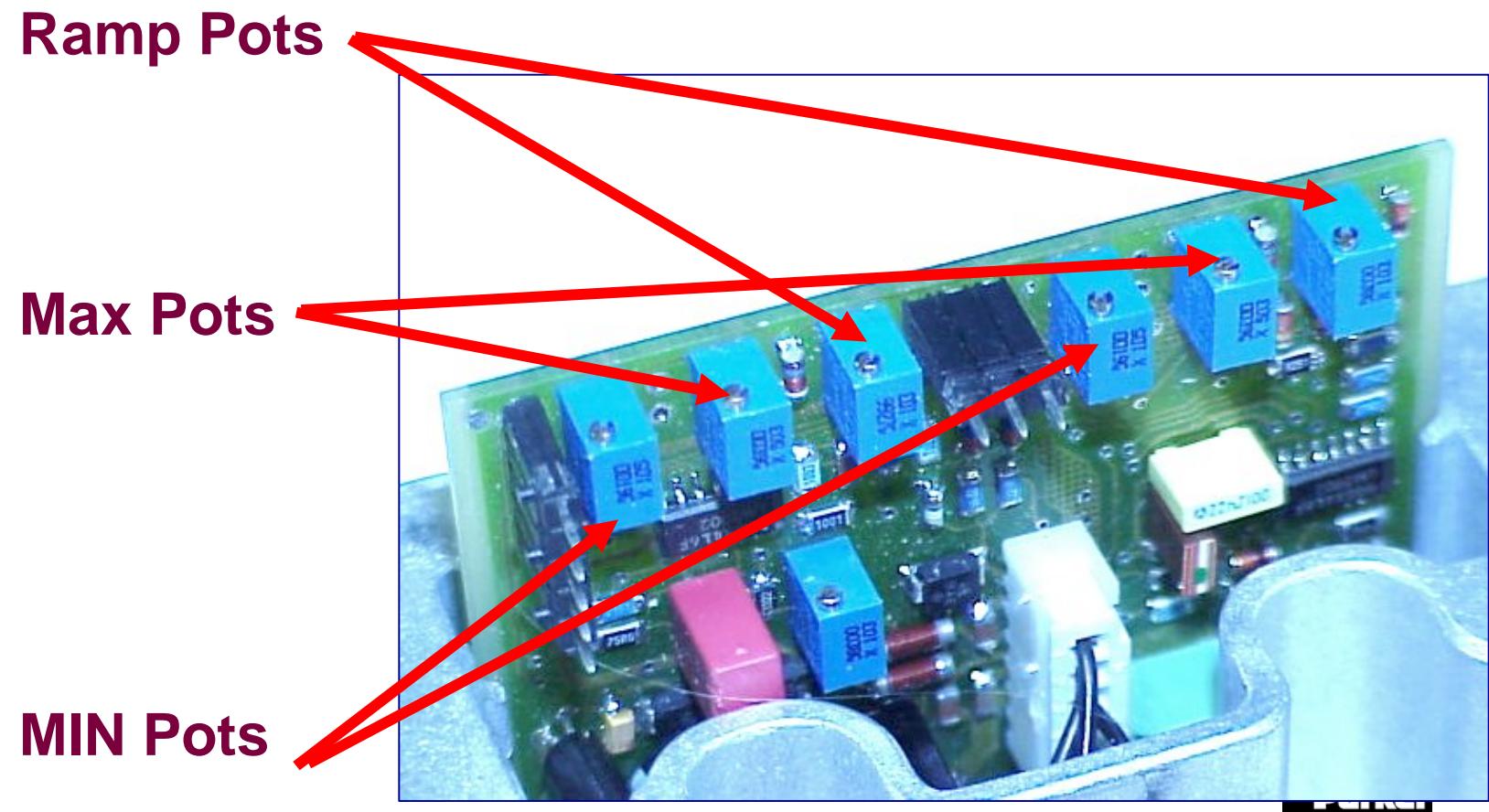


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# Valve Drivers (Open Loop) On Board Integrated Electronics

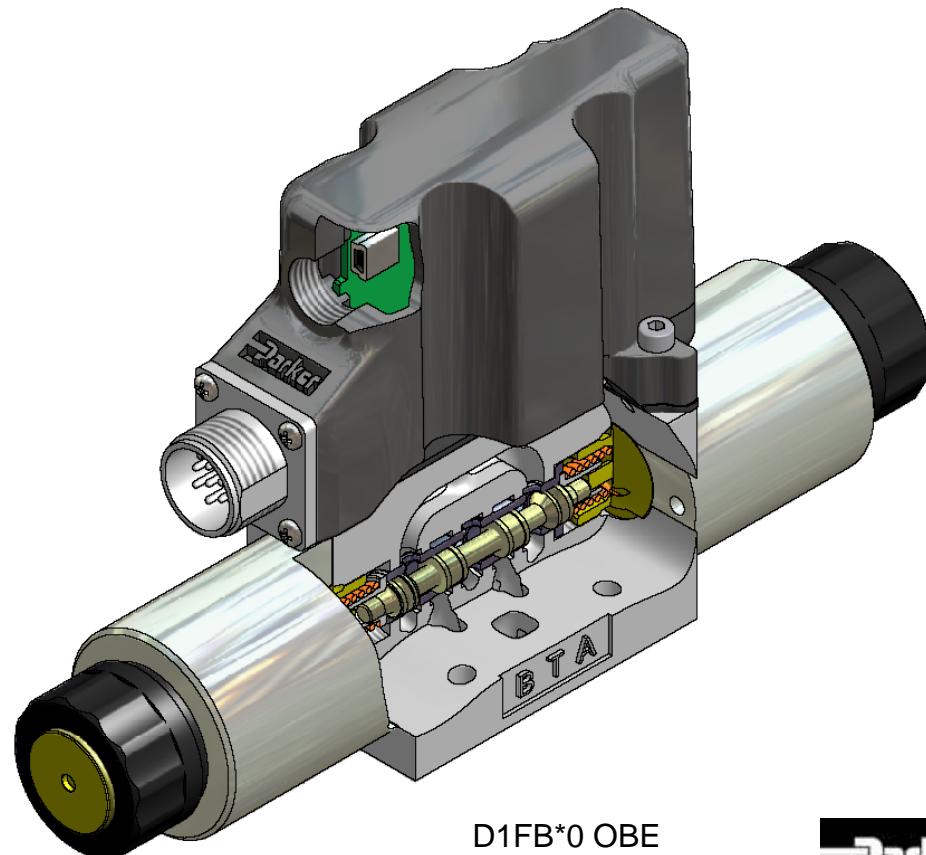
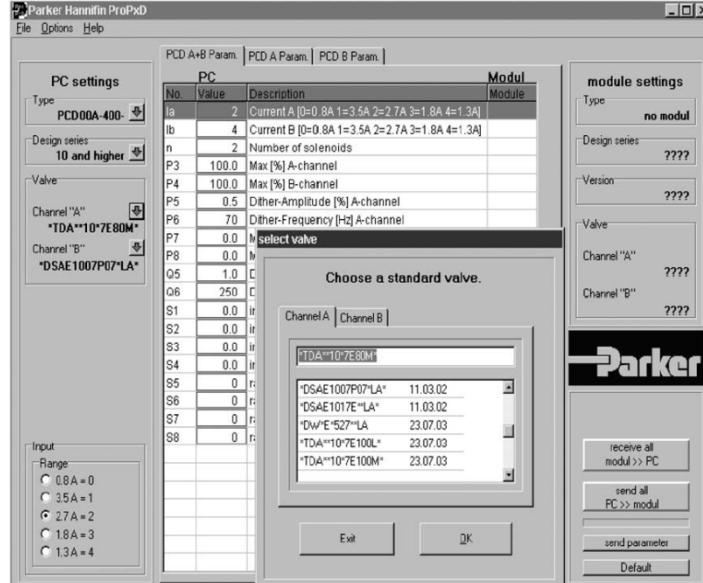


# Valve Drivers (Open Loop) On Board Integrated Electronics (Pot adjustments)



# Valve Drivers (Open Loop) On Board Integrated Electronics (PC Adjustments)

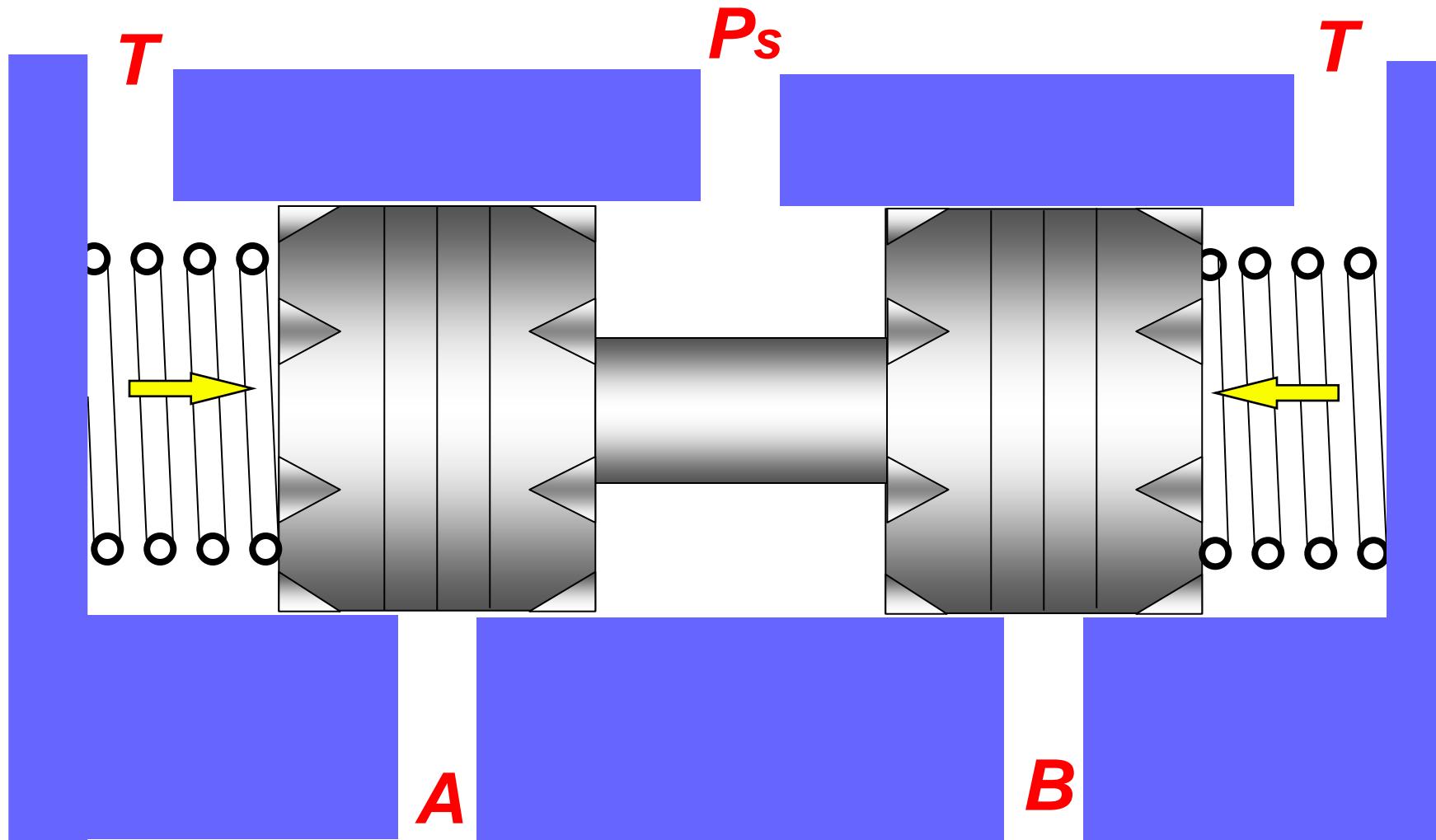
## Digital Onboard Electronics



D1FB\*0 OBE

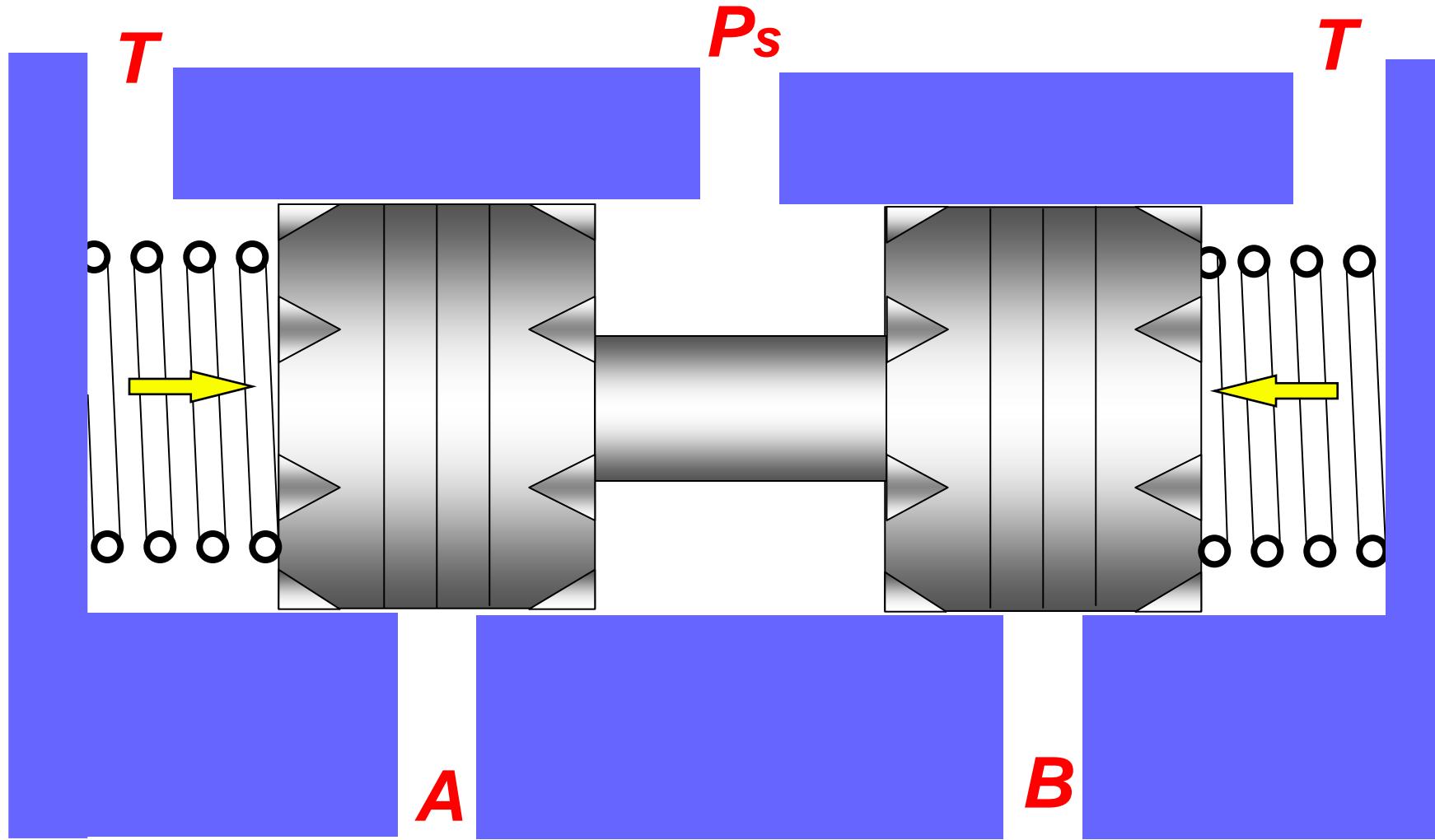


# *Deadband Eliminator P to A flow Path*



 Parker

# *Deadband Eliminator (P to B flow Path)*



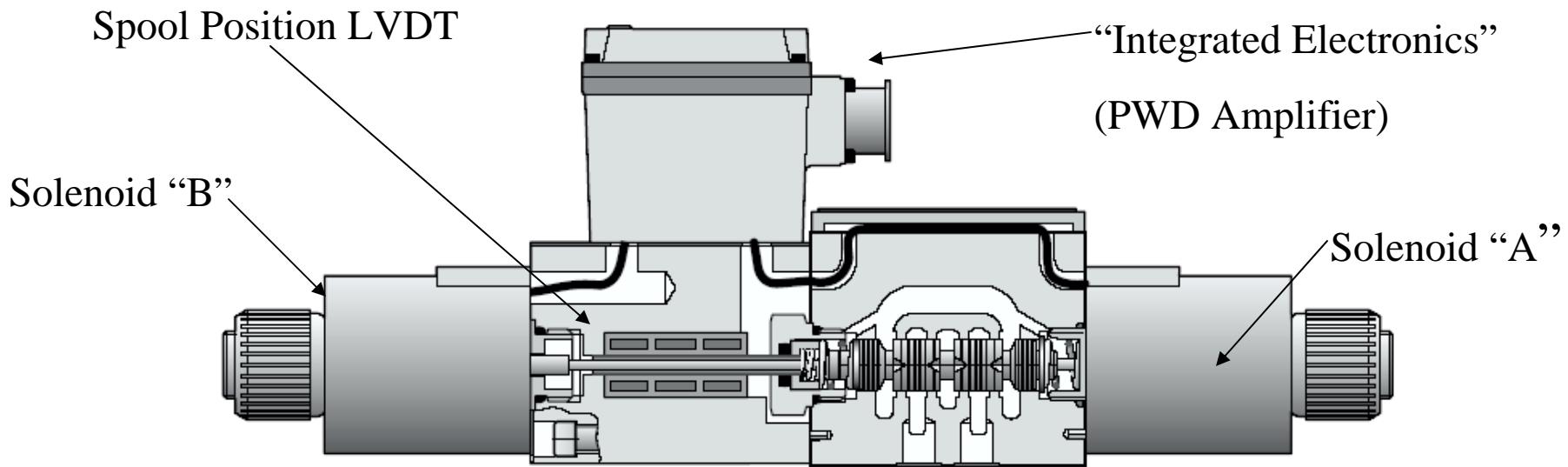
Parker

# Proportional Valves

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Applications	Open Loop Control		Low End Closed Loop Position	Closed Loop Position & Force	Closed Loop Position & Force		

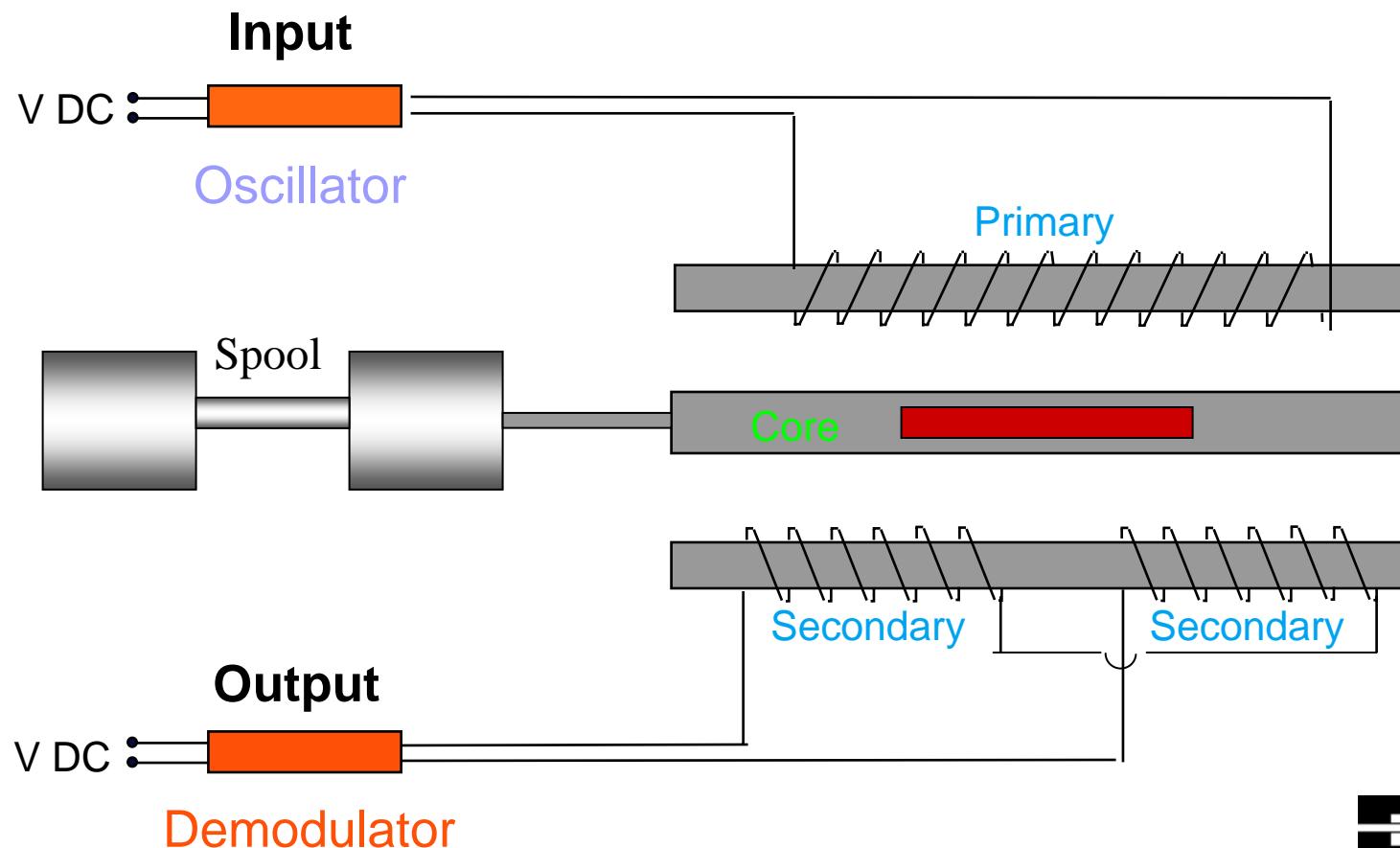
# Spool Feedback, How does it Work?



- Same basic operation as non-feedback valves, but “outcome” is measured and corrected to match desired result.
- “Closing the Loop”.

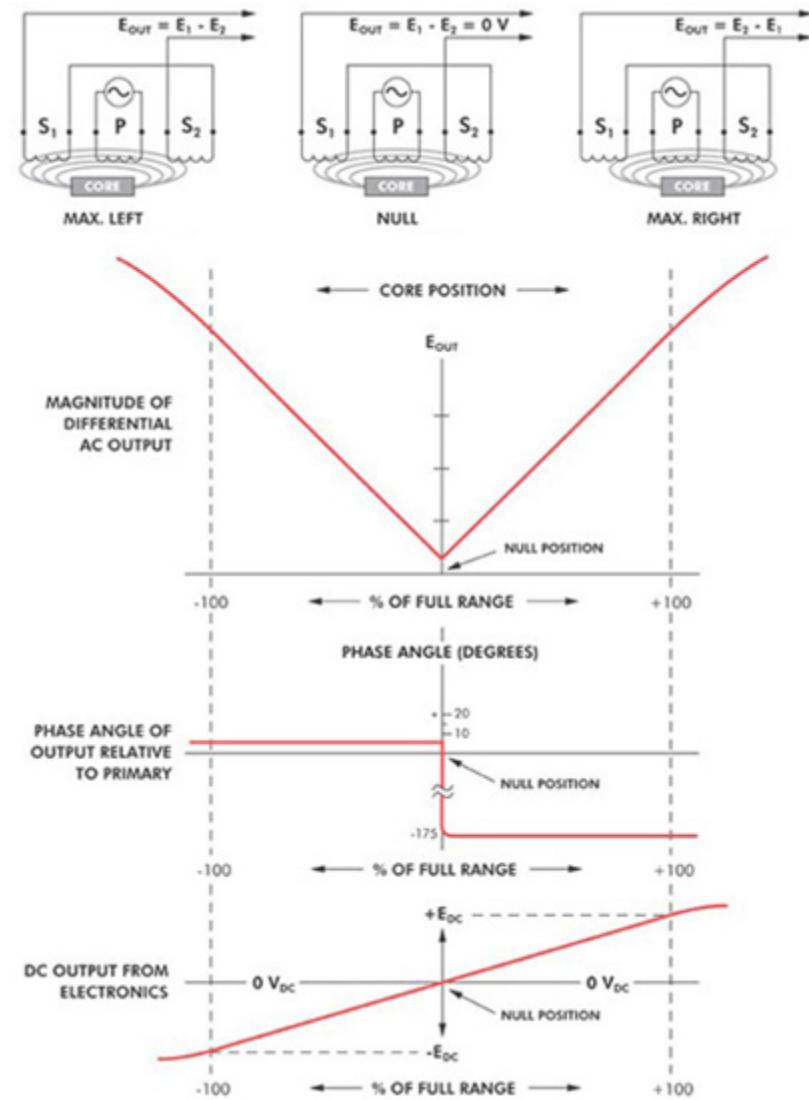
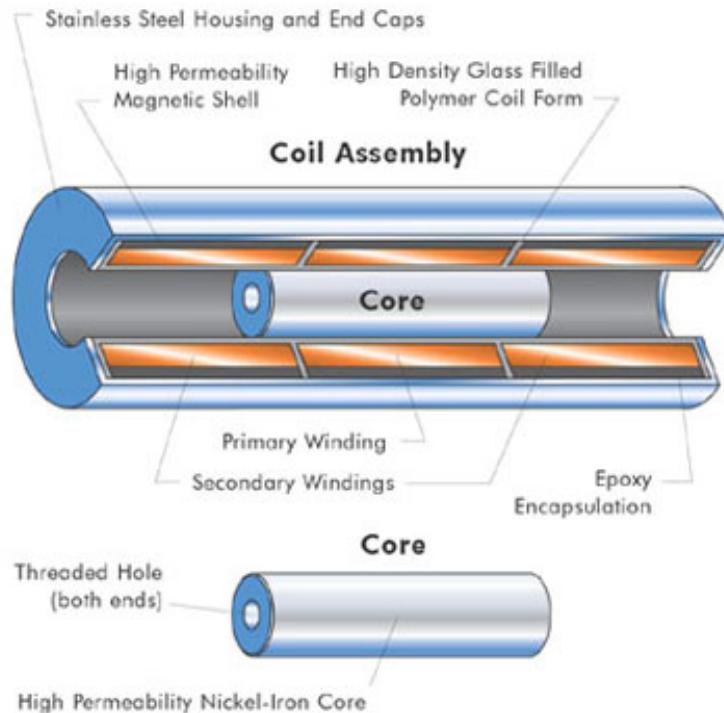
# Spool Feedback Device

## Linear Variable Differential Transformer



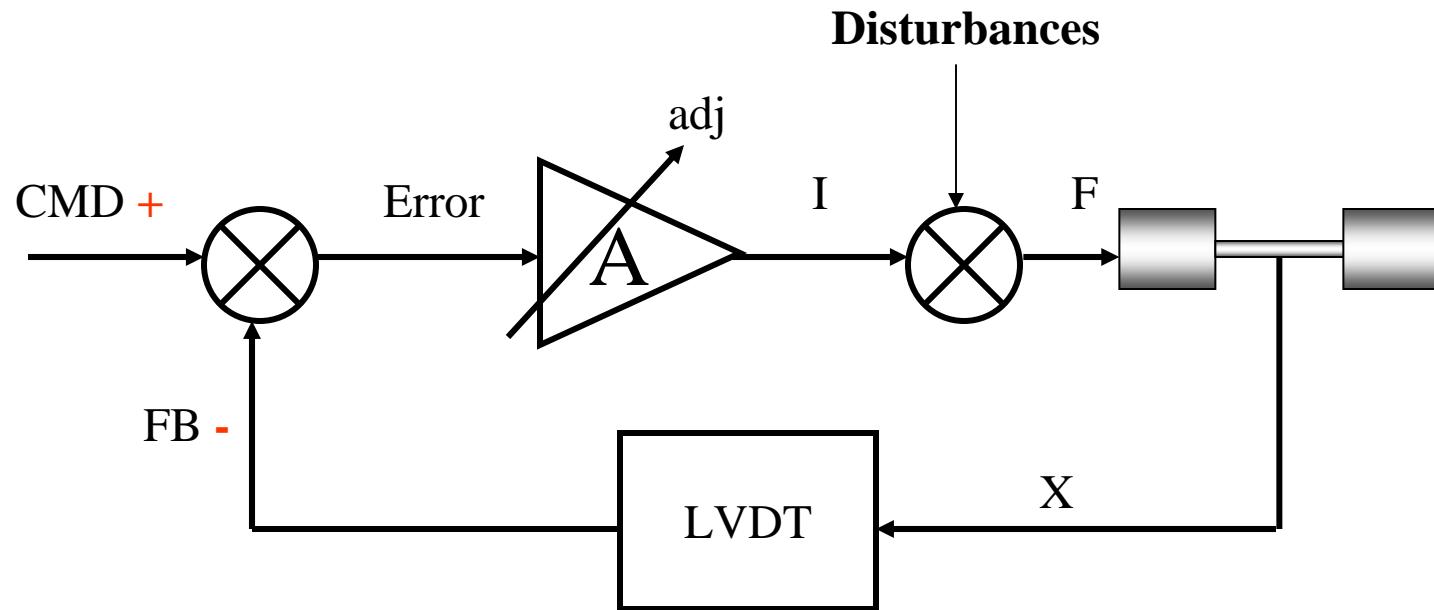
 **Parker**

# L.V.D.T.s

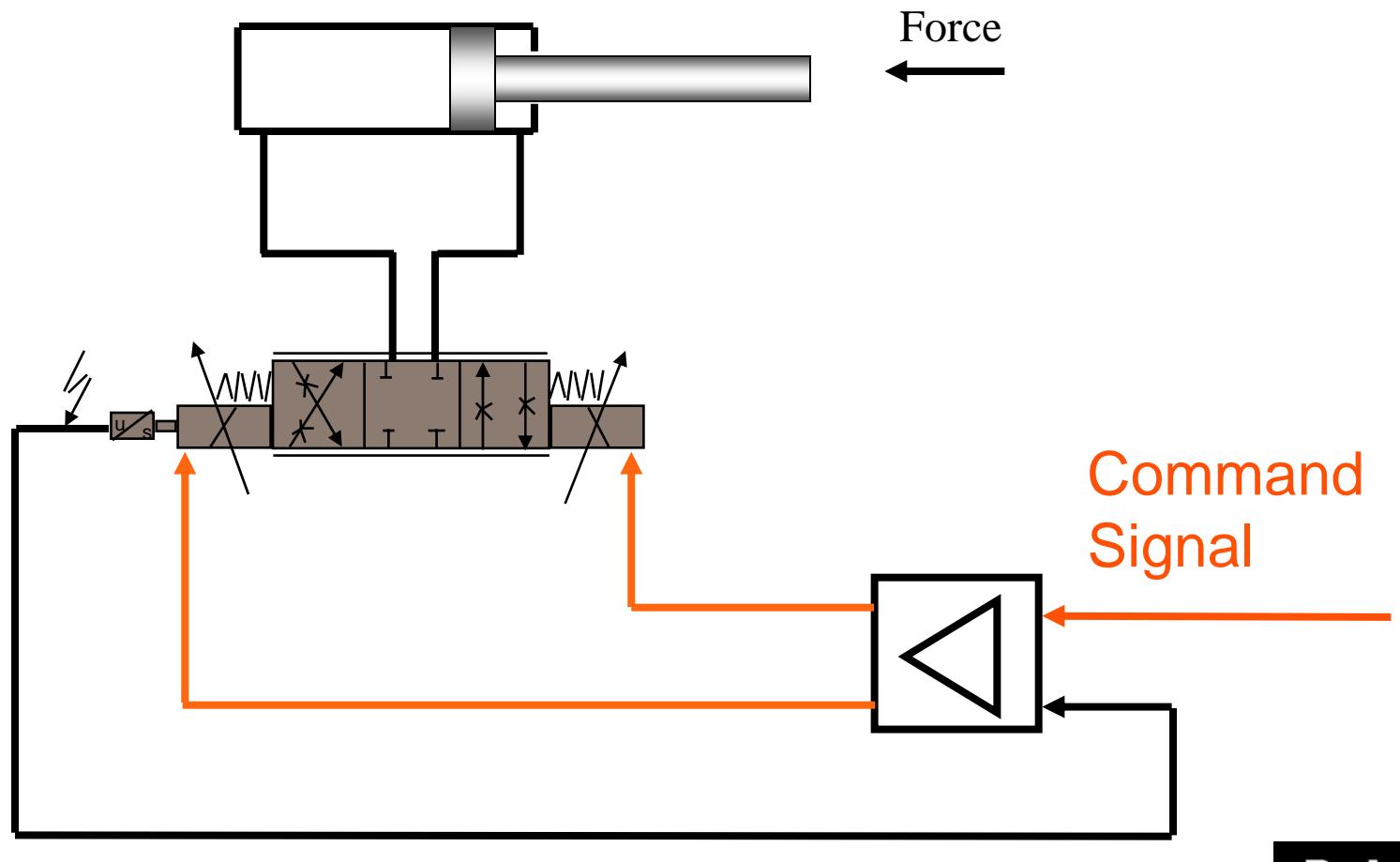


**Parker**

# Spool Feedback Devices (Electrical Schematic-Integrated Electronics)



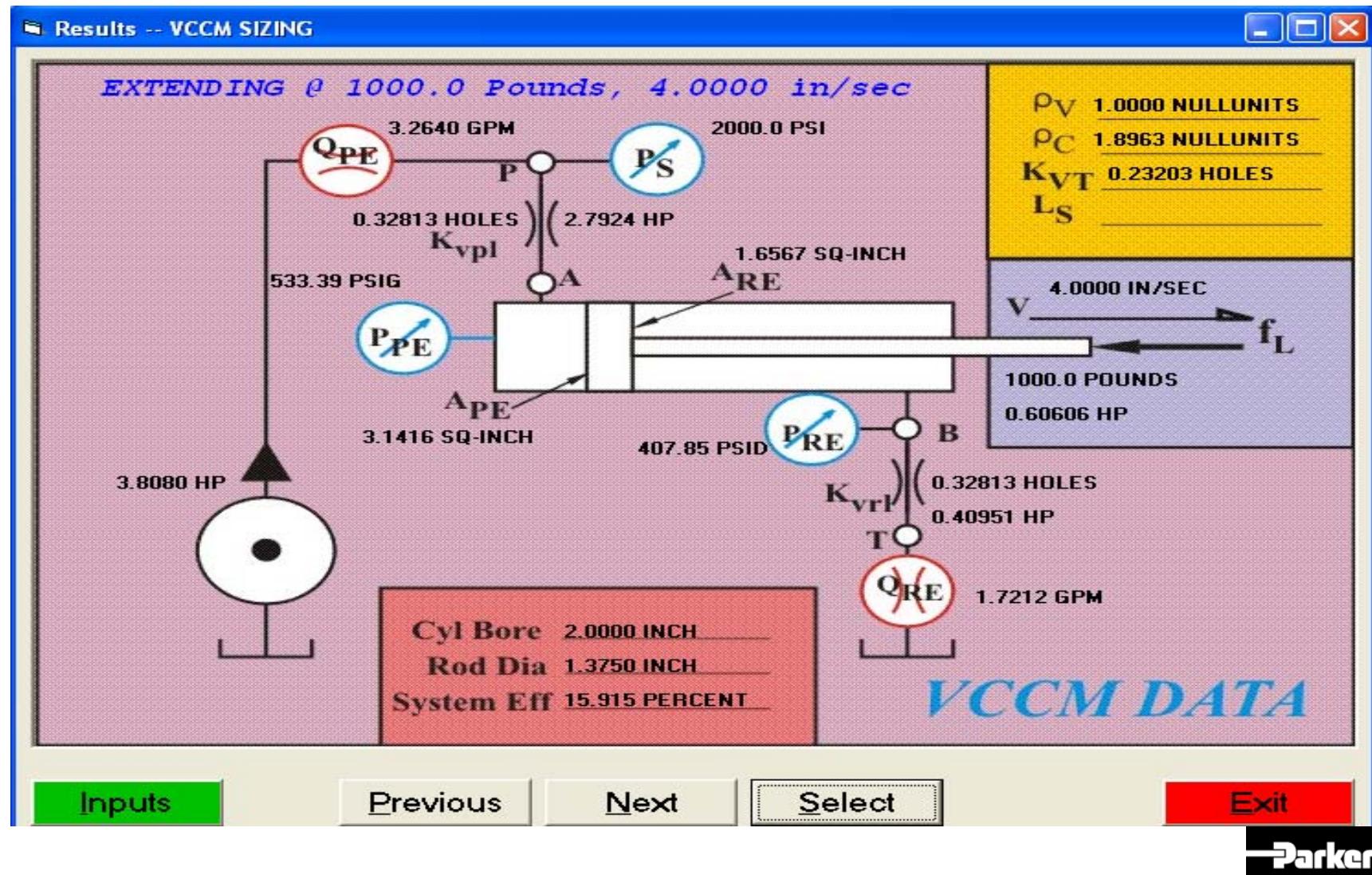
# Internal Closed Loop



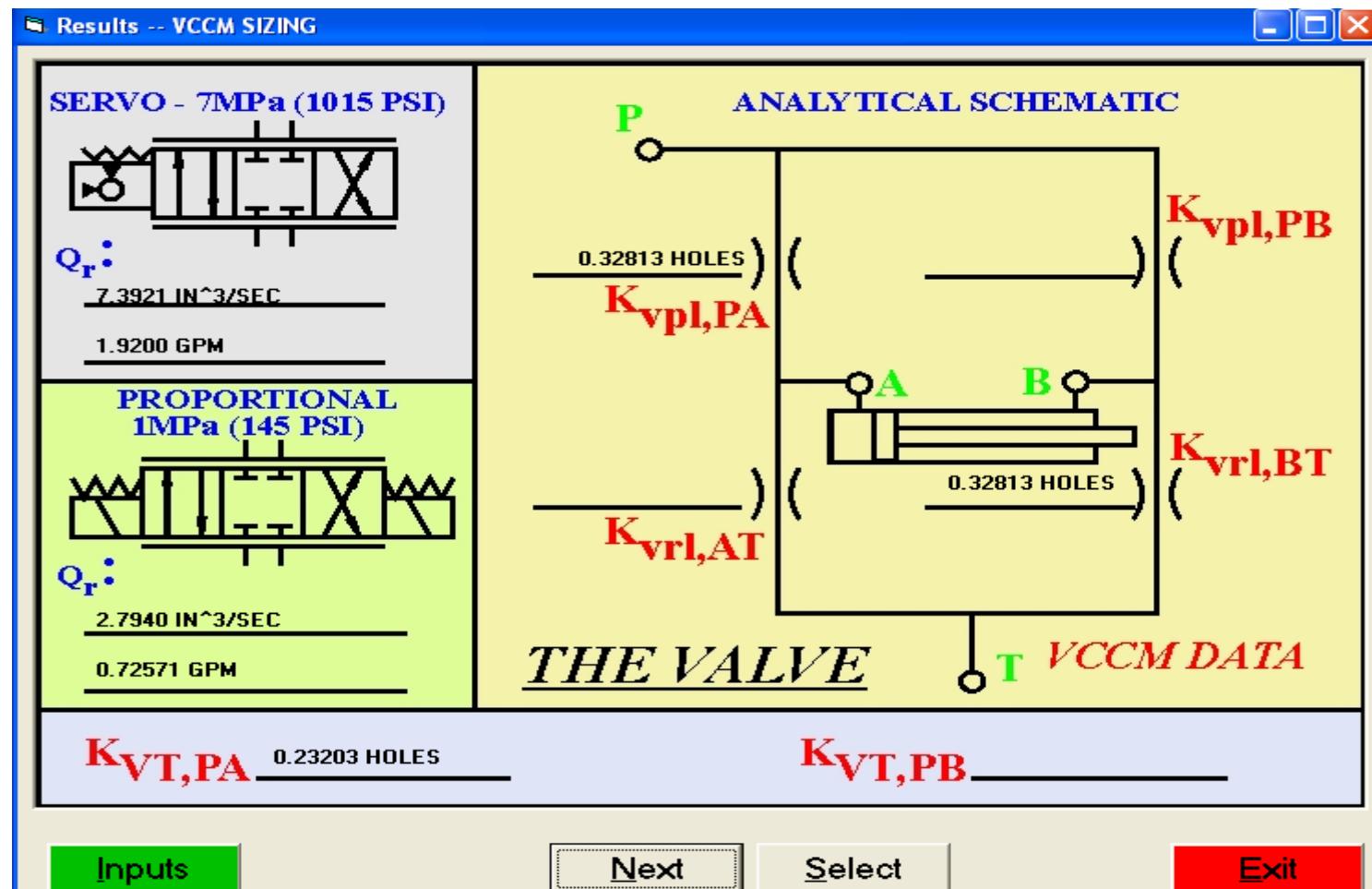
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Valve position feedback

# Sample Application

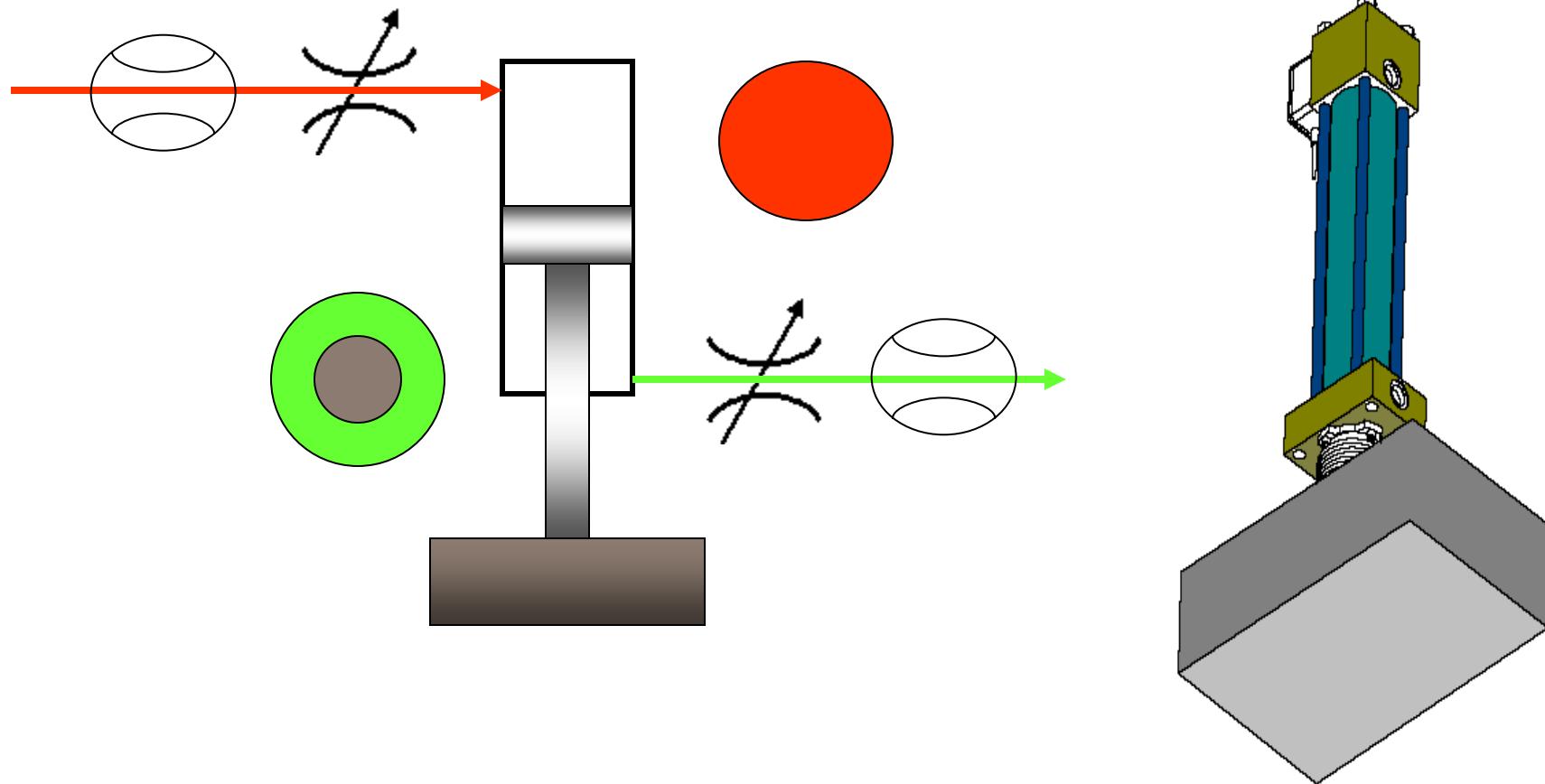


# Kv Sizing

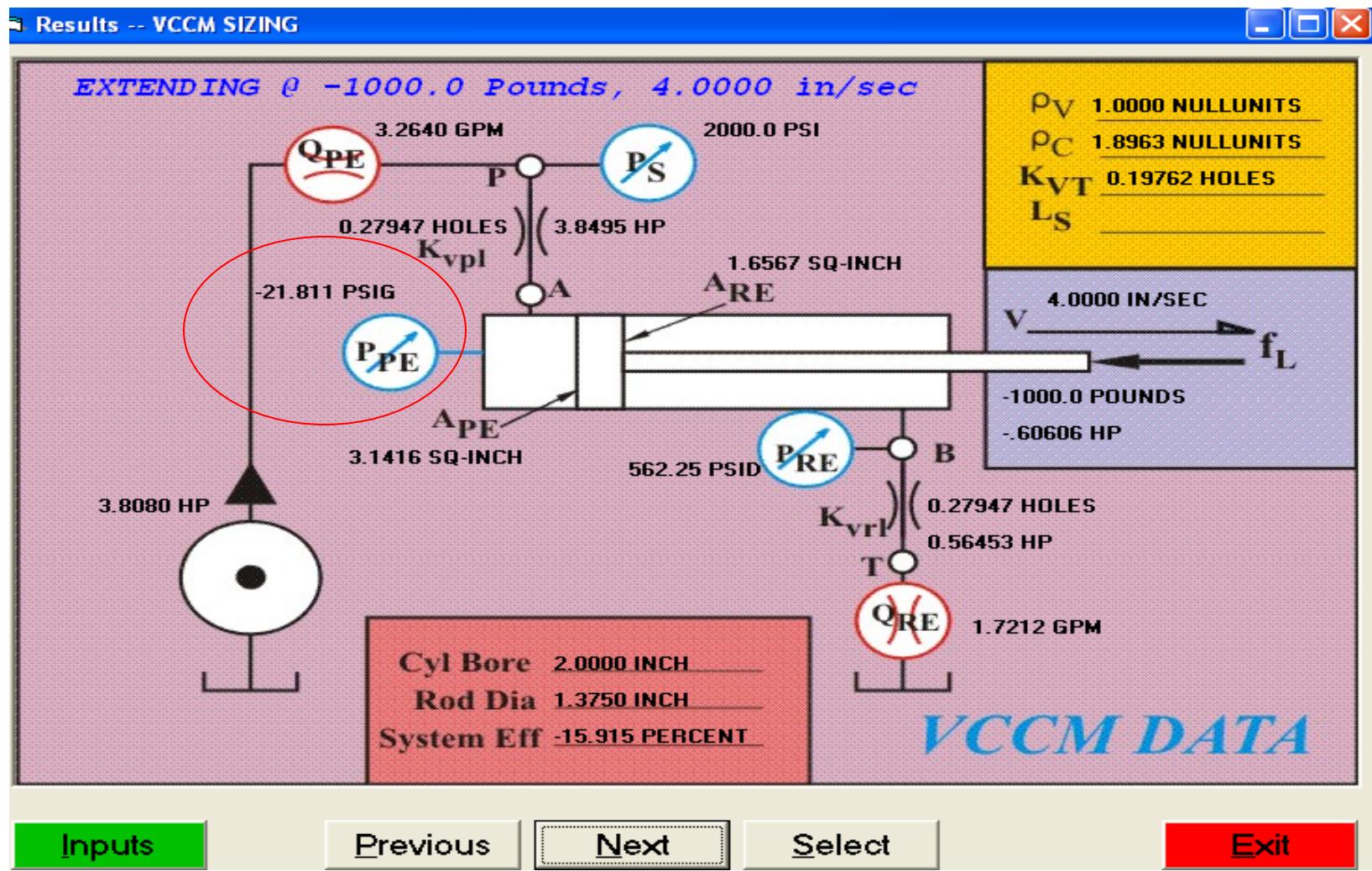


**Parker**

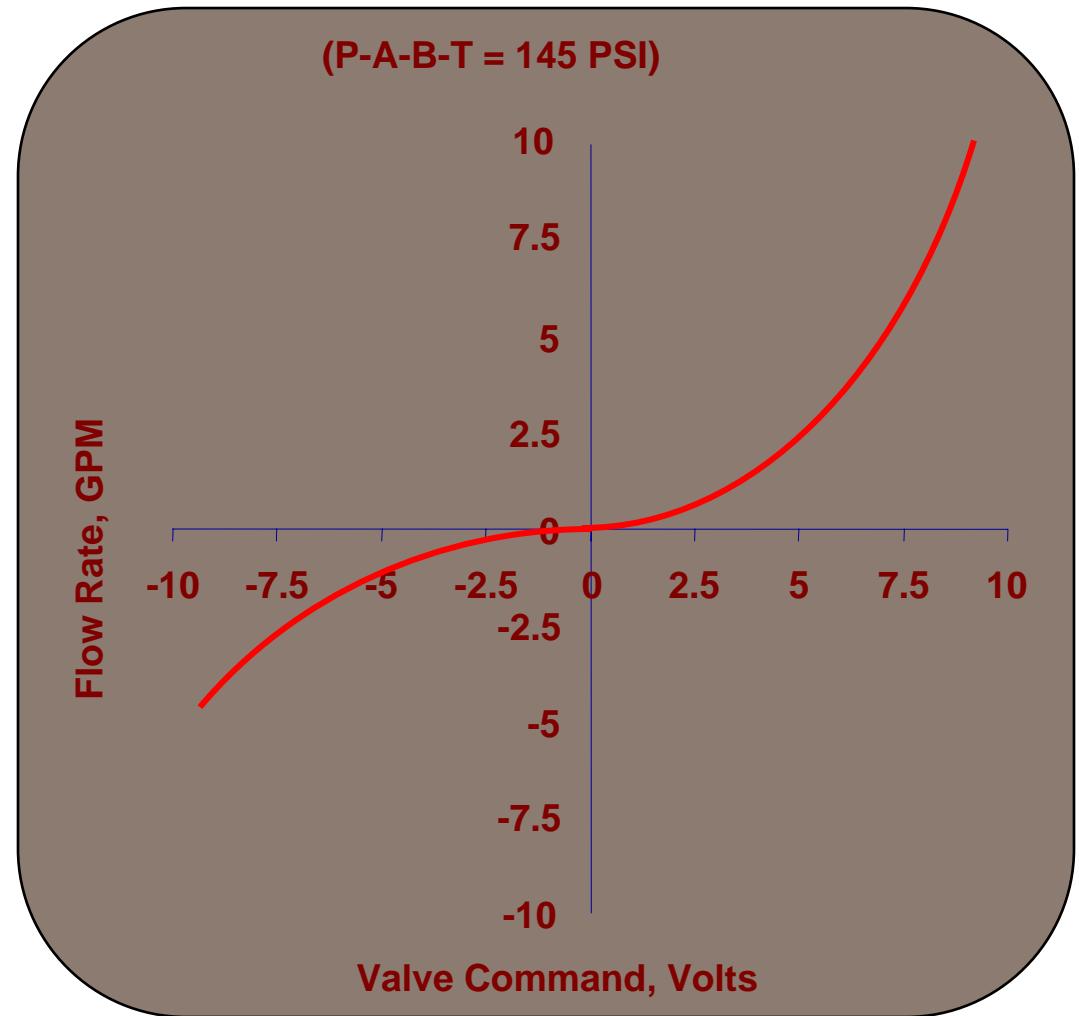
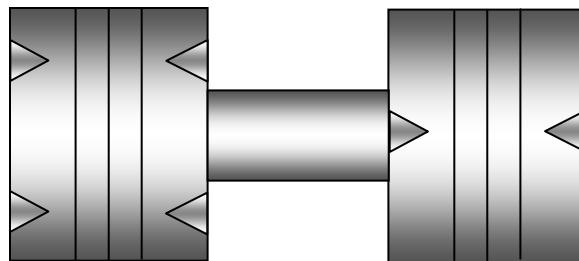
# Non-Symmetrical Spools



# Sample Application, Number 2



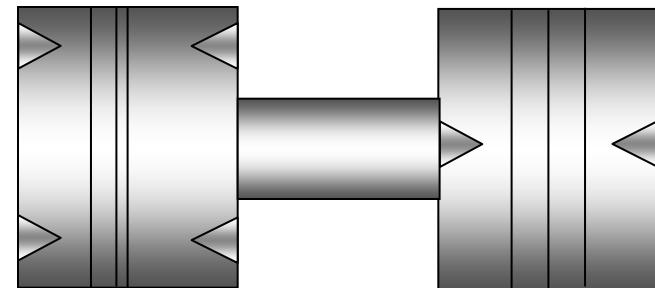
# Non-Symmetrical Spools



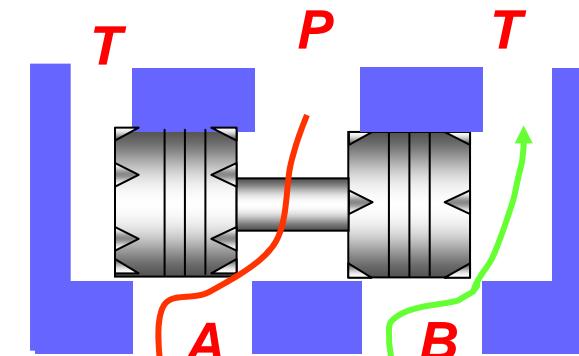
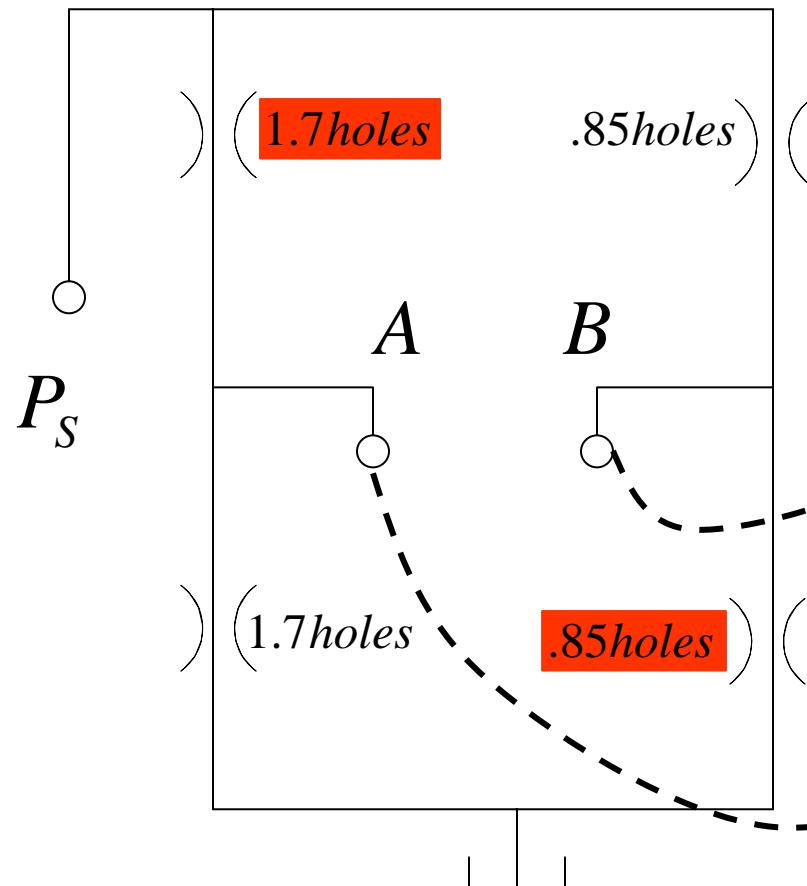
# Common Procedure

The manufacturer can choose to take a standard 10gpm valve with normally 4 notches on each land and only cut two notches in the land that will be connected to the small area of the cylinder.

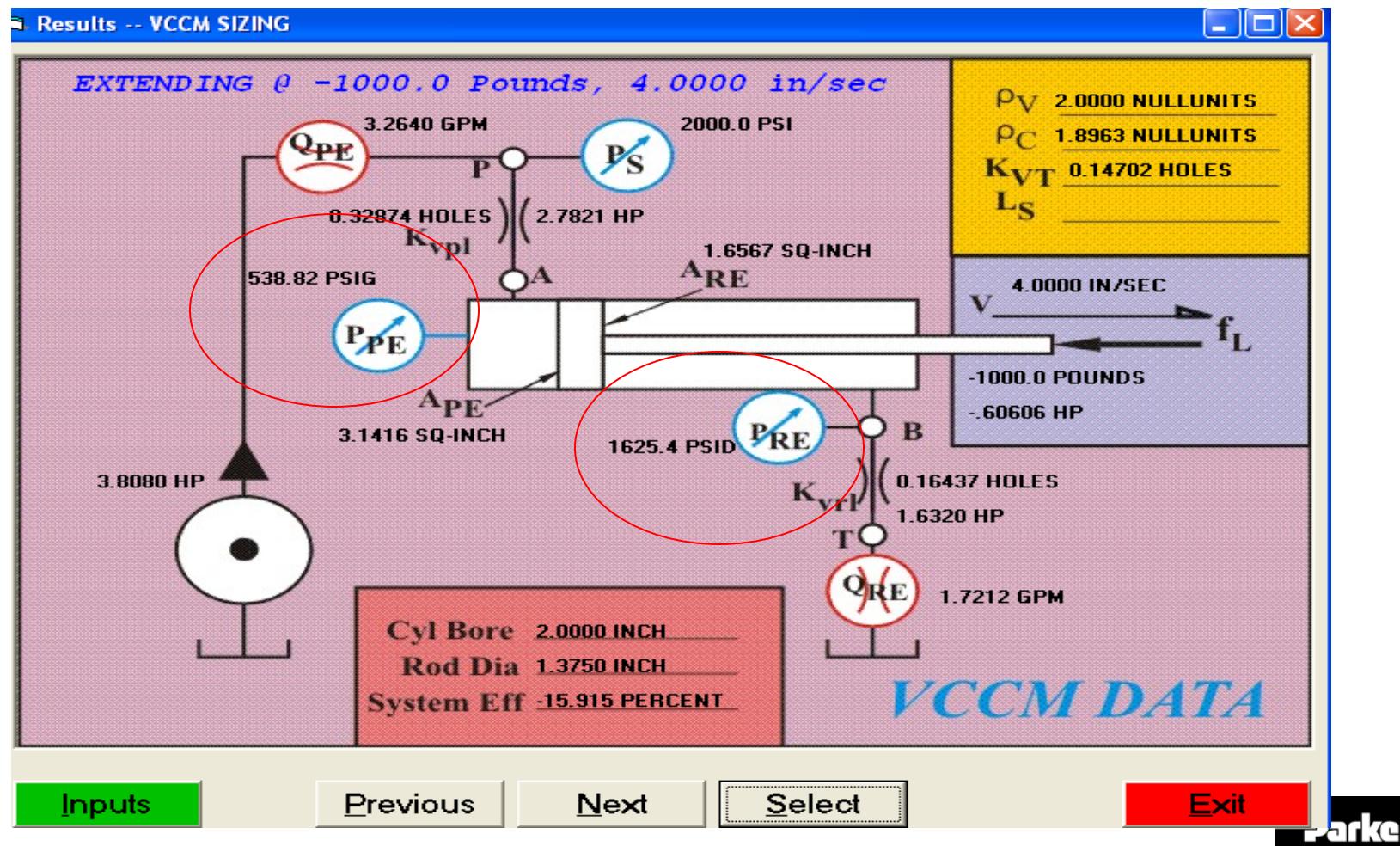
- 3 notches instead of 4
- 4 notches instead of 6
- 2 notches instead of 6...



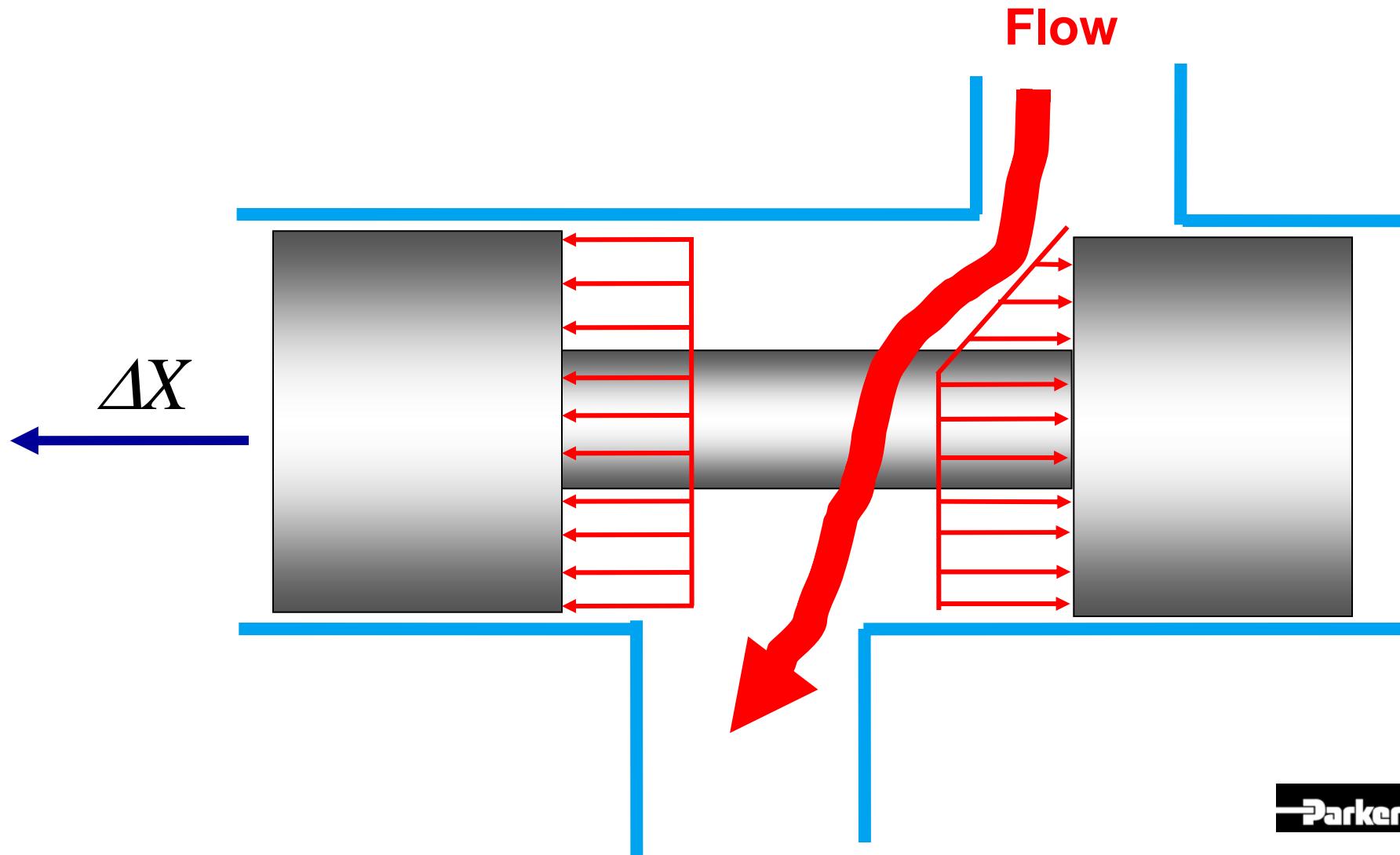
# Non-symmetrical Spools



# Sample Application No. 2 with Non-Symmetrical Spool



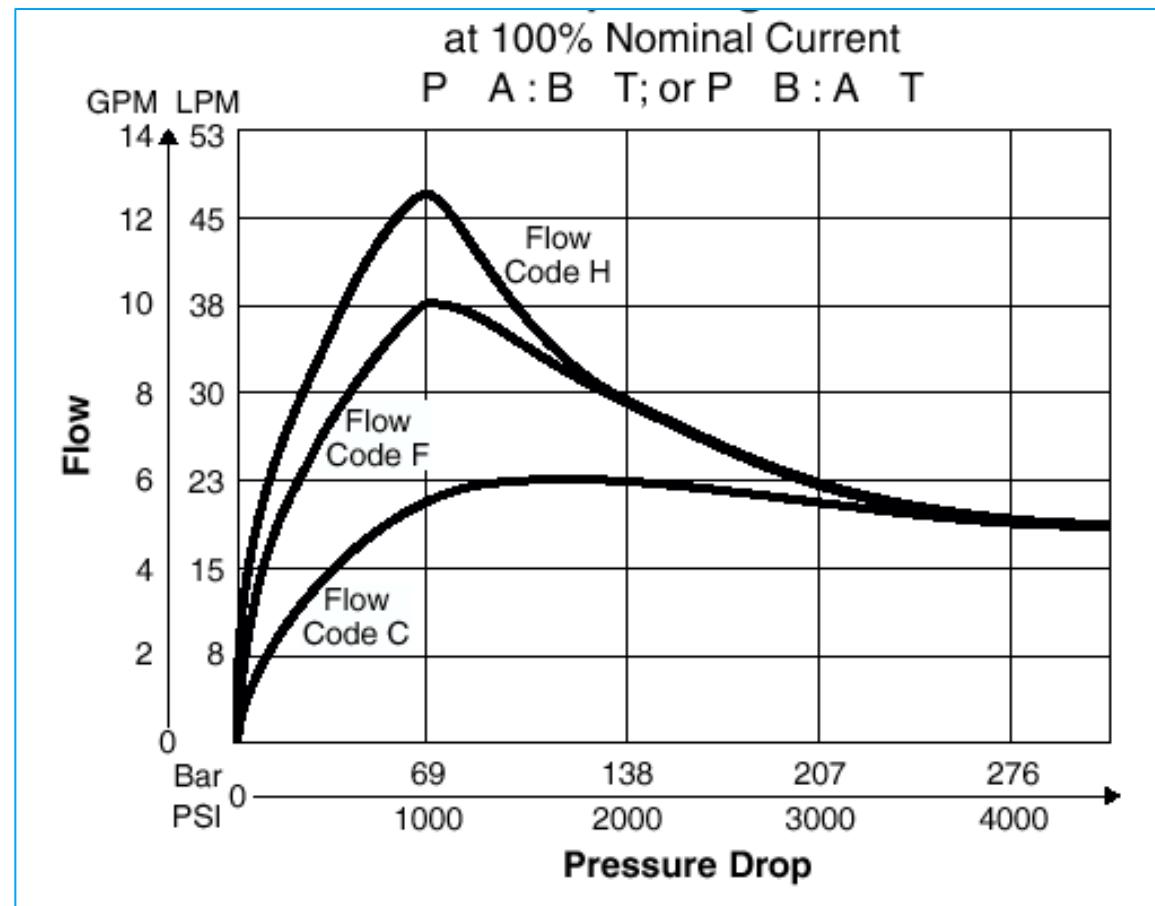
# Flow Force Effects Proportional Valves



Parker

# Flow Force Performance

Operating Limits  
Curves show Valve  
Performance over  
entire Pressure  
Range



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# VCD®...Milestone for High Performance Valves

- D1FP (NG6) valve as pilot valve for D\*1FP

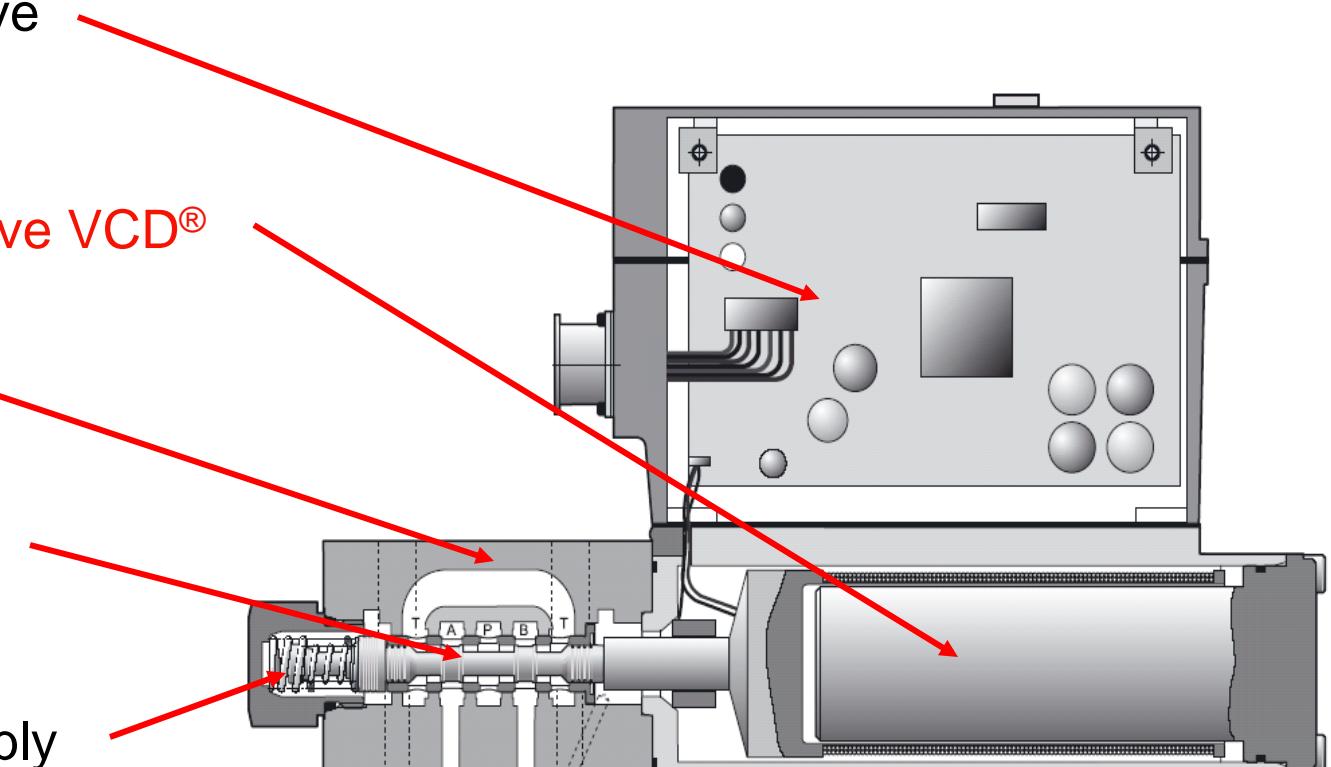
- integrated drive electronics

- Voice Coil Drive VCD®

- valve body

- spool-sleeve assembly

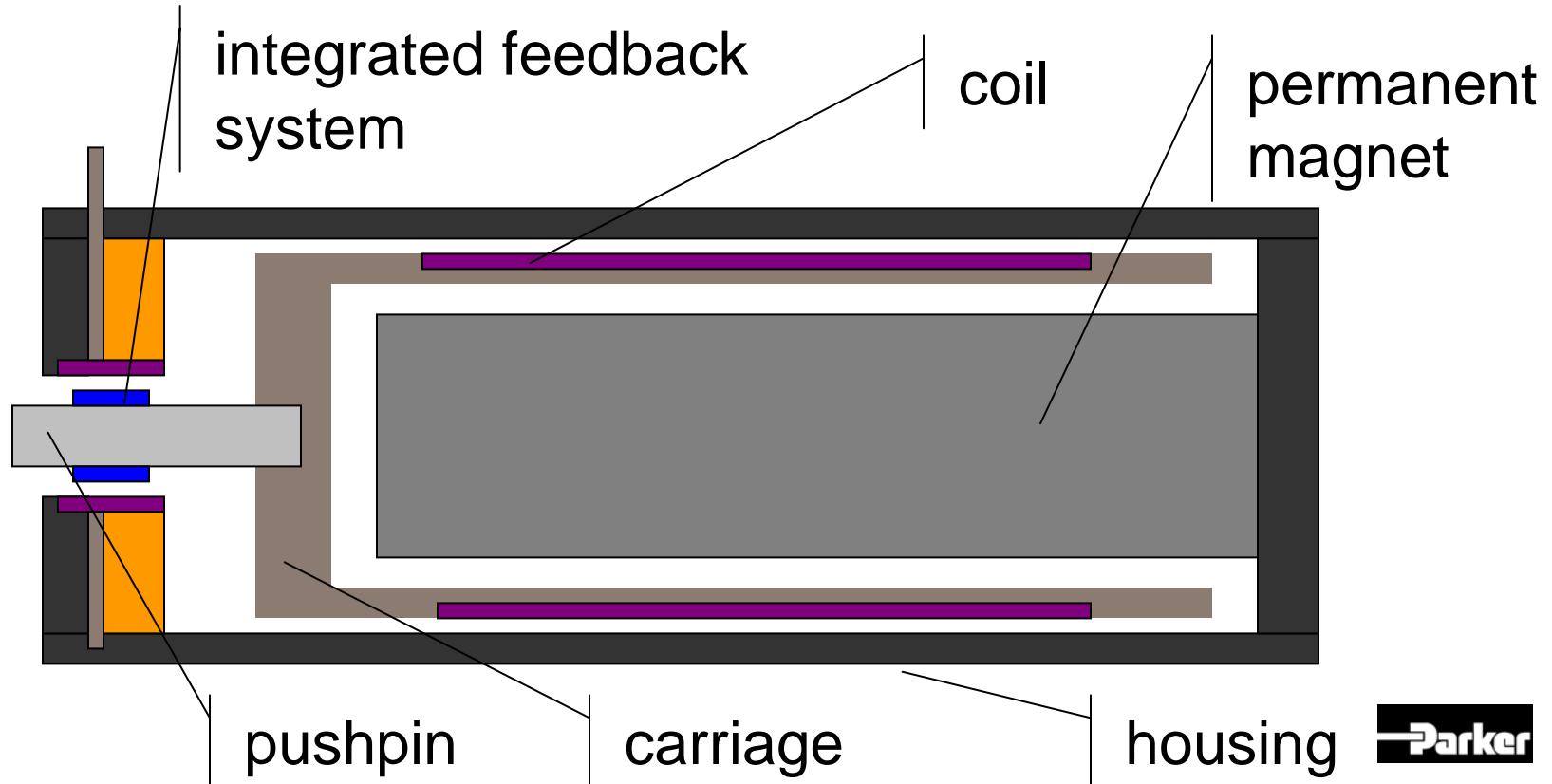
- spring assembly fail-safe position



Parker

# VCD®...Milestone for High Performance Valves

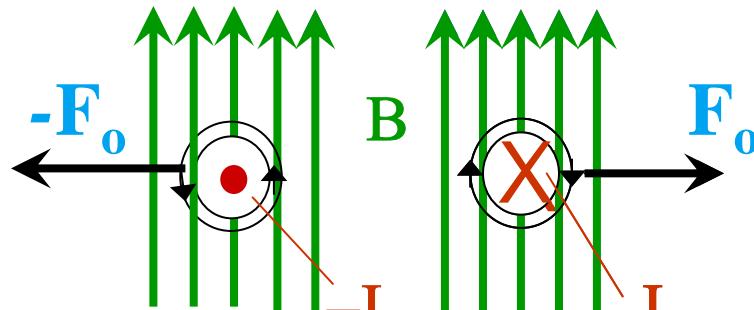
- Parker Voice Coil Drive (VCD®) technology for highest precision



Parker

# VCD®...Milestone for High Performance Valves

- VCD® principle, moved coil in magnetic field

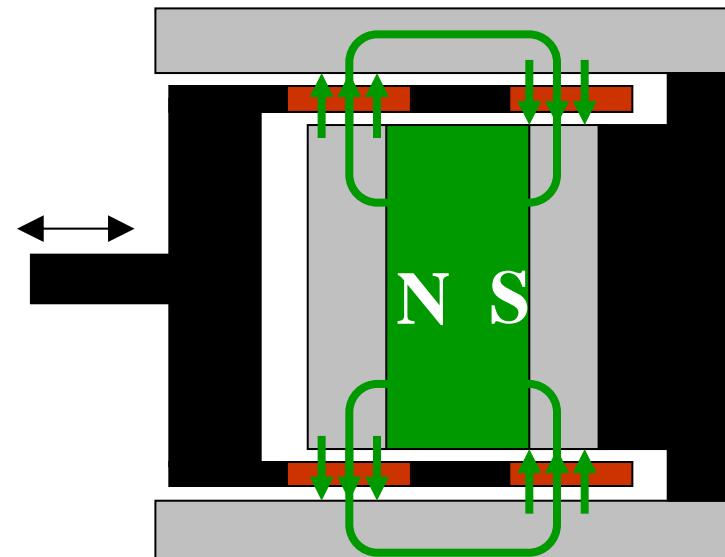


$$F_o = B \cdot I \cdot l$$

$B$  = magnetic flux density

$I$  = electrical current

$l$  = wire length (winding)



permanent magnet



winding



non-magnetic material

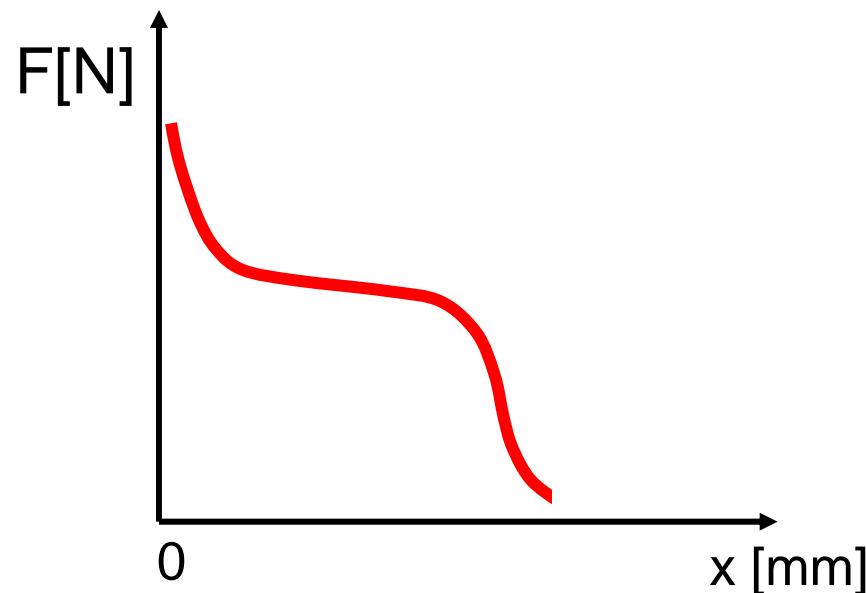


iron (magnetic)

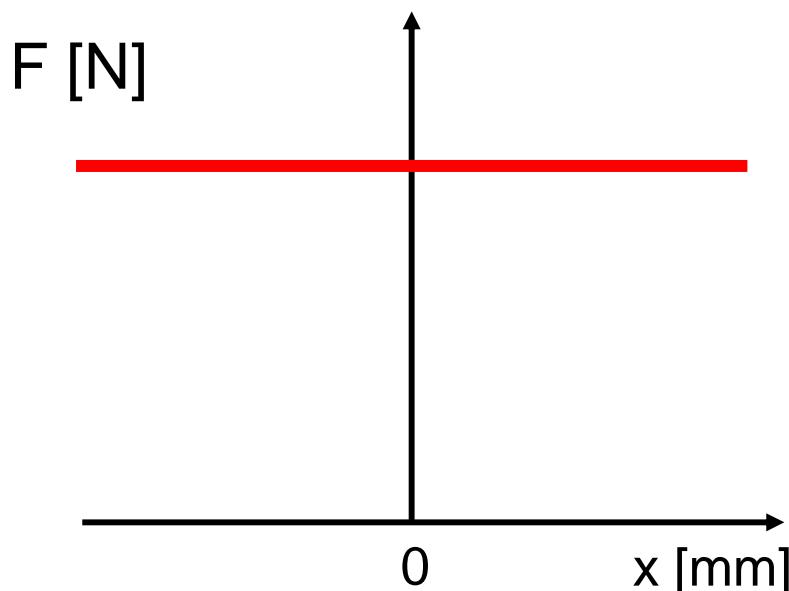


# VCD®...Milestone for High Performance Valves

- Characteristics of force in comparison



**conventional solenoid**  
force dependent of stroke



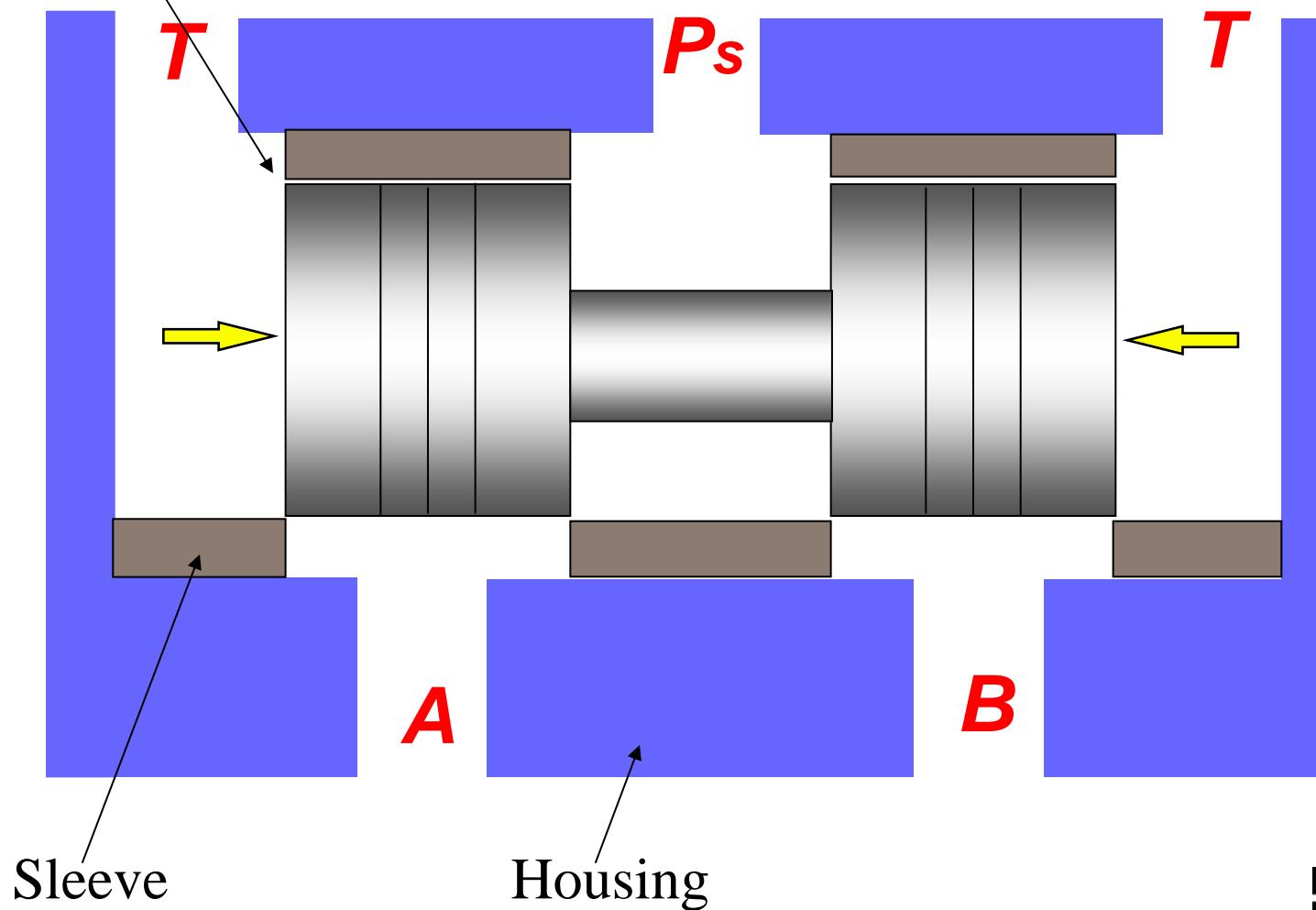
**Voice Coil Drive**  
force independent of stroke



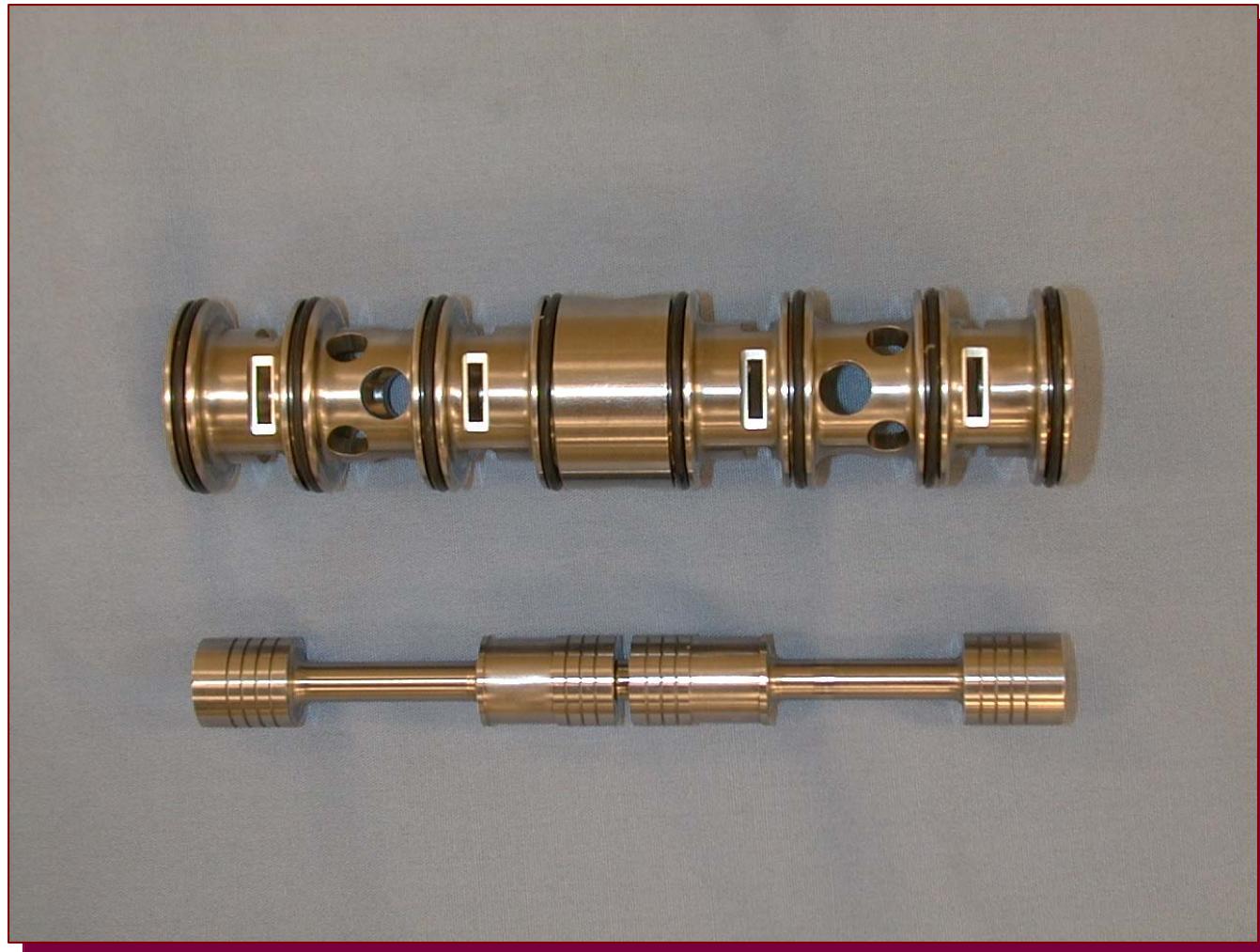
# *Servo Solenoid/Voice Coil Valves*

“Line to Line” Lap  
Or “Axis Cut”

*No Deadband*

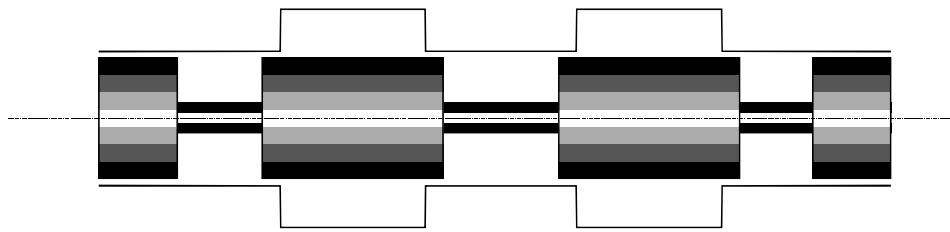


# Spool and Sleeve Arrangement

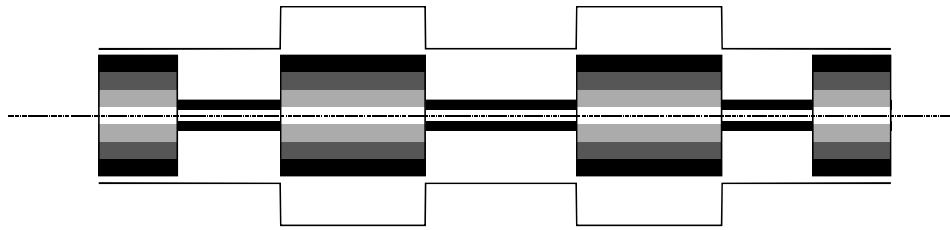


Parker

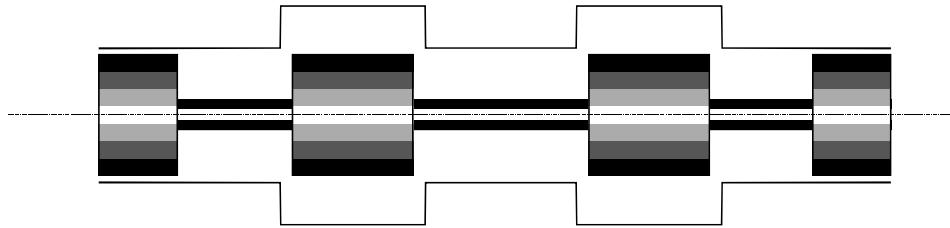
# Spool Lap Conditions



(Positive) Overlap



Zerolap

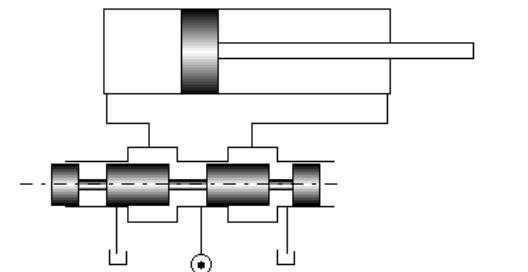


(Negative) Underlap

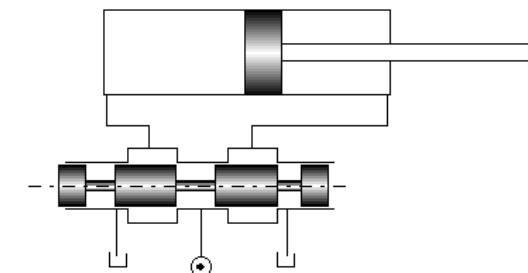
# Spool Lap Conditions

(Positive) Overlap

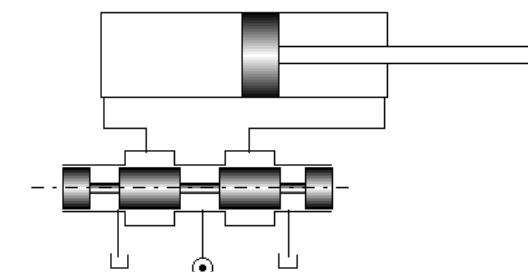
**U = 40%**



**U = 20%**

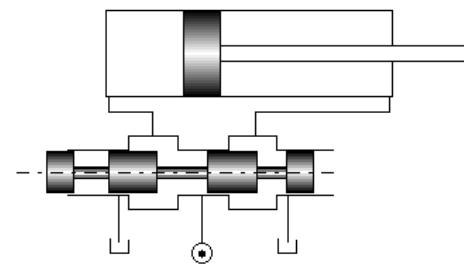


**U = 0%**

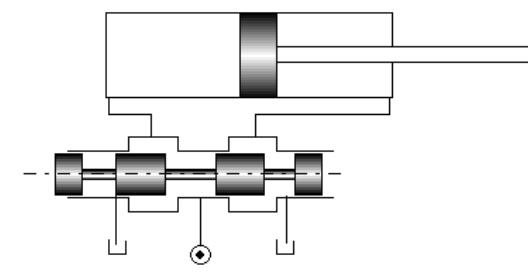


Zerolap

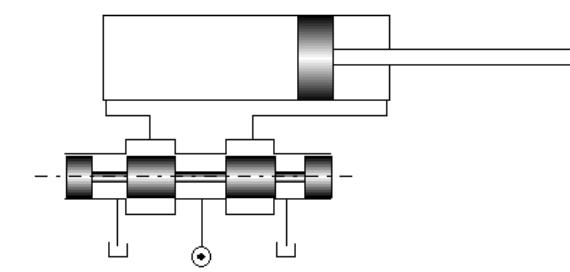
**U = 40%**



**U = 20%**

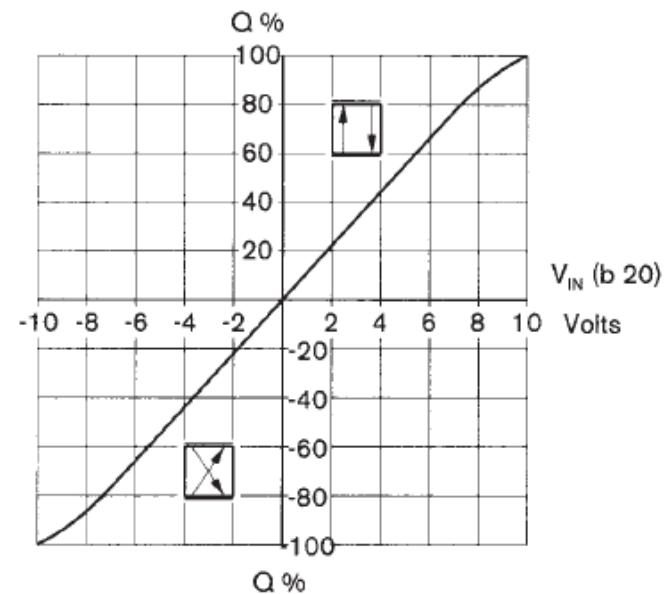
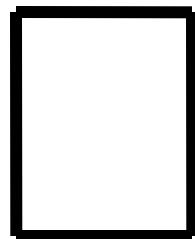
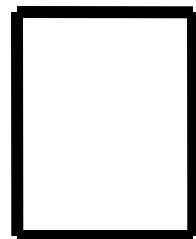


**U = 0%**

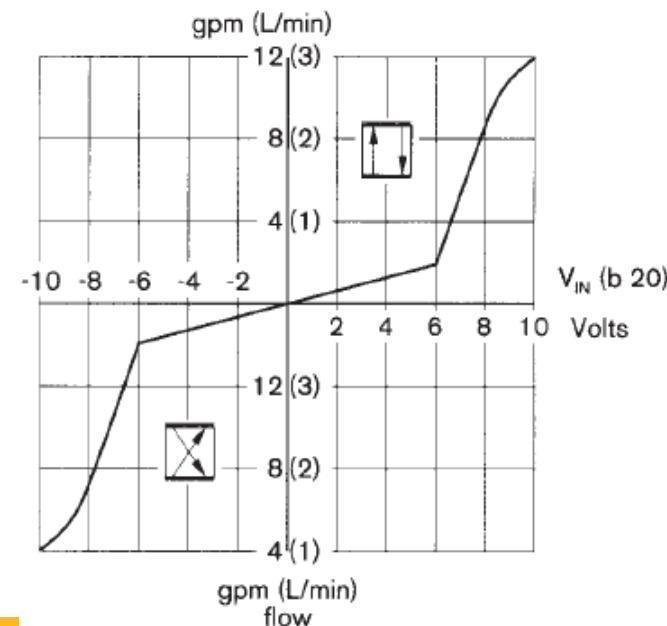
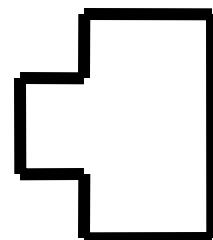
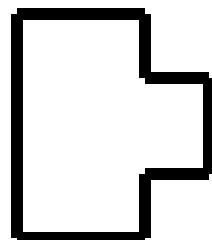


# Flow windows

- Standard Symmetrical Laps



- “Knick Servo Cuts”



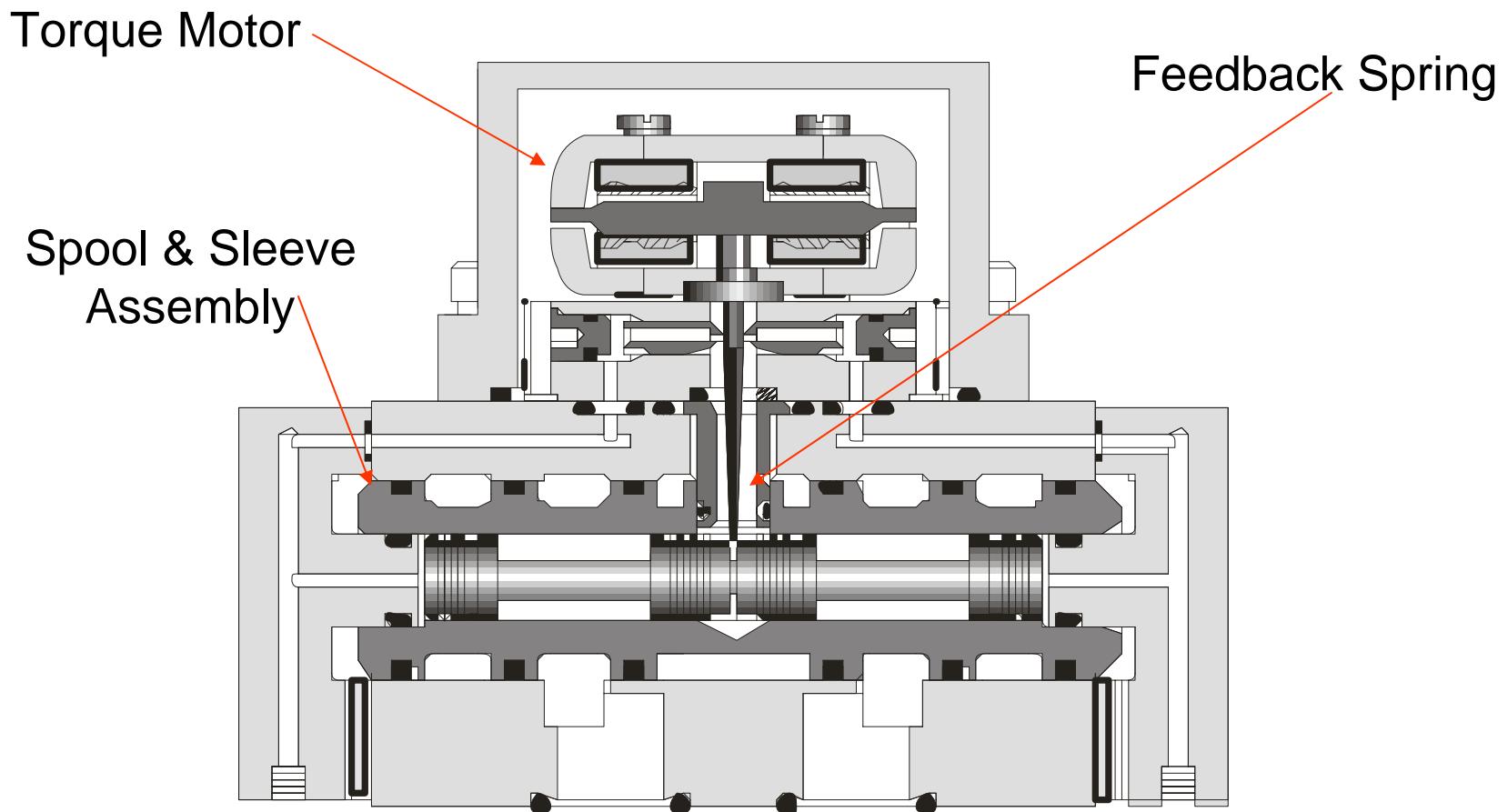
**Parker**

# Proportional Valves

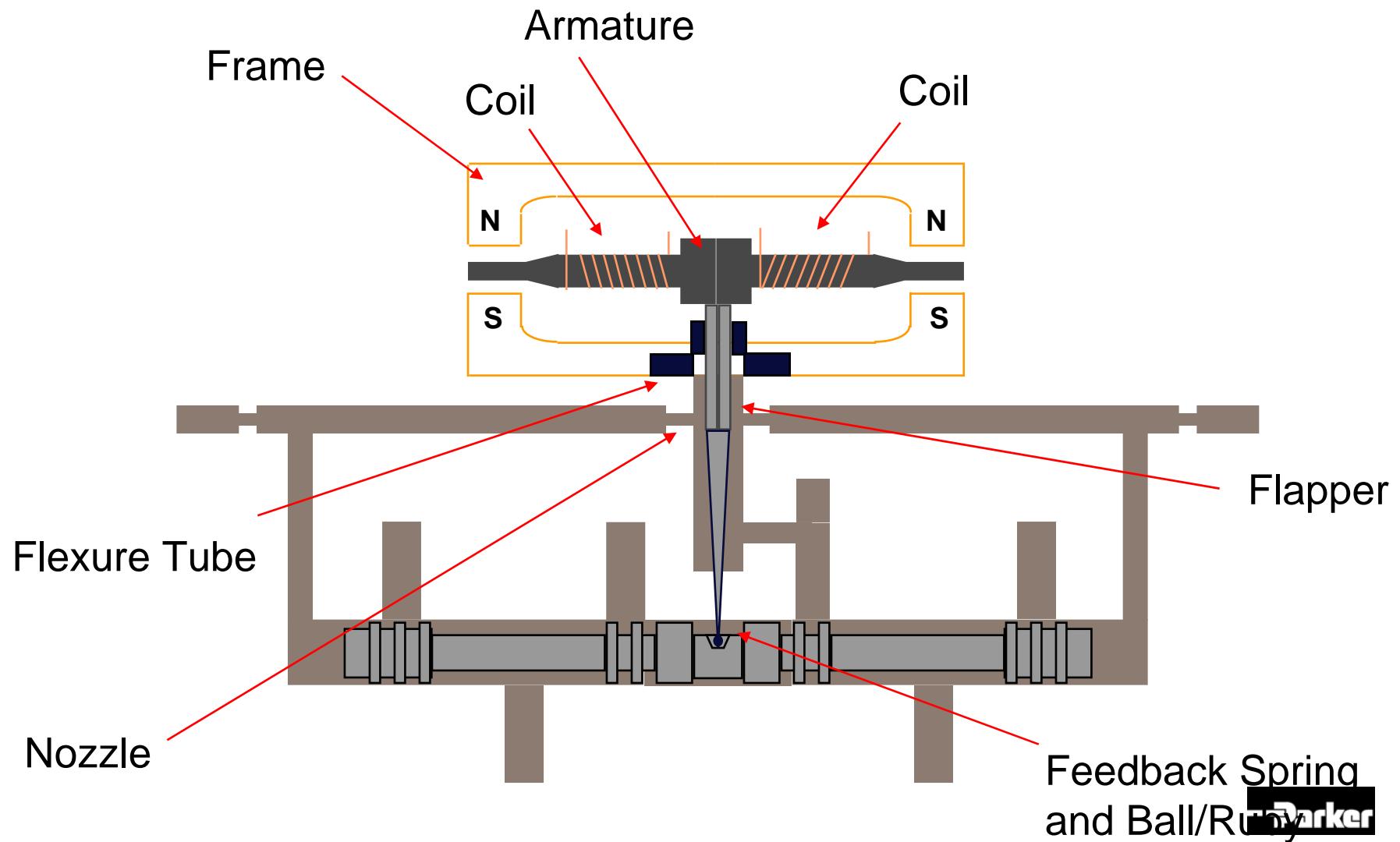
## Servo Valves

	Mobile bankable Style, Threaded Cartrdige Style	NFPA Mounting Without Spool Position Feedback	NFPA Mounting With Spool Position Feedback	Servo Performance, Closed Loop Valves with Spool Position Feedback	Servo Valves With either Mechanical or Electrical Feedback (spool position).
Parker Models	Pulsar VP, VPLs. DF**, ERVs	D*FWs, D*FTs' 	D*FXs' 	D*FH D*FMs D*1FH 	BDs' DYs' SEs' 
Mechanical Construction (spool shift)	Electro-Hydraulic Pilot, Mechanical (spring) Return	Electromagnetic Force, Mechanical (spring) Return	Electromagnetic Force, Mechanical (spring) Return	Electromagnetic Force, Mechanical (spring) Return	Torque Motor Pilot Control Balance PSI spool Control
Hysteresis	3.....7%	3...7%	0.03.....1%	0.1....5%	0.1.....0.5%
Frequency Response	< 10 Hz	10...50 Hz	10....70 Hz	50...150 Hz	100...200 Hz
Center Lap Condition	Overlap 5...20%			Zero Overlap	Zero Overlap
Operating Pressures Limits (Drop across metering edge).	1000...2000 PSI			1000...3000 PSI	None
Applications	Open Loop Control		Low End Closed Loop Position	Closed Loop Position & Force	Closed Position & Force 

# Servo Valve – Double Flapper Design

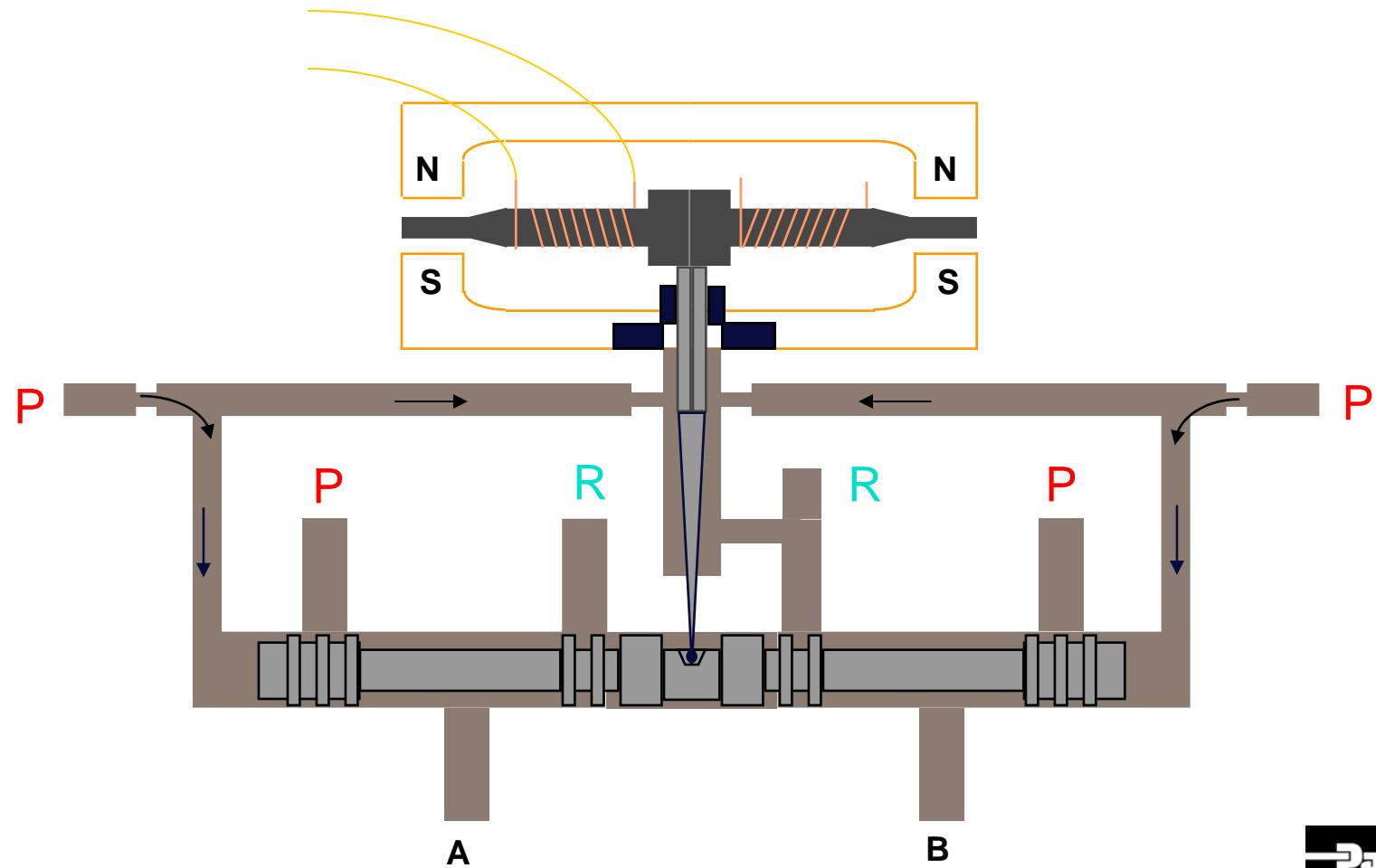


# *Servo Valve – Double Flapper Design*



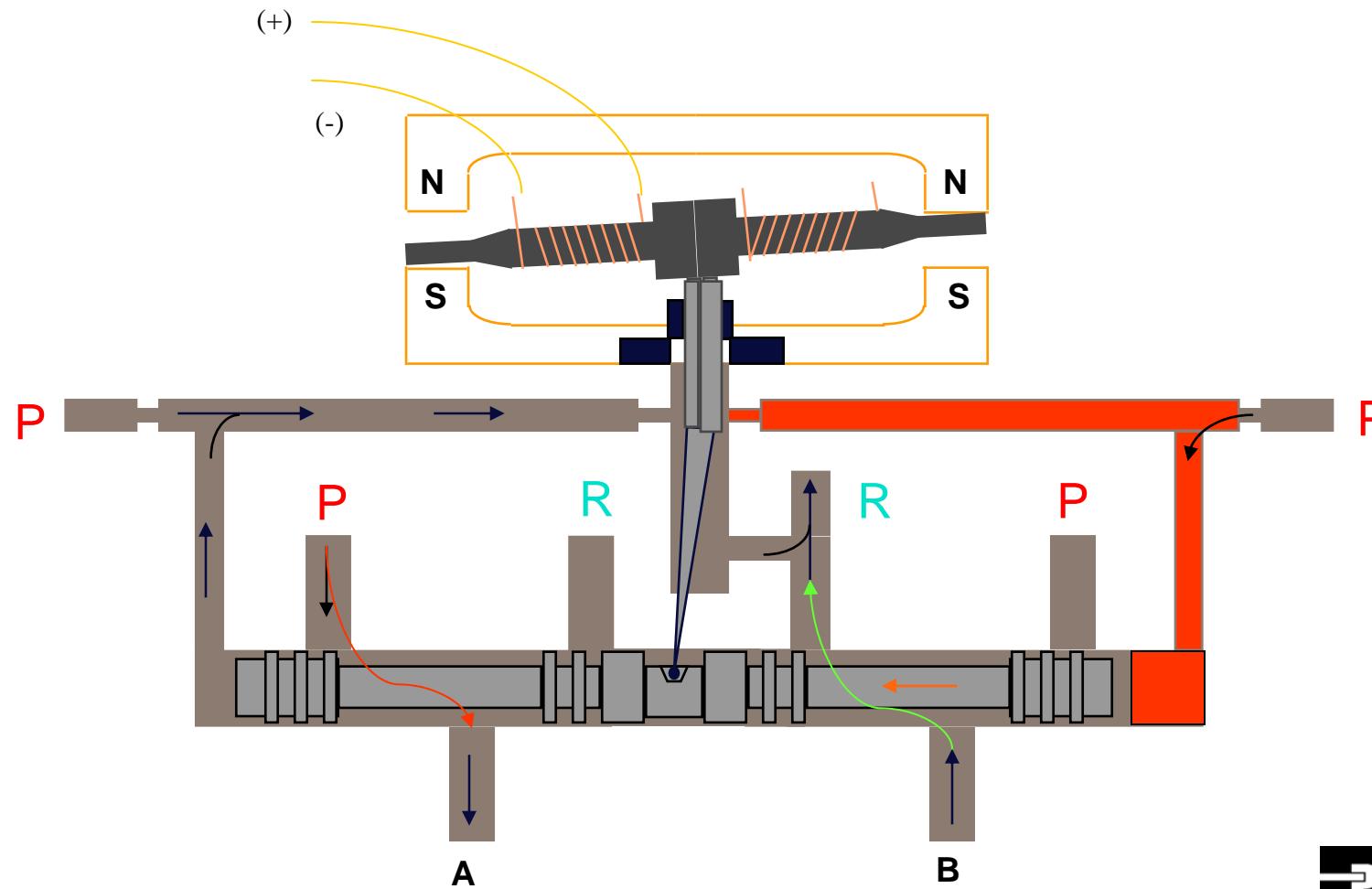
# *Servo Valve - Principles of Operation*

## *Valve at "Null"*

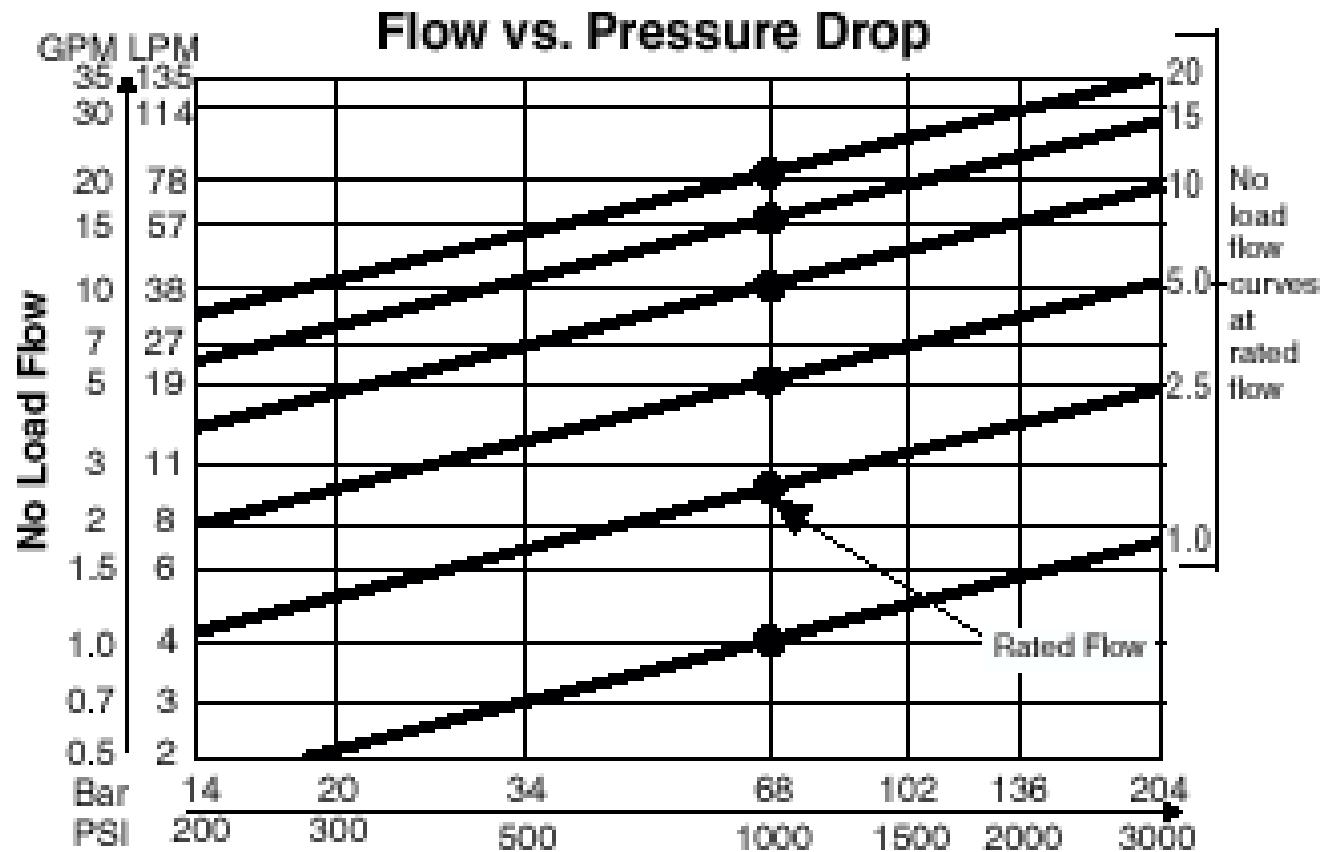


# *Servo Valve - Principles of Operation*

## *Valve With Current Applied*

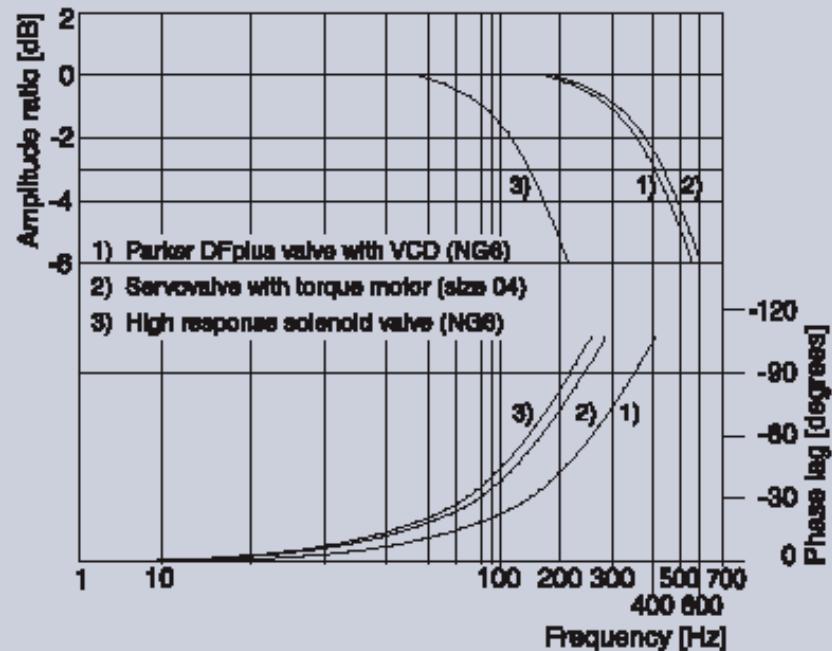


# Flow Forces Not an Issue with Servo Valves

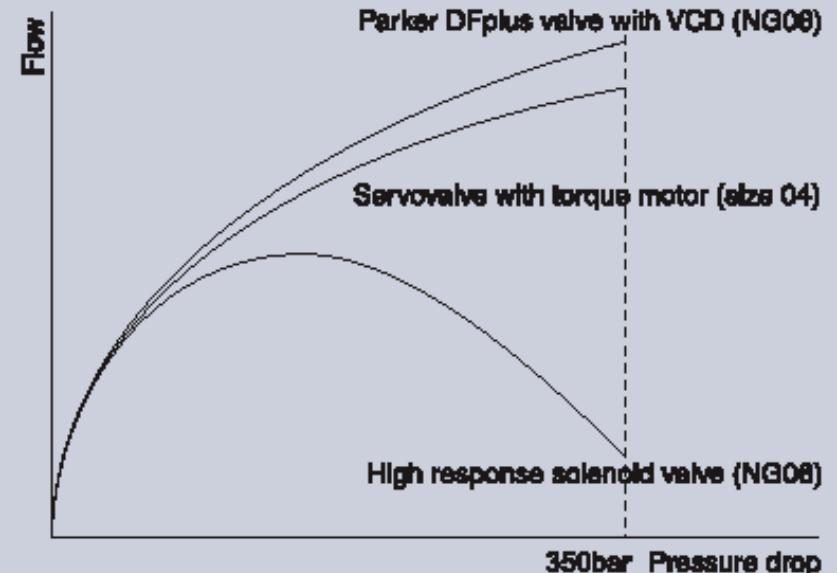


# Servo 's Performances

## Frequency response $\pm 5\%$ Command signal\*



## Flow limit\*

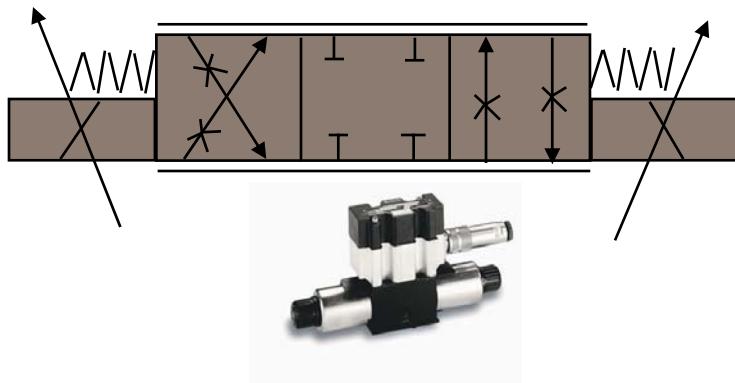


\* schematics, all valves  $Q_{\text{nom}} = 40 \text{ l/min}$

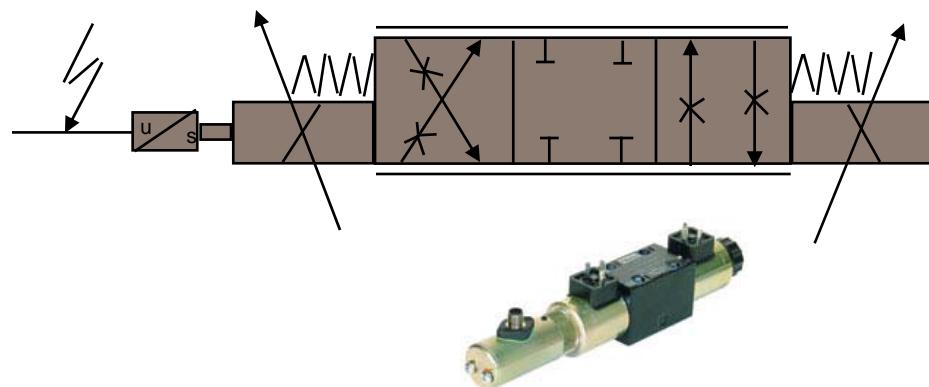


# *Symbology*

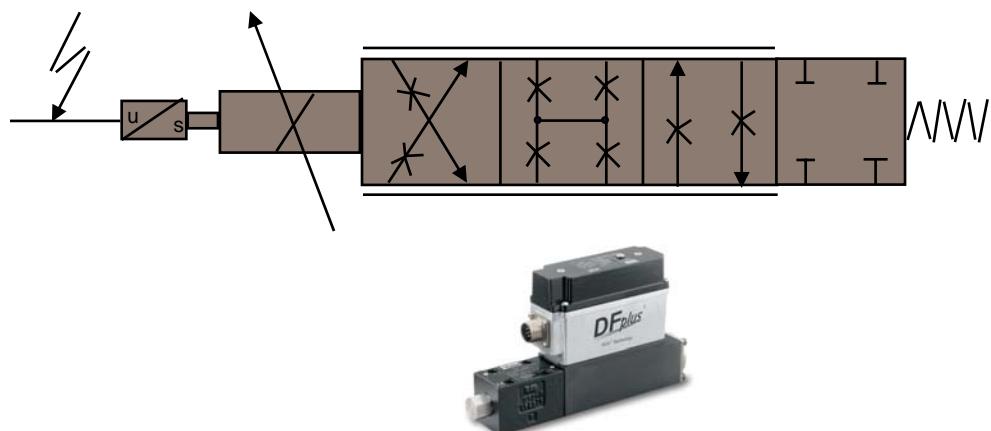
Proportional Directional Valve



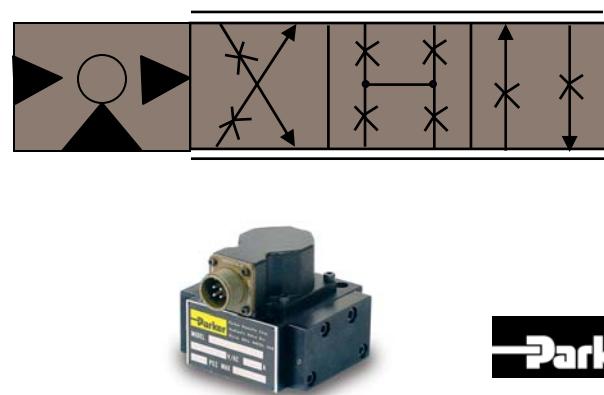
Proportional Valve with Spool Feedback



Servo Performance Proportional Valve



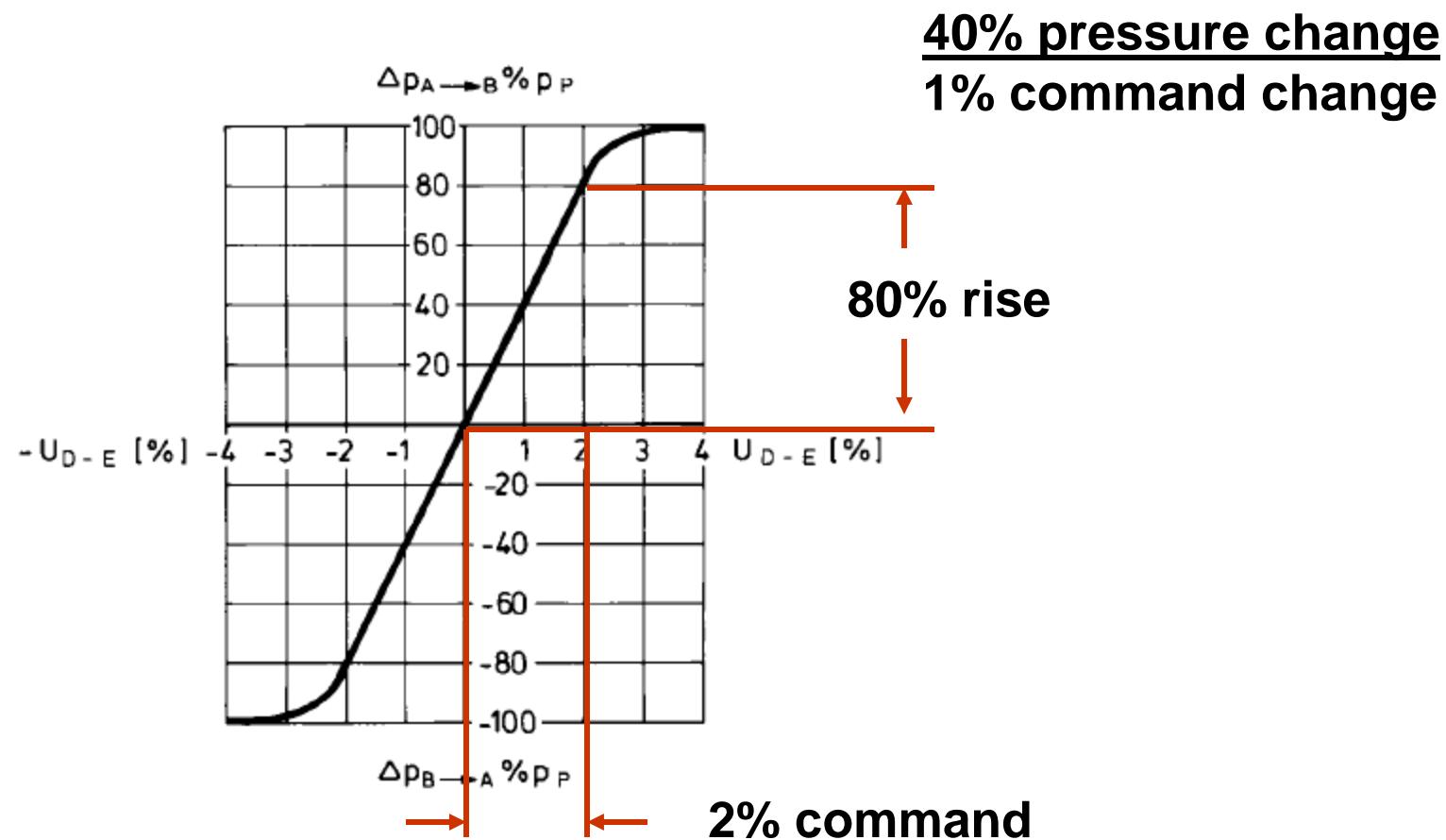
Servo Valve



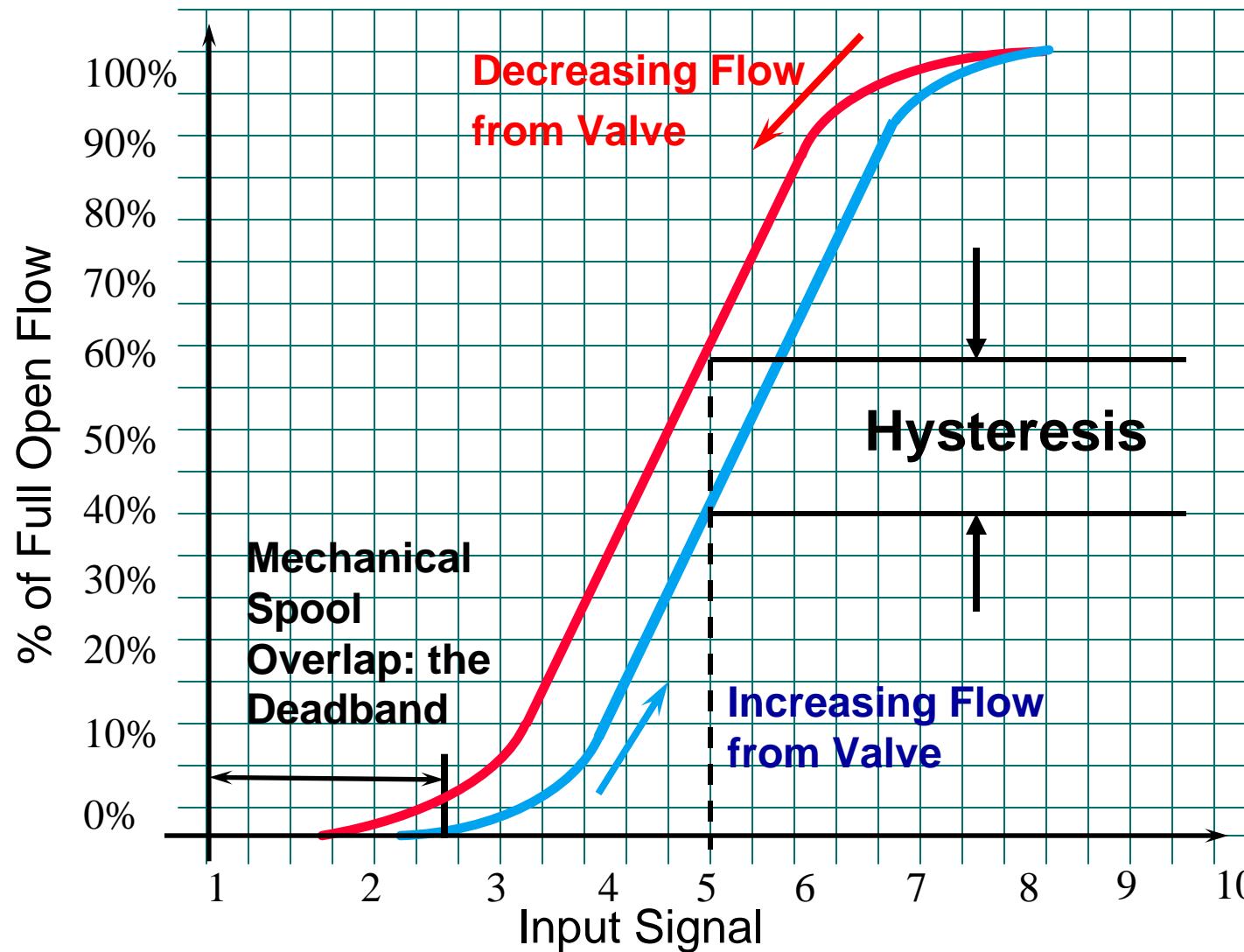
# Terminologies



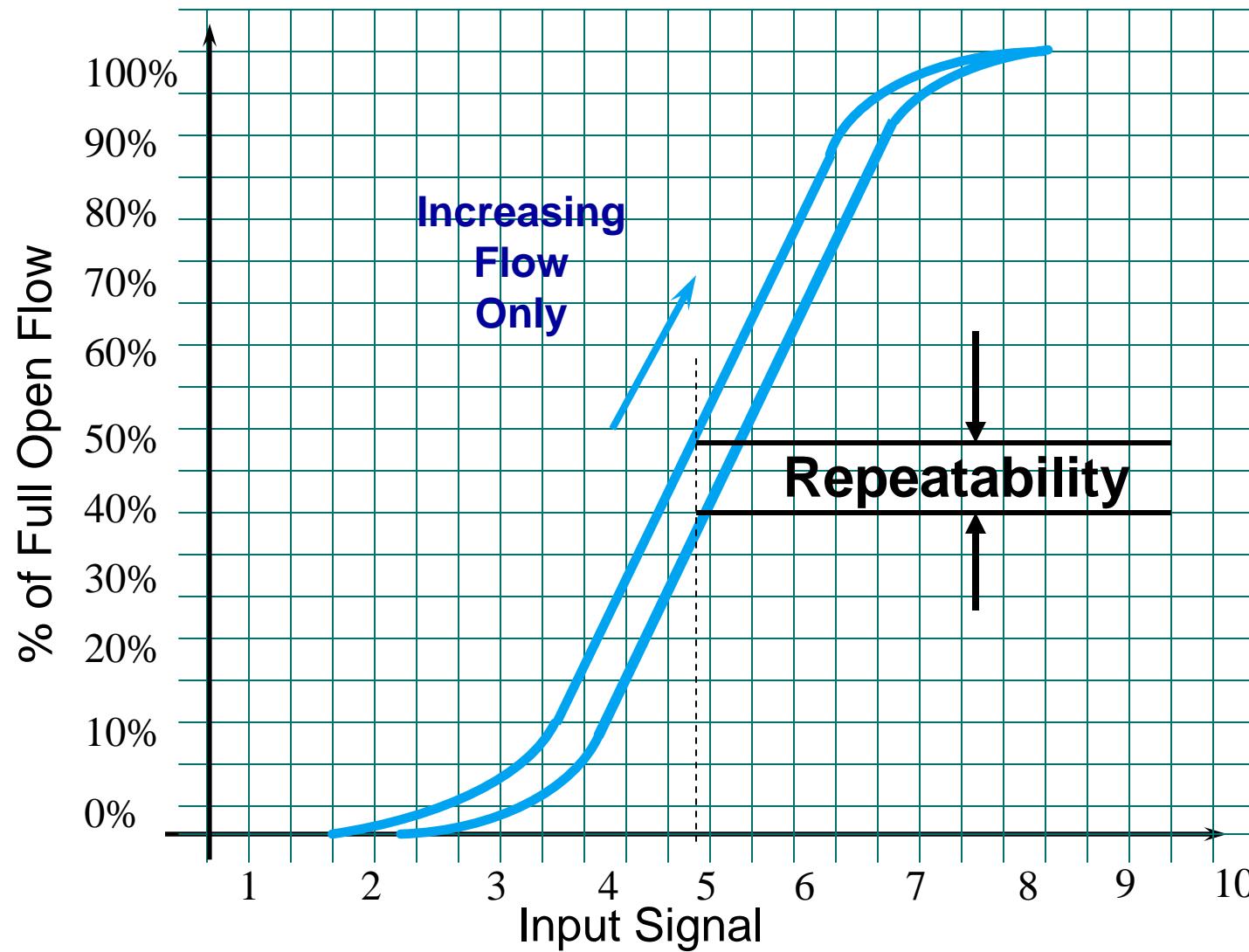
# Pressure Gain



# Hysteresis



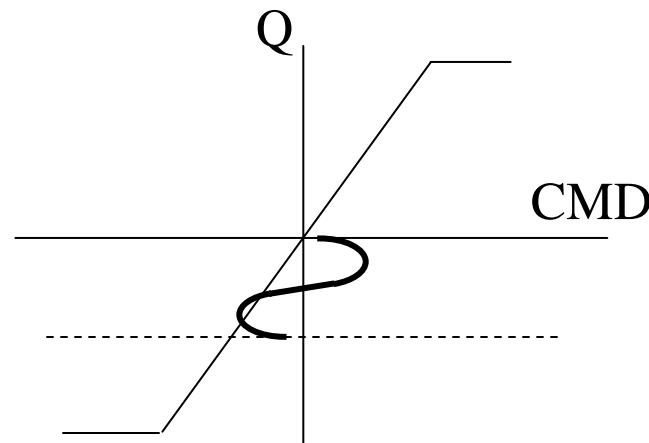
# Repeatability



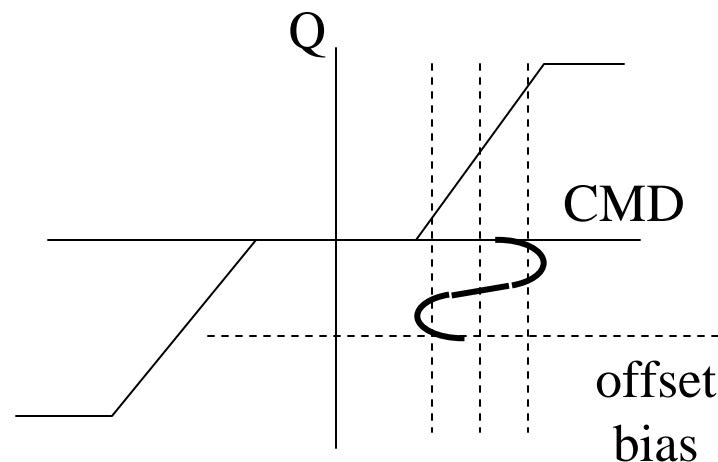
# Valve Frequency

Two methods: frequency response, step response (we'll use frequency response)

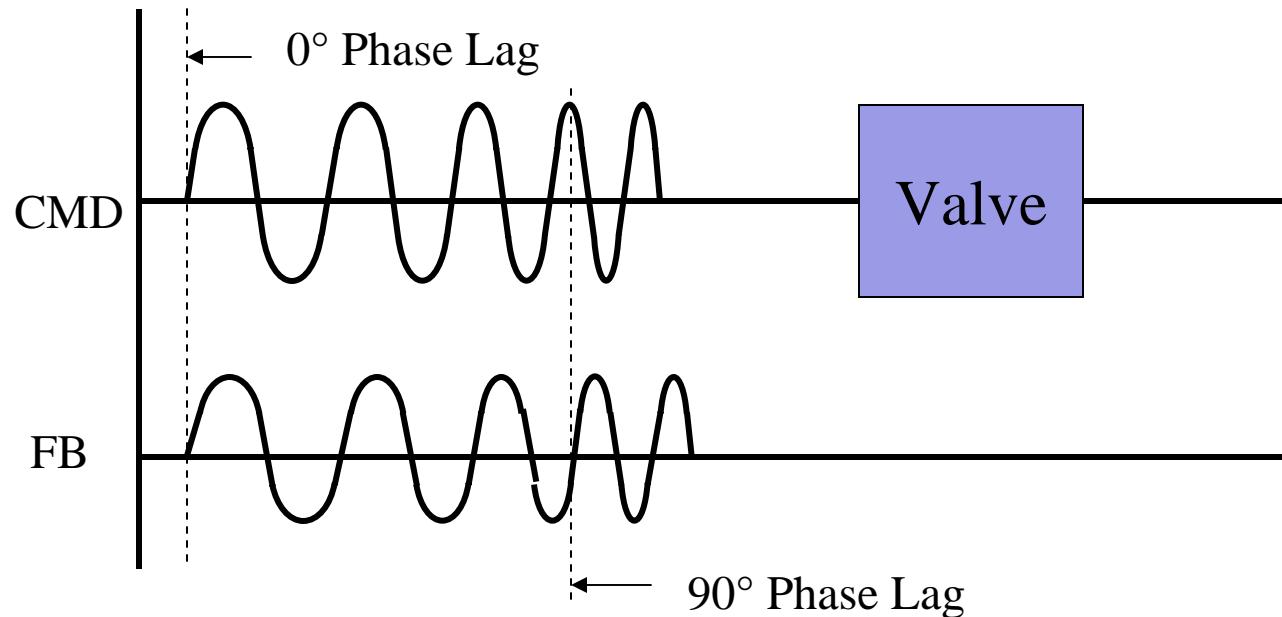
Thru center



Bias or one sided



# Valve Frequency Response



$$90^\circ \text{ Phase Lag} = \frac{1}{4} \text{ cycle}$$

The input frequency which creates a phase lag of  $90^\circ$  is the defining characteristics of a valve and is referred to as bandwidth



# Frequency Response

- To measure the response of a control valve, a sinusoidal varying input signal is applied, effectively switching the valve from one working position to the other
- At a very low frequency the valve is able to follow the demand signal closely
- As the frequency increases the valve becomes less able to follow the input signal precisely



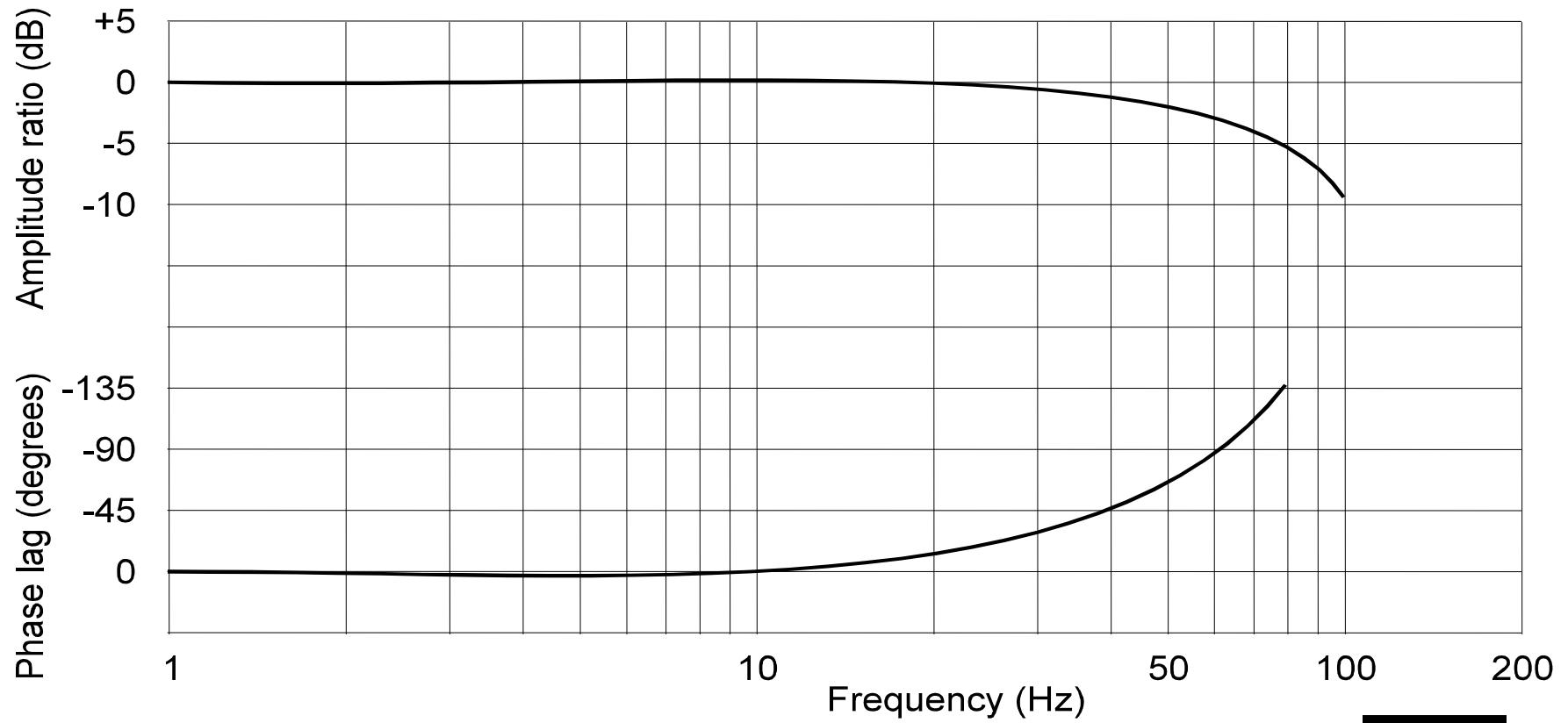
# Frequency Response

- The output starts to lag behind the input, then the valve is not able to reach the maximum output position before the input signal reverses
- The lag between the input and output is known as phase lag
- The reduced output apparent at higher frequencies is known as attenuation



# Valve Frequency

Frequency response (sine wave) to make catalog data



# Proportional Valves

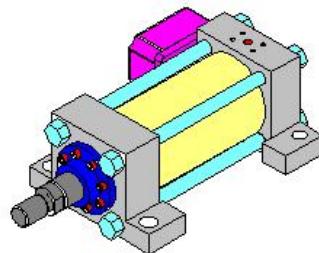
## Servo Valves

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Center Lap Condition	Overlap 5...20%			Zero Overlap	Zero Overlap
Operating Pressures Limits (Drop across metering edge).	1000...2000 PSI			1000...3000 PSI	None
Applications	Open Loop Control		Low End Closed Loop Position	Closed Loop Position & Force	Closed Position & Force

# Actuators

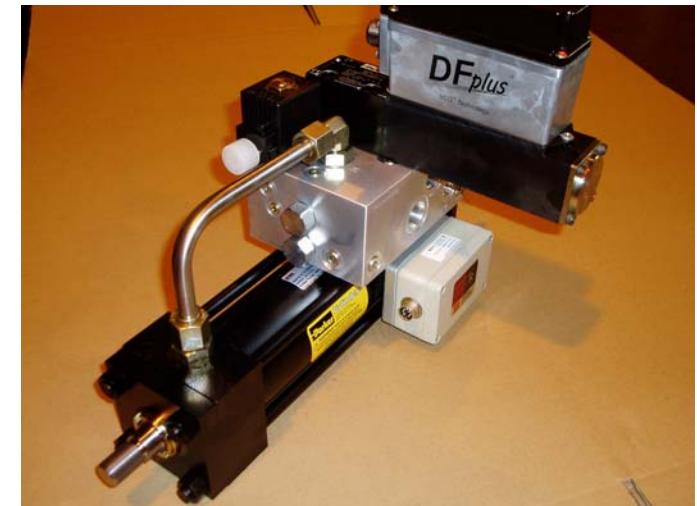
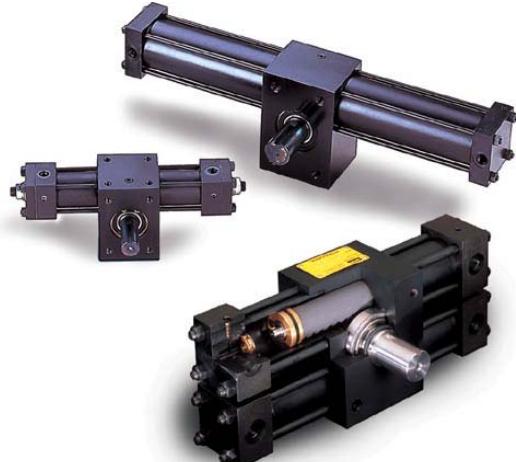
## Rotary Motion

- Position Control (Angle)
- Velocity
- Torque



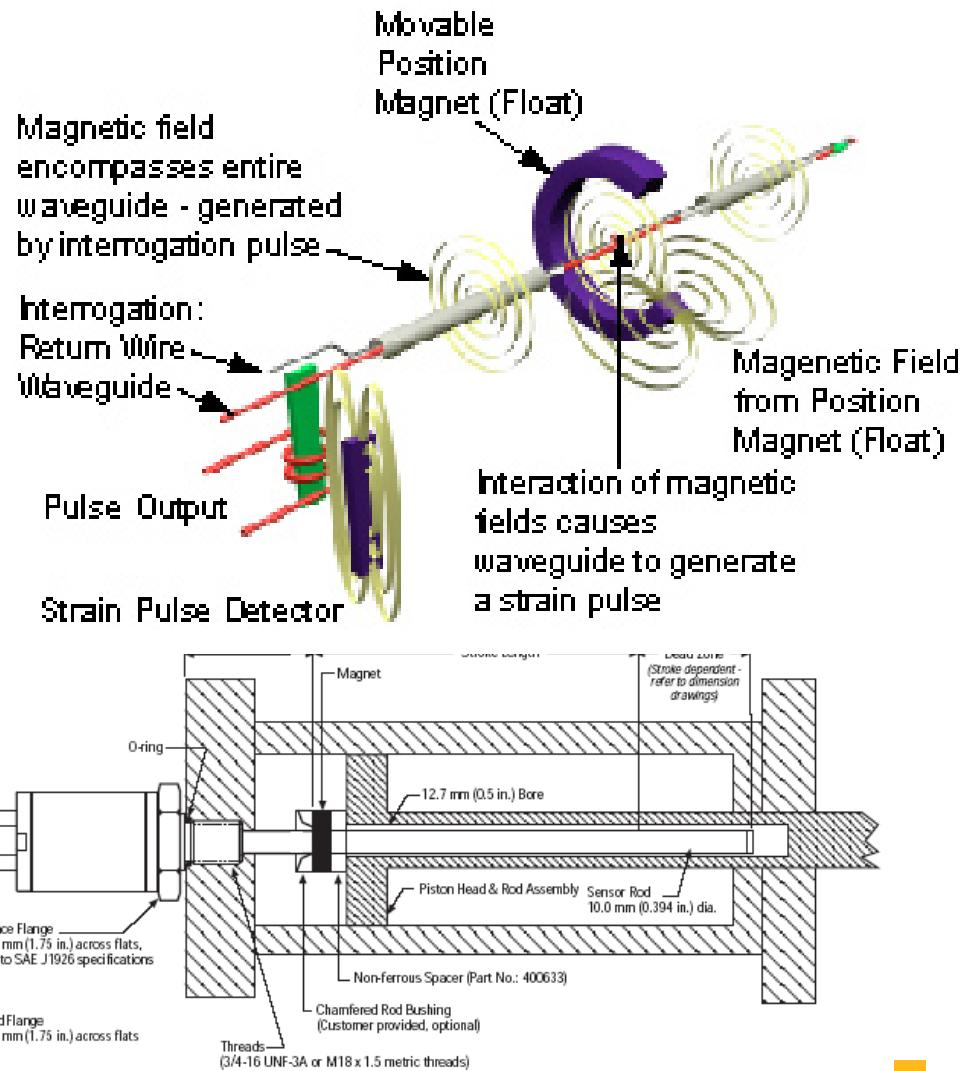
## Linear Motion

- Position Control
- Velocity
- Force



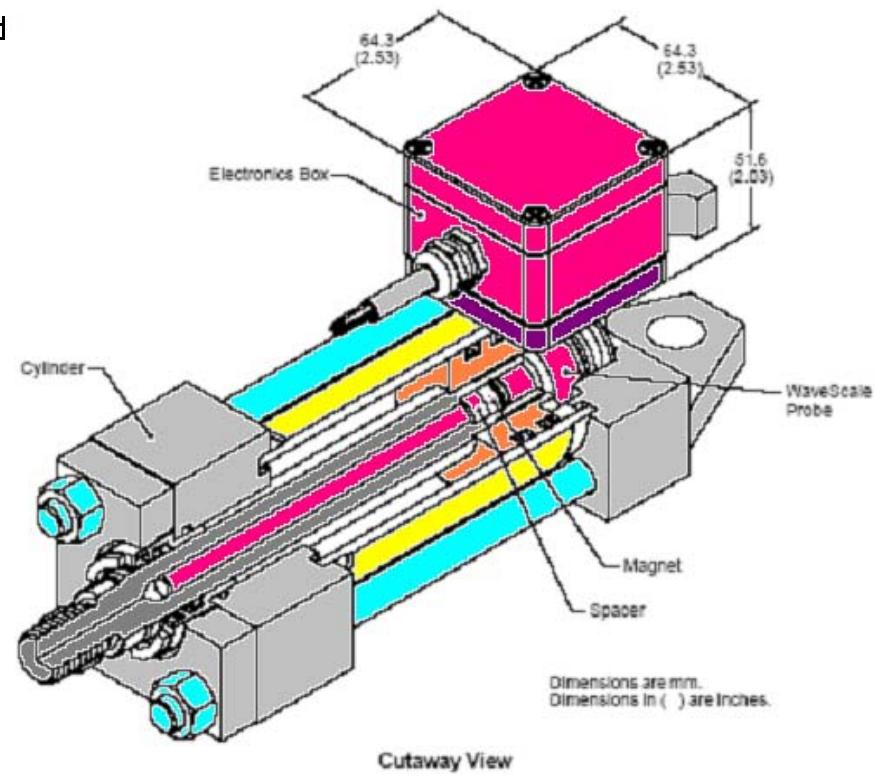
# Actuators Linear Feedback Types

## Magnetostrictive (M.D.T.s)



Pulse is sent down waveguide, when hits magnet, “twist” is sensed. Time between pulse sent to twist measured dictates distance.

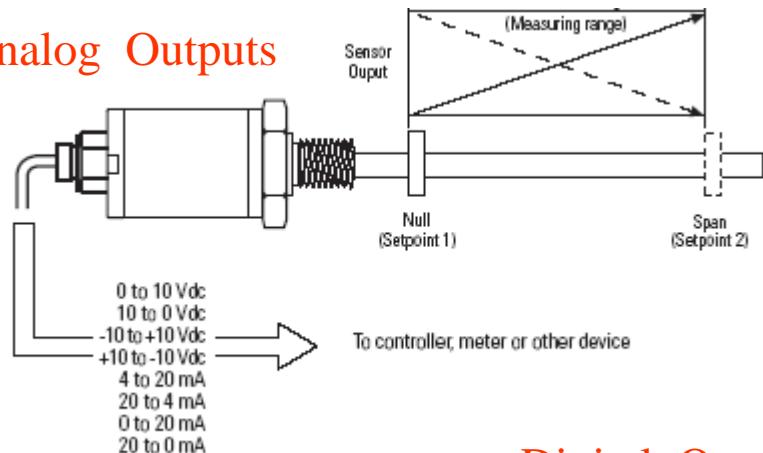
Approximately 9 microseconds = 1"



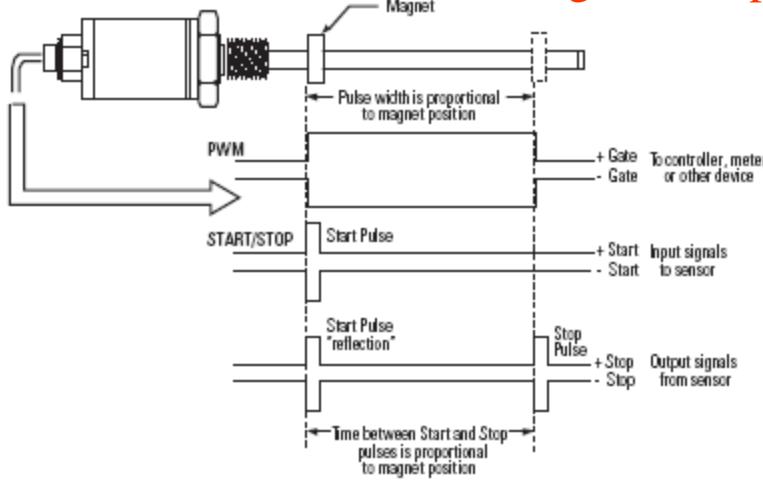
# Actuators Linear Feedback Types

## Magnetostrictive (M.D.T.s)

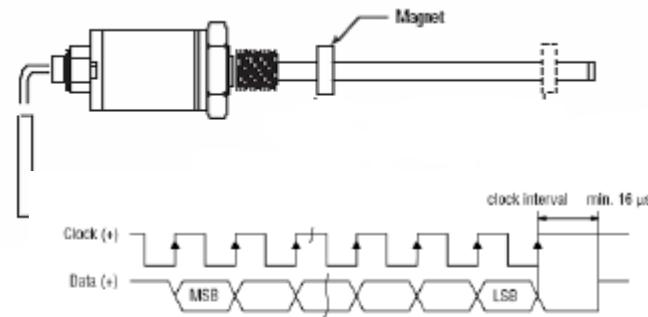
### Analog Outputs



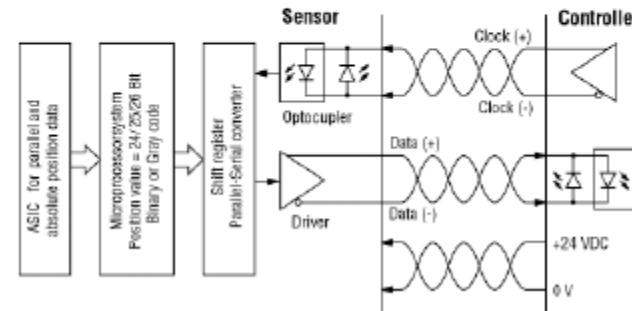
### Digital Outputs



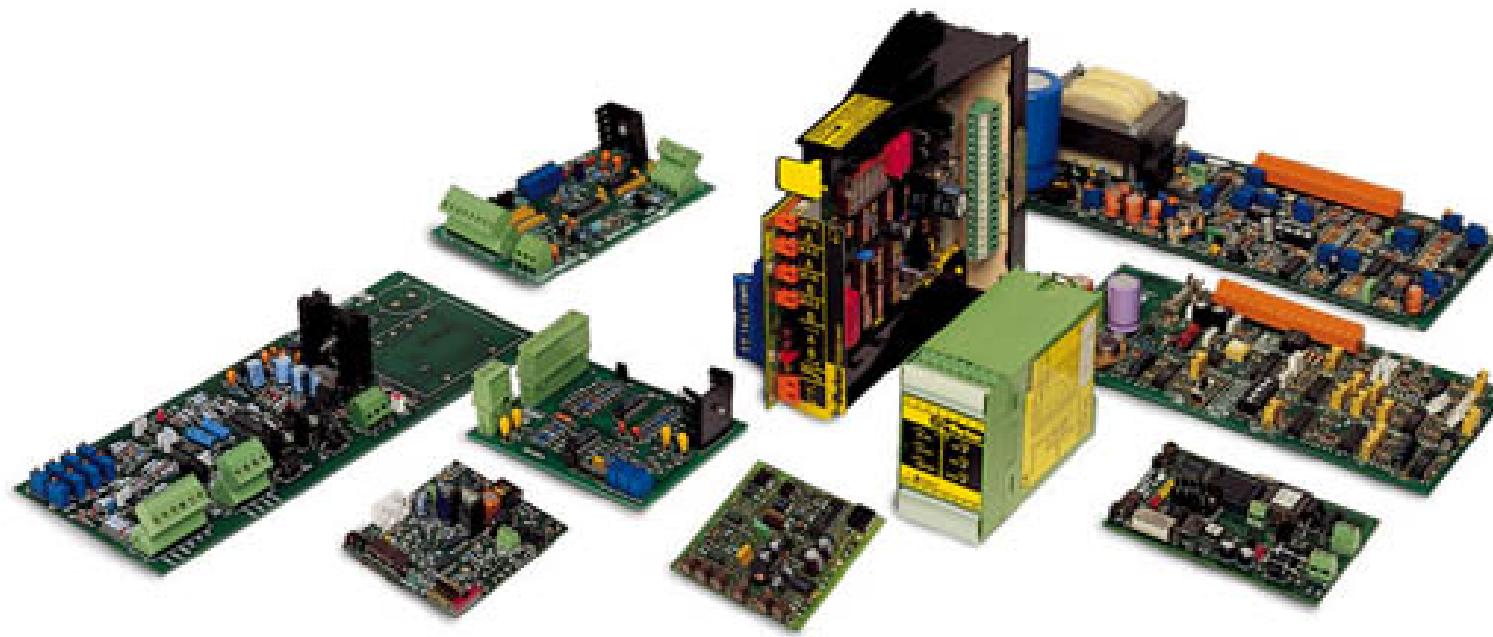
### SSI Output



### Logic diagram



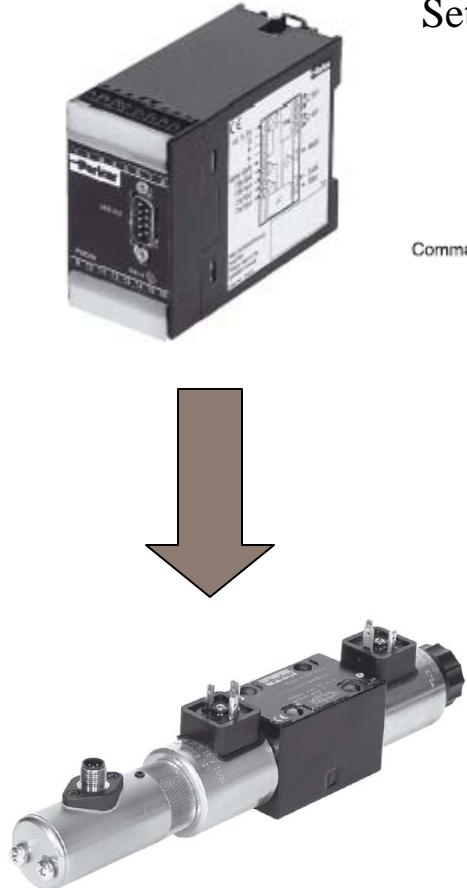
# Valve Drivers and Motion Controllers



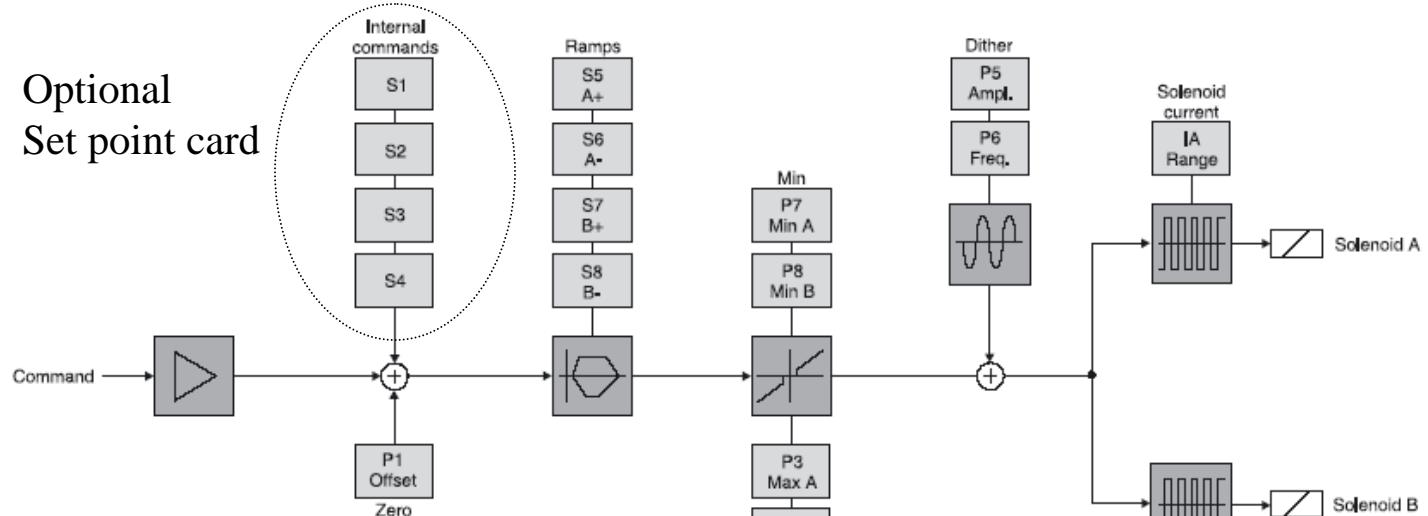
**Parker**

# Valve Drivers (Open Loop) Off Board

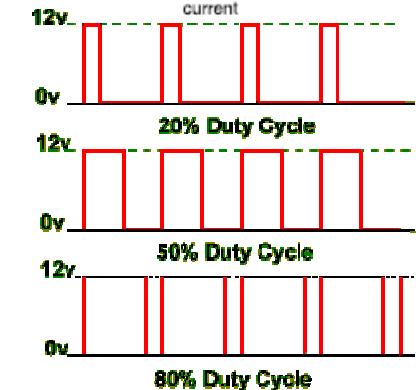
## Elec



Optional  
Set point card

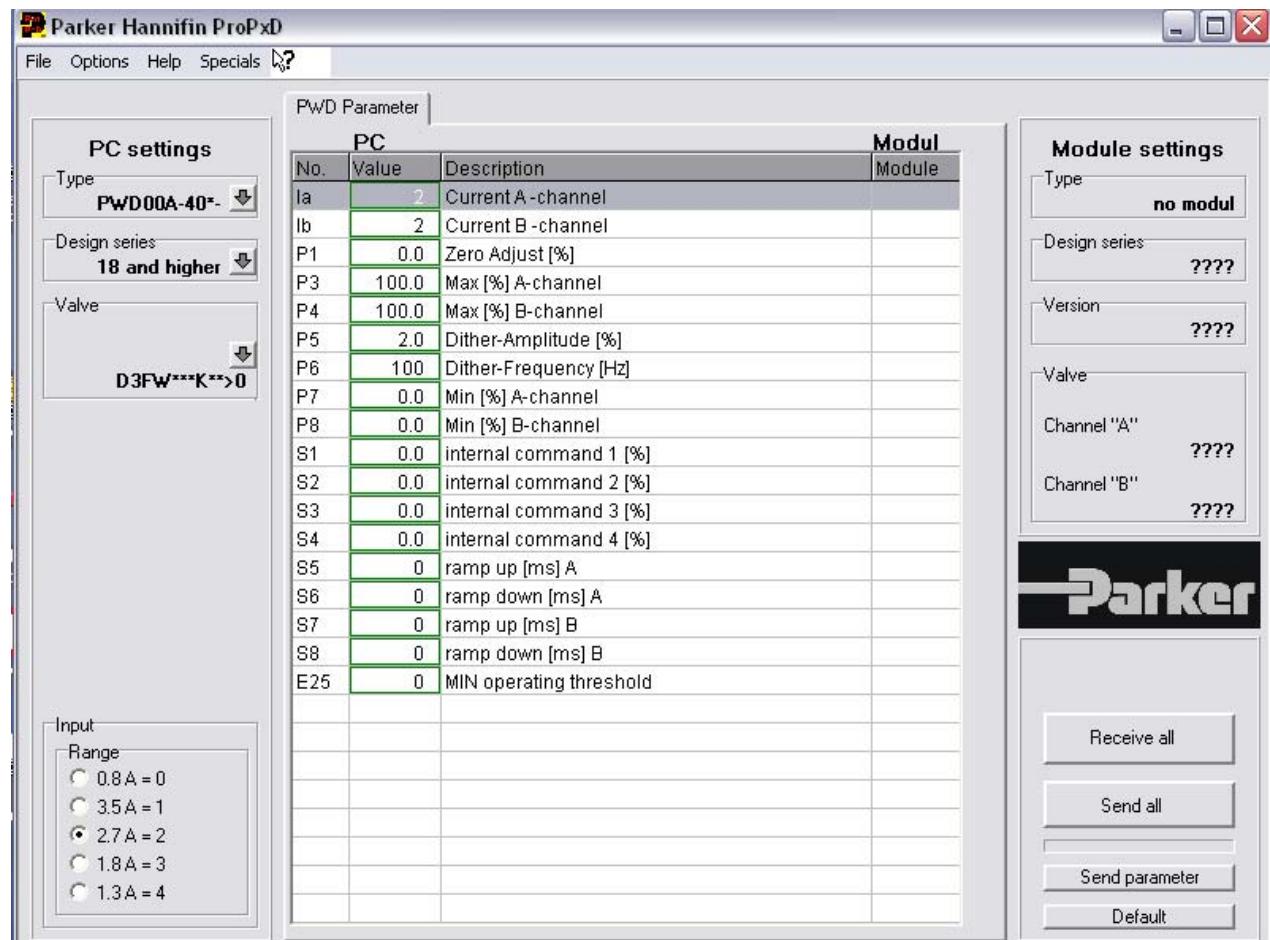
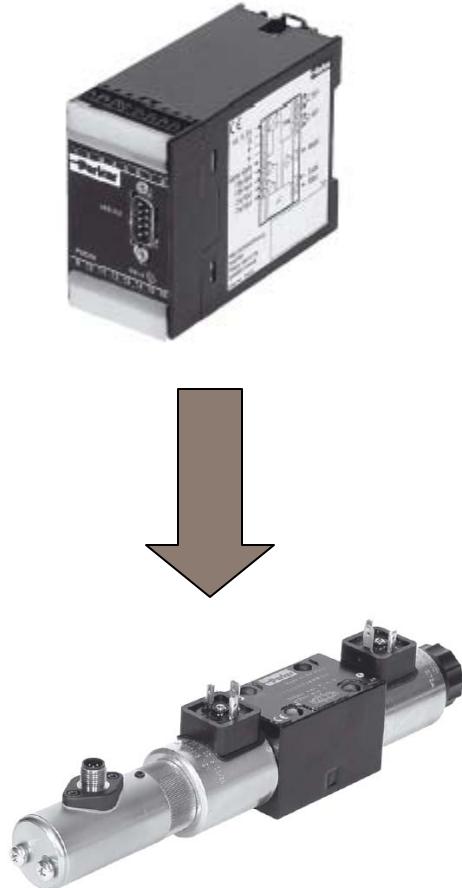


- Converts “Command Signal” to PWM signal to drive Coil.
- Digital Versions (shown) incorporate a microprocessor with numeric settings.
- Analog Versions incorporate Trim pots for

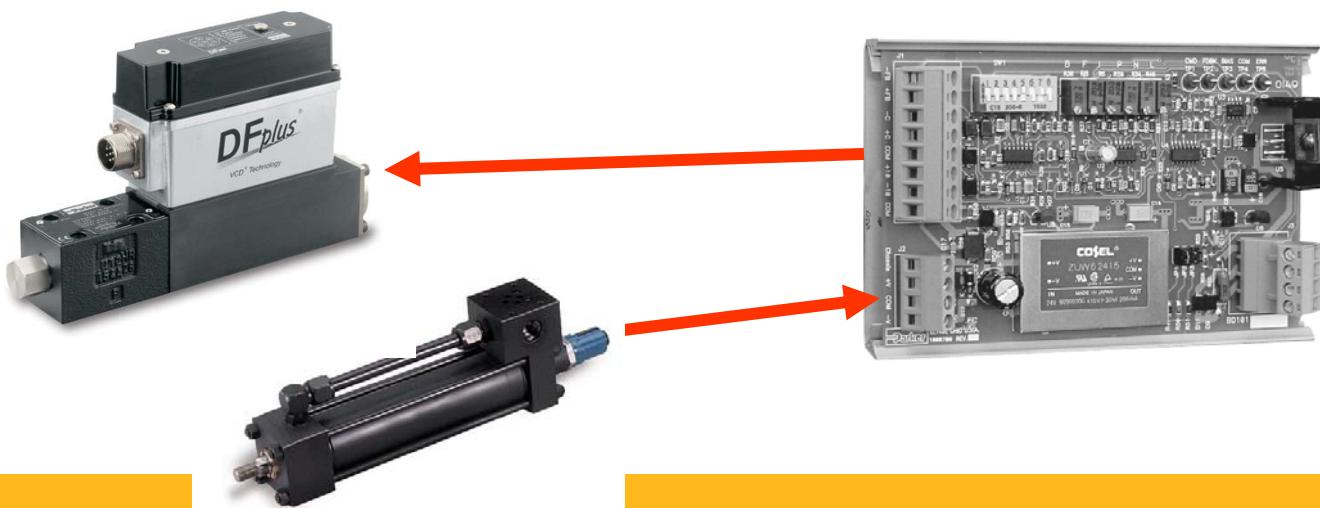
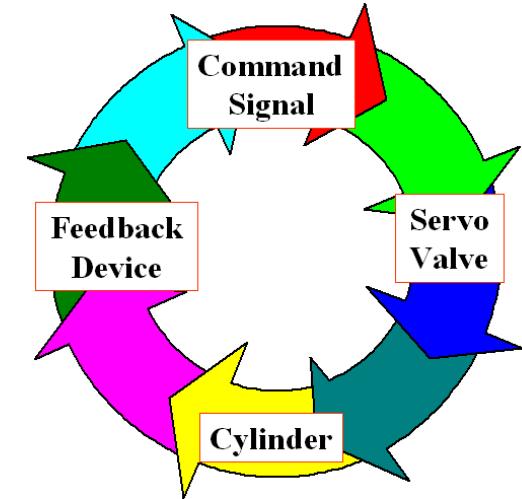
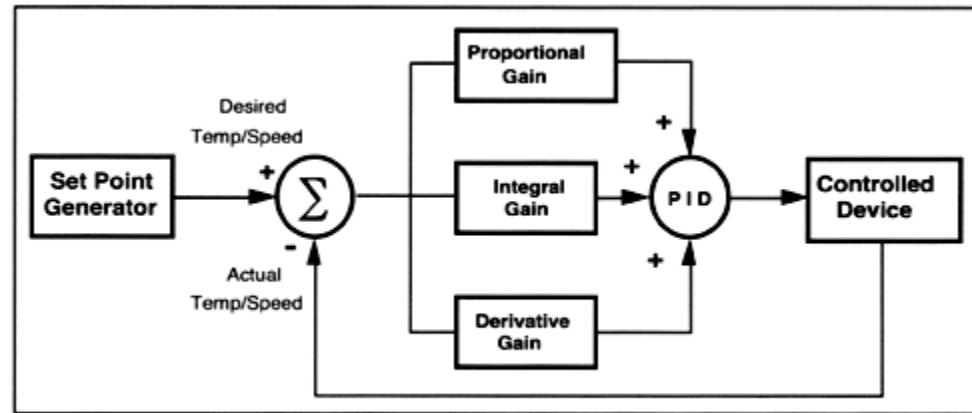


**Parker**

# Valve Drivers (Open Loop) Off Board Elec



# Valve Drivers (Closed Loop)



# Valve Drivers (Closed Loop)

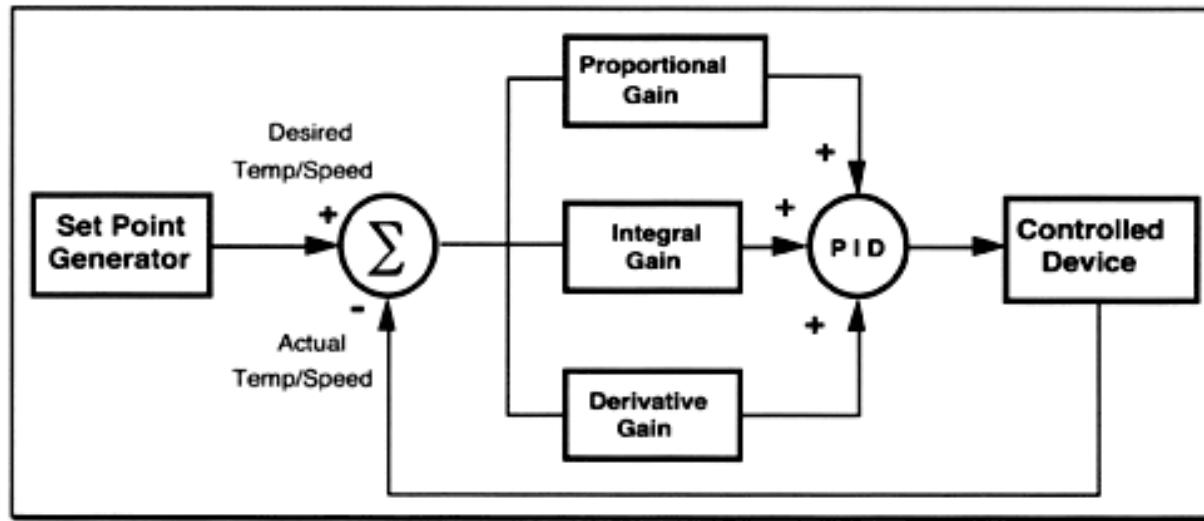
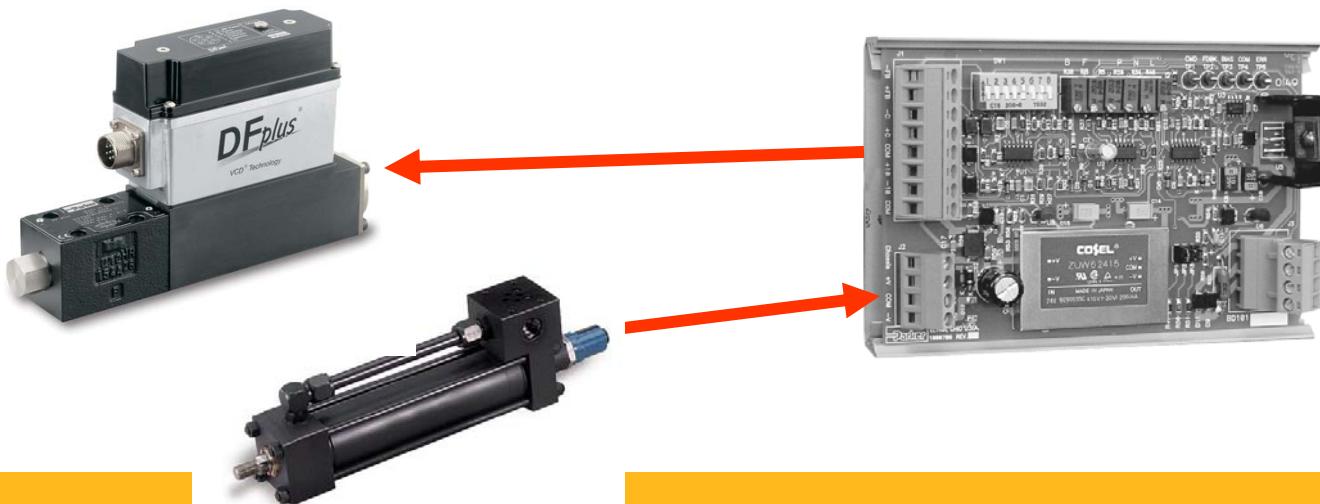
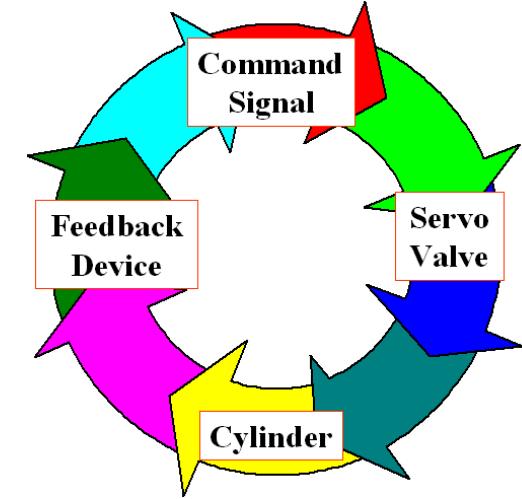
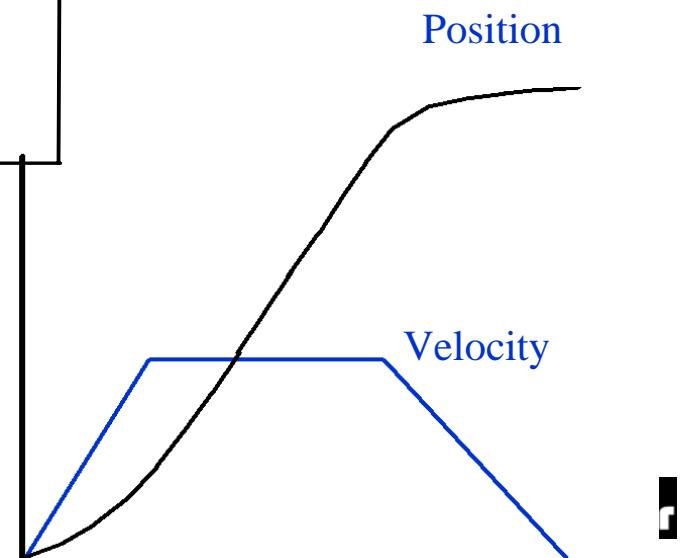
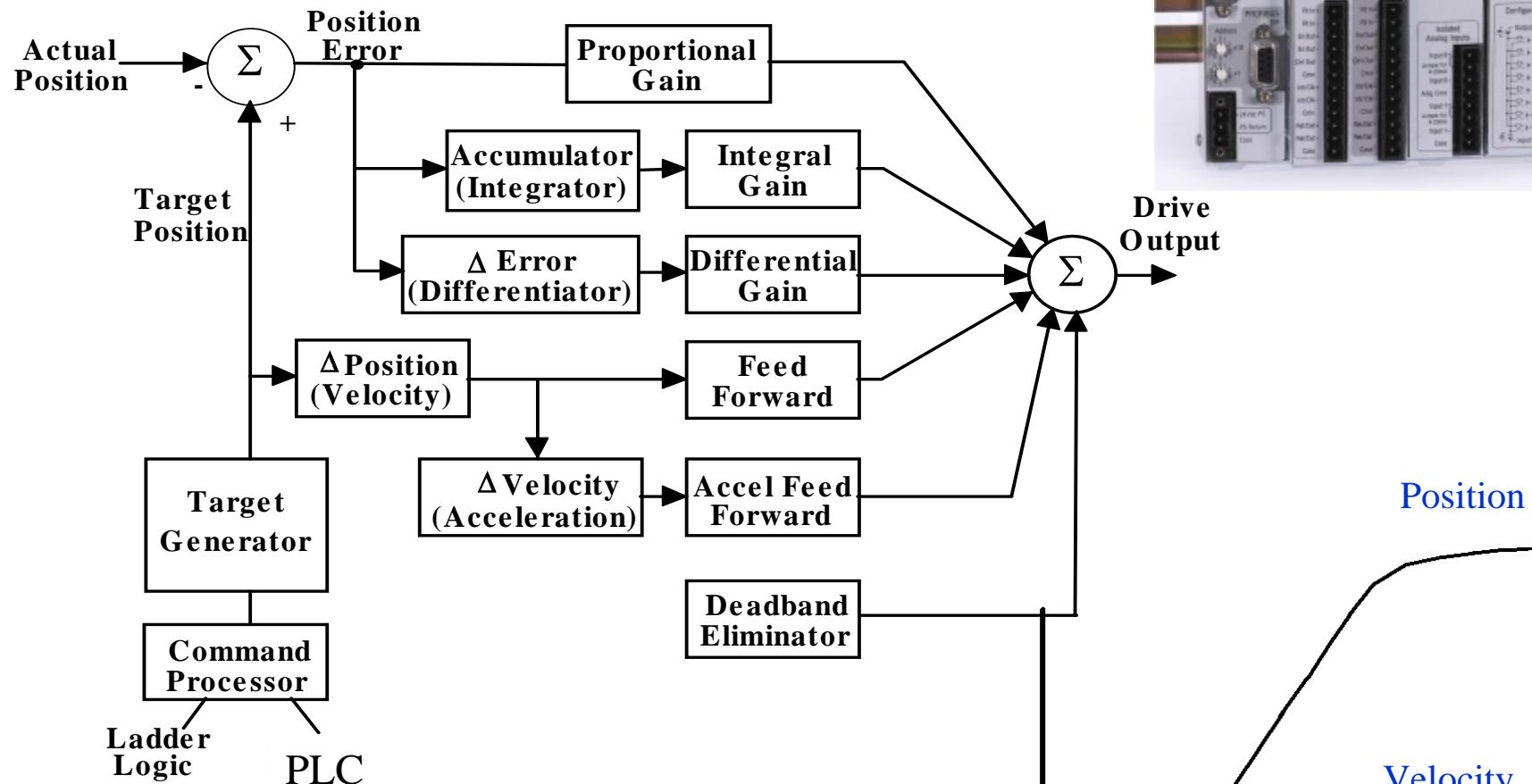


Figure 1 - PID Control Loop



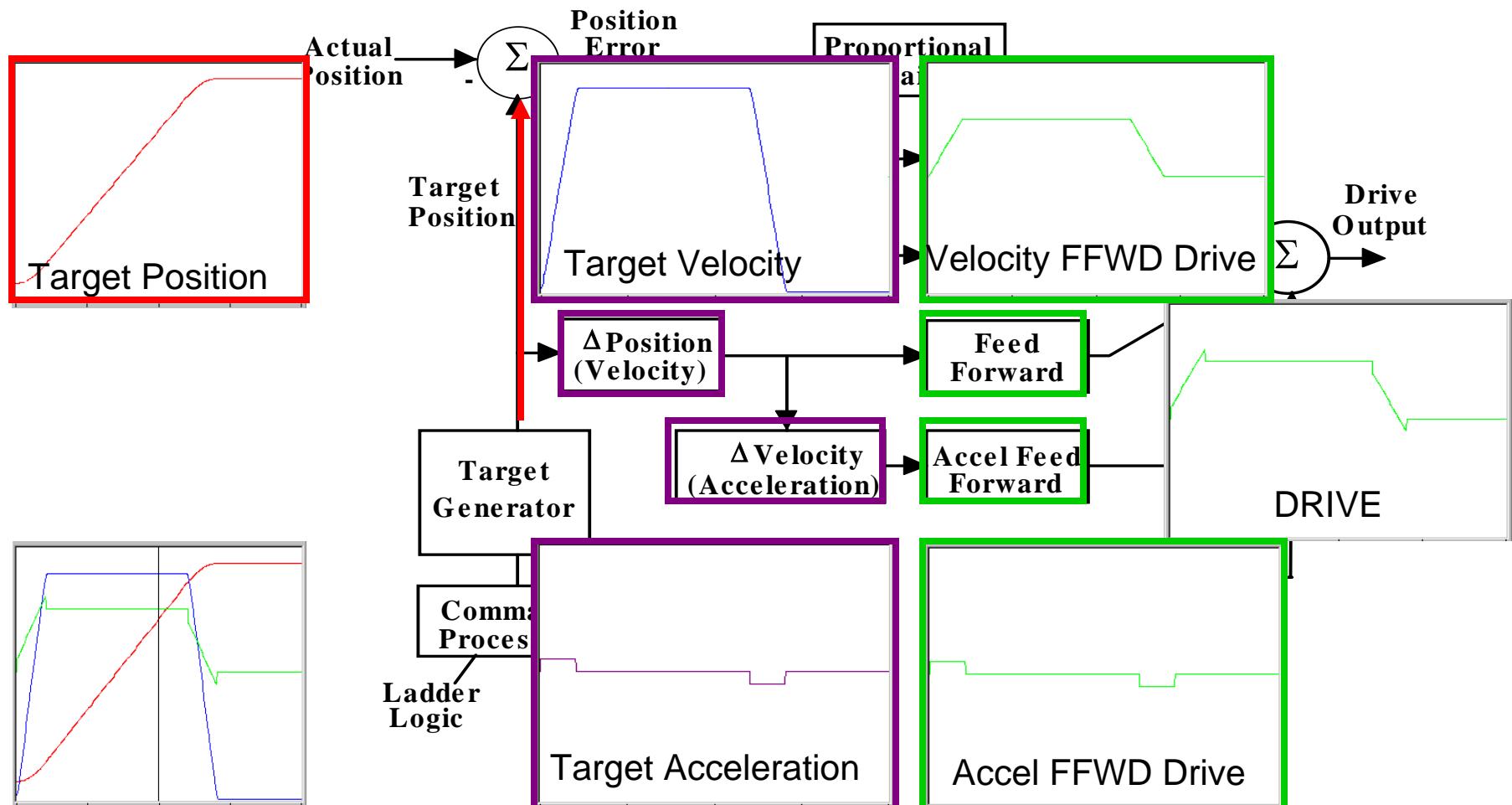
**Parker**

# Motion Controllers

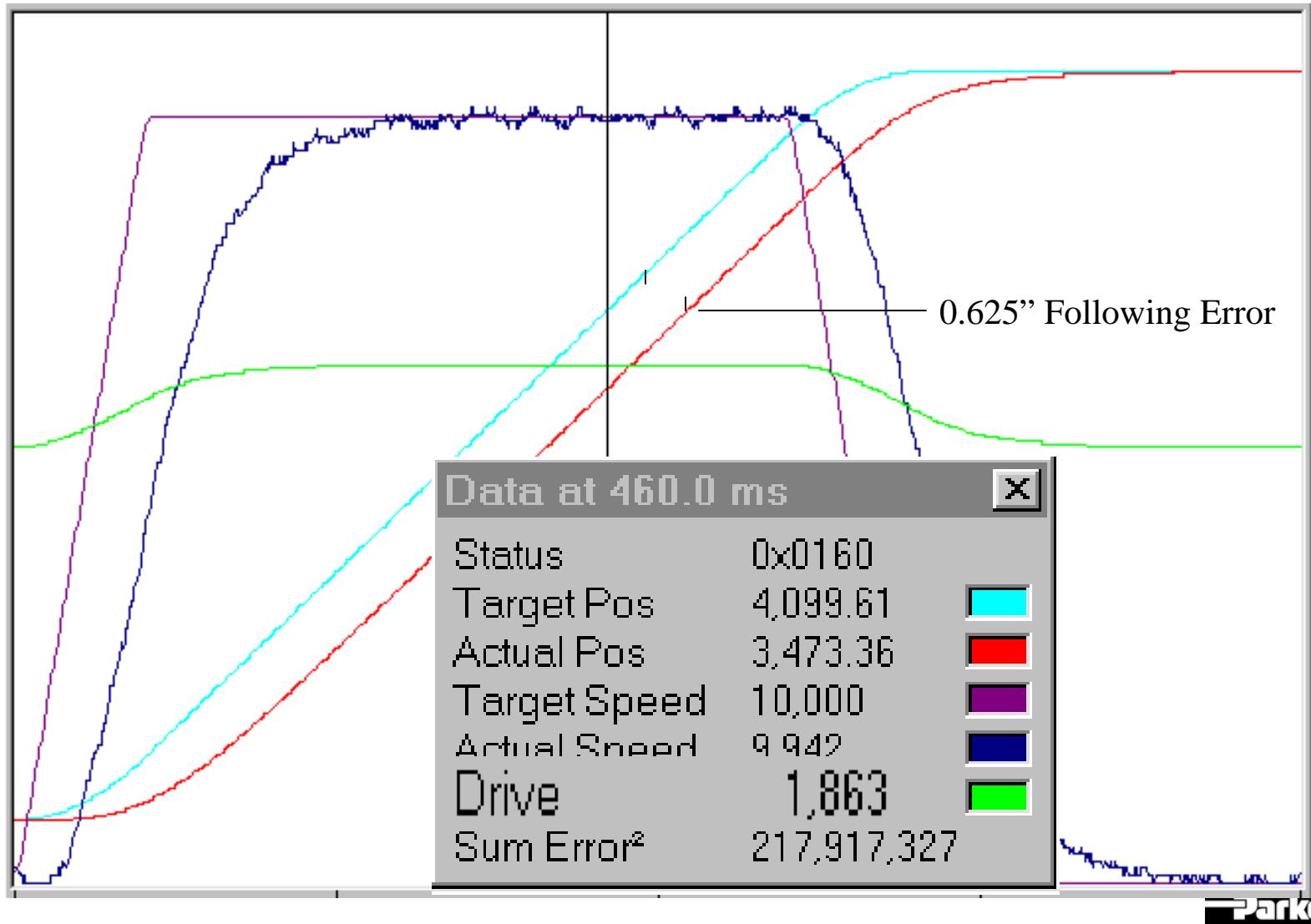


“Generate A Target Profile”

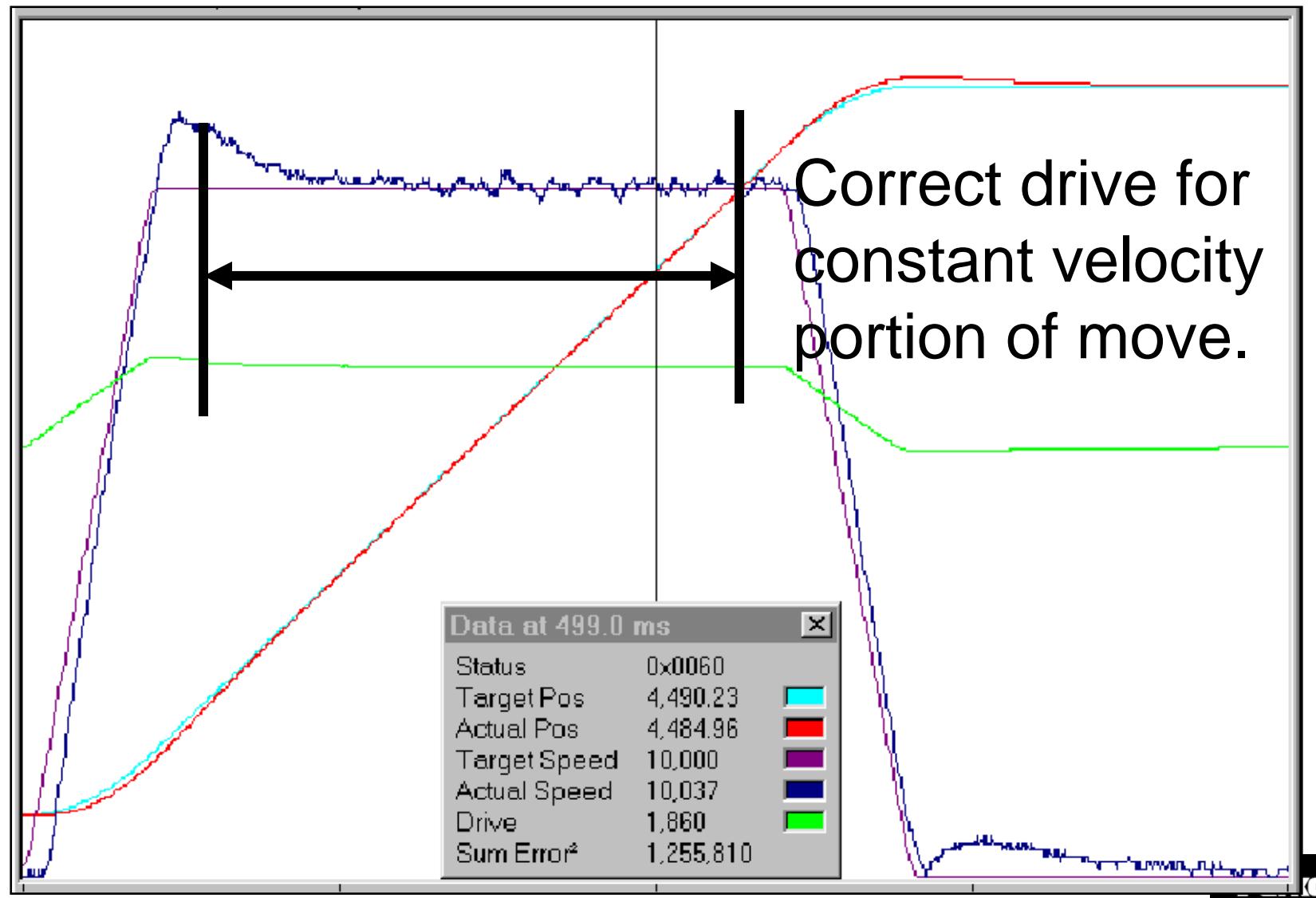
# Velocity and Accel Feed Forward



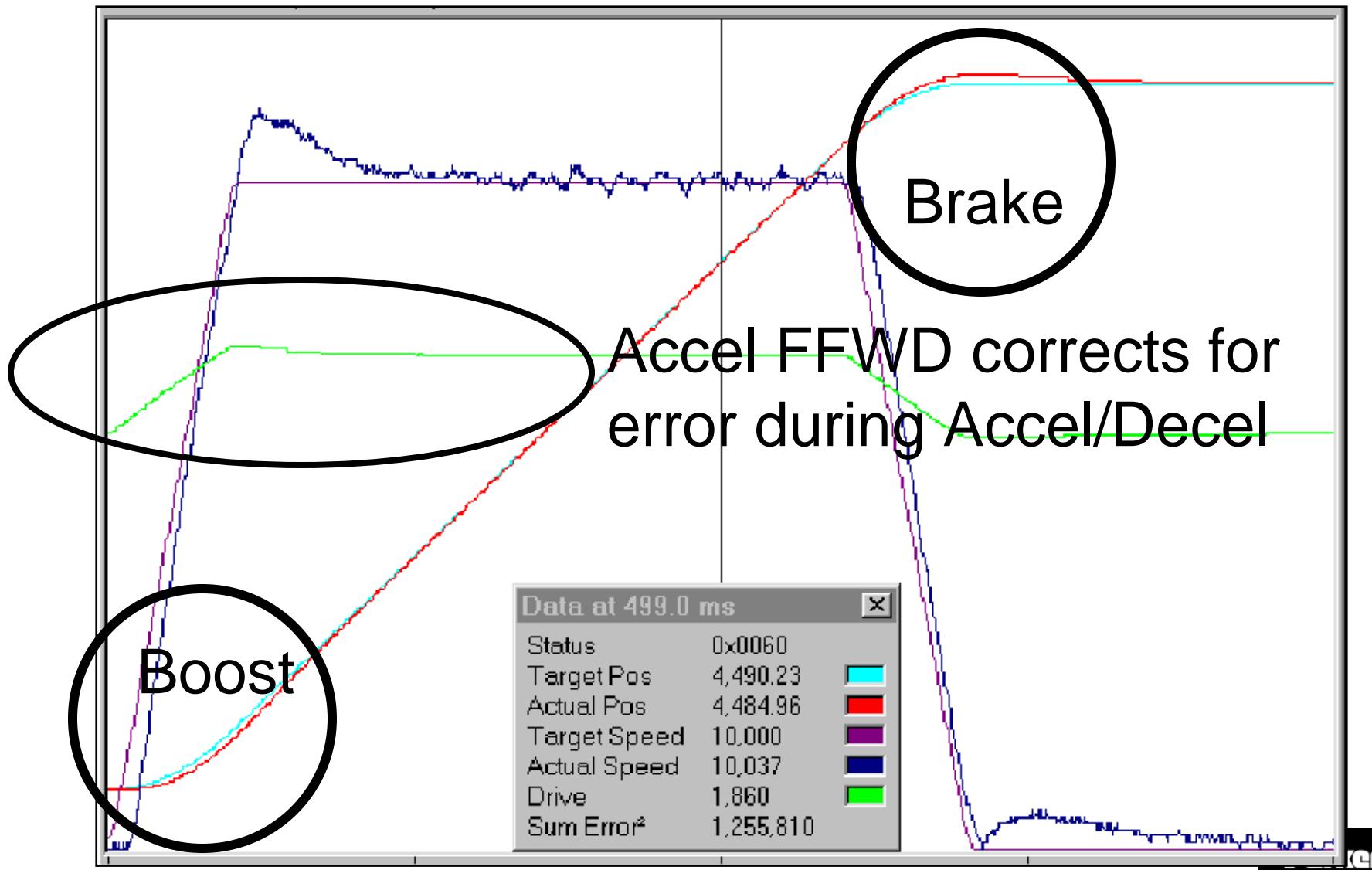
# Proportional Gain ONLY



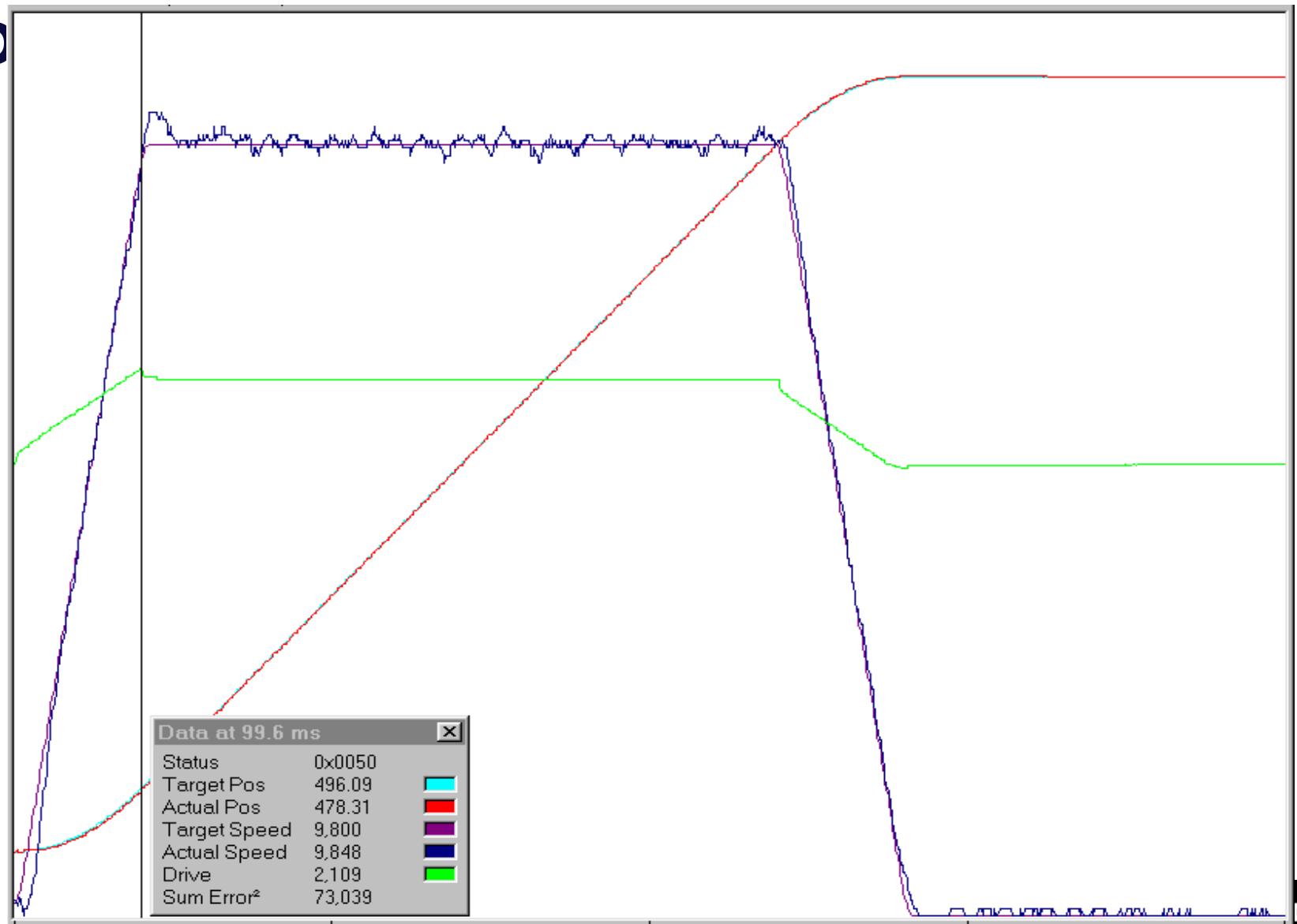
# Feed Forward Adjust “F” Cmd



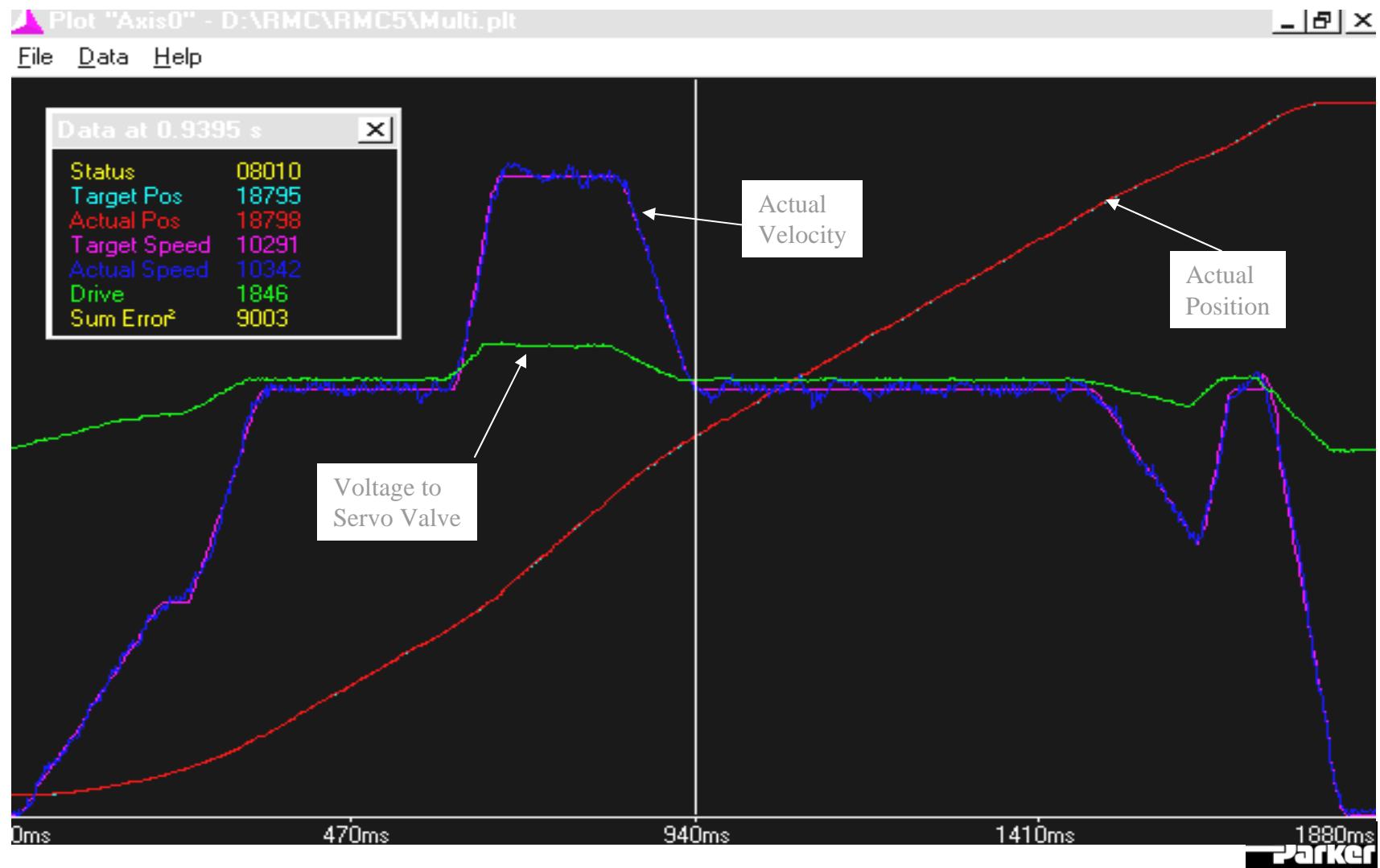
# Position Error during



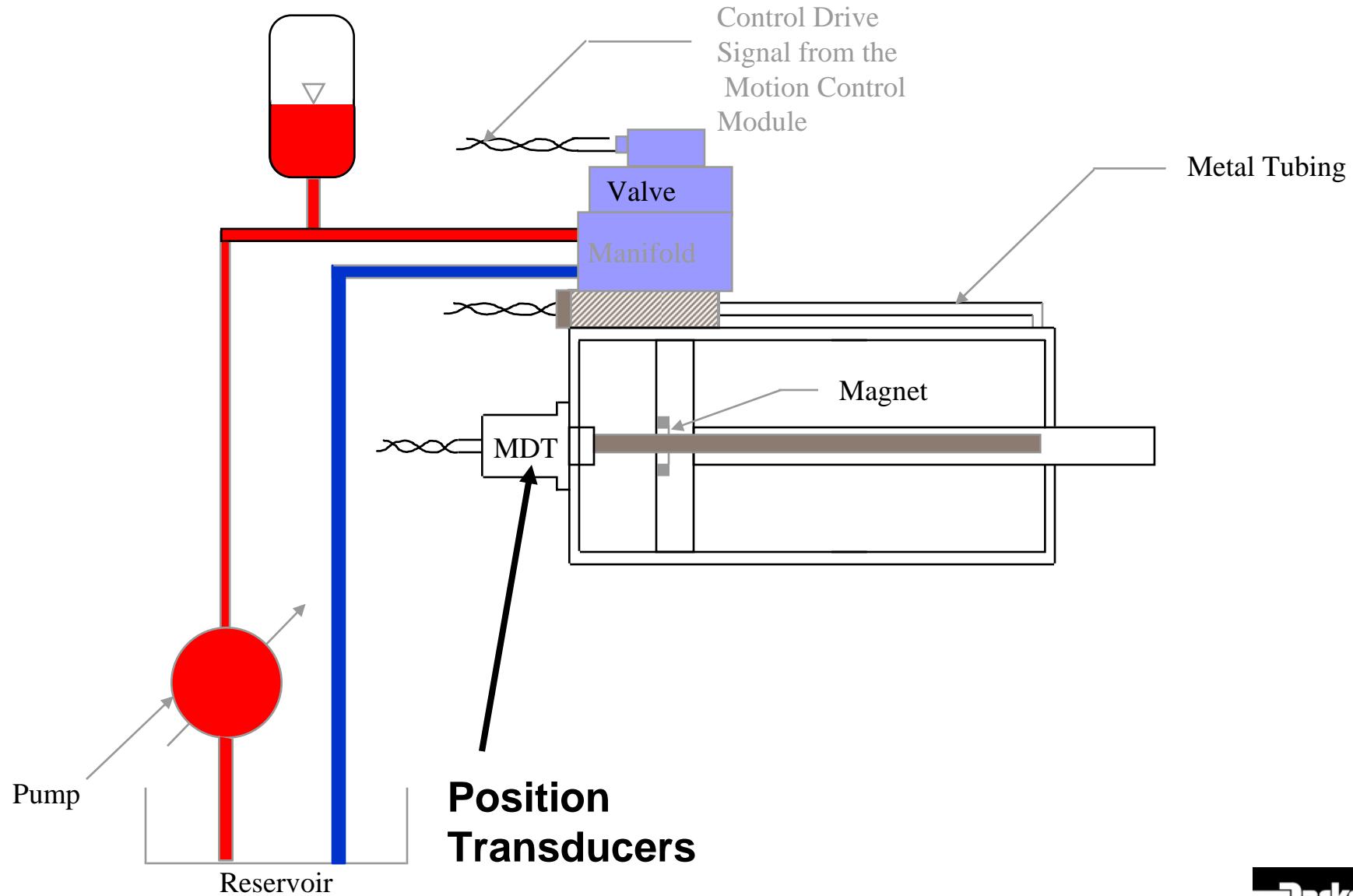
# Proper Velocity and Accel Feed for



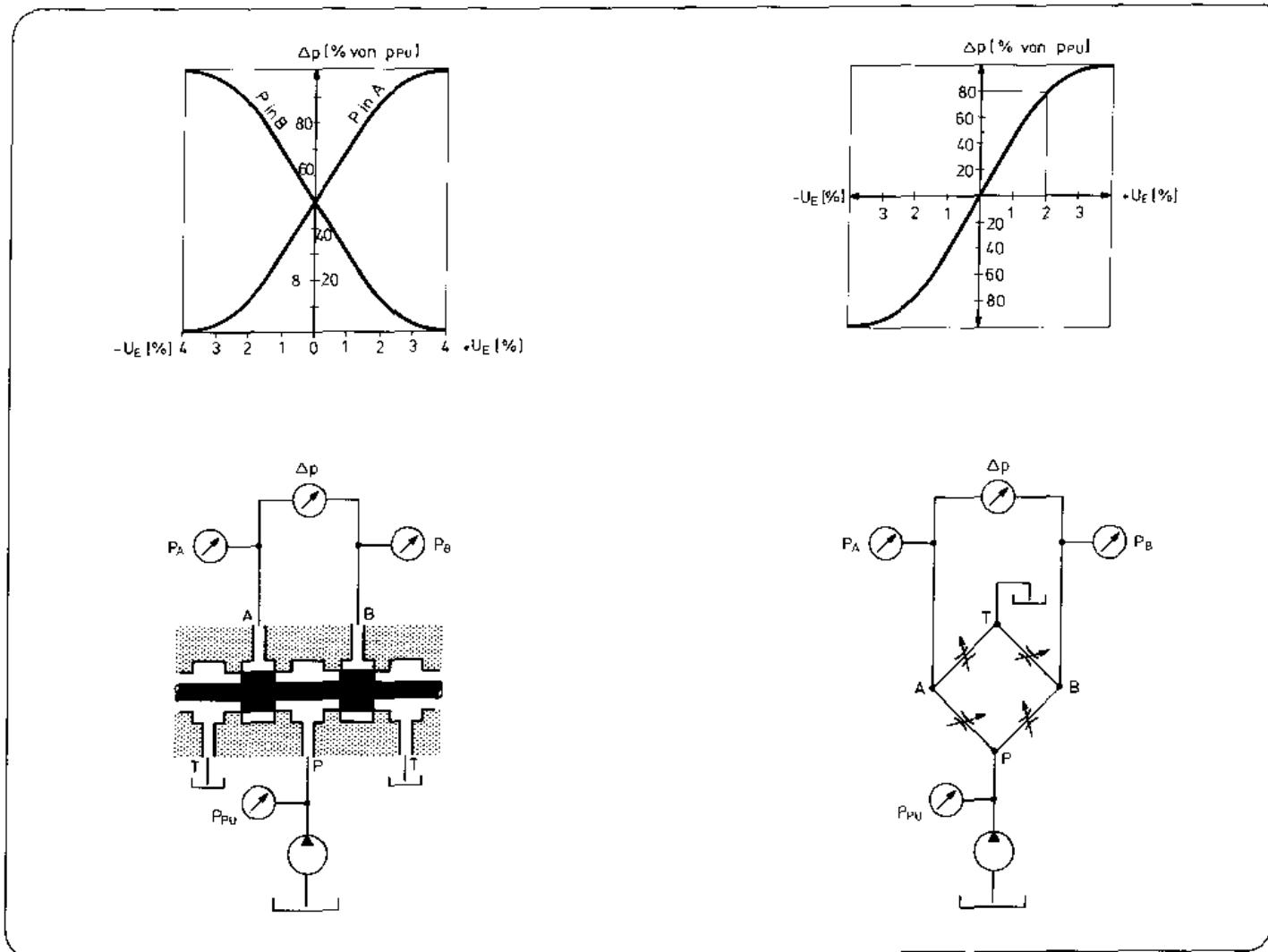
# Real Time Plot Data



# Closed Loop Position System

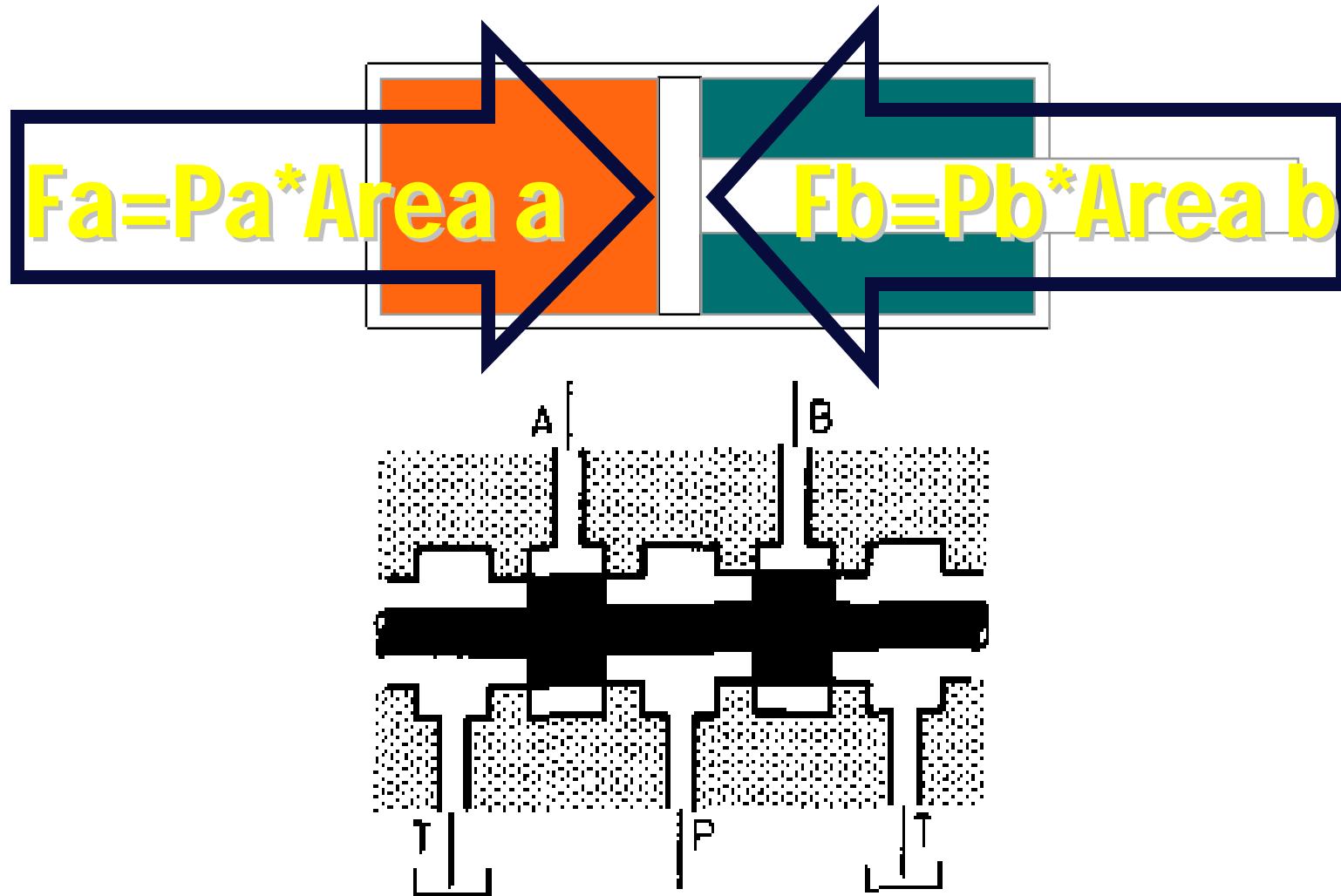


# Principle of Operation Force Balance



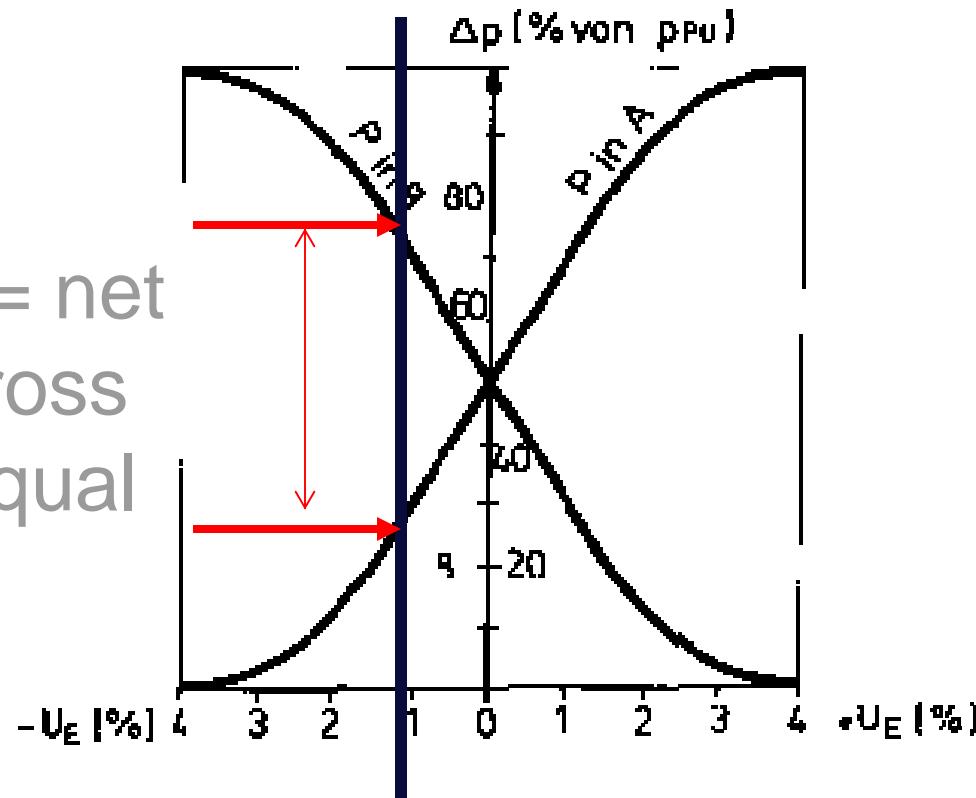
**Parker**

# Balanced Force

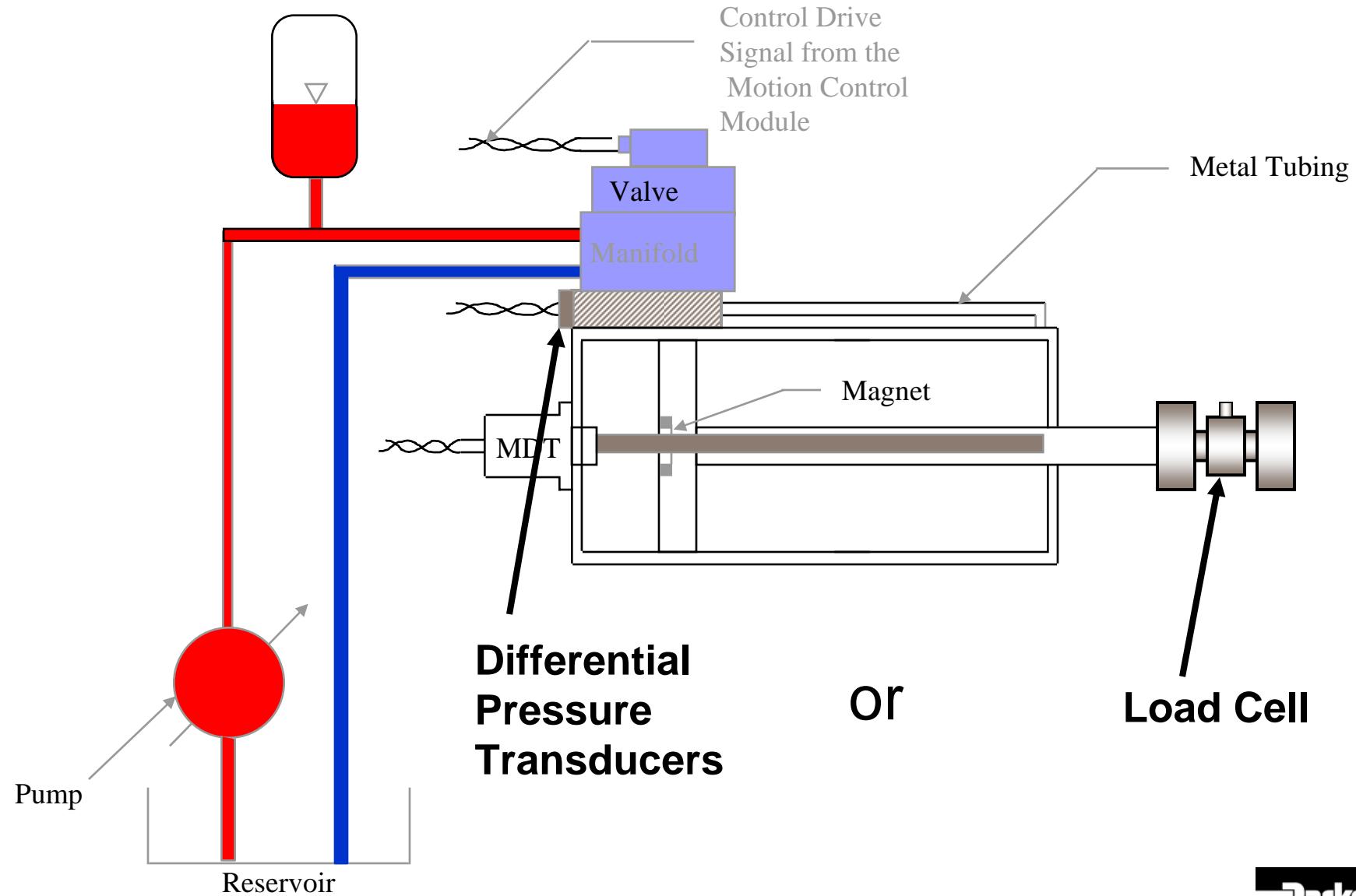


# Principle of operation (cont)

Valve null = net forces across cylinder equal zero

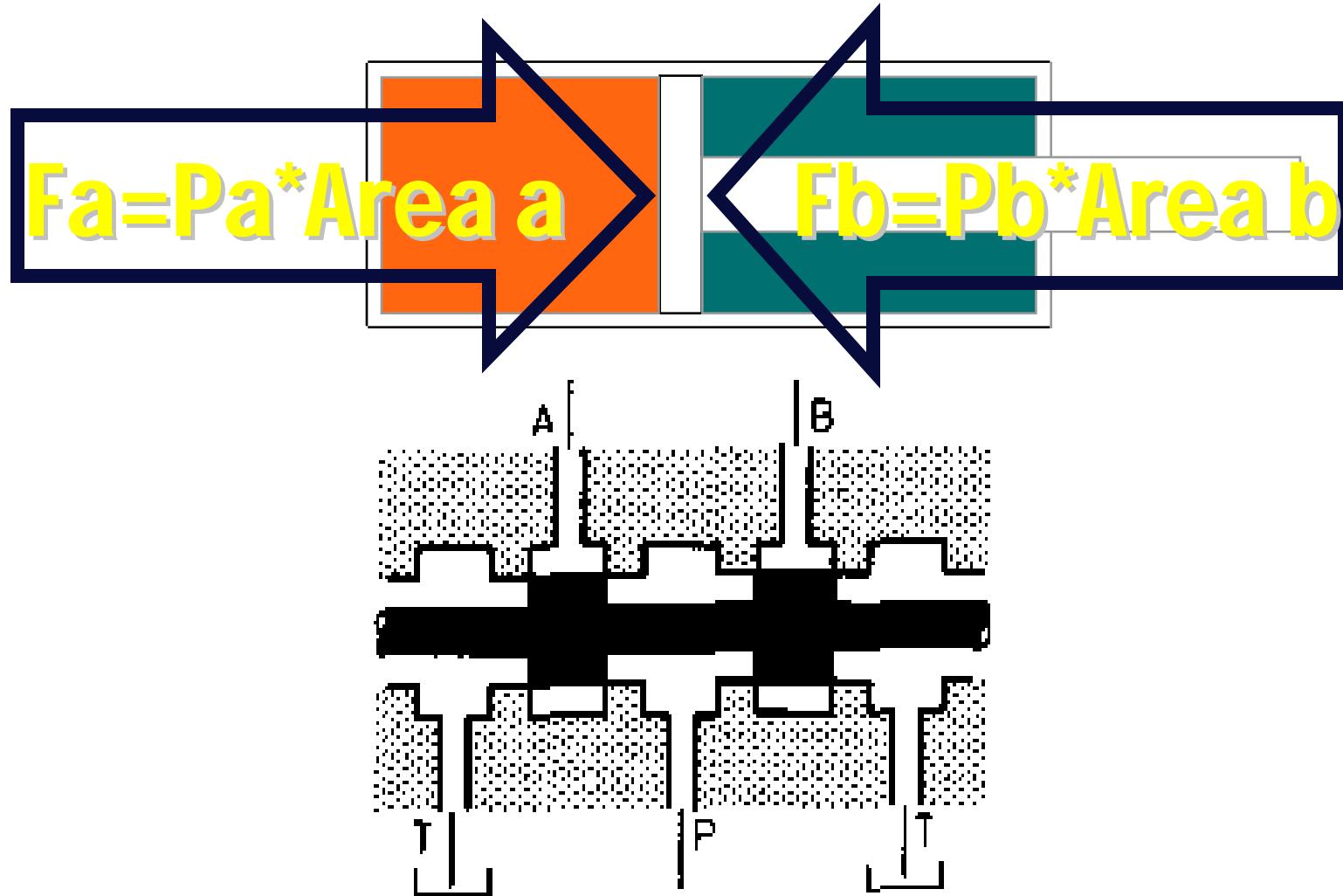


# Closed Loop Force Control



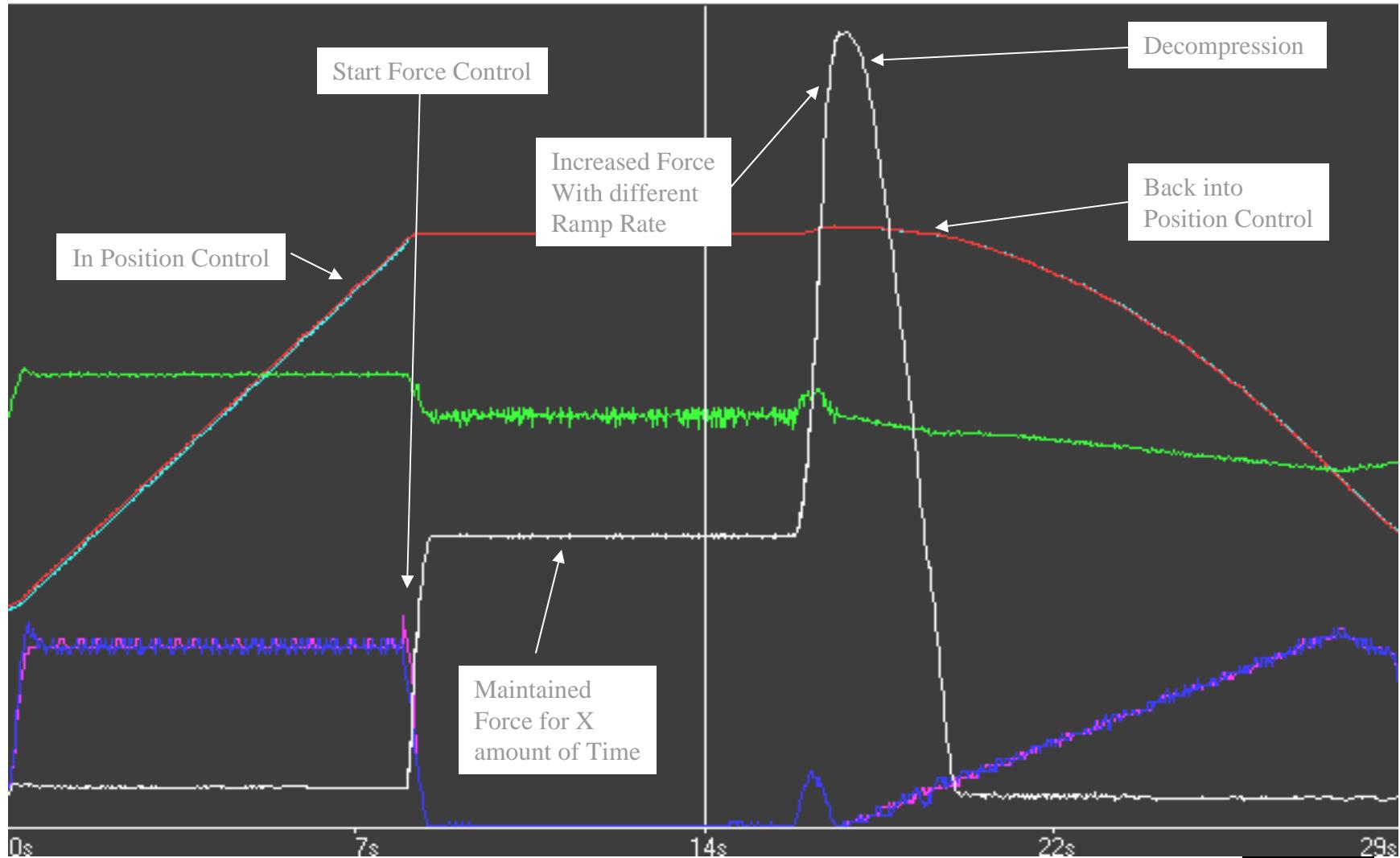
Parker

# Balanced Force



# Force Control

File Data Help



# Thank you!

