# Welcome to

## Pump

Installation & Maintenance

by Fayek Shakran

#### Course Objective

identify pump classifications

describe the purpose of pump parts.

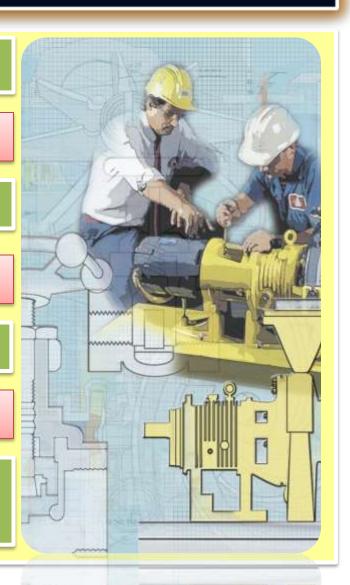
define various pump terms.

explain pump operation.

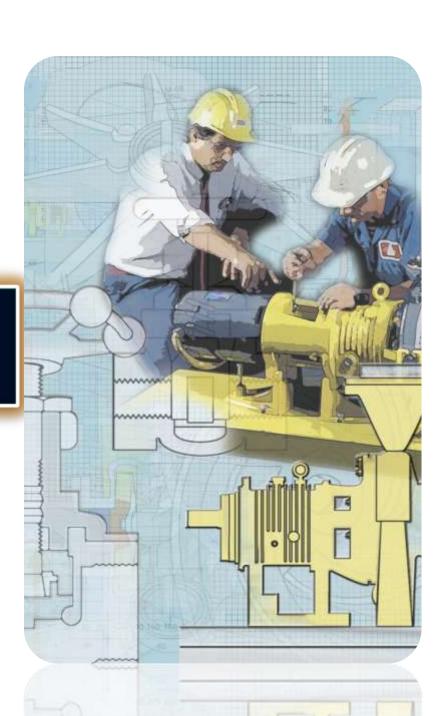
describe the components of a pump curve.

Describe a simple pumping system.

describe typical pump failures and their causes



## Introduction



#### Introduction

#### **Pump Definition**

Pump is defined as a mechanical device that rotates or reciprocates to move fluid from one place to another

#### Purpose of a pump:

A pump is designed to: transfer fluid from one point to another.

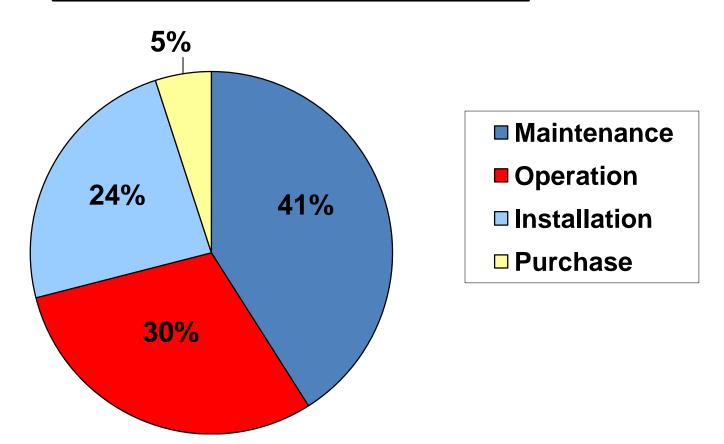
- -from low pressure areas to higher pressure Areas.
- -from lower elevations to higher Elevations.
- -From local locations to distant locations.



#### Life Cycle Cost

Energy and maintenance costs during the life of a pump system are usually more than 10 times its purchase price

## 20 Year Life Cycle Cost Chemical Process Pump

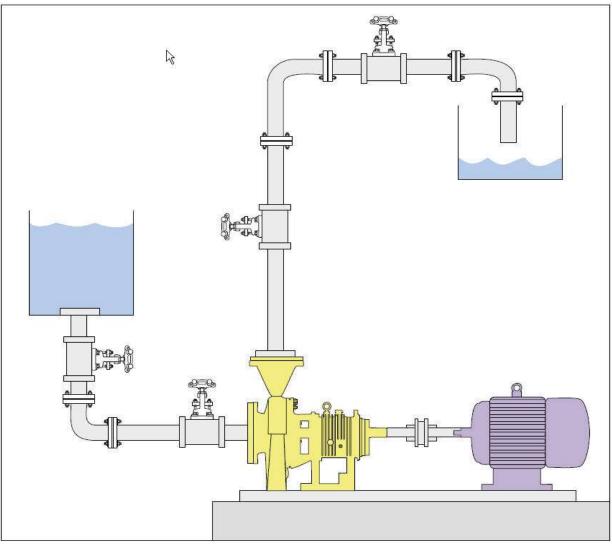


## Pumping System Basics

Typical pumping systems contain five basic components:

Pumps.
prime movers.
Piping.
Valves.
end-use equipment.

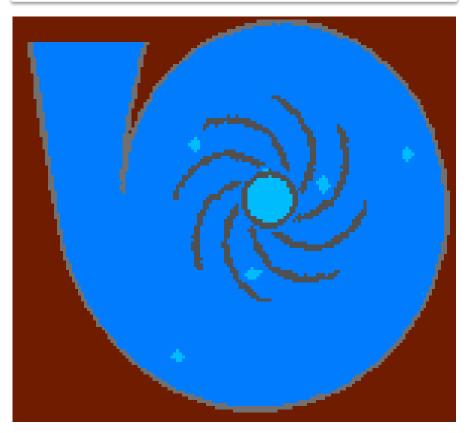




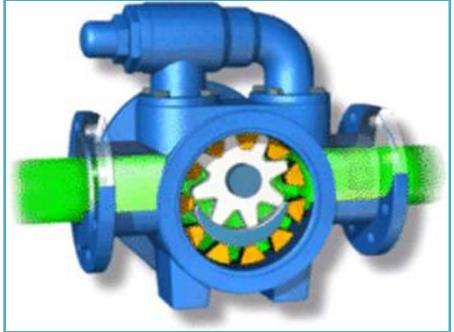
#### Pumps Types

Pumps are available in a wide range of types, sizes, and materials, two categories described—positive displacement and centrifugal.

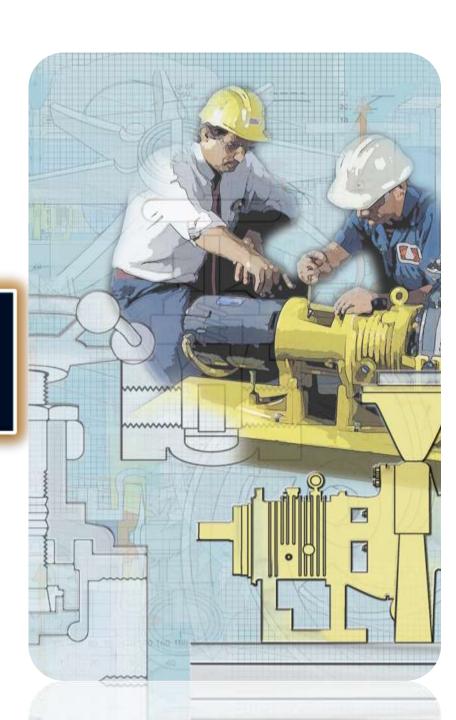
Centrifugal pumps have a variable flow/pressure relationship.



positive displacement pumps have a fixed displacement volume.
Consequently, the flow rates they generate are directly proportional to their speed.



Positive displacement pump



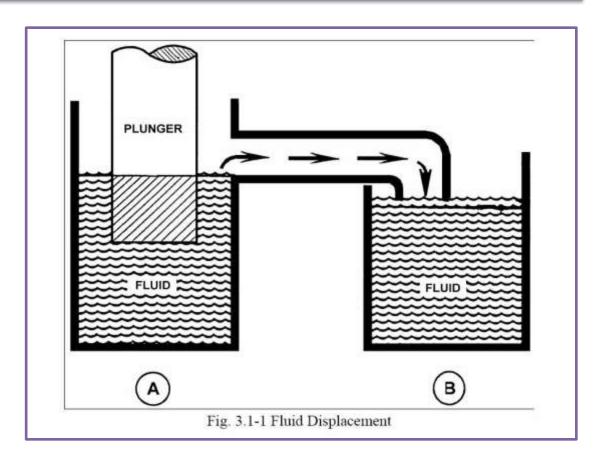
## Positive Displacement Pumps

Positive displacement pumps work on the principle that – No two objects can occupy the same space at the same time

A solid will displace a volume of fluid equal to its own volume

Reciprocating element (piston – plunger)

Rotating element (Vans, Screws or Gears)



## Positive Displacement Pumps

#### Discharge flow / rate can be controlled by:

- Change the speed of the prime mover.
- Use of bypass lin.
- Change of the piston stroke length.

#### Types of Positive Displacement Pumps

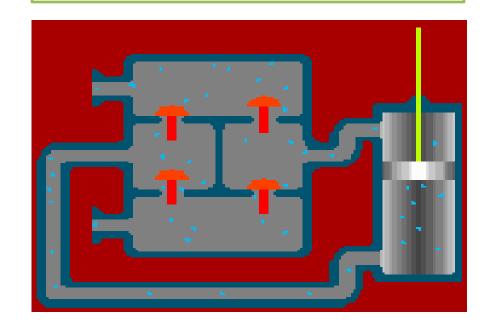
**Reciprocating Pumps** 

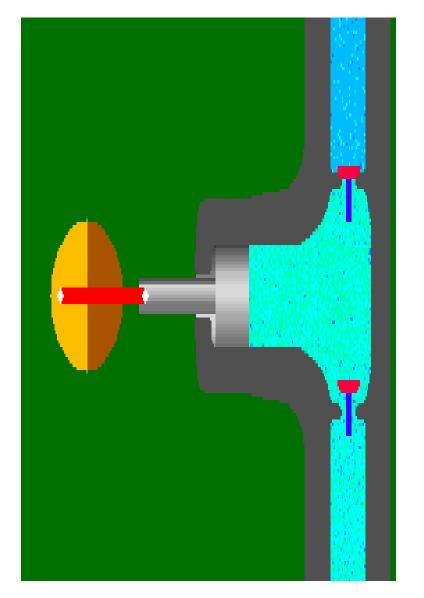
**Rotary Pumps** 

## Reciprocating Pumps

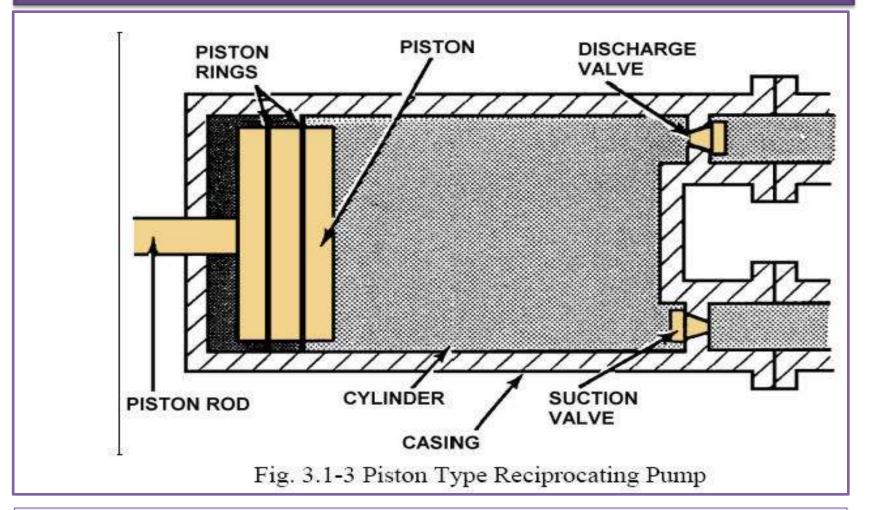
#### **Three main types:**

- Piston Type.
- Plunger Type.
- Diaphragm Type.





#### Piston Type Reciprocating Pump

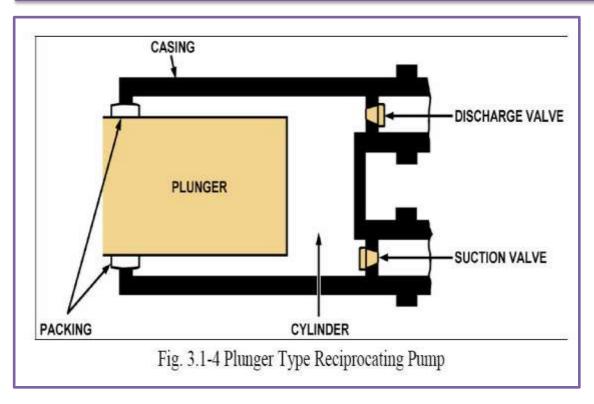


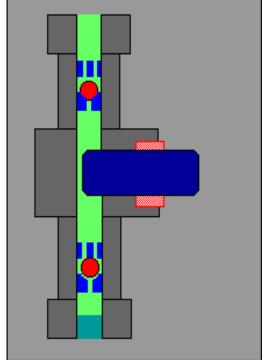
Each complete movement of the piston along the length of the cylinder is called  $\underline{A}$  STROKE

Used as portable backup pumps for the removal of surplus process fluids

#### Plunger Type Reciprocating Pump

Very similar to piston type pumps and either vertical or horizontal acting

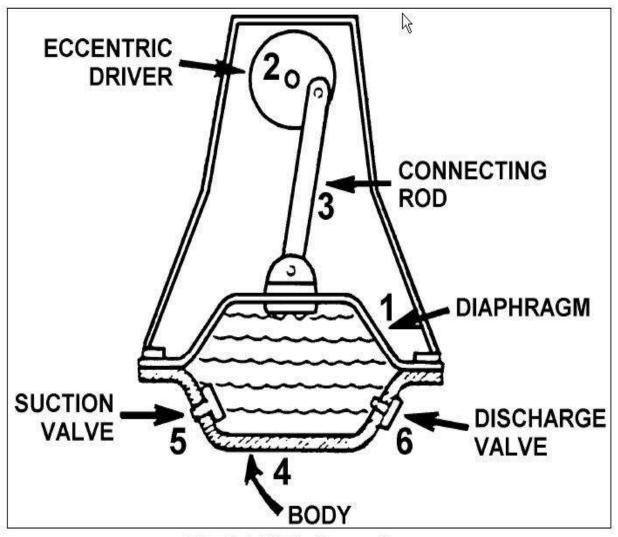


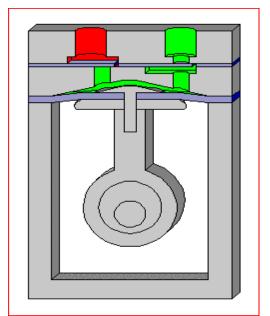


What is the difference between piston and plunger pumps?

## Diaphragm Pump

#### Get their name from the flexible diaphragm





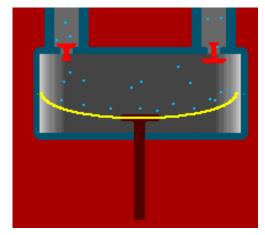
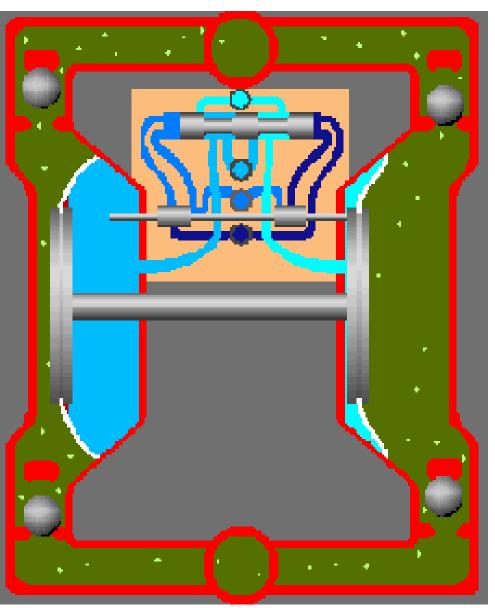


Fig. 3.1-5 Diaphragm Pump

## Diaphragm Pump

#### **Double-Diaphragm Pump**





#### Diaphragm Pump

A perfect seal, it makes Diaphragm Pump ideal for:

- 1- pumping chemicals.
- 2~ Controlled metering.

A diaphragm is usually made of a flexible rubber materials often covered with a thin metal disc where the connecting rod is attached.

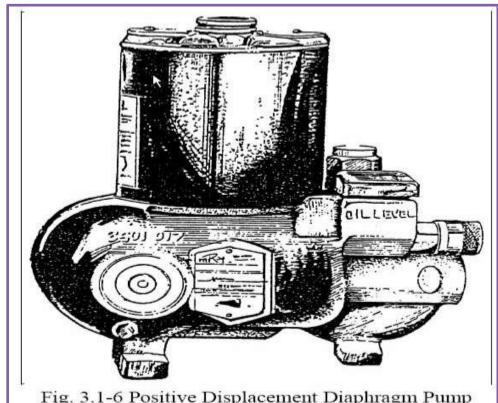


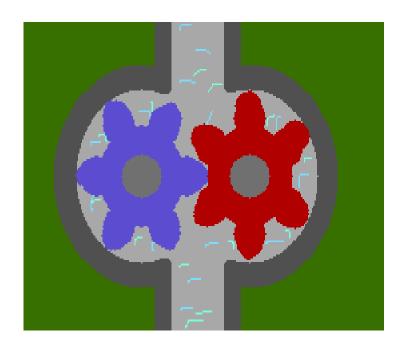
Fig. 3.1-6 Positive Displacement Diaphragm Pump

## Rotary Pumps

Types of Rotary Pumps

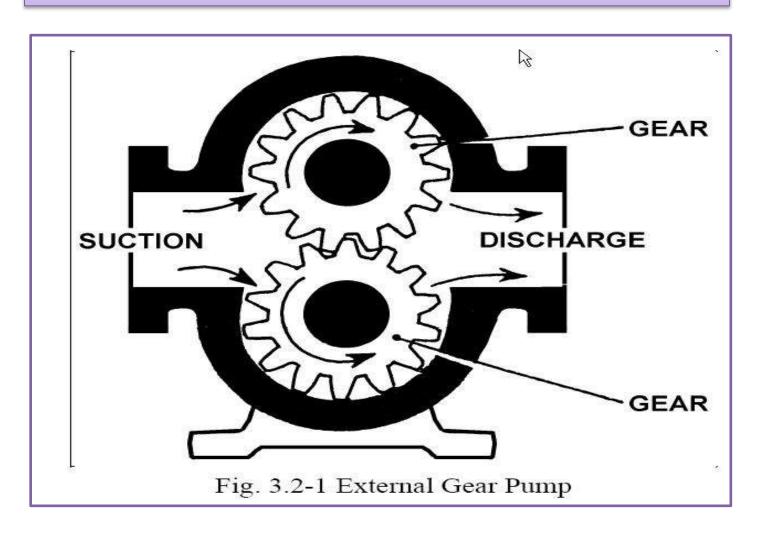
- ~Gear Pump.
- -Sliding Vane Pump.
- -Rotary Lobe Pump.
- ~Rotary Screw Pump.



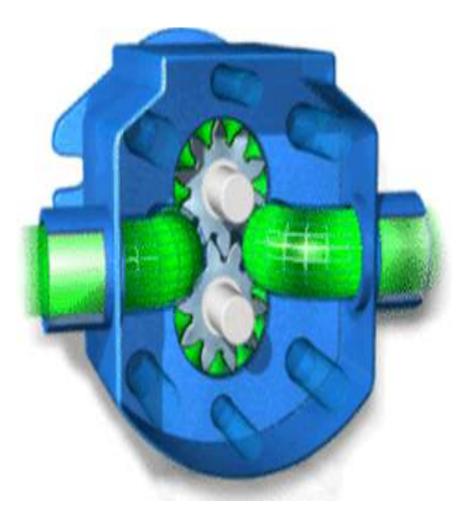


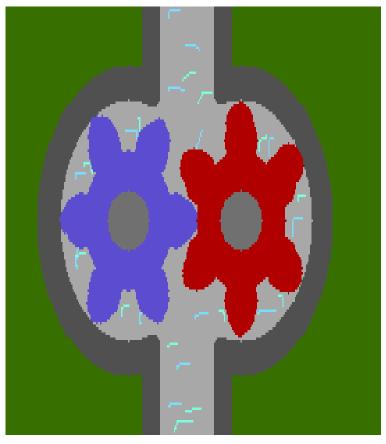
## Gear Pump / External

consists of two intermeshing gears, one driven and one idling, in a close-fitting casing



## Gear Pump / External

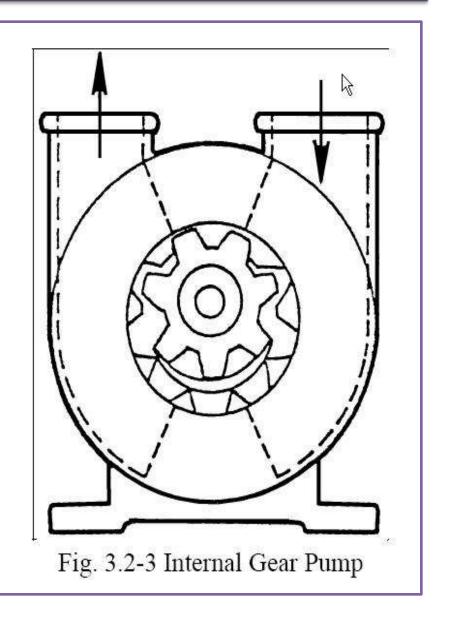




## Gear Pump / Internal

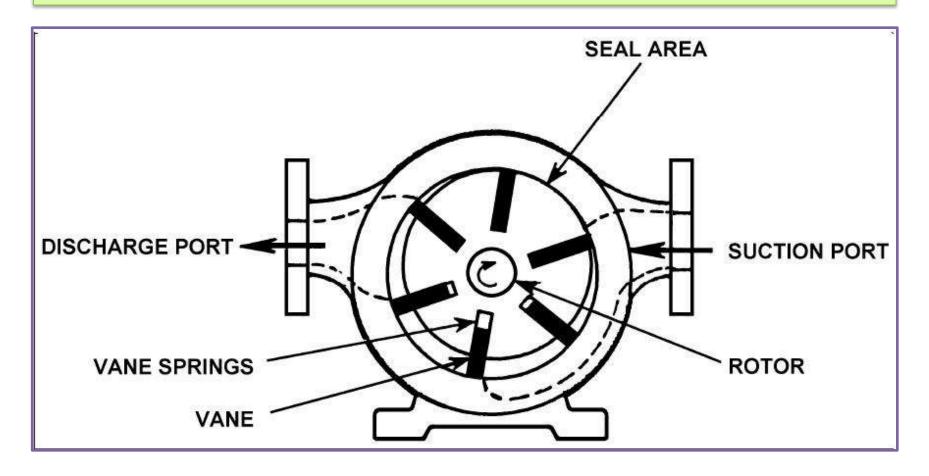
Internal gear pump has a small (driven) gear mounted eccentrically inside a larger idler gear that rotates inside a circular casing.





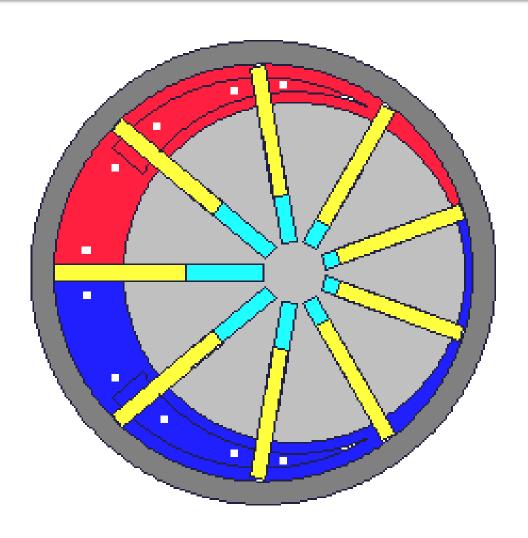
#### Vane Pump

The common vane type pump consists of a rotor with a radial slots machined into it. The rotor is mounted off-center and rotates inside a circular casing.

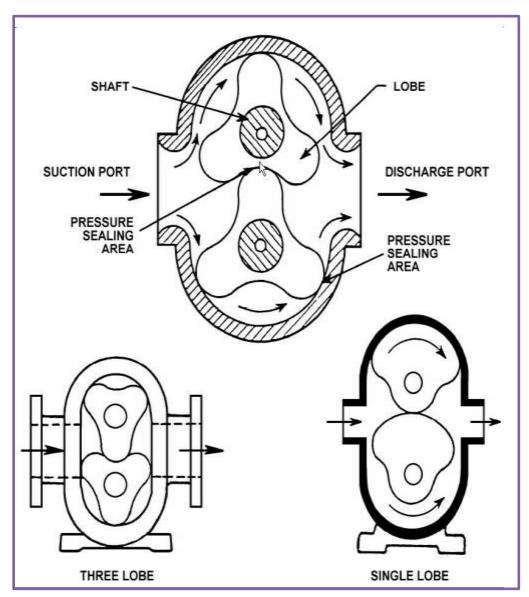


## Vane Pump

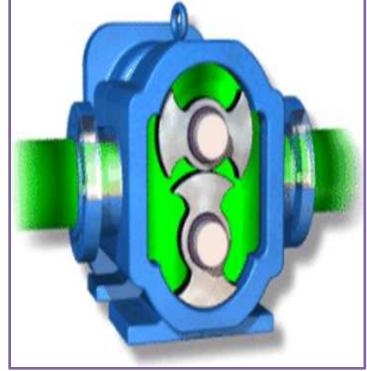
Liquid is drawn in through the suction side and squeezed out under pressure through the discharge side.



## Rotary Lobe Pump



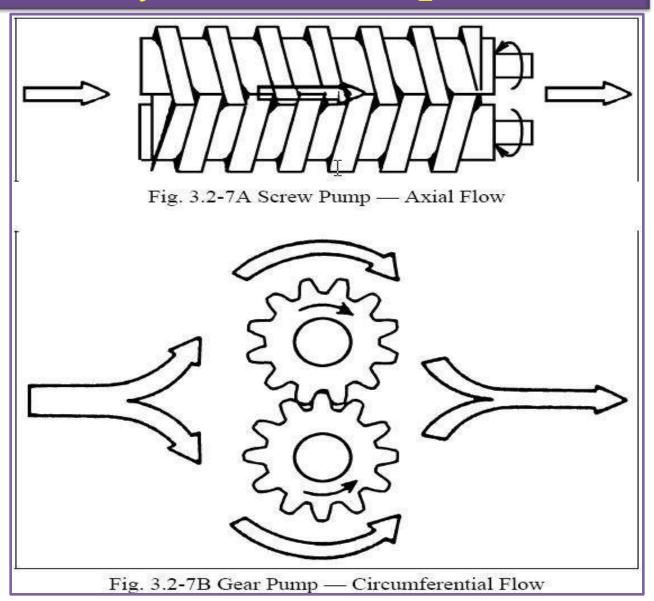
The rotary lobe pump works on a similar principle to the gear pump



#### Rotary Screw Pump

Rotary screw pumps are special types of rotary positive displacement pumps in which the flow through the pumping elements is truly axial.

Screw pumps are also called axial flow pumps

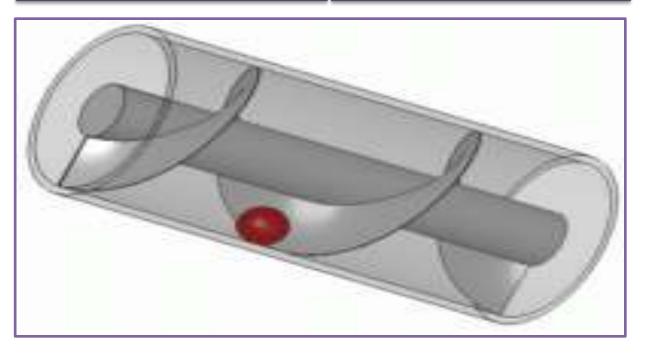


#### TYPES OF SCREW PUMPS

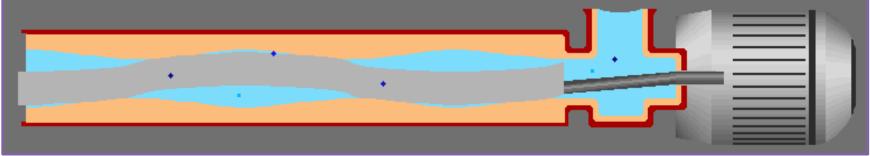
Screw pumps can have one, two or three screws

The types of screw pumps are the:

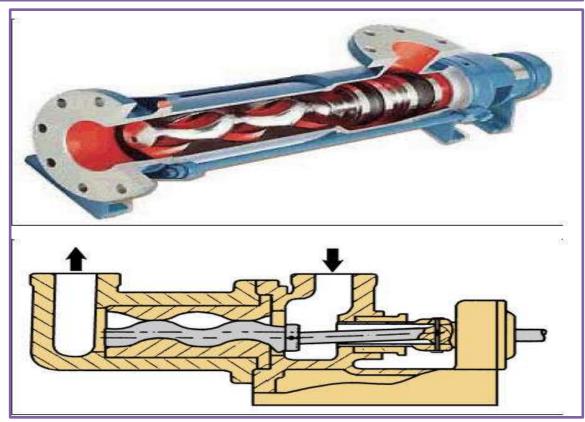
Single Screw Pump Multiple Screw Pump



## Single Screw Pumps



It consists of a spiral-shaped rotor that turns in an internal-helix liner. The rotor is usually metal. The liner is rubber.



## Multiple Screw Pumps

Multiple screw pump increases the pumping capacity of a single screw type by providing additional rotors to move fluid.

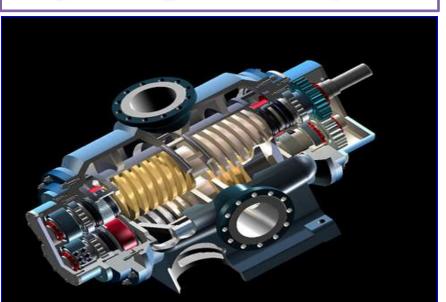
ional rotors to move

Fig. 3.2-9A Pump with Internal Timing Gears

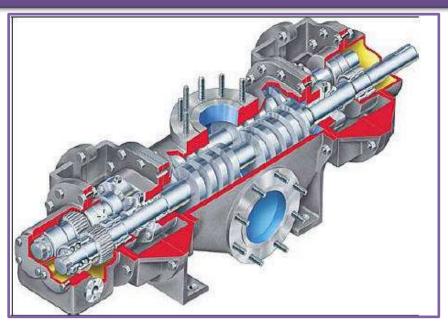
ble screw pumps are
by a single rotor called

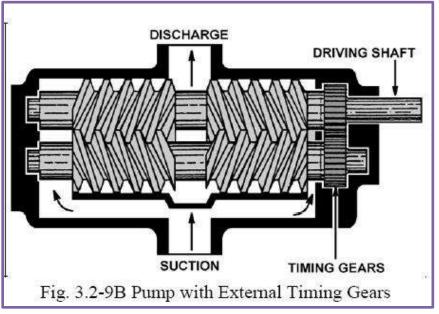
Multiple screw pumps are driven by a single rotor called the **power rotor**.

Two screw pumps are often called timed screw pumps



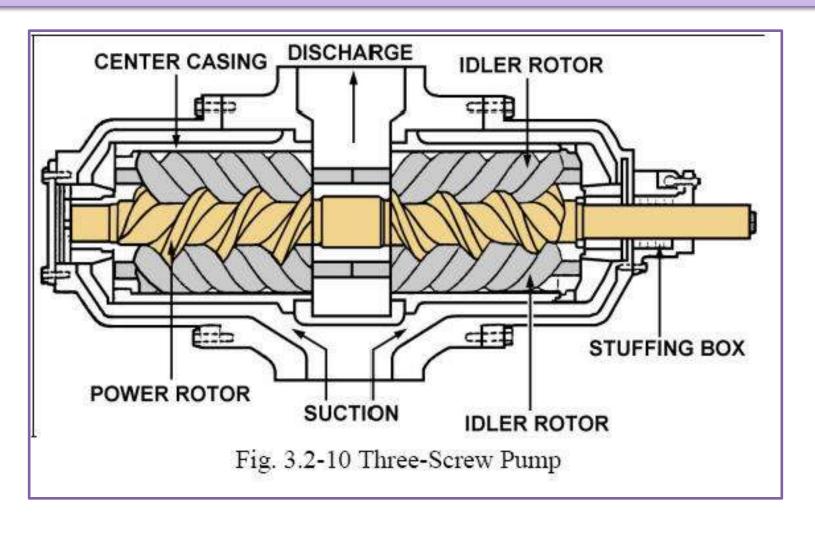
#### Multiple Screw Pumps



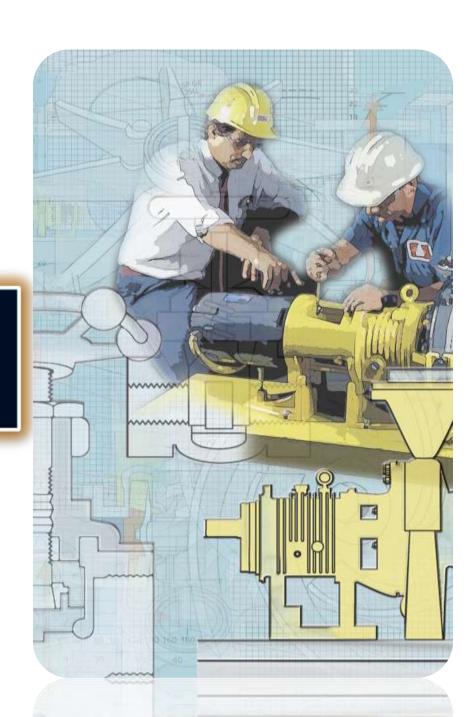


#### Multiple Screw Pumps

Three-screw pumps have two idler screws. The idlers are threaded to mesh with the power rotor. (untimed screw pumps)



# Centrifugal and axial pumps

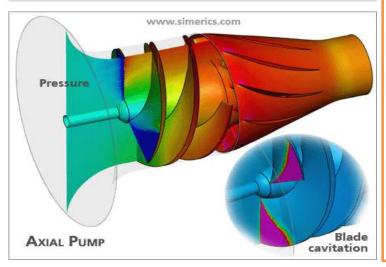


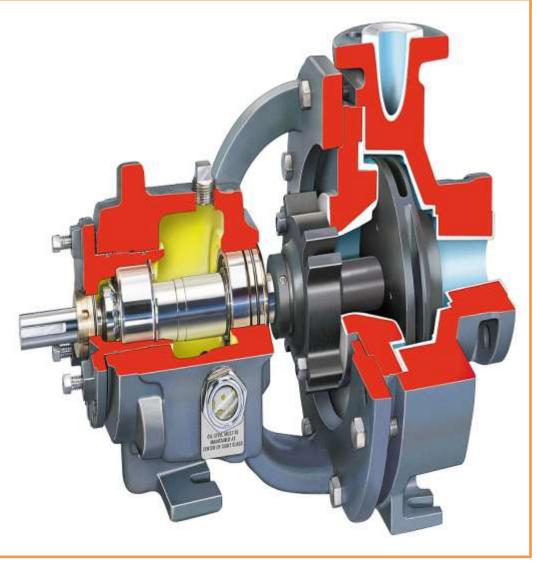
#### Initial Pump Selection

#### Centrifugal and axial pumps

Centrifugal and axial pumps impart kinetic energy to a fluid and rely on the conversion of this kinetic energy to potential energy to increase fluid pressure

centrifugal pumps are used typically in high-flow, low-head applications in which fluid viscosity is not prohibitively high

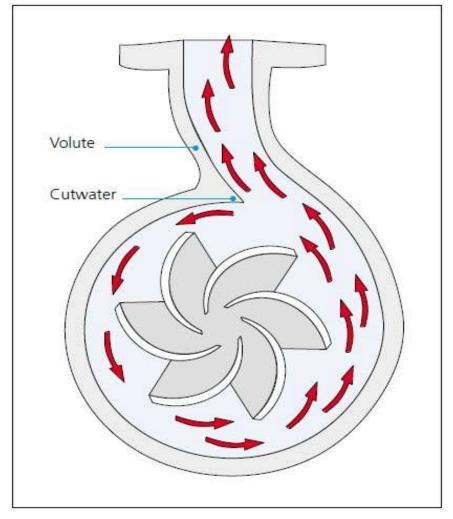




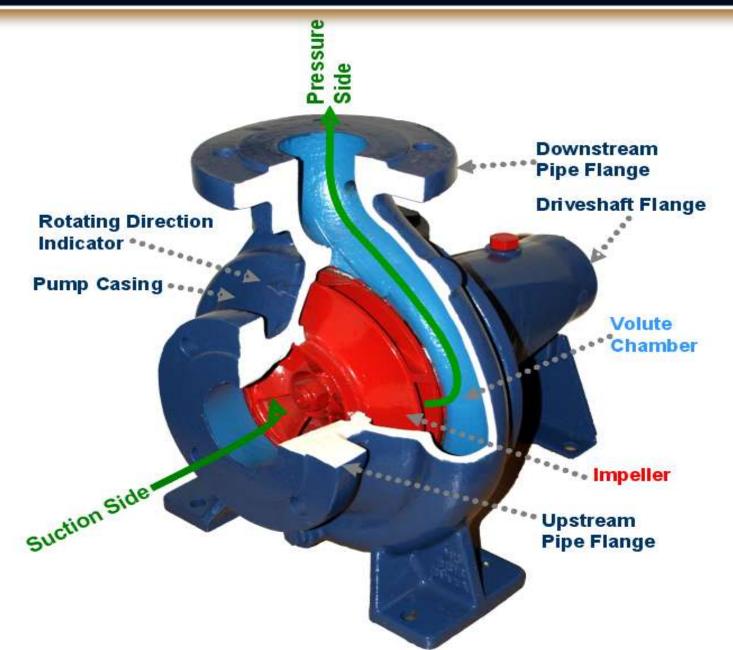
## Initial Pump Selection

Centrifugal pumps



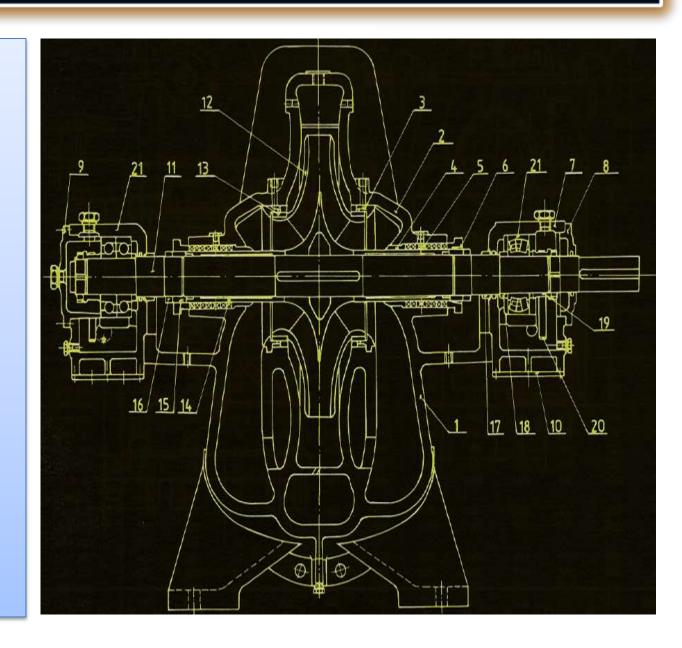


#### Pumps sectional drawing



#### pumps, typical sectional drawing

- 1. Casing half, lower
- 2. Casing half, upper
- 3. Casing wear ring
- 4. Neck bush
- 5. Lantern ring
- 6. Stuffing gland
- 7. Bearing housing
- 8. Bearing cower I.
- 9. Bearing cover II.
- 10. Cooling cover
- 11. Shaft
- 12. Impeller
- 13. Impeller ring
- 14. Shaft protecting sleeve
- 15. Shaft nut
- 16. Lock nut
- 17. Labyrinth ring
- 18. Shoulder ring
- 19. Distance sleeve
- 20. Lubrication ring
- 21. Bearing



#### Pumps, typical sectional drawing



Centrifugal Pump Disassembling Manual



#### Initial Pump Selection

#### Axial pumps

Axial-flow pumps with pull-out rotor and with blading adjustable during operation





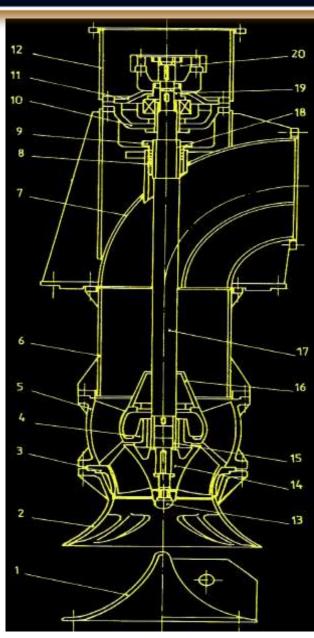


### Mixed-flow pumps



#### Vertical shaft



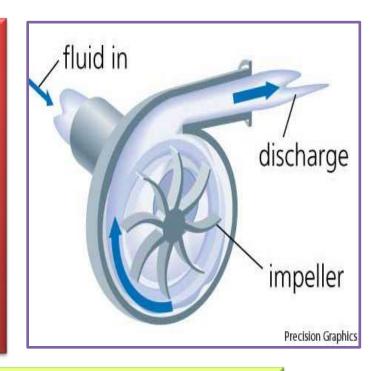


- 1. Suction cone
- 2. Suction bell
- 3. Wearing ring
- 4. Guide bearing casing
- 5. Guide vanes casing
- 6. Tube
- 7. Delivery elbow
- 8. Neck bush
- 9. Stuffing box
- 10. Thrust bearing casing
- 11. Thrust bearing cover
- 12. Motor stool
- 13. Cover
- 14. Shaft nut
- 15. Blade
- 16. Fixing plate
- 17. Impeller hub
- 18. Shaft sleeve
- 19. Shaft
- 20. Packing
- 21. Shaft sleeve
- 22. Oil retaining tube
- 23. Bearing bell
- 24. Coupling

#### **ADVANTAGES OF CENTRIGUGAL PUMPS**

#### Advantages

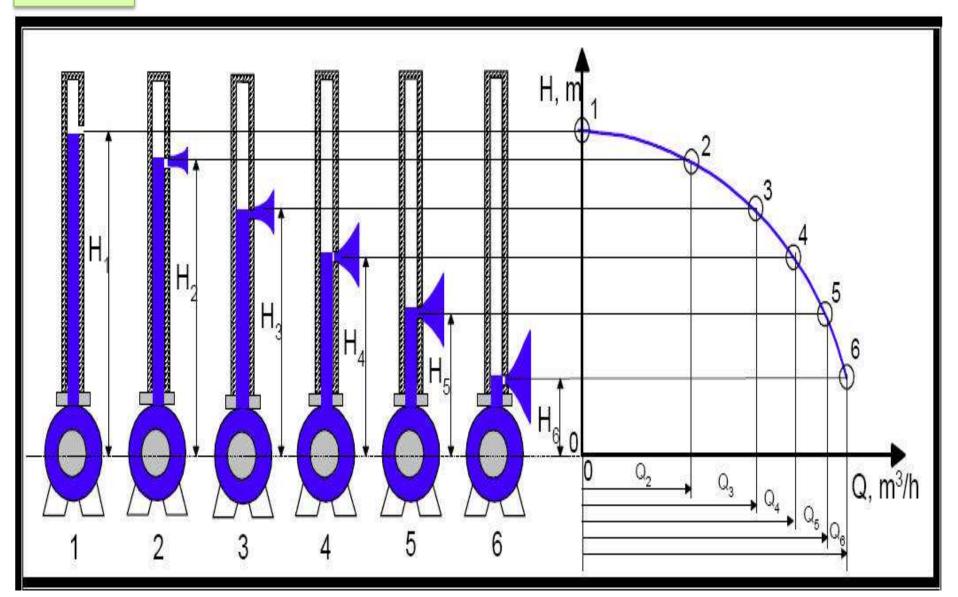
- Simple in construction and cheap
- Handle liquid with large amounts of solids
- No metal to metal fits
- No valves involved in pump operation
- Maintenance costs are lower



#### Disadvantages

Cannot handle highly viscous fluids efficiently
Cannot be operated at high heads
Maximum efficiency holds over a narrow range of conditions

**HQ** Curve



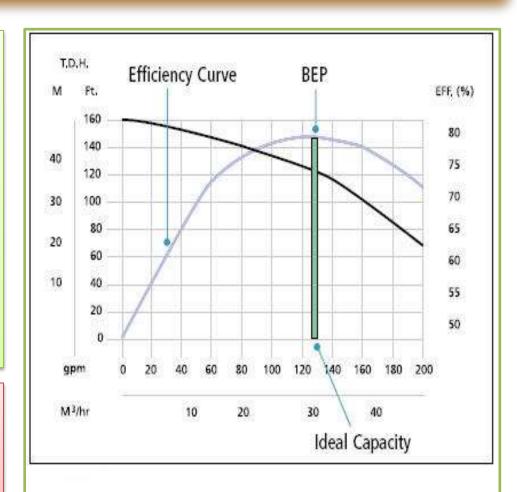
The amount of fluid power that a system consumes is a product of head and flow, according to this equation: Fluid power =  $\frac{HQ(s.g.)}{3,960}$ 

where

H = head (ft)

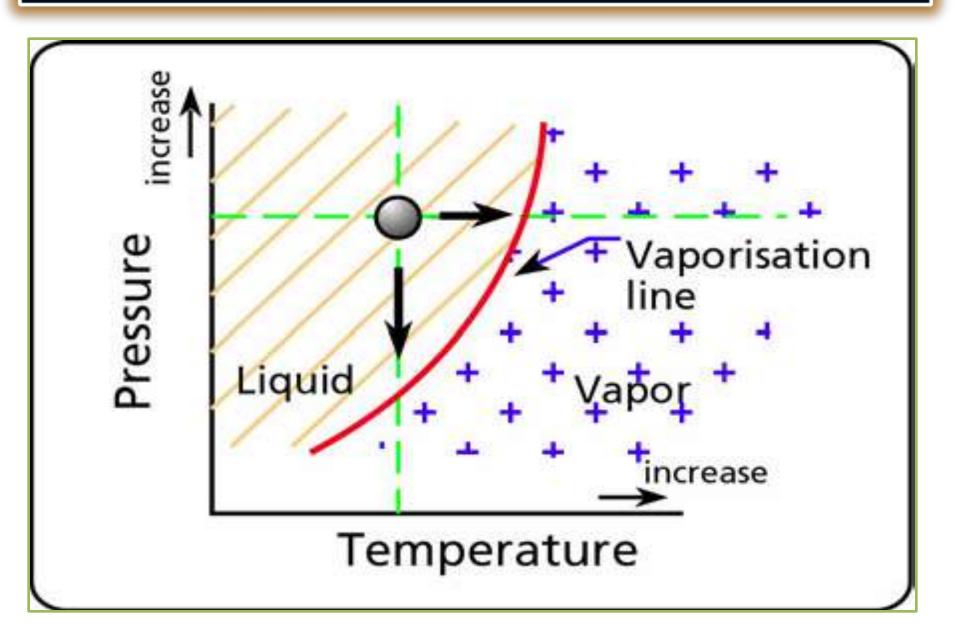
Q = flow rate (gallons per minute [gpm]) s.g. = specific gravity of the fluid 3,960 is a units conversion to state fluid power in terms of horsepower

The operating point of centrifugal pumps at which their efficiency is highest is known as the best efficiency point (BEP). Efficiencies range widely, from 35% to more than 90%, and they are a function of many design characteristics

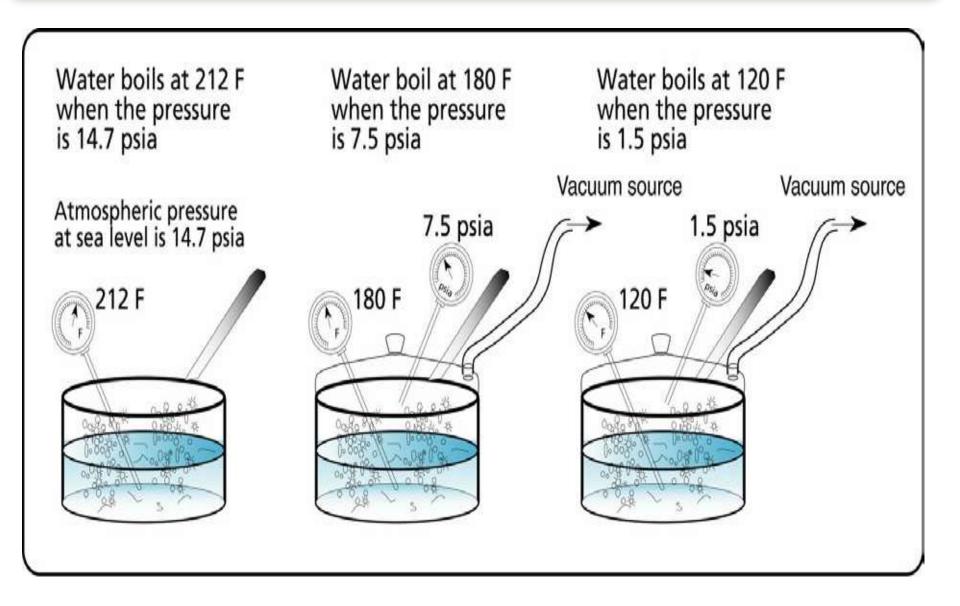


Efficiency curve illustrating decreasing efficiencies as the capacity moves away from the ideal capacity and BEP

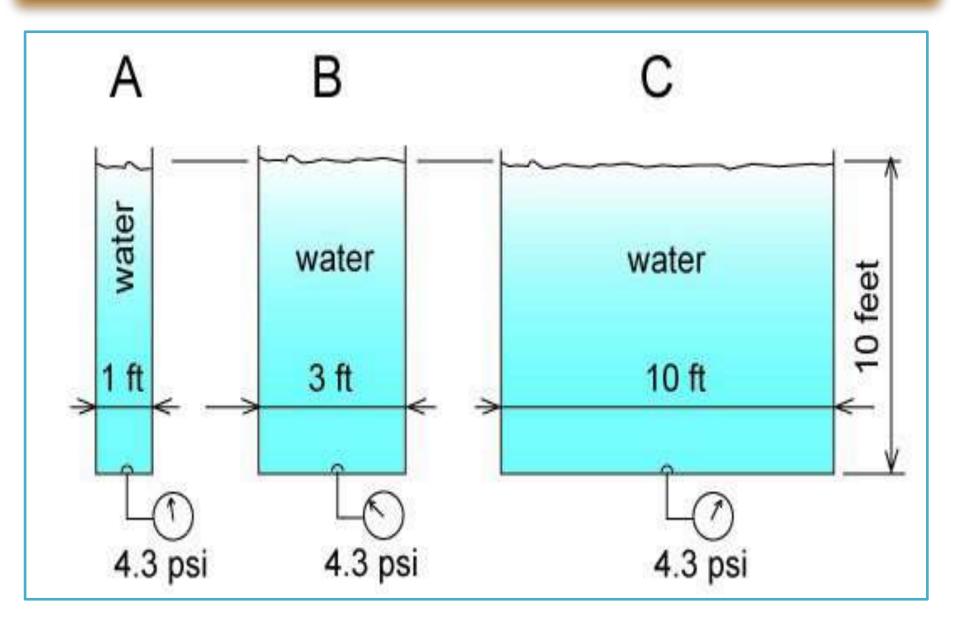
### Vapor pressure vs. temperature



### Boiling water with low pressure



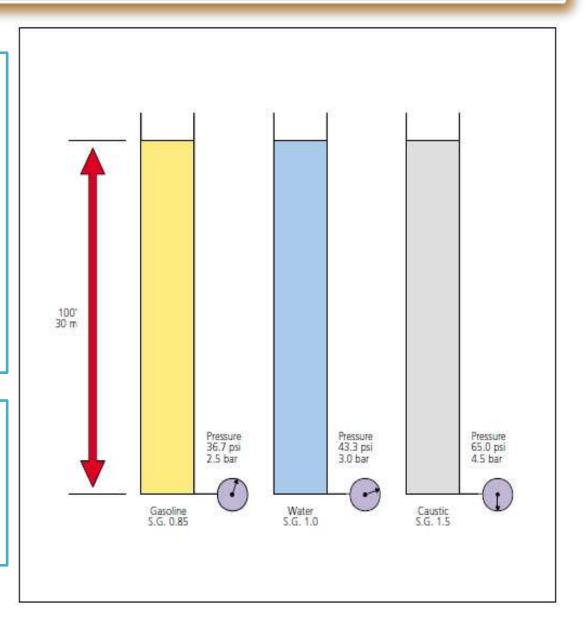
#### Pressure



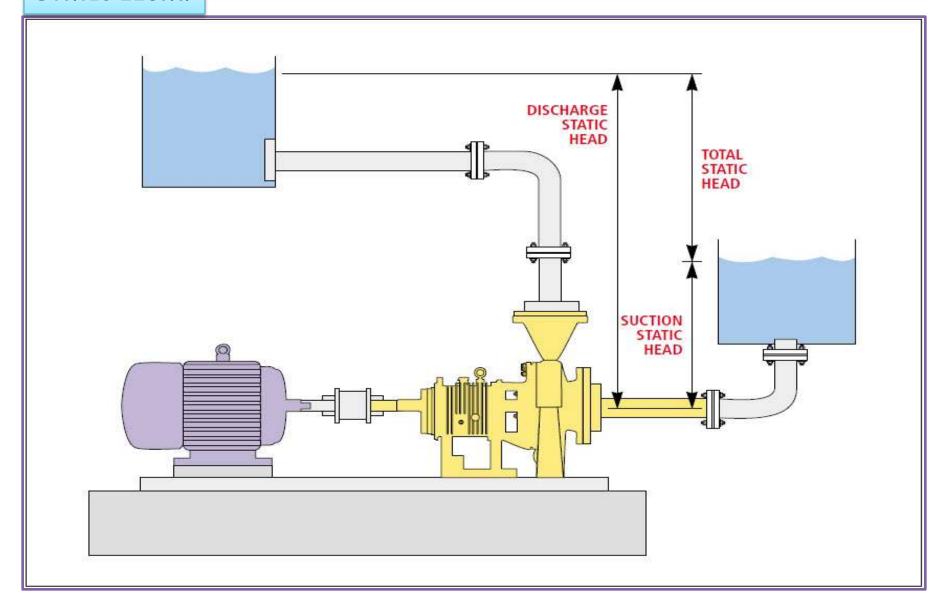
### Specific Gravity

Specific gravity is the density ratio of a liquid as compared to water at a given temperature. Water is used as the standard at 14.69 psia (1.013 bar abs) and at 60°F (15.5°C).

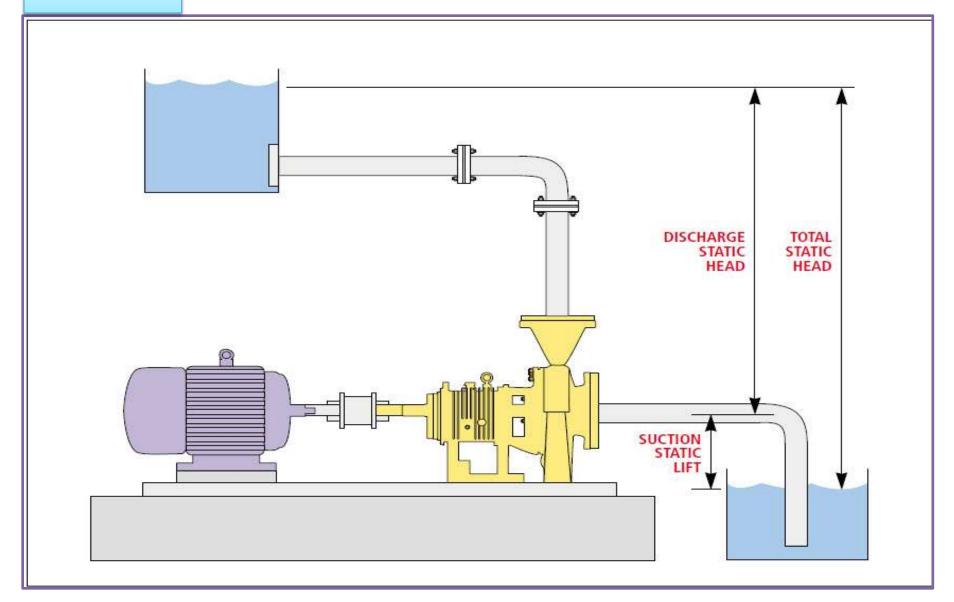
Its specific gravity is 1.0 at this standard temperature and pressure.



#### Static Head

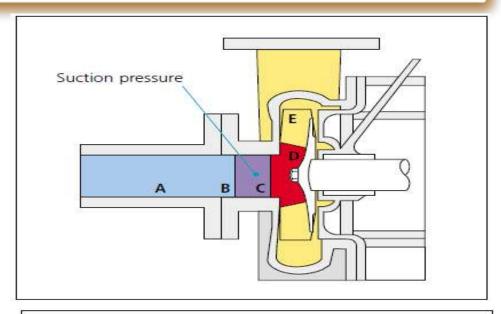


#### Static Lift

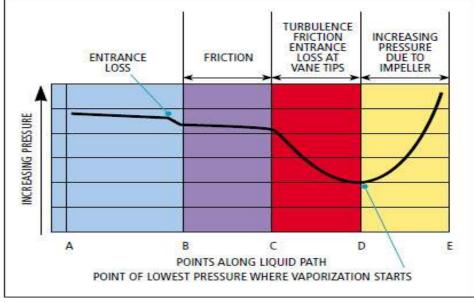


### Net Positive Suction Head (NPSH)

is defined as the minimum hydraulic head condition in which a pump can meet its head and capacity requirements without the liquid vaporizing inside the pump.



Vaporization of the liquid causes cavitations. This cavitations reduces a pump's performance and may damage the pump.



# Cavitations

LOADING

NPSHA = 
$$\frac{2.31 (P_a - P_V)}{spgr} + (H_e - H_f)$$

#### where

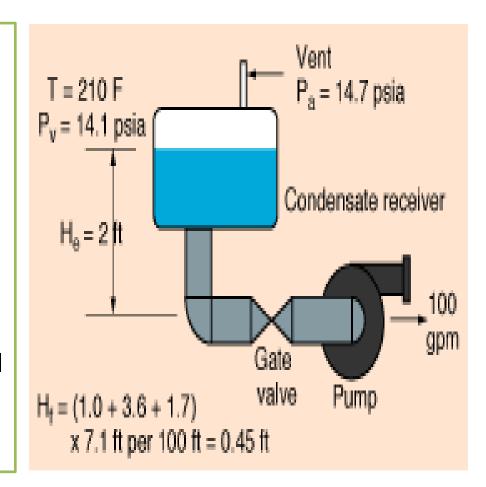
 $P_a$  = Pressure in the receiver, psia

 $P_V$  = Vapor pressure of the liquid at its maximum temperature, psia

 $H_e$  = Elevation head, ft.

He = Friction losses in the suction piping at the required flow rate, ft

spgr = Specific gravity

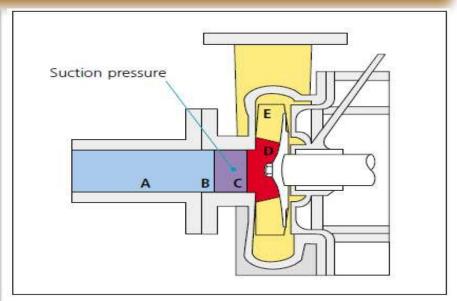


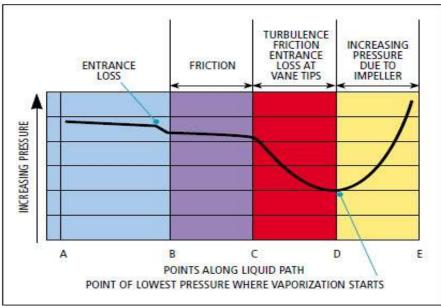
### NPSH Available and NPSH Required

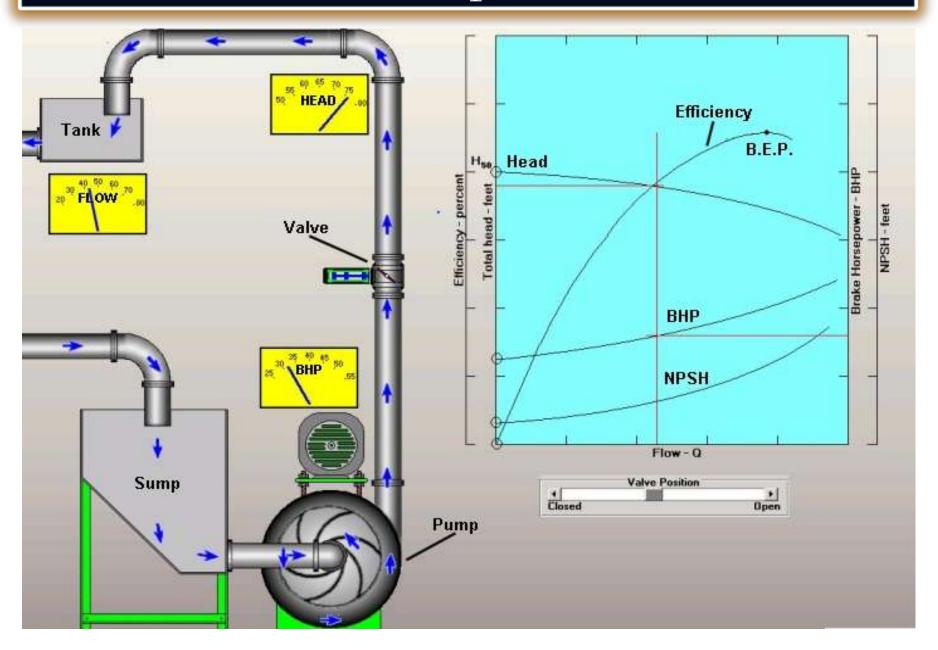
The standard tests for NPSHR tell us that even if NPSHA equals NPSHR there still is still a mild incipient cavitation occurring. Therefore, we need a little safety margin. A good margin to use is:

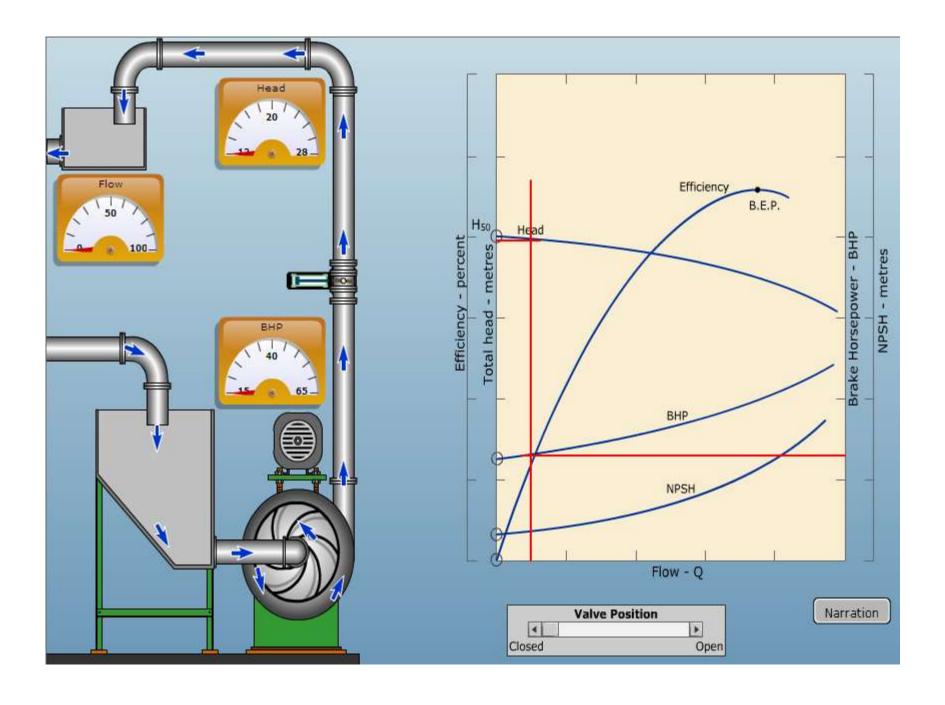
#### NPSHA > NPSHR +3 ft. (lm)

This margin can vary with pump type, impeller type, and fluid being pumped. However for most Overhung Impeller Centrifugal Pumps the 3 ft (lm) safety margin is usually satisfactory.









### Prime Movers

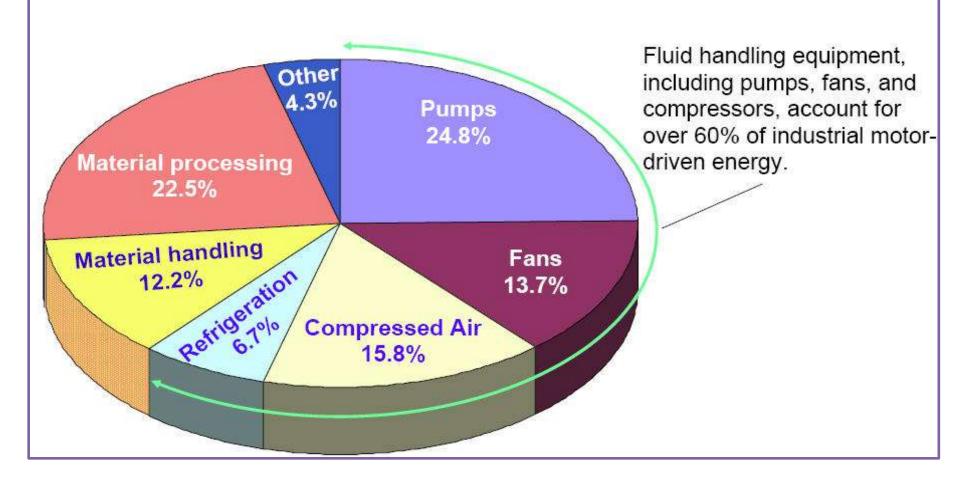
Most pumps are driven by electric motors. Although some pumps are driven by direct current (dc) motors, the low cost and high reliability of alternating current (ac) motors make them the most common type of pump prime mover.





#### Prime Movers







# Piping

Piping is used to contain the fluid and carry it from the pump to the point of use.

aspects of piping are its dimensions, material type, and cost.

The flow resistance at a specified flow rate of a pipe decreases as the pipe diameter gets larger; however, larger pipes are heavier, take up more floor space, and cost more than smaller pipe.



# Piping

In systems that operate at high pressures small-diameter pipes can have thinner walls than large-diameter pipes and are easier to route and install.

Small-diameter pipes restrict flow, can be especially problematic in systems with surging flow characteristics. Smaller pipes also operate at higher liquid velocity, increasing erosion effects, wear, and friction head. Increased friction head affects the energy required for pumping



### Valves

The flow in a pumping system may be controlled by valves.

Some valves have:

distinct positions.

shut or open.

throttle flow.

selecting the correct valve for an

application depends on:

ease of maintenance.

Reliability.

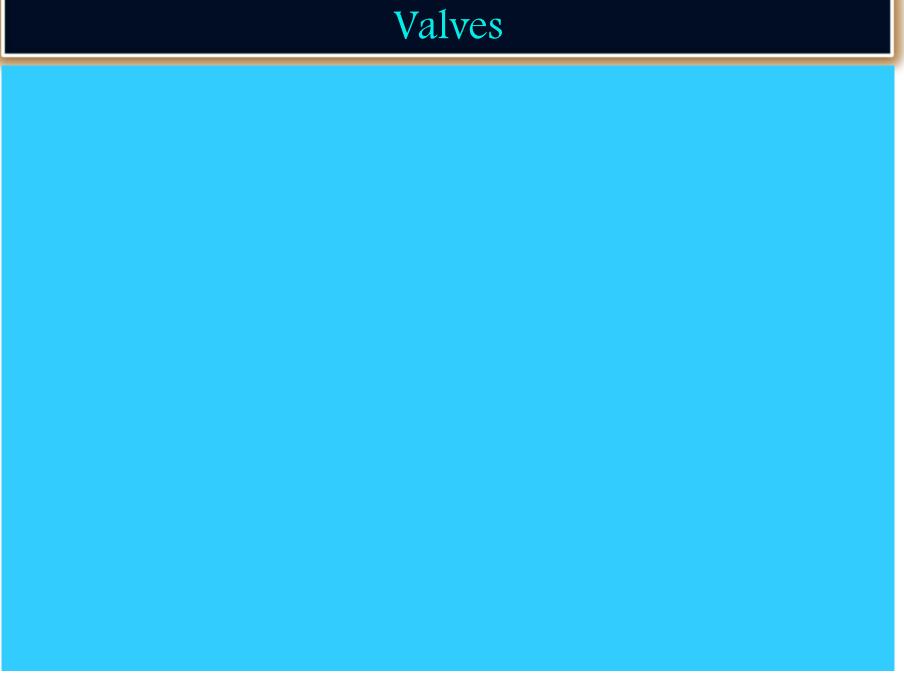
leakage tendencies.

Cost.

frequency with which the valve will be open and shut.

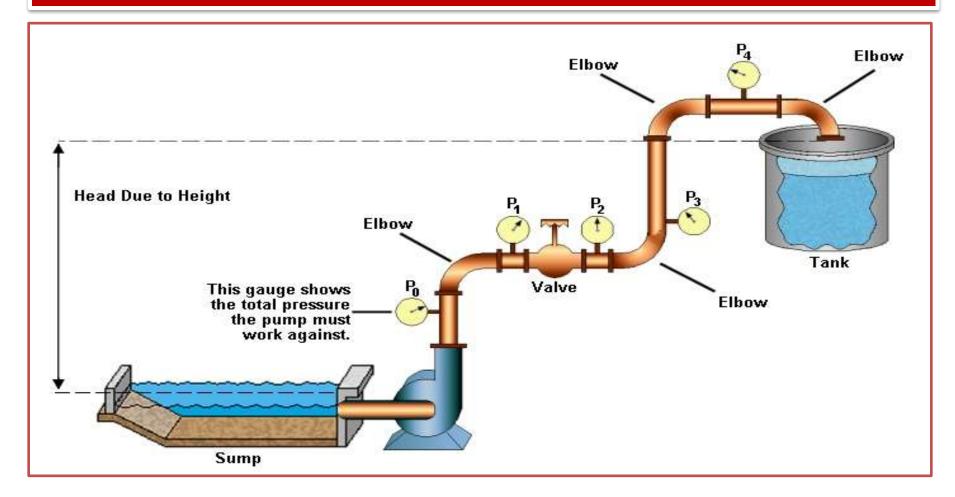




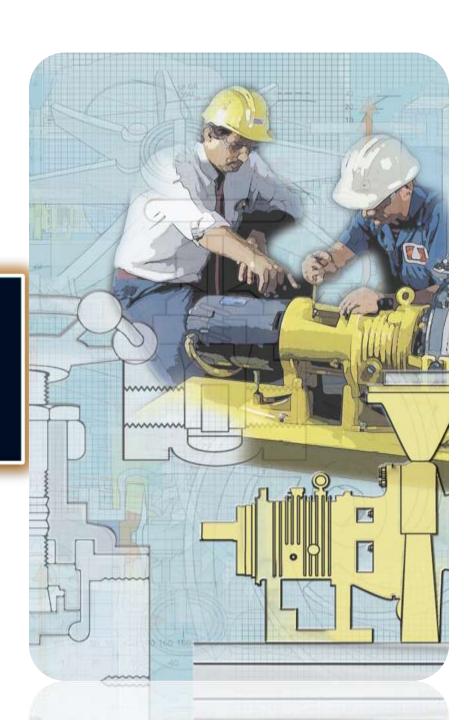


# **End-Use Equipment**

The essential purpose of a pumping system may be to provide cooling, to supply or drain a tank or reservoir, or to provide hydraulic power to a machine.



# Pumping System Basics



# Design Practices

Fluid system designs are usually developed to support the needs of other systems.

In cooling system applications, the requirements flow is determine. Pump capabilities are then calculated based on the system layout and equipment characteristics

The most challenging aspect of the design process is cost-effectively matching the pump and motor characteristics to the needs of the system

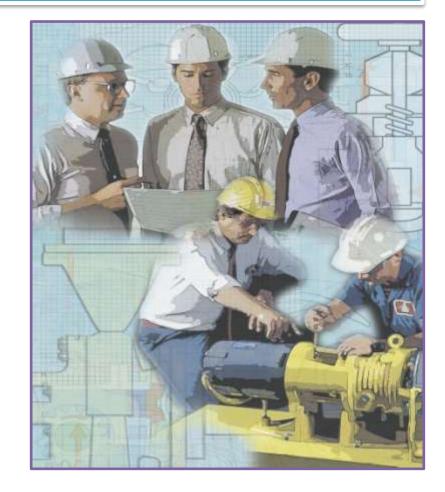




# Analyzing System Requirements

fully understand system requirements (peak demand, average demand, and the variability of demand) with respect to time of day and time of year

Problems with oversized pumps often develop because the system is designed for peak loads, while normal operating loads are much smaller. Excess flow energy is then forced into the system. In addition to increasing operating costs, this excess flow energy creates unnecessary wear on components such as valves, piping, and piping supports.



Pump selection starts with a basic knowledge of:

System operating conditions.

Fluid properties.

Pressures.

Temperatures.

System layout.

**System Operating Costs.** 

These conditions determine the type of pump that is required to meet certain service needs.

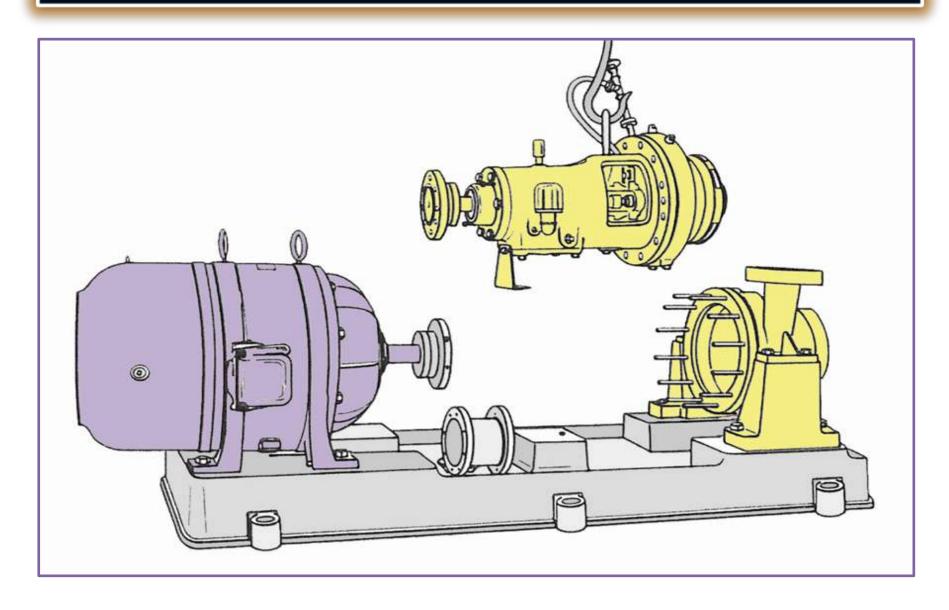
positive displacement centrifugal.

Axial flow pumps

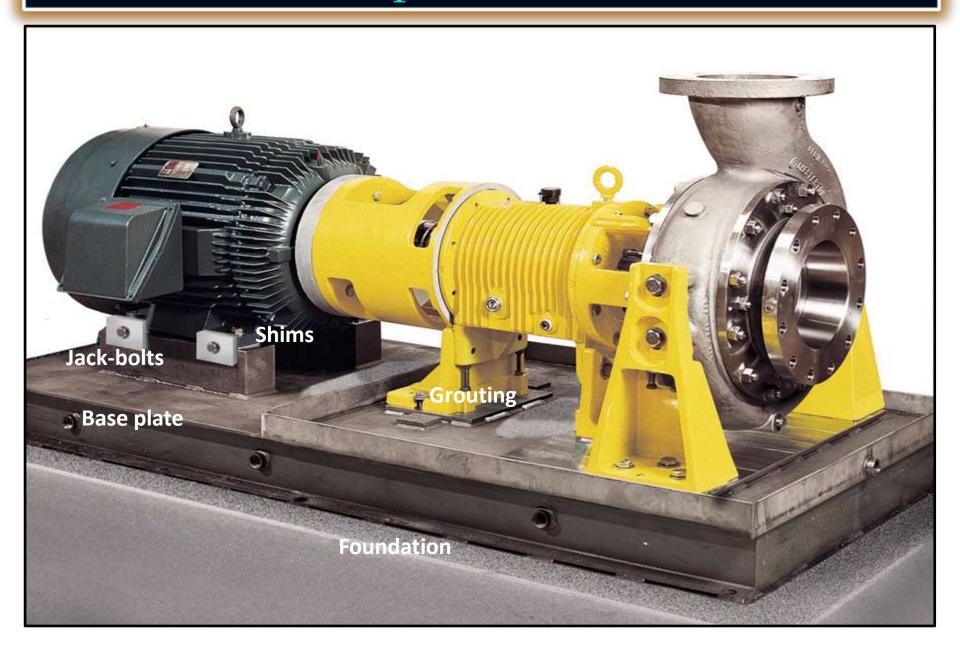
flow rate and head!
efficiency
suction inlet conditions
operating life
maintenance.
Costs



# Pumps Installation



# Pumps Installation

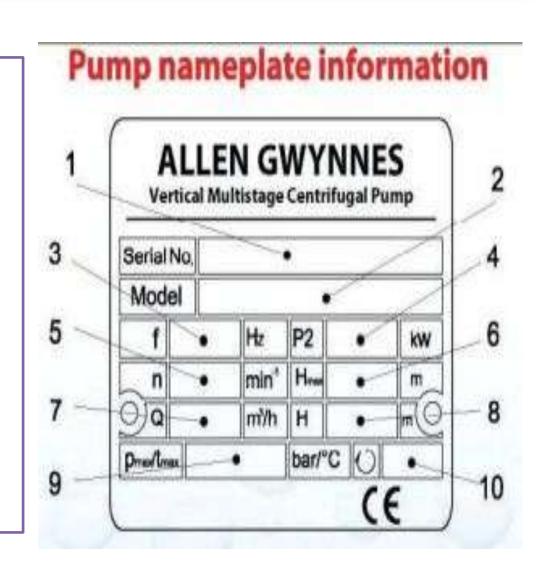


# Pump nameplate information

- 1. Serial number
- 2. Pump Model
- 3. Frequency
- 4. Rated Power
- 5. Speed
- 6. Maximum Head
- 7. Capacity
- 8. Head Range
- 9. Maximum Operating

Pressure

10. Rotating Direction



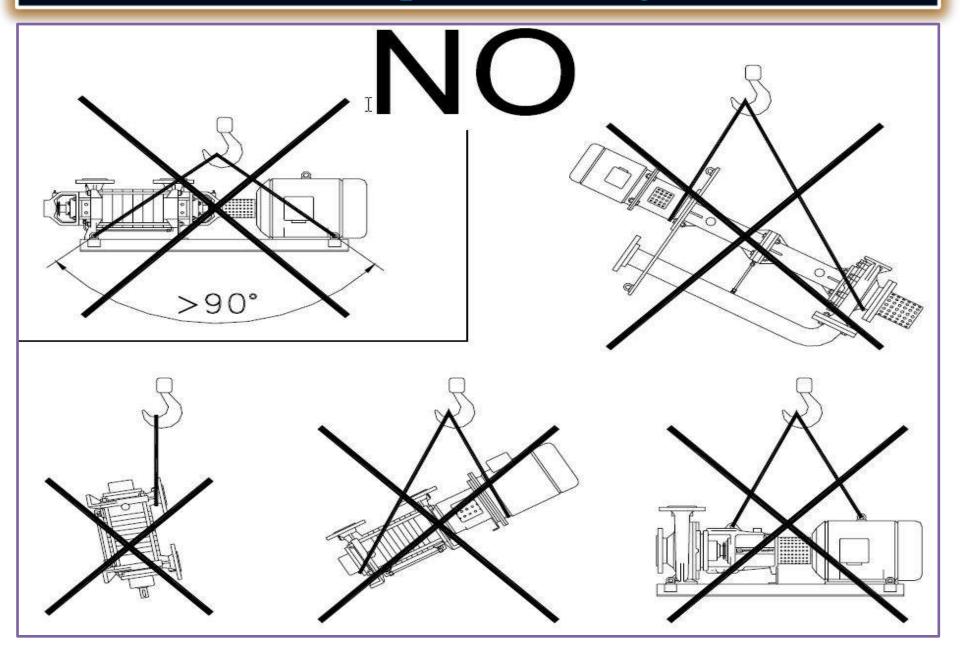
- I-Check the pump/pump unit immediately upon delivery/receipt of dispatch for damage or missing parts.
- 2-The pump/pump unit must be transported carefully and by competent personnel. Avoid serious impacts.
- 3-Keep the pump/pump unit in the same position in which it was supplied from the factory.
- 4-Take note of the instructions on the packaging.
- 5-The intake and discharge side of the pump must be closed with plugs during transport and storage

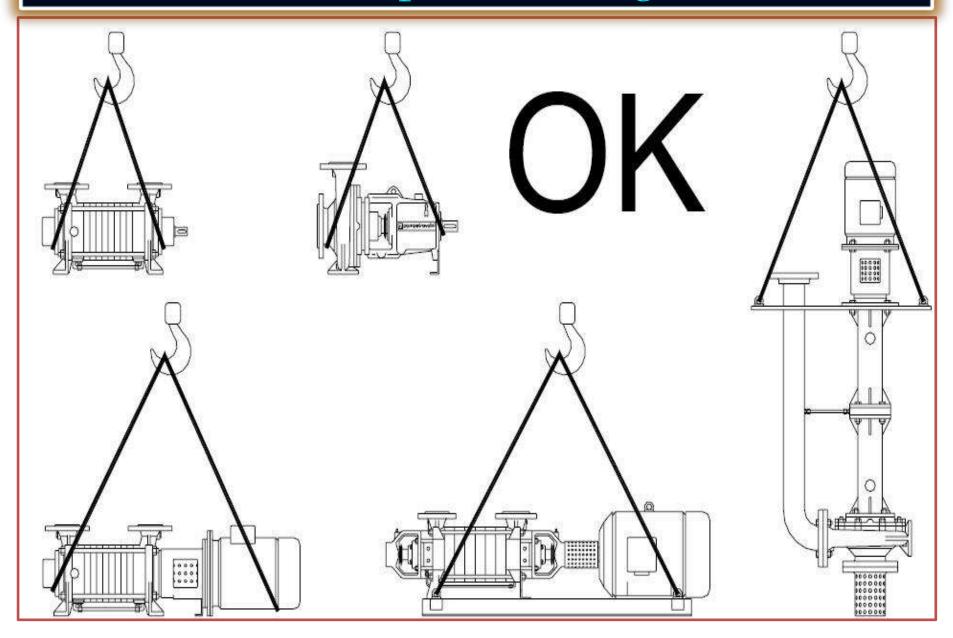




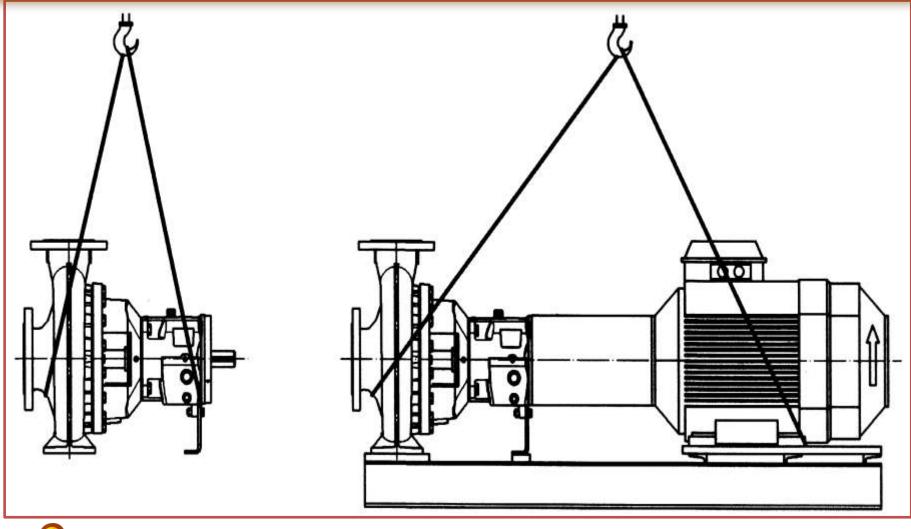
Dispose of all packing materials in accordance with local regulations.

- 6-Lifting devices (e.g. fork-lift truck, crane, crane device, pulleys, sling ropes, etc.) must be sufficiently strong and must only be used by authorized persons.
- 7-The weight of the pump/pump unit is given in the Data Sheet.
- 8-The pump/pump unit may only be lifted by solid points such as the casing, flanges or frame.
- 9-The following illustration shows the correct method of carrying by crane.





## Transport, Handling





Do not stand underneath suspended loads.

## Pumps Storage

Pumps or pump units that are stored for a long time (6 months max) before use must be protected against moisture, vibrations and dirt (e.g. by wrapping in oil paper or plastic sheeting).

Pumps must basically be stored in a place where they are protected from the weather, e.g. under cover. During this time, all suction and discharge branches and all other intakes and outlets must be closed with dummy flanges or plugs.

Please contact factory for storage instructions for storage periods longer than 6 months

Cover the equipment with industrial strength plastic, preferably transparent to allow its visual inspection, including its nameplate, without uncovering the unit.

## Pumps Storage

Drain the casing completely and dry it including:

bearing housing.

stuffing box.

seal chamber.

Apply a coat of soluble rust preventive solution both internally and externally

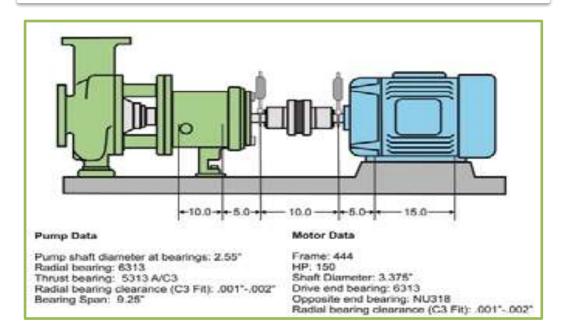
Remove the shaft coupling; it may cause the shaft to develop a permanent sag during prolonged storage.

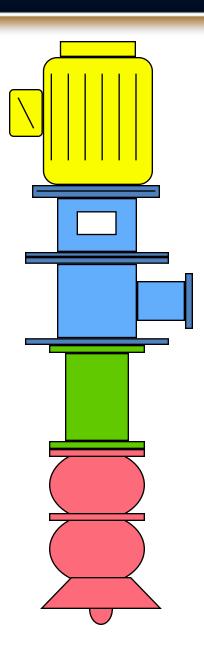
Protect the bearing housing from moisture by placing bags of vapor phase inhibitor crystals around the housing

Store the unit in its normal position in a dry place.

## Pumps Storage

Inspect the unit periodically and turn the shaft a few times plus 1/4 turn. Turning the shaft prevents pitting of finished surfaces. The extra 1/4 turn is to displace the sag and prevent the shaft from developing a permanent bow.





Mounting of pump onto a base frame

Installation and Alignment of Coupling

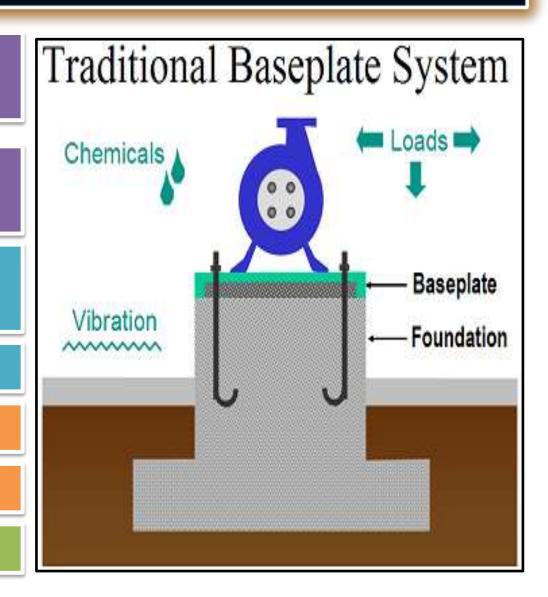
Connecting the pipes to the pump

Electrical connection

Starting up

Operation and Monitoring

Shutting down



## Mounting of pump onto a base frame

The pump unit must be provided with a shared base frame made of steel or cast iron or a welded steel profile frame. This base frame must be placed on a foundation which can withstand all loads that arise during operation.



coupling

#### following is to be noticed

The base must withstand all loads occurring during operation. The mounting surfaces of the pump feet and motor on the base frame must be flat safe fastening

Space between pump and motor depending on the used

#### Installation and Alignment of Coupling



Make sure that nobody can start the motor during work on the coupling.

#### Before starting installation coupling

carefully clean shaft ends do not hit the coupling coupling may be heated before hand in an oil bath approx. 100°C Secure coupling hubs against axial sliding



#### Installation and Alignment of Coupling

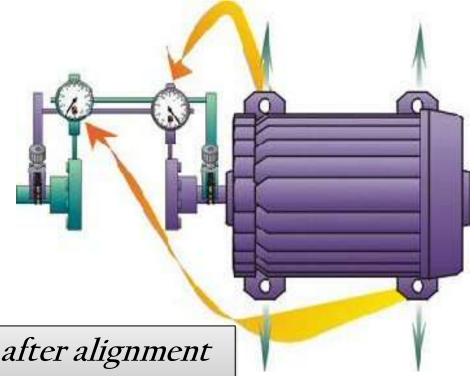


The installation and alignment of the coupling must be carried out with the utmost care and attention.

Badly aligned couplings cause noise.

vibration.

increased wear on bearings, couplings and shaft seals





Mount coupling guard after alignment and before starting the unit.

#### Connecting the pipes to the pump

The pipes must be of a size and design that liquid can flow freely

it is recommended that a check valve is installed in the discharge pipe shortly after the pump.

Particular attention is to be paid to ensuring that suction pipes are air tight and that the NPSH values are observed.

Do not install fittings or bends right before the suction nozzle.

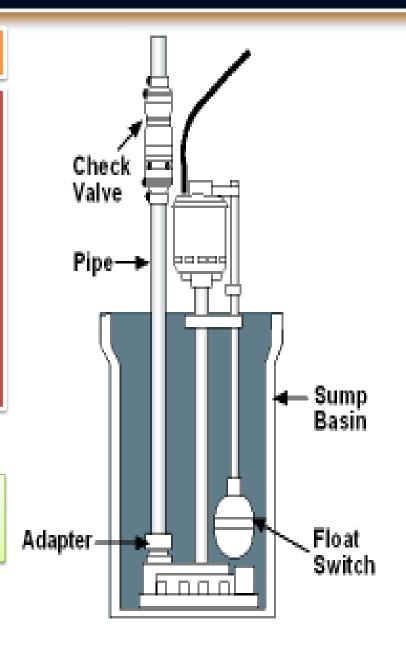
When laying the pipes, make sure that the pump is accessible for maintenance, installation and disassembly.

#### Connecting the pipes to the pump

If the pipe system is tested with the pump installed, do not exceed the maximum permitted casing pressure of the pump and/or shaft sealing.

In the case of pumps with stuffing boxes, replace packing after pressure test

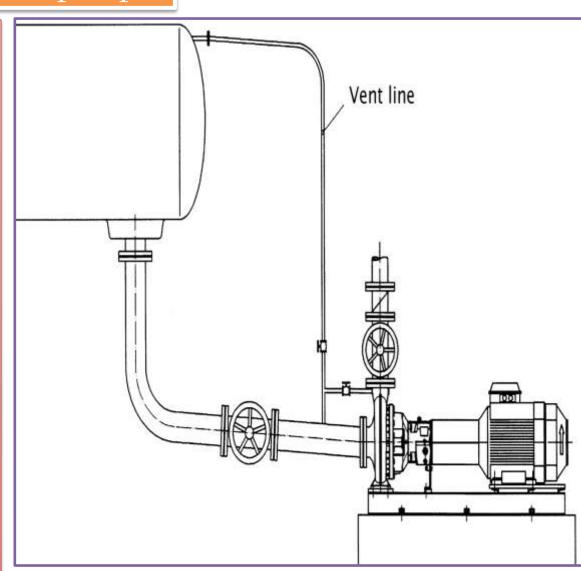
Any sealing, flushing or cooling pipe connections must be installed



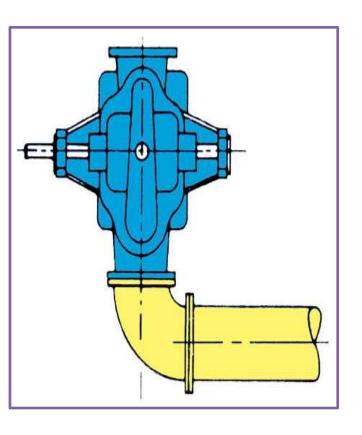
#### Connecting the pipes to the pump

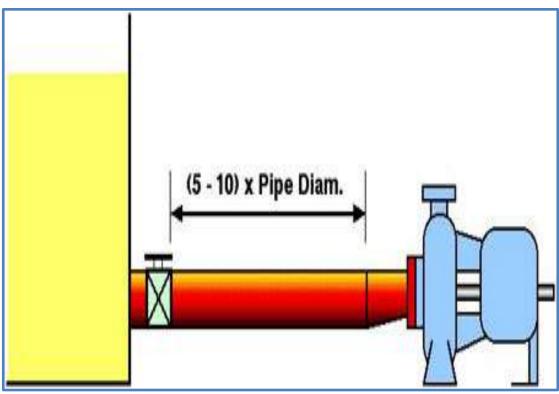
This vent will prevent air binding problem during operation and facilitate proper filling and vent of the pump during startup.

An additional flushed piping — discharge branch-vent line — makes it easier to deaerate the pump before start-up.



## Connecting the pipes to the pump





#### Electrical connection

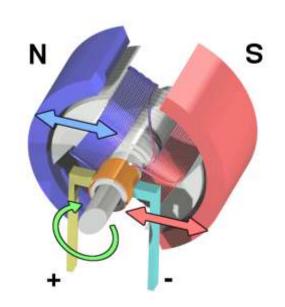


Electrical connection work may only be carried out by an authorized professional

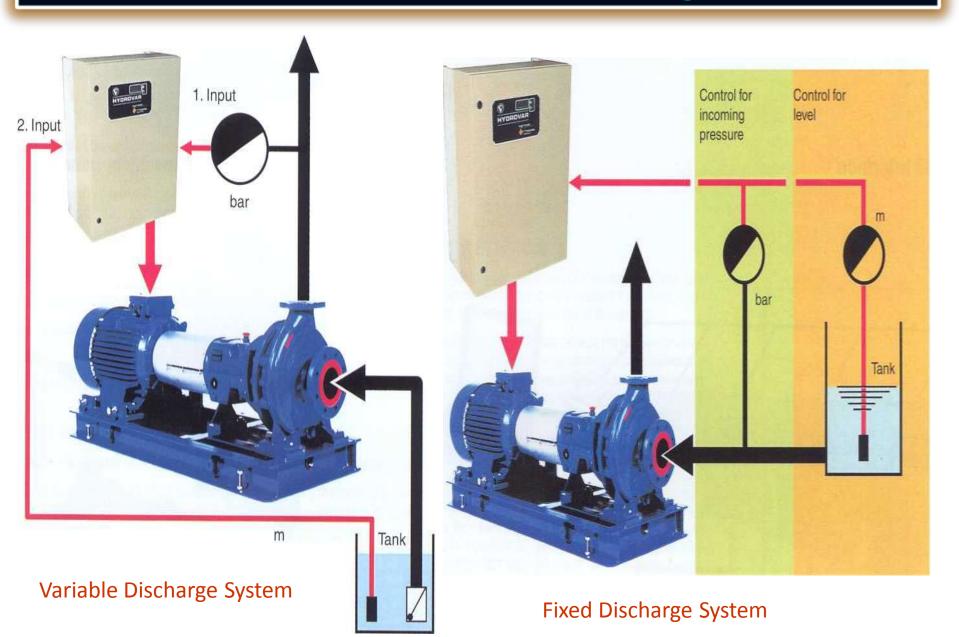
Before starting work, check that the information on the motor rating. The power supply cable accordance with the wiring diagram. A protective motor switch is to be provided.



The direction of rotation should only be checked when the pump is full. Dry running will cause damage to the pump.



# Tank Transfer\Unloading



#### Starting up

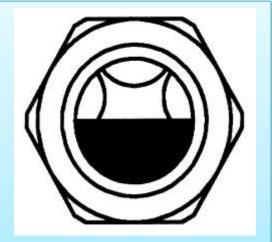


The plant may only be started up by people who are familiar with the local safety regulations and with these Operating Instructions

#### Starting up for the first time

If pump is oil lubricated, first open oil drain and drain off any liquid that may have collected.

Fill until the fluid level is at the mid of the oil level sight glass.



For pumps with grease lubrication, no further lubrication is needed before initial start-up.

#### Starting up

- ·Pump and suction pipe must be filled completely with liquid when starting up.
- ·Turn pump unit once again by hand and check that it moves smoothly and evenly.
- ·Check that coupling guard is installed and that all safety devices are operational.
- ·Switch on any sealing, flushing or cooling devices that are provided.
- ·Open isolation valve in suction /intake pipe.
- ·Set discharge side isolation valve to approx. 25% of rated flow quantity.
- With pumps with a discharge
- branch rated width less than 200, the isolation valve can remain closed when starting up
- ·Check direction of rotation by switching on and off briefly. It must be the same as the directional arrow on the bearing frame.
- ·Start drive device.
- ·As soon as it reaches normal operating speed, open discharge valve immediately and adjust the required operating point.

#### Starting up / Packing

During the first few hours of operation, slowly reduce the leakage rate as the pump is running by gradually tightening the packing gland. The guideline value is around 30 - 100 drops/minute



Packing that run dry will harden and then destroy the shaft sleeve and/or the shaft.

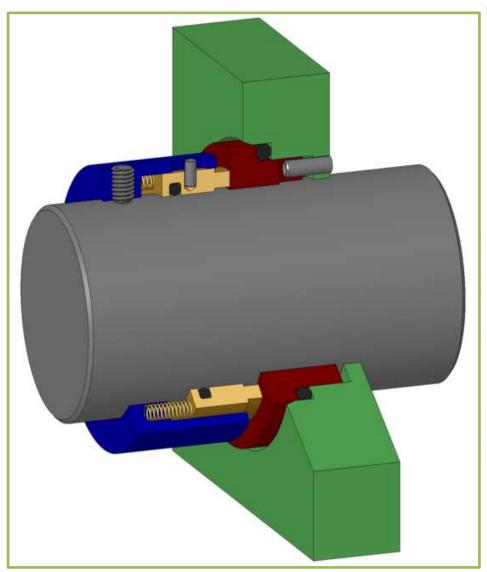


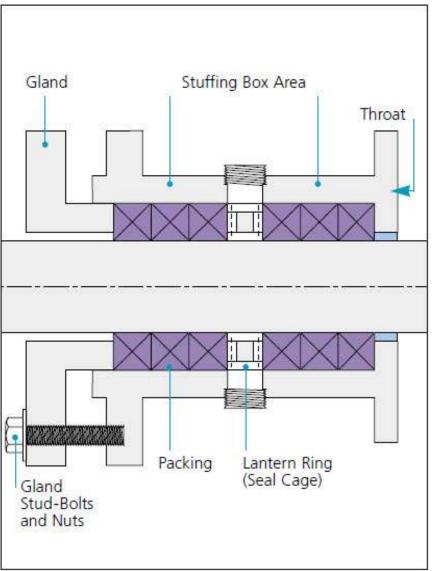
If pump does not reach attended head or if atypical sounds or vibrations do occur:
Switch off pump and seek for causes

## Starting up /Packing

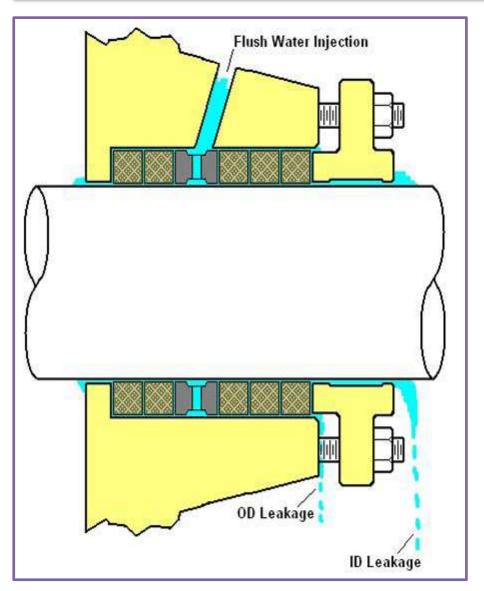


# Packing & Mechanical Seal





# The Basic Sealing Problem:

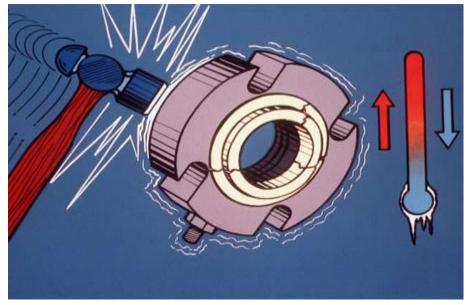




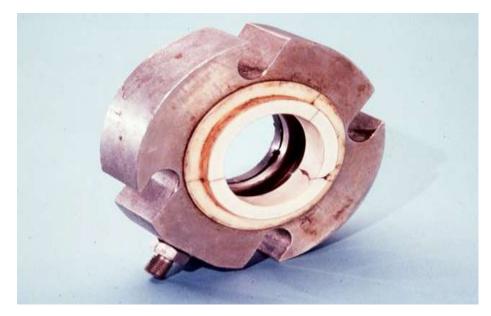


## MECHANICAL SEAL COMMON FAILURES



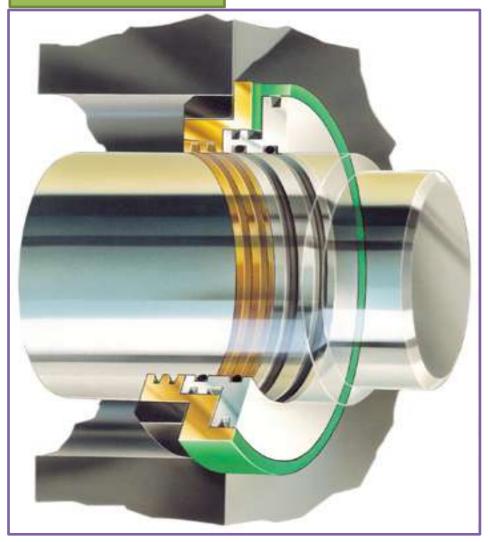


- RUN-DRY
- DEAD HEADING
- TEMPERATURE
- ALIGNMENT
- VIBRATION
- PARTICULATE / ABRASIVES
- CHEMICAL INCOMPATIBILITY





## Labyrinth seals





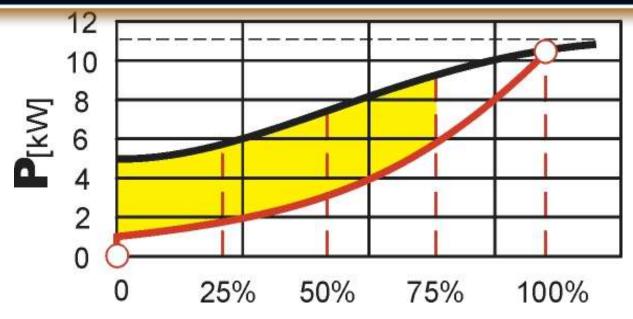
#### Operation and Monitoring

#### suction pressure:

Suction pressure is the actual pressure, positive or negative, at the pump suction connection as measured on a gauge. Pumps do not "suck" fluid as the pump suction



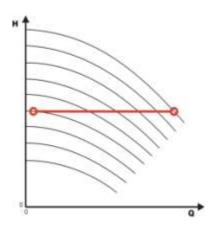
## Reduces Operating Cost



	power consum	ption as per curve		
capacity in %	pumpat constant speed	pump at variable speed	saving in kW	saving per 1/3 year (2920 hours)
25 % 50 %	5,8 kW 7,6 kW	1,8 kW 3,2 kW	4,0 kW 4,4 kW	11.680 kWh 12.848 kWh
75%	9,2 kW	5,7 kW	3,5 kW	10.220 kWh
				34.748 kWh

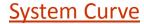
Energy saving within 1 year (8.760 hours)

## **Process Application**

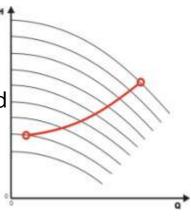


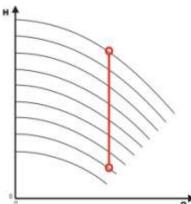
#### **Constant Pressure**

- Spray Nozzles
- Decoking
- Boiler Feed
- Fire Suppression



- Pilot Plant
- Heat Exchanger Feed



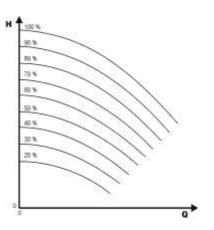


#### **Constant Flow**

- Distillation Tower
- Reactor Feed
- Filter Supply
- Pipe line

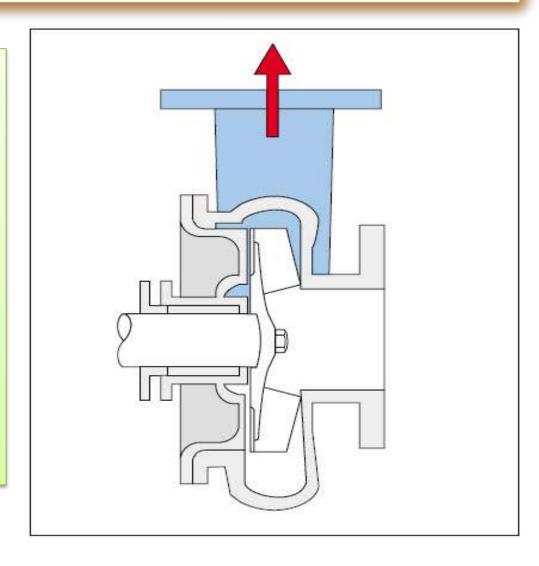
#### **Actuator Mode**

- Process Metering
- Tank\Sump Level Control
- Tank Unloading



#### pump discharge?

The pump discharge is the outlet or flange area where the fluid leaves the olute (casing). The discharge flange is usually oriented up (or vertically), but can also be mounted sideways (or horizontally) if the application requires it.



# Troubleshooting a System Problem

Some pumping system problems are sufficiently expensive to justify a system assessment. Examples of these problems include inefficient operation, cavitations, poor flow control, and high maintenance

#### Inefficient Operation.

Inefficient system operation can be caused by a number of problems as: improper pump selection. poor system design. excessive wear-ring clearances.

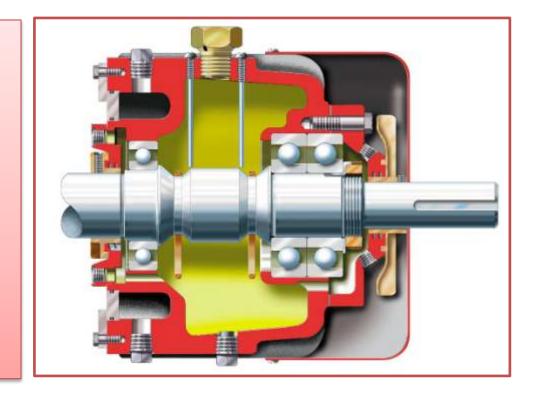
wasteful flow control practices.

Indications of inefficient system operation include:

high energy costs.

excessive noise in the pipes and across valves.

high maintenance requirements

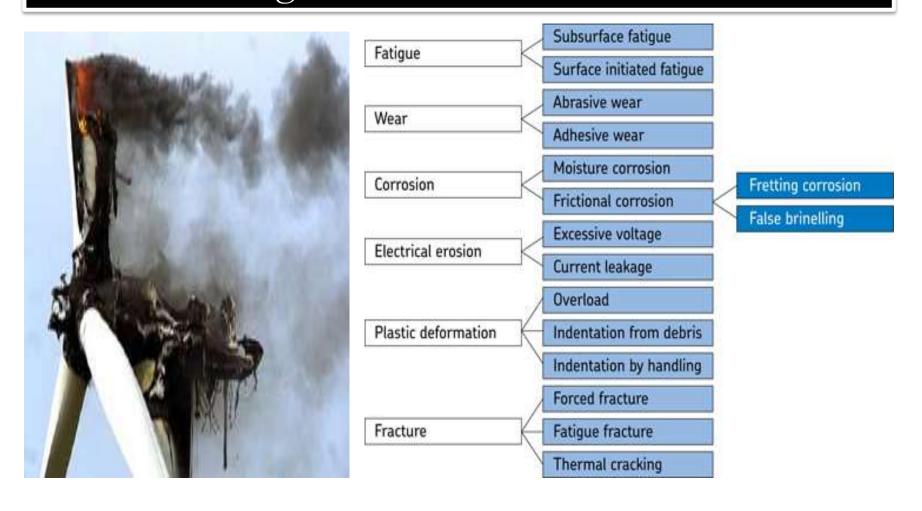


## Bearing Failures and Their Causes





# Bearing Failures and Their Causes



# Troubleshooting a System Problem



## PUMP FAILURE ANALYSIS

#### 6 month period in a typical process plant

CAUSE	NUMBER	% of TOTAL
Bearing	25	10.50
Bearing housing	1	0.42
Case wearing ring	2	0.84
lm p e lle r	8	3.36
Rotating face	1	0.42
Screws /set screws	1	0.42
Seals - mechanical	179	75.21
Shaft	12	5.04
Sleeve	9	3.78
TOTAL	238	100.00%

## Results of Operating Off BEP

High Temp. Rise <u>Low Flow Cavitati</u>on **Discharge Recirculation** Low Brg. & Seal Life Reduced Impeller Life Head Suction Recirculation **BEP** Low Brg. & Seal Life Cavitation **Flow** 

## Shaft Deflection

Shaft deflection is the result of unbalanced radial loads.

The amount of shaft deflection or bending depends on the amount of unbalance radial forces and a pump's slenderness ratio.

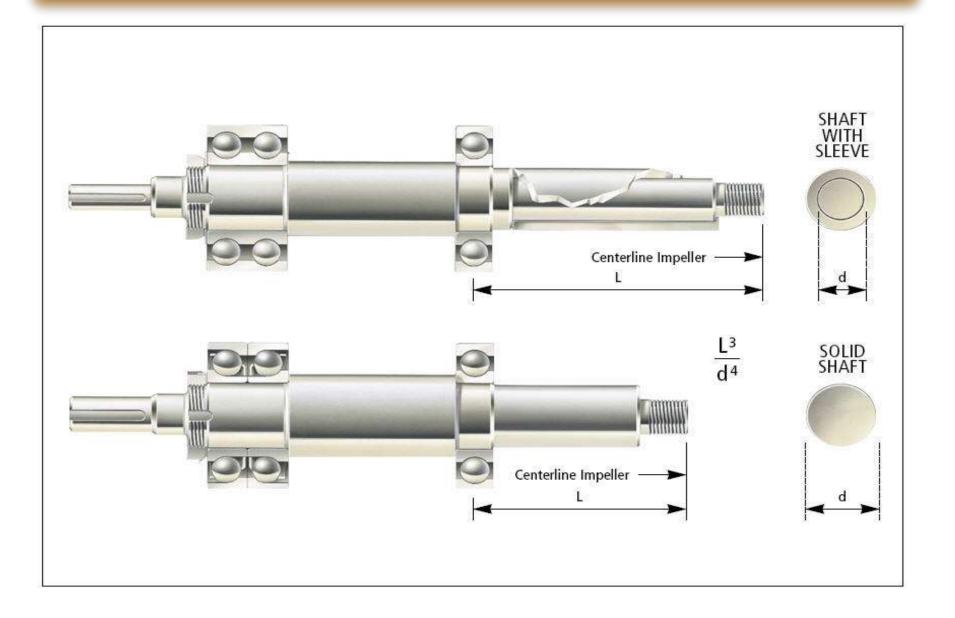
The higher the forces or the larger the slenderness ratio, the more shaft deflection will occur. Shaft deflection results in sealing device, bearing, and other pump mechanical failures.

#### **Shaft Slenderness Ratio**

The Shaft
Slenderness Ratio is
a ratio of shaft
length to shaft
diameter.

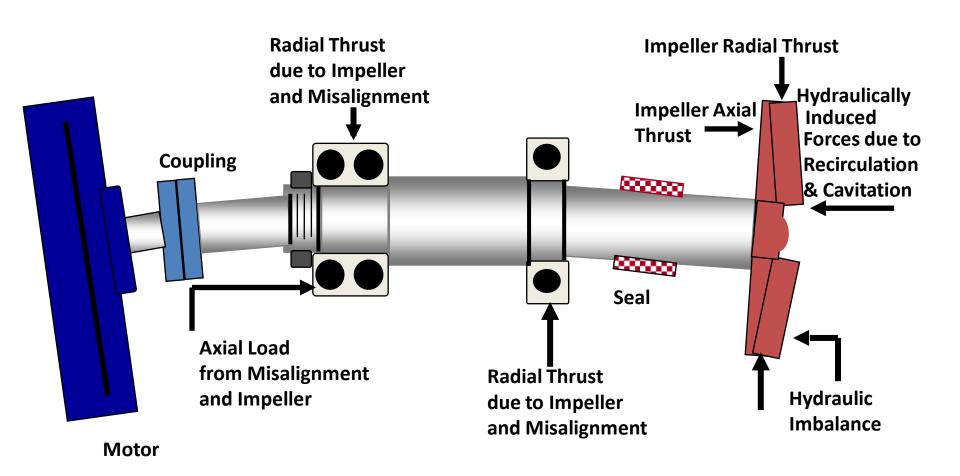


## Shaft Deflection



## Shaft Deflection

#### SIMULTANEOUS DYNAMIC LOADS ON PUMP SHAFT



#### TEMPERATURE RISE

#### Overheating of the liquid in the casing can cause:

- Rubbing or seizure from thermal expansion
- Vaporization of the liquid and excessive vibration
- Accelerated corrosive attack by certain chemicals

Temperature rise per minute at shutoff is:

```
\DeltaT °F (°C) / min.= HP (KW)<sub>so</sub> x K Gal (m³) x S.G. x S.H.
```

 $HP_{so}$  = HP (KW) @ shutoff from curve

Gal.  $(m^3)$  = Liquid in casing

S.G. = Specific gravity of fluid

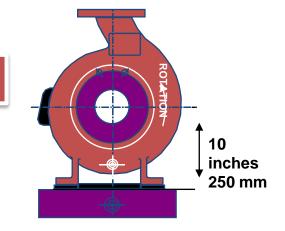
S.H. = Specific heat of fluid

Ex.: Pump w/ 100HP (75KW) @s.o., 6.8 gal casing (.03 $m^3$ ) w/ 60°F (16°C) water would reach boiling in 2 min.

A recirculation line is a possible solution to the low flow or shut off operation problems....

#### CASING GROWTH DUE TO HIGH TEMPERATURE

#### High temperature requires more clearance



COEFFICIENT OF THERMAL EXPANSION FOR 316 S/S IS 9.7X10<sup>-6</sup> IN/IN/°F OR 17.5 X10<sup>-6</sup> MM/MM/°C CALCULATION IS T x 9.7 X10<sup>-6</sup> X LENGTH IN INCHES

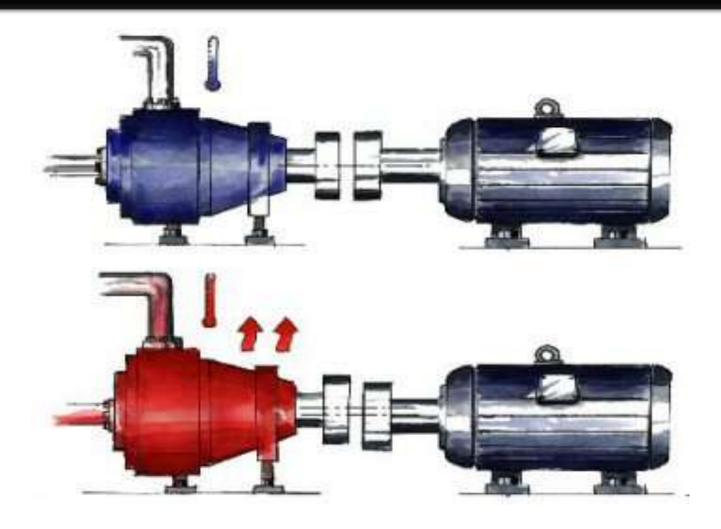
Δ

T x 17.5X10-6 X LENGTH IN

#### **MILLIMETERS**

	1 1	EXPANSION								
<u>∆T° F</u>	△T° C	INCHES	MILLIMETERS							
100 F	55 C	0.0097 IN	0.245 MM							
200 F	110 C	0.0190 IN	0.490 MM							
300 F	165 C	0.0291 IN	0.735 MM							
400 F	220 C	0.0388 IN	0.900 MM							
500 F	275 C	0.0485 IN	1.230 MM							
600 F	330 C	0.0582 IN	1.470 MM							

## PUMP GROWTH DUE TO HIGH TEMPERATURE



## **IMPELLER BALANCE**

#### *EXAMPLE ANICAL*

- Weight offset from center of impeller
- Balance by metal removal from vane

#### **← HYDRAULIC**

- Vane in eye offset from impeller C/L
- Variation in vane thickness
- Results in uneven flow paths thru impeller
- Investment cast impeller eliminates problem
- Careful machining setup can help



#### **Centrifugal Pump Troubleshooting Guide**

Table 4-6 Table 4-6 (cont.) Troubleshooting Guide -- Centrifugal Process Pumps Symptoms Symptoms Symptoms Insufficient Disch. Pressure E E D Short Bearing Life D Insufficient Disch. Pressure Short Bearing Life F C Intermittent Operation Short Mech. Seal Life C Intermittent Operation Short Mech. Seal Life F Insufficient Capacity В Vibration & Noise G В Insufficient Capacity Vibration & Noise G No Liquid Delivery Power Demand Excessive H Power Demand Excessive No Liquid Delivery Possible Causes C DFF G H # Possible Remedies # A BCDEFG Possible Causes Possible Remedies Pump Is Cavitating 2 1 1 \* Check NPSHa/ NPSHr Strainer Partially \* Inspect And Clean (Symptom For Liquid Margin Clogged Vaporizing In Suction \* If Pump Is Above Pump Impeller 8 8 8 5 \* Check For Damage System) -Horizontal Liquid Level, Raise Clogged Pumps And Clean Liquid Level Closer To Suction And/Or 9 9 \* Shut Down And Open Pump Discharge Valve(s) Valves \* If Liquid Is Above Closed Pump, Increase Liquid Viscosity Too High 10 5 \* Heat Up Liquid To Level Elevation Reduce Viscosity Insufficient 1 1 1 1 1 \* Lower Suction Pipe SUCTION PROBLEMS Immersion Of Suction \* Increase Size Of Or Raise Sump Level Pipe Or Bell Discharge Piping To \* Increase System (VertTurbinePump) Reduce Pressure Loss Resistance HYDRAULIC SYSTEM \* Use Larger Driver Or Pump Not Primed 3 1 2 \* Fill Pump And Suction Change Type Of Pump Piping \* Slow Pump Down Complete With Liquid \* Eliminate High Points Specific Gravity Too 11 2 11 \* Check Design Specific In Suction High Gravity \* Remove All Non-Total System Head 12 4 11 \* Increase System Condensibles (Air From Lower Than Design Resistance To Obtain Pump, Piping and Head Of Pump Design Flow Valves) \* Check Design \* Eliminate High Points Parameters Such As In Suction Piping Impeller Size, Etc. \* Check For Faulty Foot Total System Head 13 6 5 10 2 \* Decrease System Valve Or Check Valve Higher Than Design Resistance To Obtain Head Of Pump Design Flow Non-Condensibles In 3 2 | 1 \* Check For Gas/Air HYDRAULIC SYSTEM \* Check Design Liquid Ingress Through Suction Parameters Such As System/Piping Impeller Size, Etc. \* Install Gas Separation Unsuitable Pumps In 14 6 \* Check Design Chamber Parallel Operation Parameters \* Refill Supply Tank Supply Tank Empty 5 3 5 Improper Mechanical \* Check Mechanical Obstructions In Lines 9 \* Inspect And Clear Seal Seal Selection Strategy Or Pump Housing Possible Causes # A Possible Remedies Possible Causes BCDE Possible Remedies BCDEFGH#

#### **Centrifugal Pump Troubleshooting Guide**

	A	0.0000		- 35	Tabl	e 4-	6 (c	ont.)				CARAMETER
	Sympto	0000001	- 7	8			P 78					mptoms
D Insufficient Disch. Pressure						Sh	_		ing I		E	
	C Intermittent Operation							Short Mech. Seal Life F				
В	Insufficie	ent C	ent Capacity						Vibration & Noise			Noise G
A	No Liquid	Deliv	ery	L						Po	wer	Demand Excessive H
	Possible Causes	#	Α	В	С	D	Ε	F	G	Н	#	Possible Remedies
	Speed Too High	16								1	16	* Check Motor Voltage - Slow Down Driver
	Speed Too Low	17	4	4		2			. te		17	* Consult Driver Troubleshooting Guide
	Wrong Direction Of Rotation	18	5			3				6	18	* Check Rotation With Arrow On Casing - Reverse Polarity On Motor
	Impeller Installed Backward (Double Suction Imp.)	19	3:	10						12	19	* Inspect
MECHANICAL SYSTEM	Misalignment	20					1	2	4	7	20	* Check Angular And Parallel Alignment Between Pump And Driver
	Casing Distorted From Excessive Pipe Strain	21	33 6°				2	3	5		21	* Check For Misalignment  * Check Pump For Wear Between Casing And Rotating Elements  * Analyze Piping Loads
	Inadequate Grouting Of Base	22	5 T					3	6	5-5 5-5	22	* Check Grouting And Regrout If Required
	Bent Shaft	23	8	J.	0		3	4	7	8	23	* Check Deflection (Should Not Exceed 0.002"). Replace Shaft And Bearings If Necessary
	Internal Wear	24				8				9	24	* Check Impeller Clearances
	Possible Causes	#	Α	В	С	D	Ε	F	G	Н	#	Possible Remedies

				1.00	Tabl	e 4-	6 (cc	ont.)	0				_	
Symptoms								Symptoms						
	D Insufficient Disch. Pressure							Short Bearing Life E						
C Intermittent Ope					on	1	8	Sh	ort	Mec	h. Se	al Life F		
В	B Insufficient Capac								Vil	brati	on &	Noise G		
Α	No Liquid Del		elivery						Pow		wer	er Demand Excessive H		
SYSTEM	75000		A	В	C	D	E	<b>F</b> 5	8	10	# 25	* Inspect Parts For Defects - Repair Or Replace. Use Bearing Failure Analysis Guide * Check Lubrication Procedures	WAR AND S	
MECHANICAL	Unbalance - Driver	26	8	82	55		5	7	9		26	* Run Driver Disconnected From Pump Unit - Perform Vibration Analysis		
ME	Unbalance - Pump	27	i i				4	6	3		27	* Investigate Natural Frequency	- 3	
	Motor Troubles	28					6	8	10	11	28	* Consult Motor Troubleshooting Guide	8	
	Possible Causes	#	Α	В	C	D	E	F	G	Н	#	Possible Remedies		



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# Any Questions?

