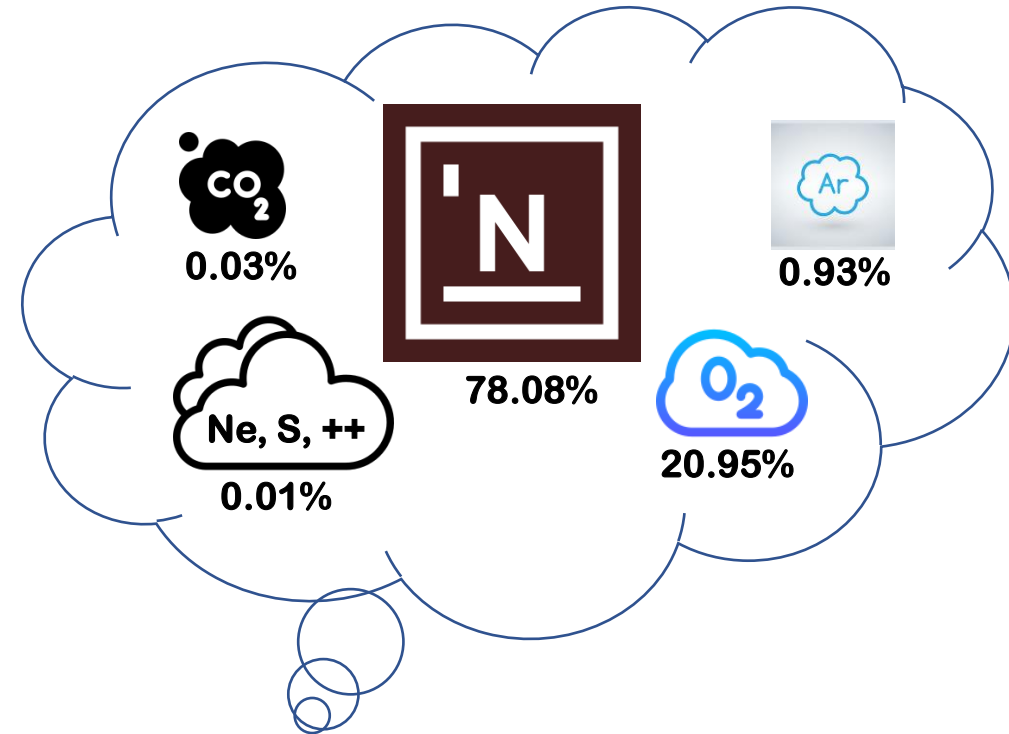
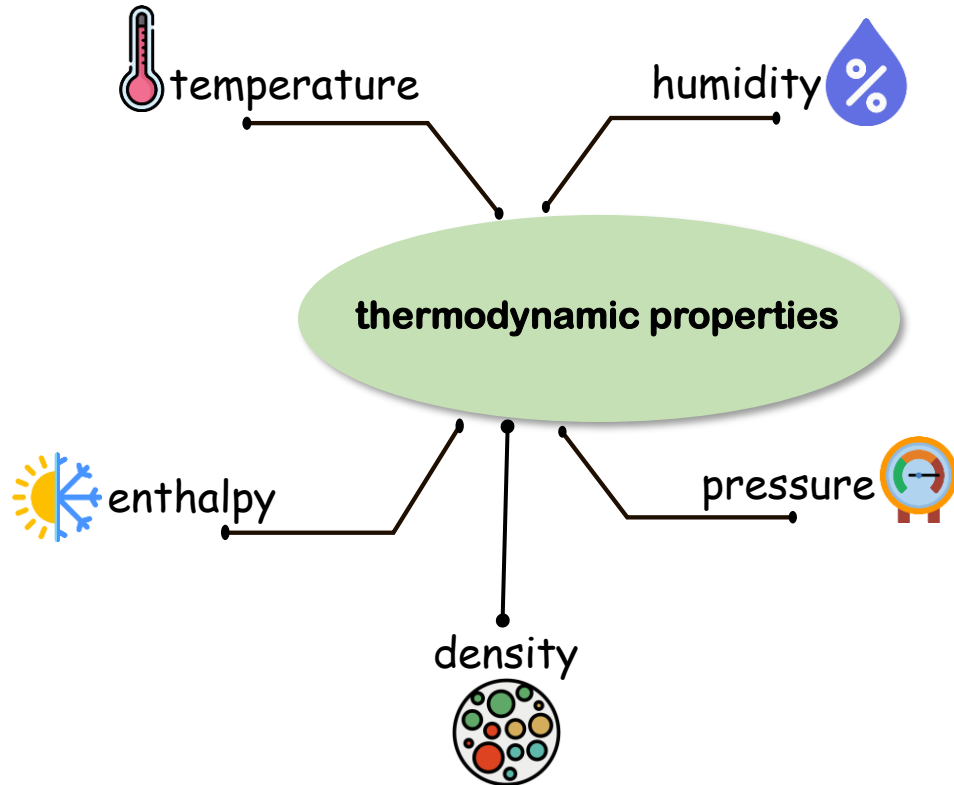


study of the thermodynamic properties of moist air (atmospheric air)

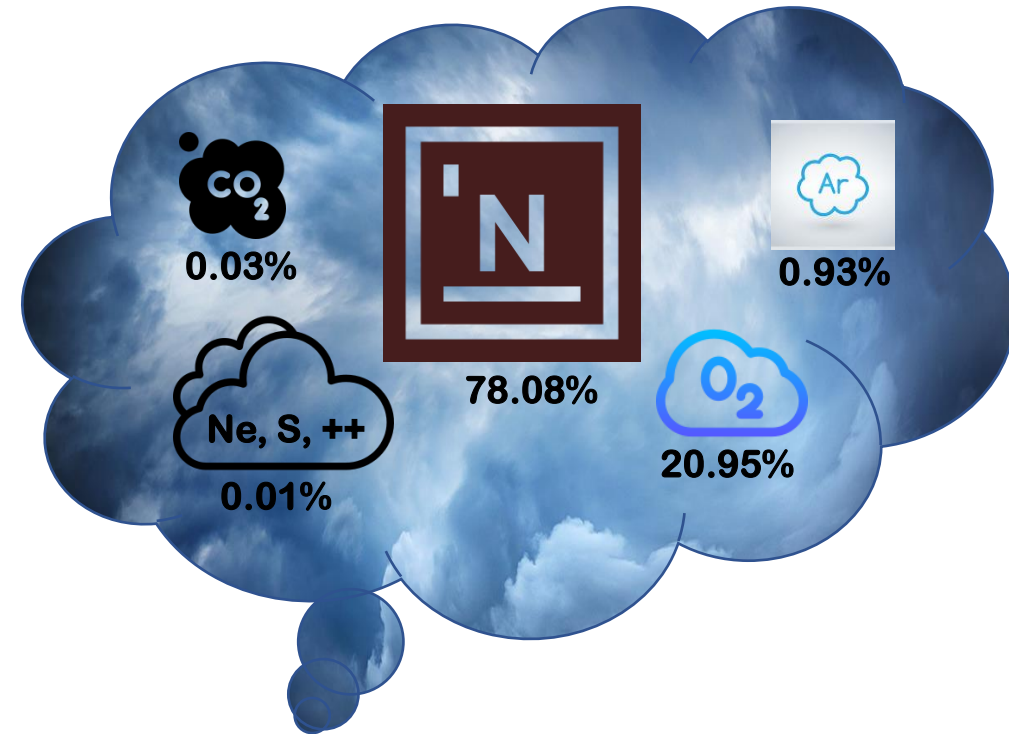
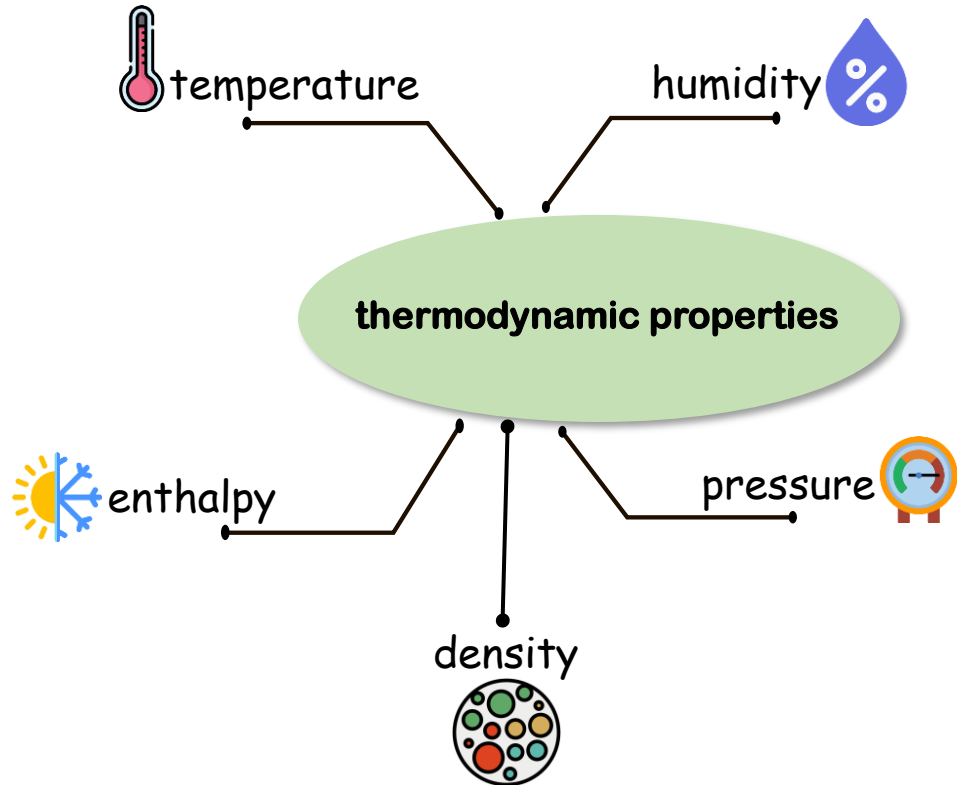
dry air + water vapor(moisture)



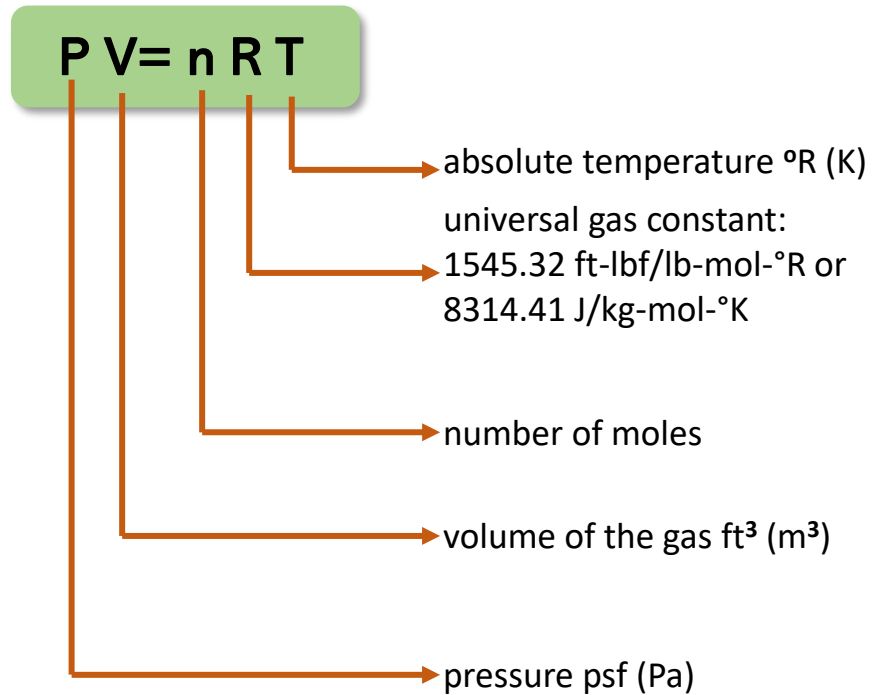
psychrometry

study of the thermodynamic properties of moist air (atmospheric air)

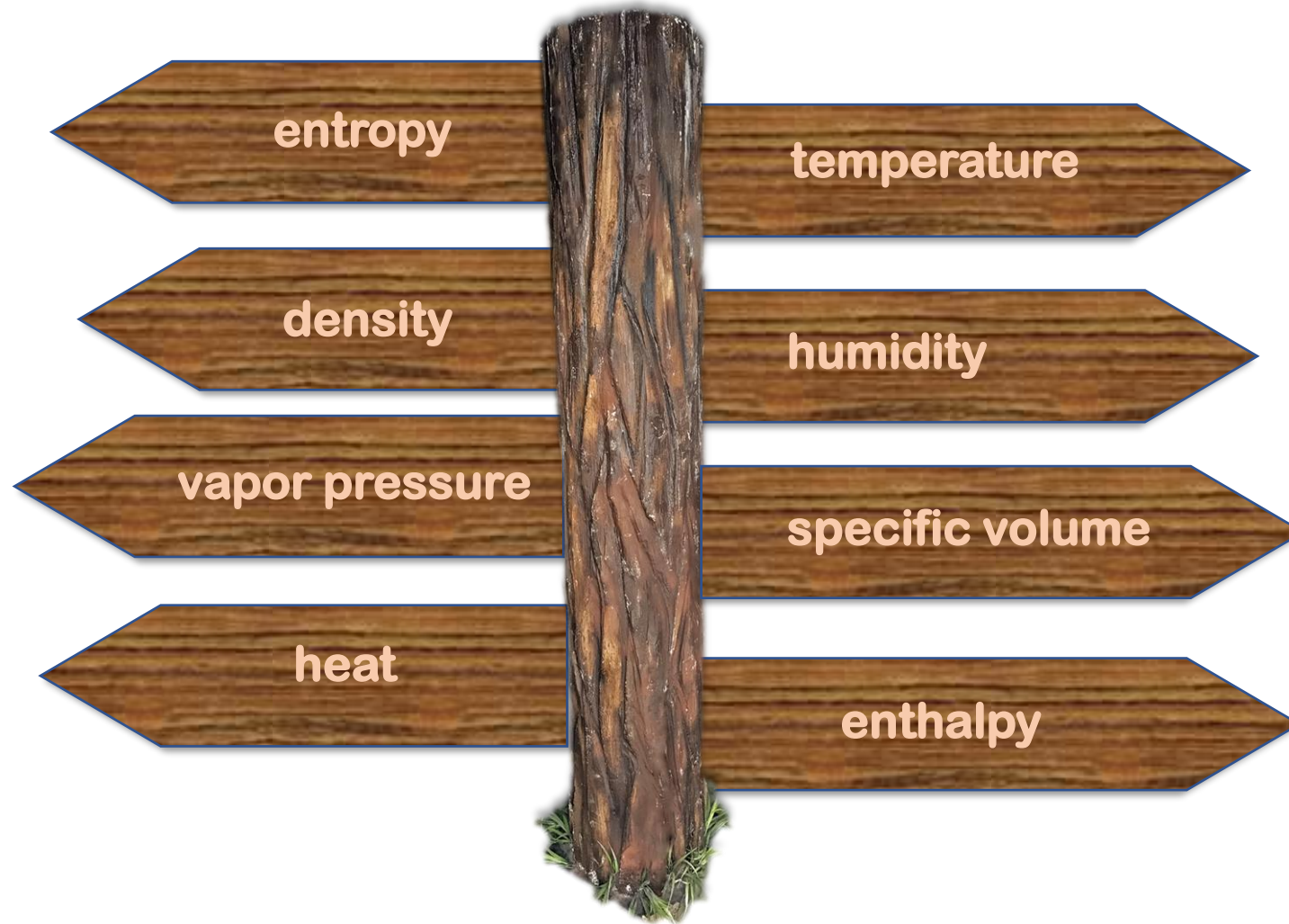
dry air + water vapor(moisture)



Ideal Gas Equation

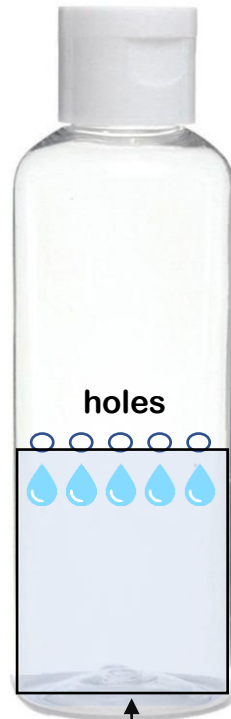


psychrometry



humidity

air has the ability to hold water



maximum water vapor
the air can hold



at higher temperature,
air can hold more
water vapor



unsaturated air



saturated air

humidity

humidity
ratio



$$\text{humidity ratio} = \frac{\text{mass of water vapor}}{\text{mass of dry air}}$$

$$W = \frac{m_v}{m_a}$$

$$W = \frac{0.6219P_v}{P_T - P_v}$$

humidity

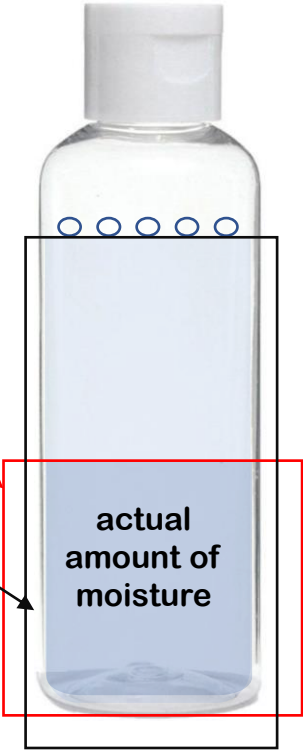
humidity ratio

relative humidity

$$\text{Relative Humidity} = \frac{\text{actual moisture in the air}}{\text{maximum moisture air can hold}} \times 100\%$$

$$\phi = \frac{m_v}{m_{vs}} \quad \phi = \frac{P_v}{P_{vs}} \quad \phi = \frac{WP_a}{0.6219P_{vs}}$$

example:
At 25°C,
max. moisture holding capacity= 60g
actual moisture present in air= 30g
 $RH = \frac{30}{60} * 100\%$
 $RH = 50\%$



humidity

humidity ratio

relative humidity

$$\text{Relative Humidity} = \frac{\text{actual moisture in the air}}{\text{maximum moisture air can hold}} \times 100\%$$



T= 25°C
max. moisture holding capacity= **60g**
actual moisture present in air= 30g
 $RH = \frac{30}{60} * 100\%$
 $RH = 50\%$



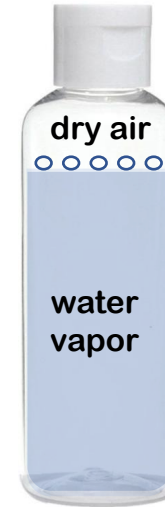
T= 40°C
max. moisture holding capacity=**80g**
actual moisture present in air= 30g
 $RH = \frac{30}{80} * 100\%$
 $RH = 37.5\%$

humidity

humidity ratio

relative humidity

degree of saturation



degree of saturation μ \equiv $\frac{\text{humidity ratio}}{\text{humidity ratio of saturated air}}$ at same T and P

$$\mu \equiv \frac{W}{W_s}$$

humidity

humidity
ratio

relative
humidity

degree of
saturation

absolute
humidity

Absolute Humidity is the total mass of water vapor present in unit volume of moist air



$$\text{absolute humidity} = \frac{\text{mass of water vapor}}{\text{volume of moist air}}$$

moist air

total volume of moist air= 2m^3

mass of moisture present= 60g

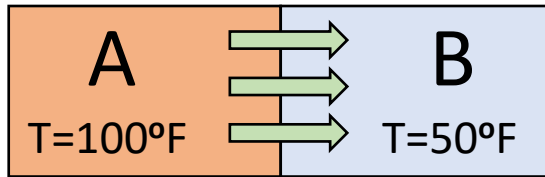
$$AH = \frac{60\text{g}}{2\text{m}^3}$$

$$AH = 30\text{g}/\text{m}^3$$

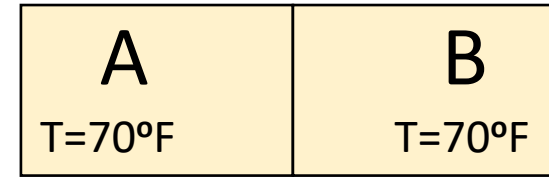
temperature



Temperature is the measure of hotness or coldness of a body



- ✓ heat flows due to the difference in temperature



- ✓ no heat flow once the temperature of both bodies become same
- ✓ thermal equilibrium is achieved (same T for both)

Units of temperature

Kelvin(K) → International Unit (SI)
Celsius (°C)
Fahrenheit (°F)
Rankine (°R)

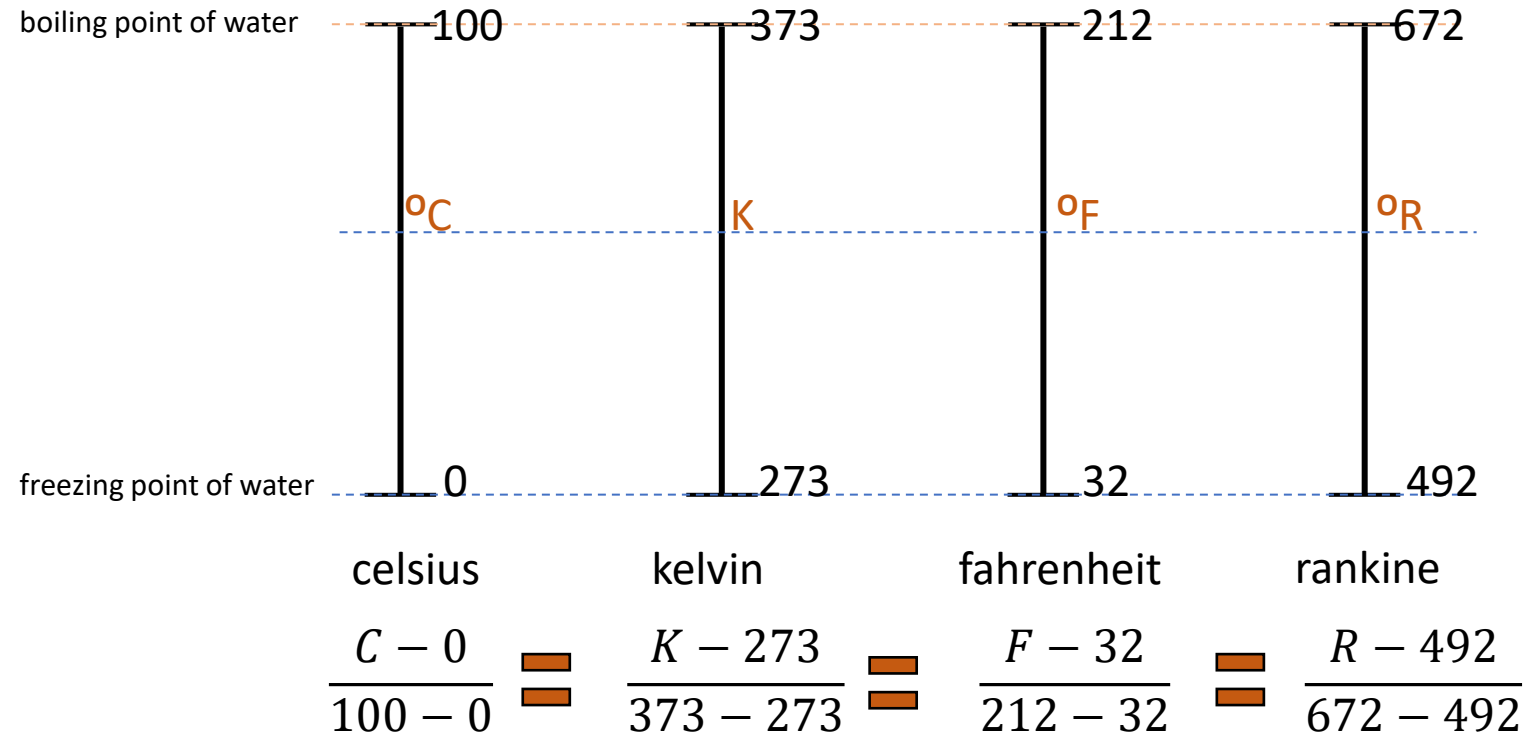
temperature



Units of temperature

Kelvin(K)
Celsius (°C)
Fahrenheit (°F)
Rankine (°R)

conversion of units

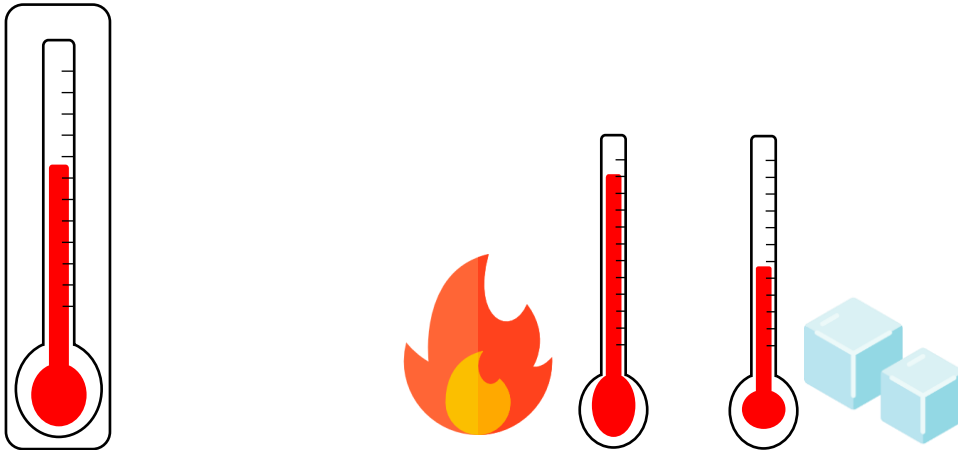


temperature

- ✓ **The dry-bulb temperature (DBT)** is the temperature of air measured by a thermometer freely exposed to the air, but isolated from radiation and moisture
- ✓ DBT is the temperature that is usually thought of as air temperature
- ✓ It shows the amount of heat content in the air

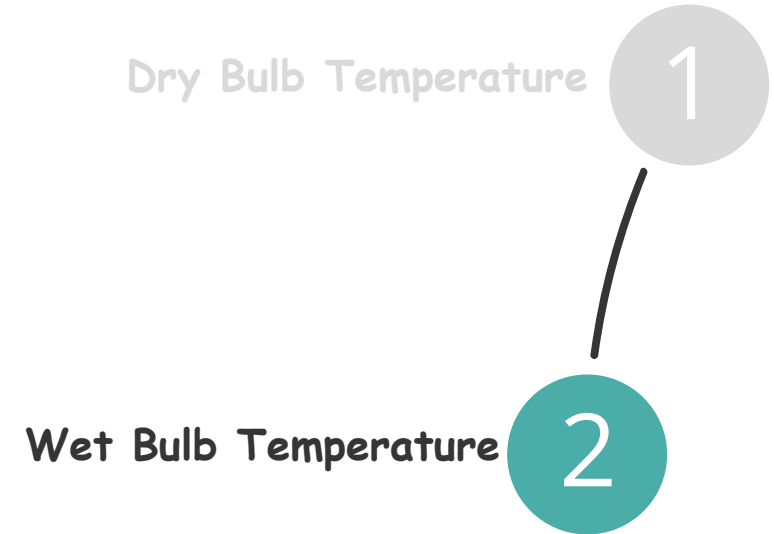
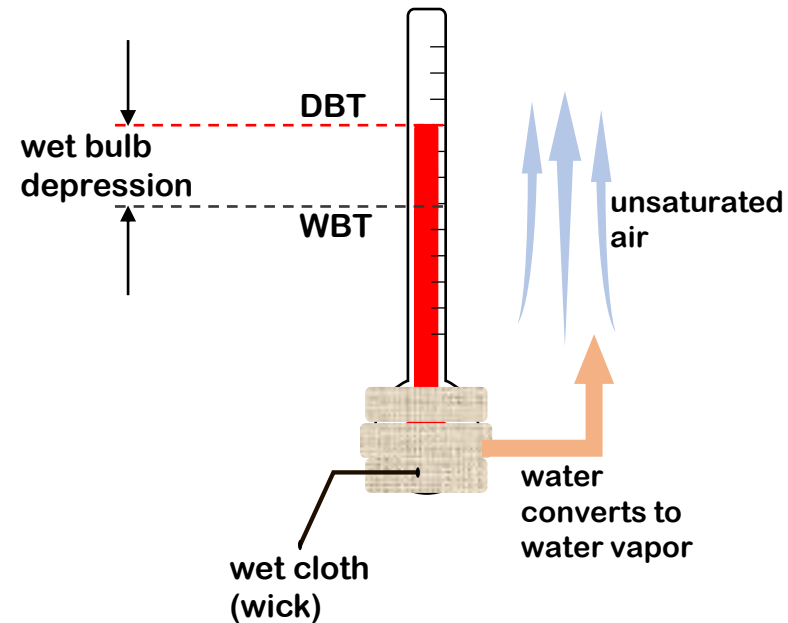
Dry Bulb Temperature

1



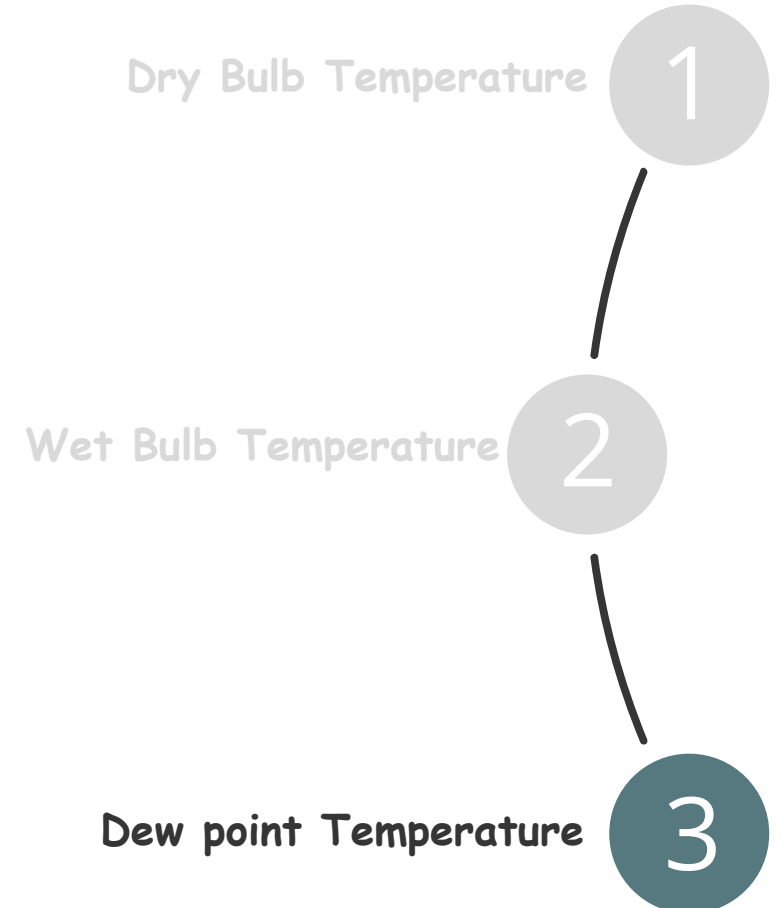
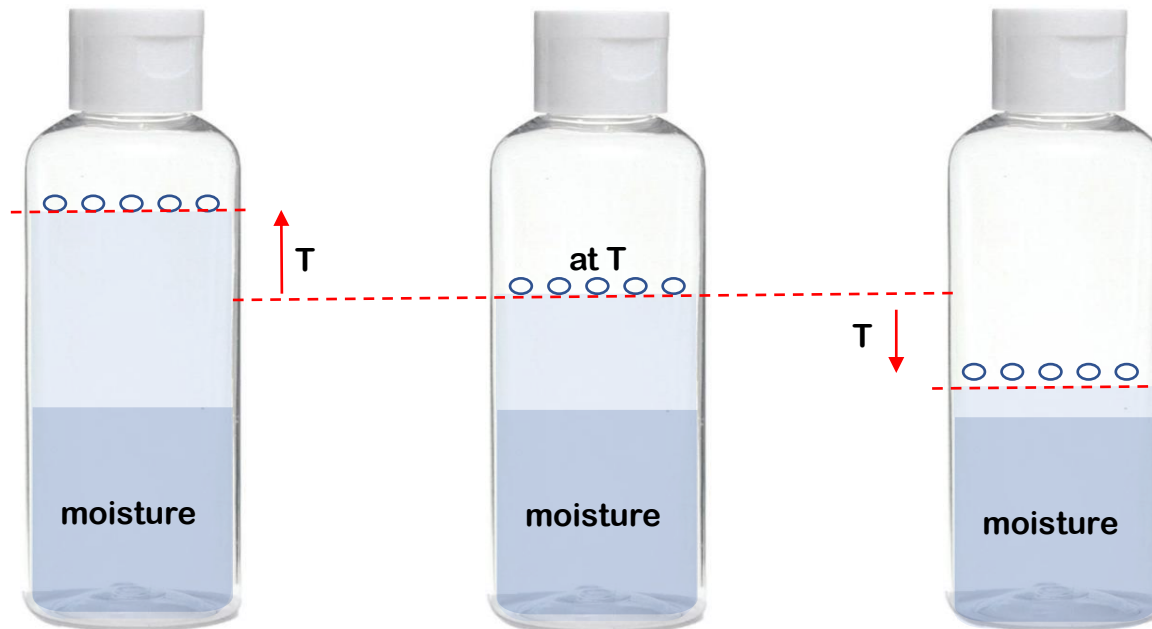
temperature

- ✓ The **wet-bulb temperature (WBT)** is the lowest temperature to which air can be cooled by the evaporation of water into the air at a constant pressure
- ✓ when the air is fully saturated (RH=100%), then DBT=WBT



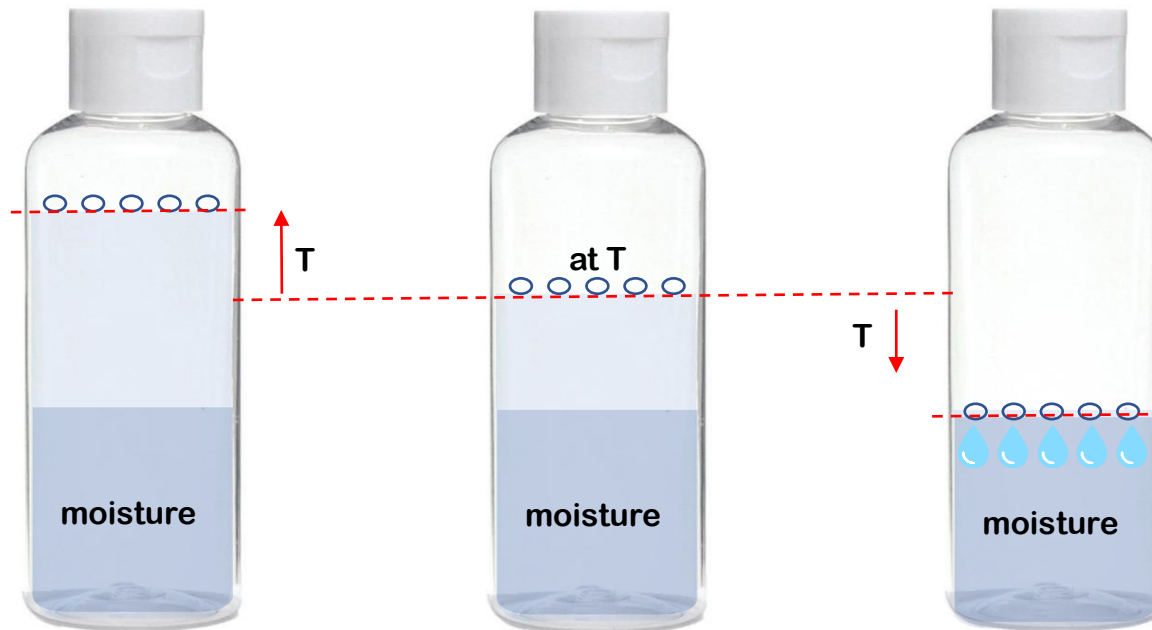
temperature

- ✓ The **dew-point temperature (DPT)** is the temperature to which air must be cooled to become saturated with water vapor it contains
- ✓ On cooling further, the water vapor will condense to form liquid water (dew)



temperature

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- ✓ On cooling further, the water vapor will condense to form liquid water (dew)



Dry Bulb Temperature

1

Wet Bulb Temperature

2

Dew point Temperature

3

specific volume



It is the volume of air per unit mass of dry air

It is measured in ft³/lb

$$\text{specific volume} = \frac{\text{volume of air}}{\text{mass of dry air}}$$

$$\text{specific volume of dry air} = \frac{\text{volume of dry air}}{\text{mass of dry air}}$$

$$\text{specific volume of moist air} = \frac{\text{volume of moist air}}{\text{mass of dry air}}$$

$$\text{specific volume of saturated air} = \frac{\text{volume of moist air at saturation}}{\text{mass of dry air}}$$

specific enthalpy

Enthalpy is the sum of internal energy and pressure-volume energy of air

- $H=U+PV$
- $H=mC_p\Delta T$ (at constant pressure)

Specific Enthalpy is the sum of internal energy and pressure-volume energy of air per unit mass of **dry air**

- $h=C_p\Delta T$
- it is measured in kJ/kg or Btu/lb



- ✓ @ zero K: enthalpy is zero
- ✓ enthalpy increases with temperature

$$\text{for air: } h_{\text{moist_air}} = h_{\text{dry_air}} + W \cdot h_{\text{vapor}}$$

$$h_{\text{dry air}} = C_p T$$

C_p = specific heat of dry air
 $C_p = 0.240$ Btu/lb

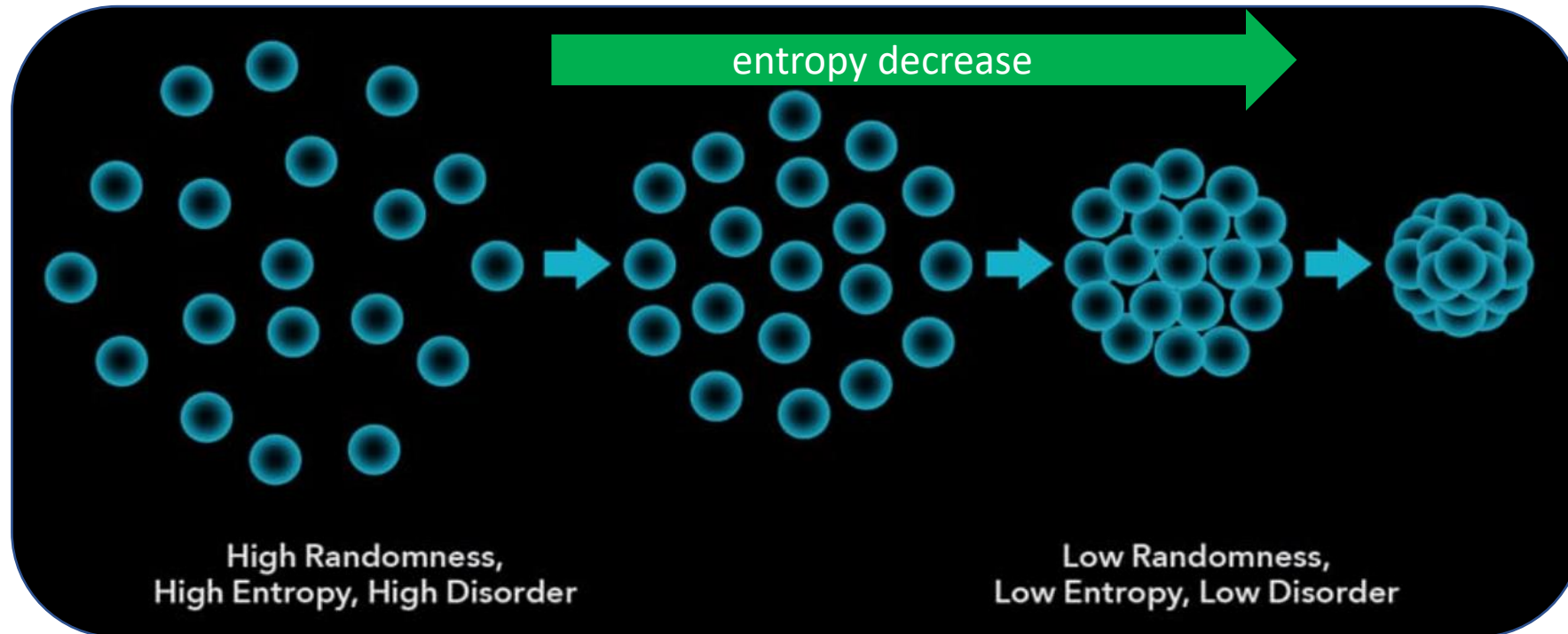
$$h_{\text{vapor}} = h_g + C_{pv} T$$

h_g = sp. enthalpy of saturated vapor
@ 0°F: $h_g = 1061$ Btu/lb
 C_{pv} = specific heat of vapor
 $C_{pv} = 0.444$ Btu/lb

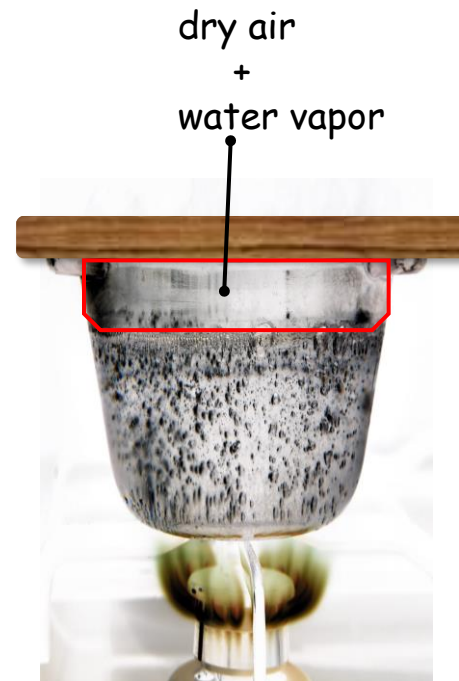
$$h = [0.24T + W(1061 + 0.444T)] \text{ Btu/lb}$$

specific entropy

- ✓ **Entropy** is measure of the molecular disorder or randomness of a system
- ✓ It the measure of a system's thermal energy per unit temperature that is unavailable for doing useful work
- ✓ **Specific Entropy**= (total entropy of the system/mass of the system)
- ✓ It is measured in Btu/lb-°F



vapor pressure



when air is unsaturated

- evaporation > condensation
- P_{w_vapor} = partial pressure of water vapor

when air is saturated

- evaporation = condensation
- $P_{w_vapor} = P_{max}$

→ VAPOR PRESSURE

density



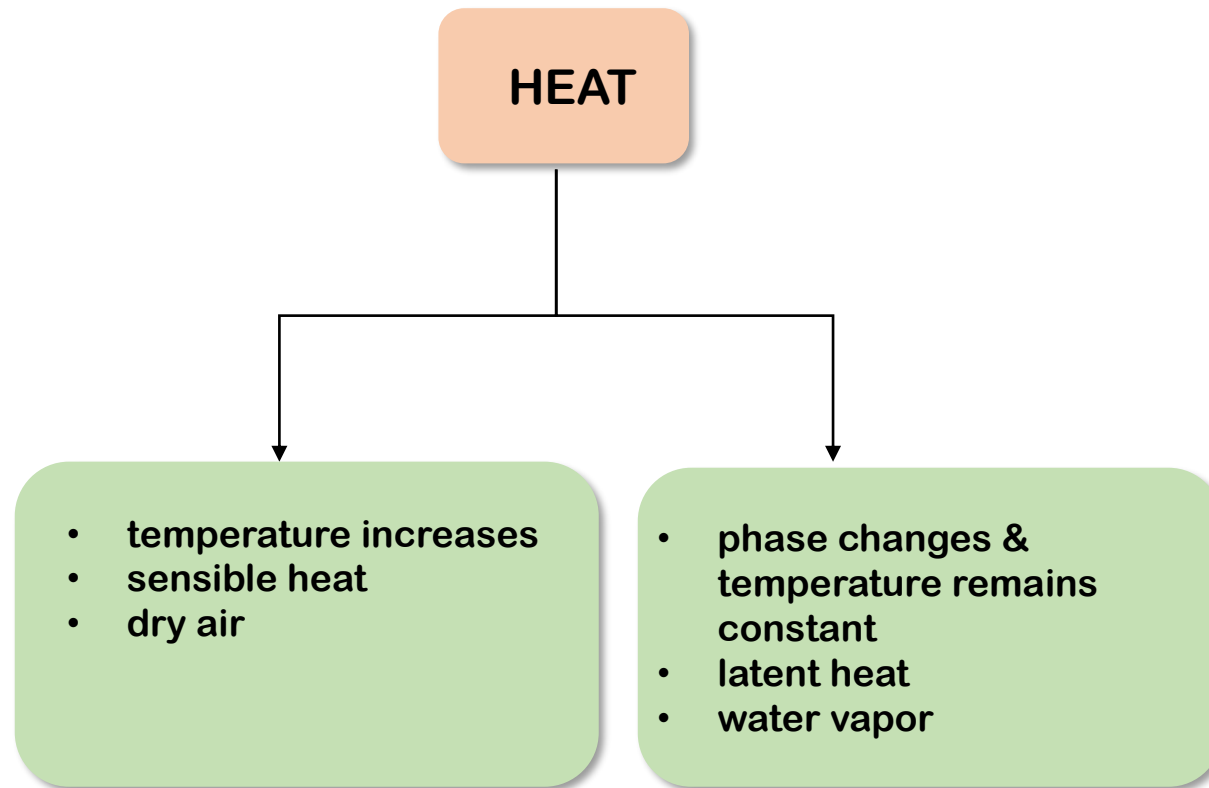
It is the mass of unit volume of air

$$\text{Density of air} = \frac{\text{Mass of air}}{\text{Volume of air}}$$

At STP:

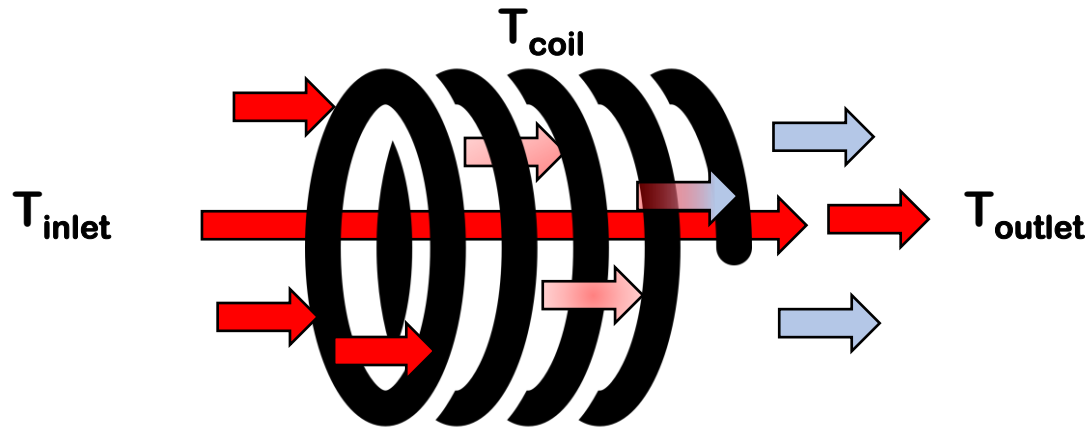
- In Metric units: $\rho_{\text{air}}=1.225 \text{ kg/m}^3$
- In Imperial units: $\rho_{\text{air}}=0.0765 \text{ lb/ft}^3$

heat



bypass factor

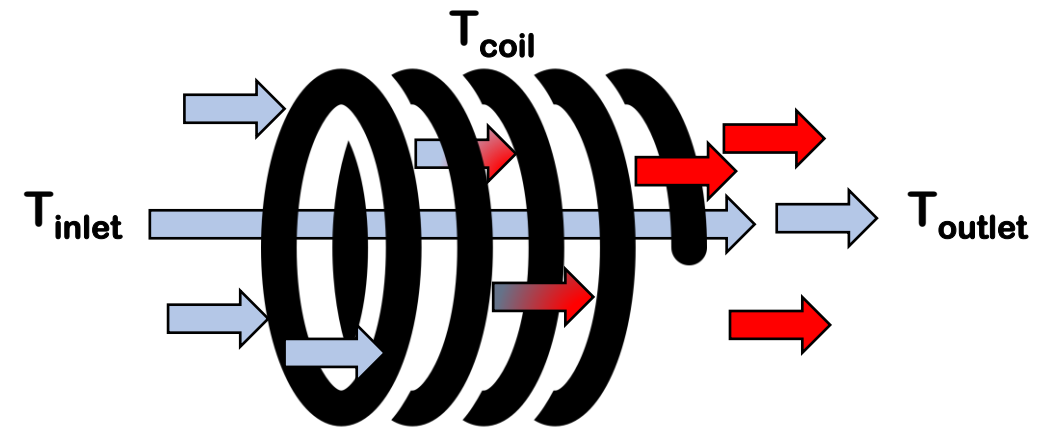
$$T_{in} > T_{out} > T_{coil}$$



cooling coil

$$BPF = \frac{T_{out} - T_{coil}}{T_{in} - T_{coil}}$$

$$T_{coil} > T_{out} > T_{in}$$

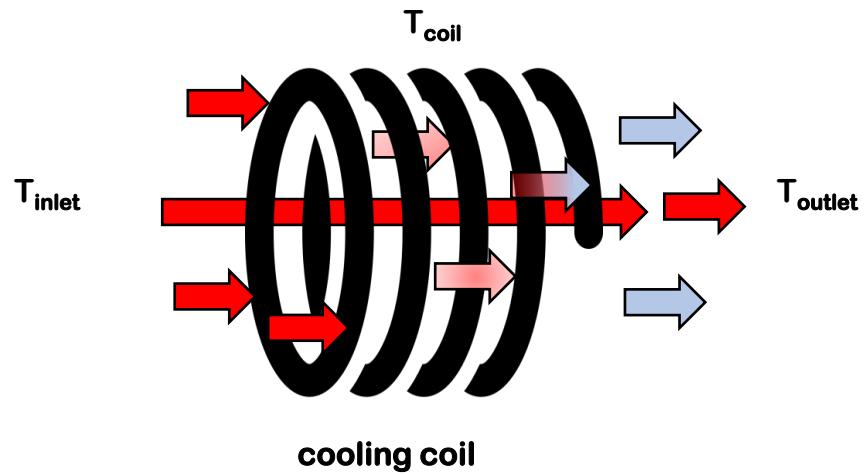


heating coil

$$BPF = \frac{T_{coil} - T_{out}}{T_{coil} - T_{in}}$$

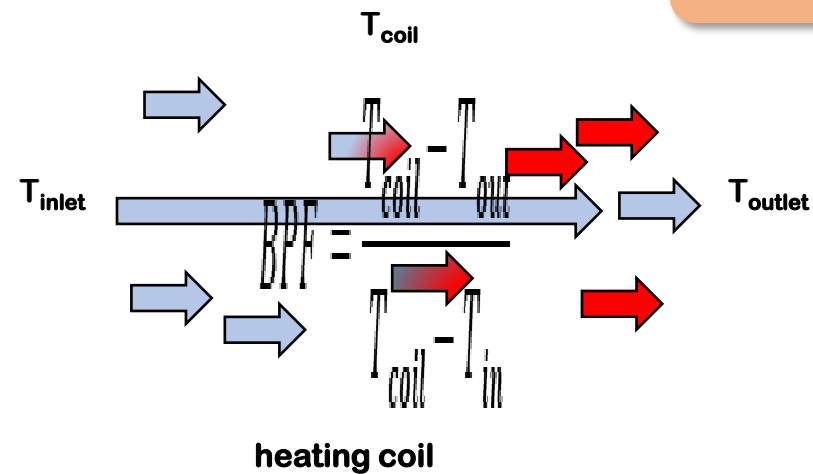
contact factor

$$T_{in} > T_{out} > T_{coil}$$



$$BPF = \frac{T_{out} - T_{coil}}{T_{in} - T_{coil}}$$

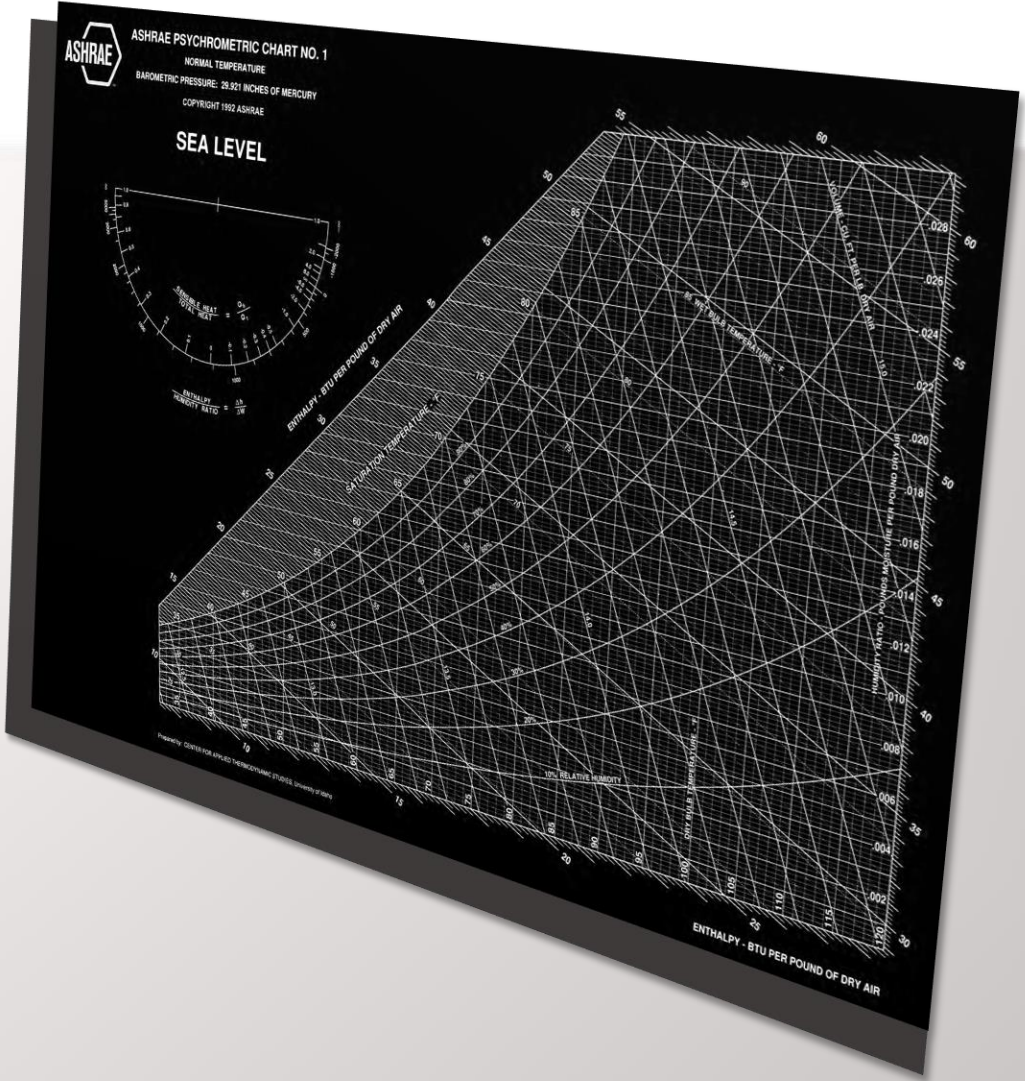
$$T_{coil} > T_{out} > T_{in}$$



$$BPF = \frac{T_{coil} - T_{out}}{T_{coil} - T_{in}}$$

Contact Factor= 1- bypass Factor

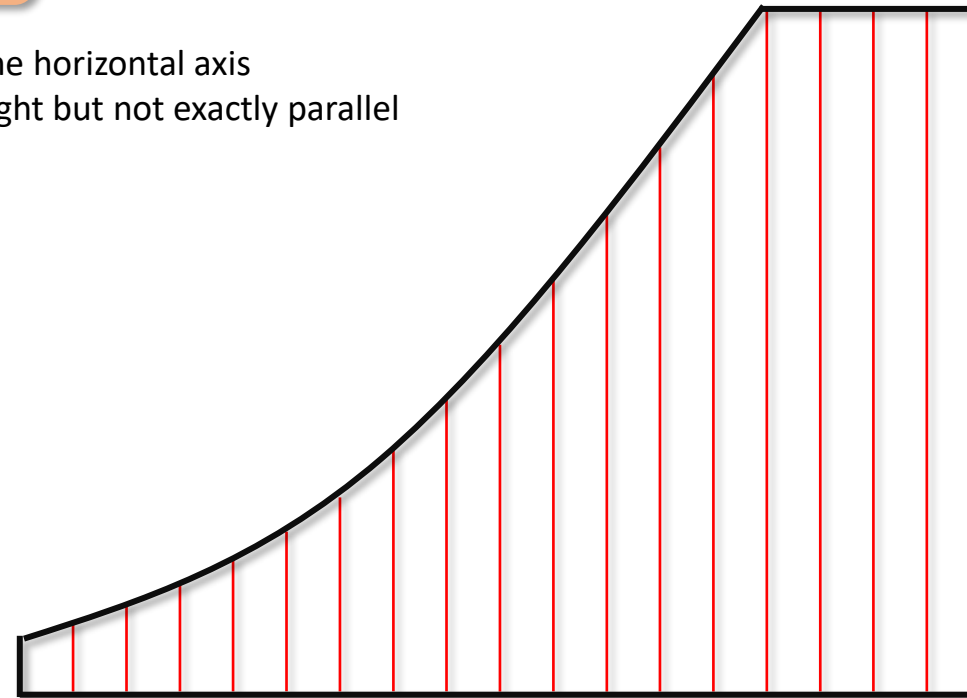
PSYCHROMETRIC CHART



psychrometric chart

Dry Bulb Temperature (DBT)

- Dry bulb temperature is plotted along the horizontal axis
- The dry bulb temperature lines are straight but not exactly parallel and incline slightly to the left

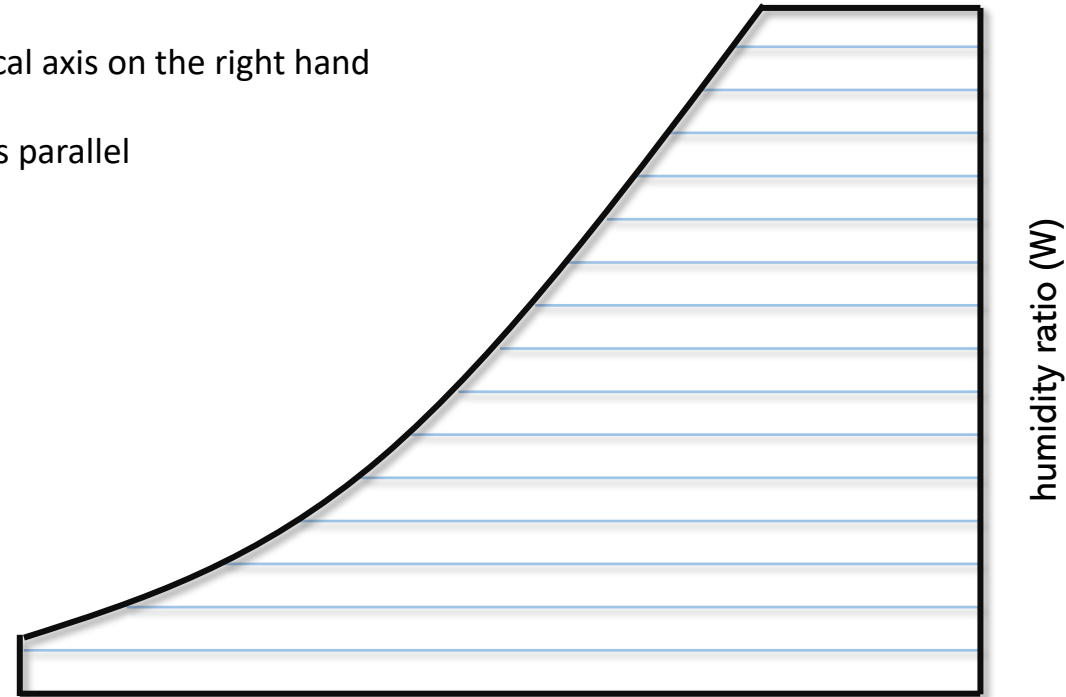


dry bulb temperature

psychrometric chart

humidity ratio (W)

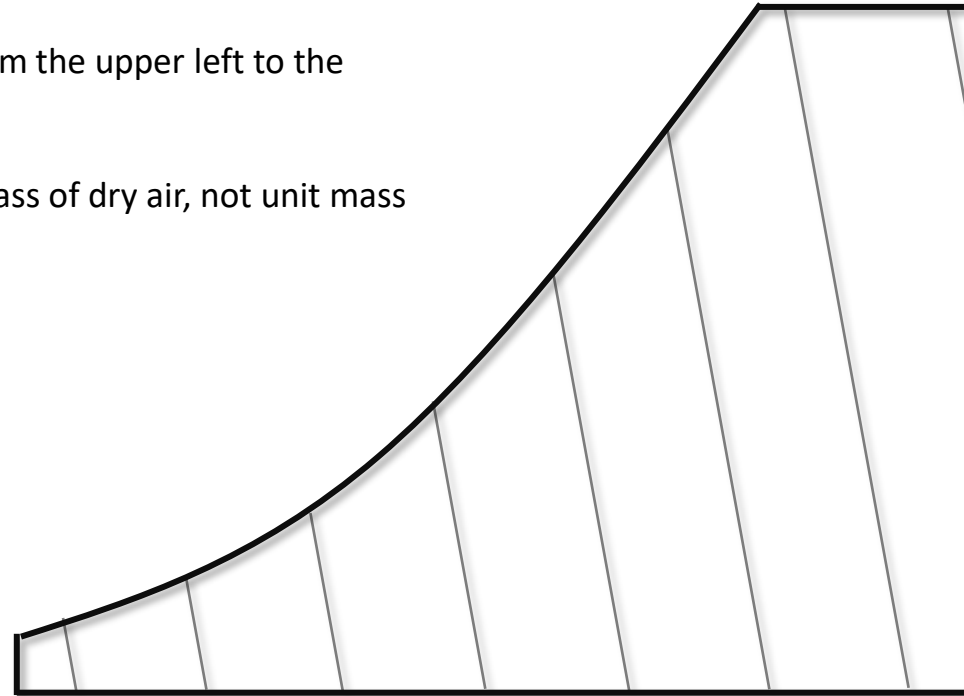
- Humidity ratio is plotted along the vertical axis on the right hand side of the chart
- The scale is uniform with horizontal lines parallel



psychrometric chart

specific volume (v)

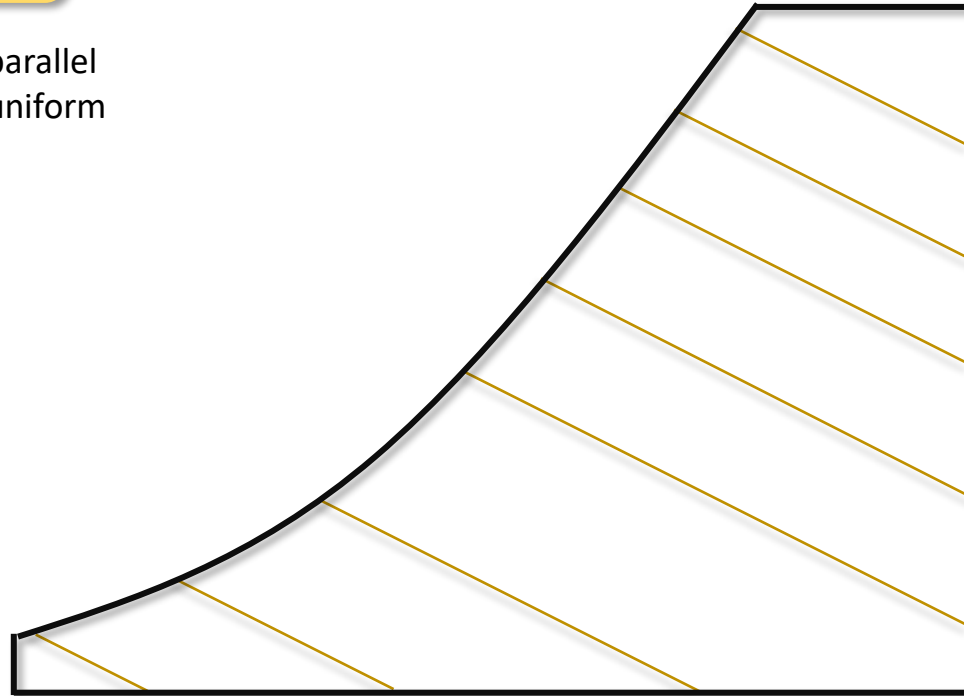
- specific volume lines appear inclined from the upper left to the lower right
- these lines are not parallel
- specific volume scale is based on unit mass of dry air, not unit mass of moist air



psychrometric chart

Wet Bulb Temperature (WBT)

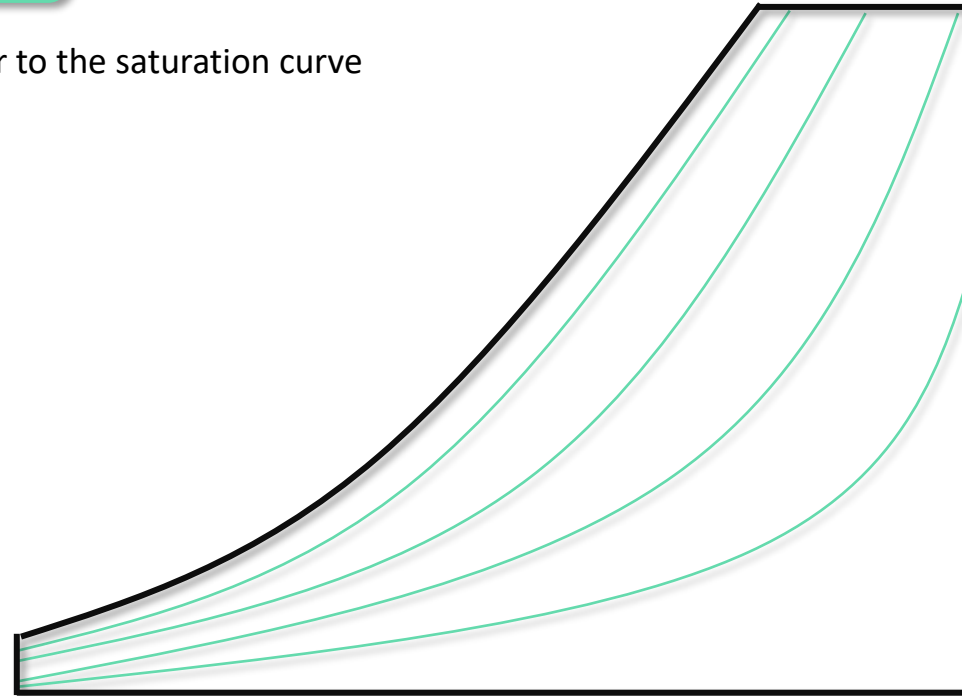
- the wet bulb temperature lines are not parallel
- The spacing of the wet bulb lines is not uniform



psychrometric chart

Relative Humidity (ϕ)

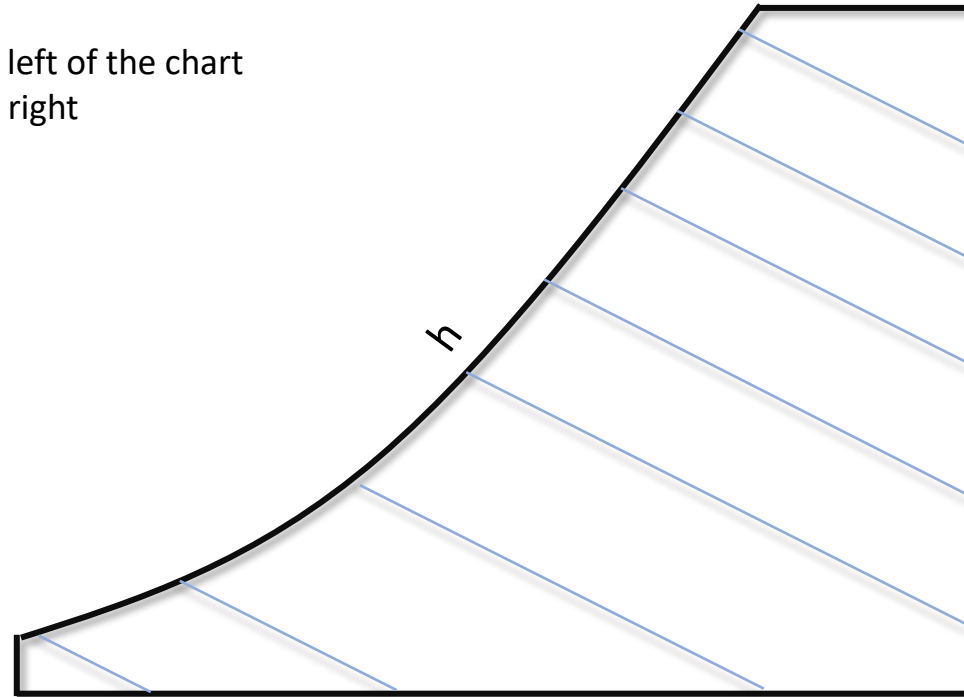
- relative humidity lines are shapes similar to the saturation curve
- they appear at regular intervals



psychrometric chart

Enthalpy (h)

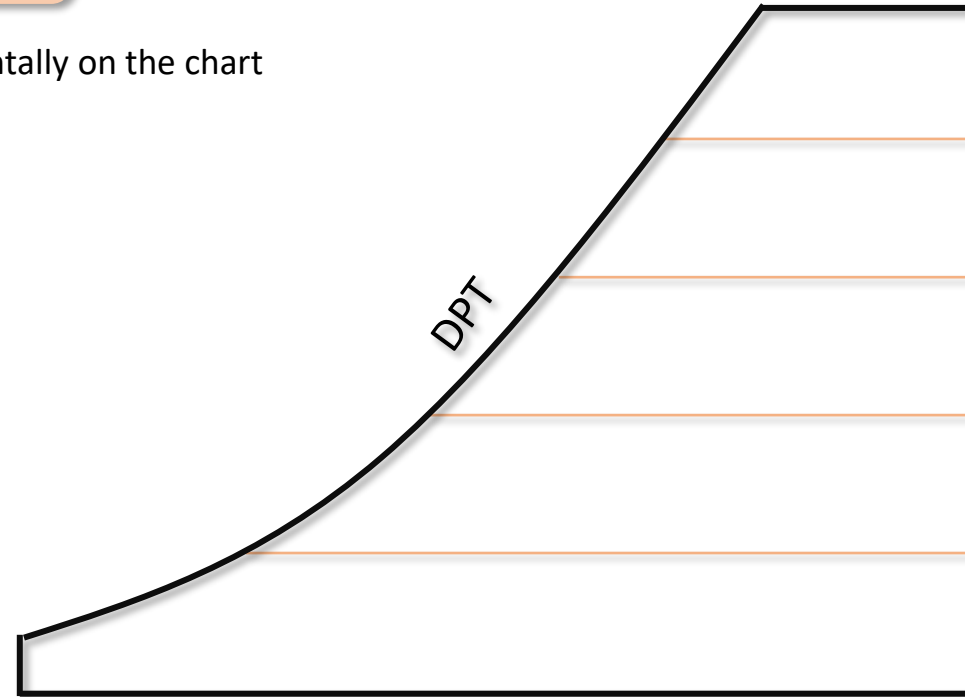
- enthalpy scale is drawn obliquely on the left of the chart
- enthalpy lines inclined downward left to right



psychrometric chart








Dew Point Temperature (DPT)

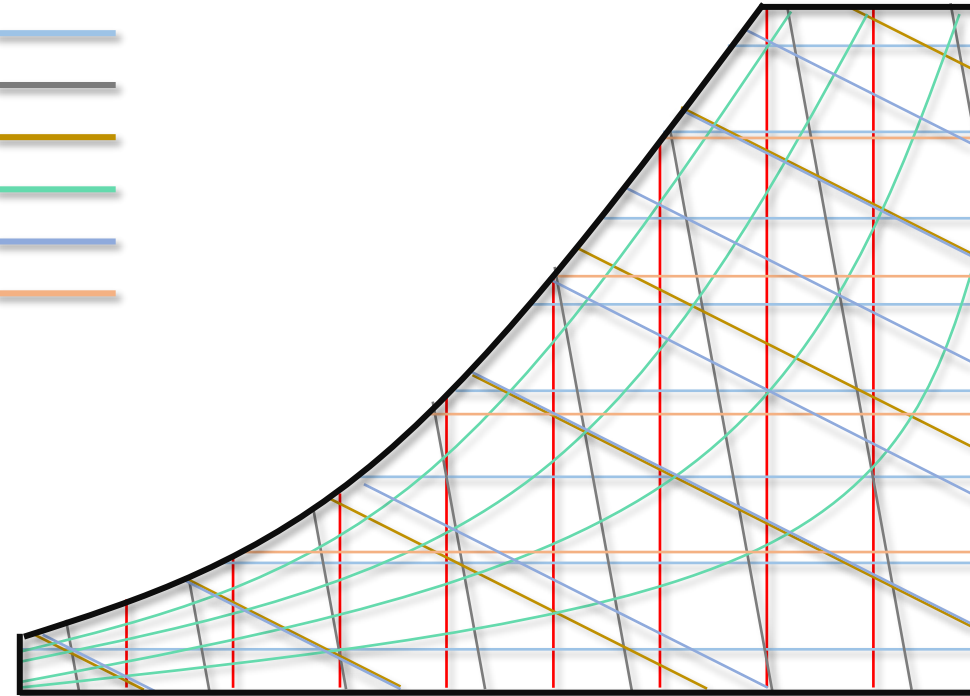
- dew point temperature is drawn horizontally on the chart



psychrometric chart



- ✓ dry bulb temperature 
- ✓ humidity ratio 
- ✓ specific volume 
- ✓ wet bulb temperature 
- ✓ relative humidity 
- ✓ enthalpy 
- ✓ dew point temperature 

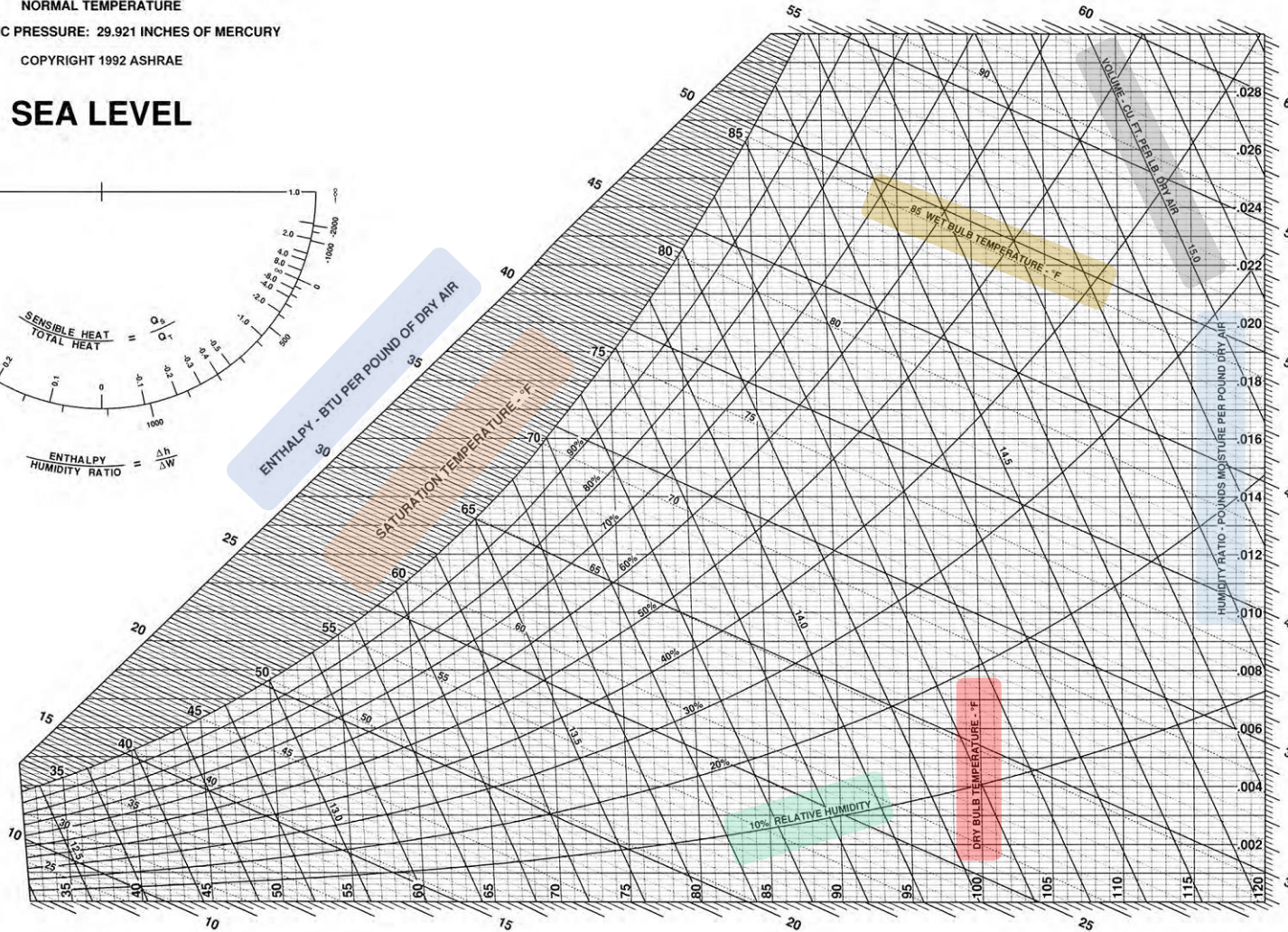
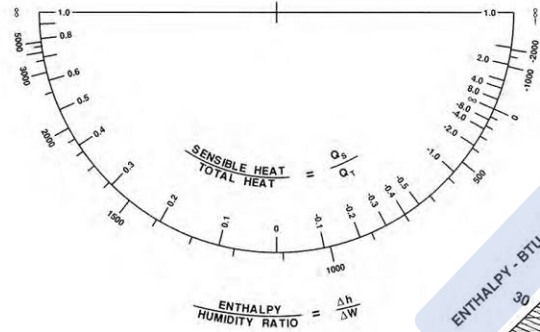


psychrometric chart



ASHRAE PSYCHROMETRIC CHART NO. 1
NORMAL TEMPERATURE
BAROMETRIC PRESSURE: 29.921 INCHES OF MERCURY
COPYRIGHT 1992 ASHRAE

SEA LEVEL



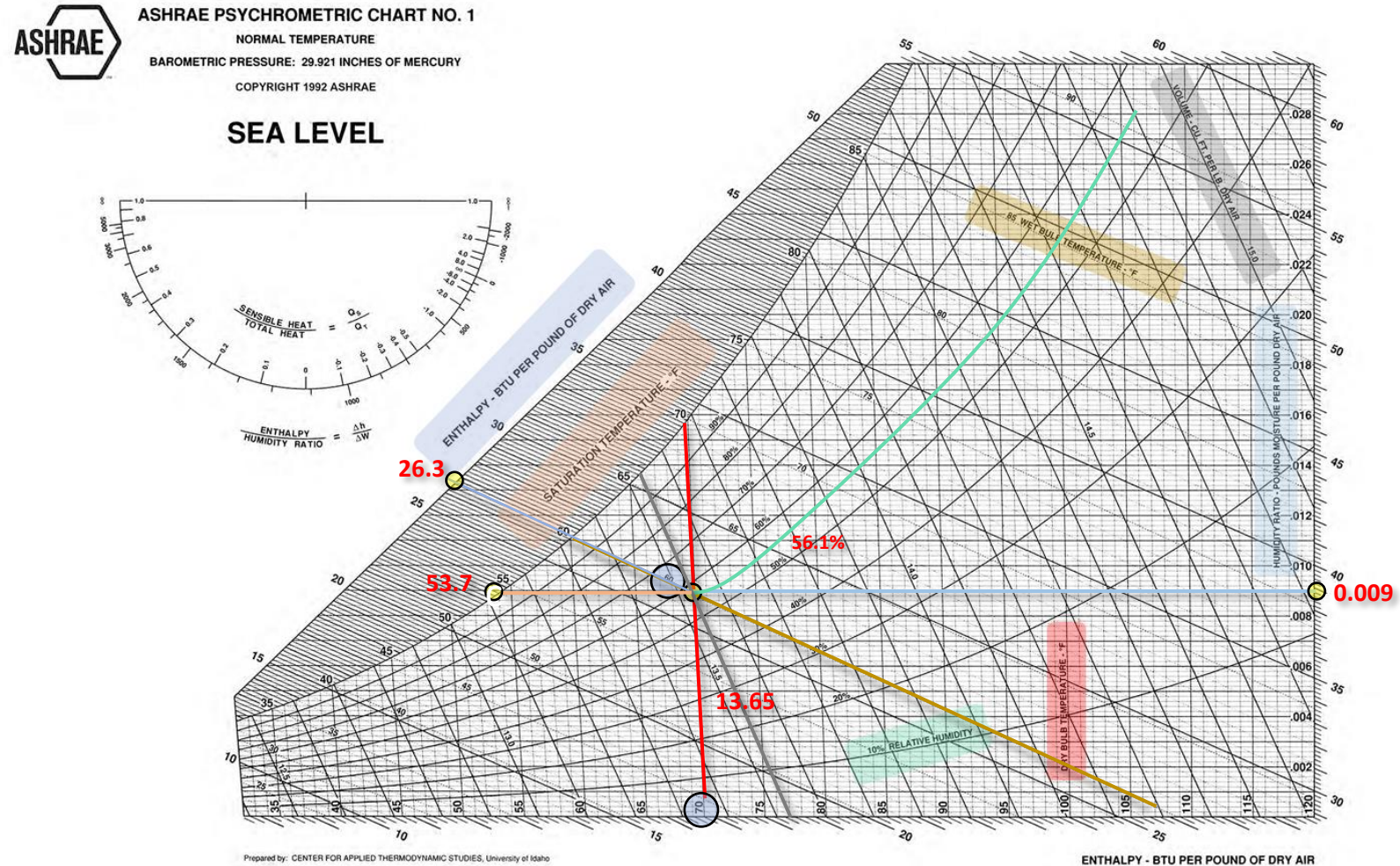
Prepared by: CENTER FOR APPLIED THERMODYNAMIC STUDIES, University of Idaho

ENTHALPY - BTU PER POUND OF DRY AIR

psychrometric chart

The air leaves a cooling coil is at 70°F T_d and 60°F T_w

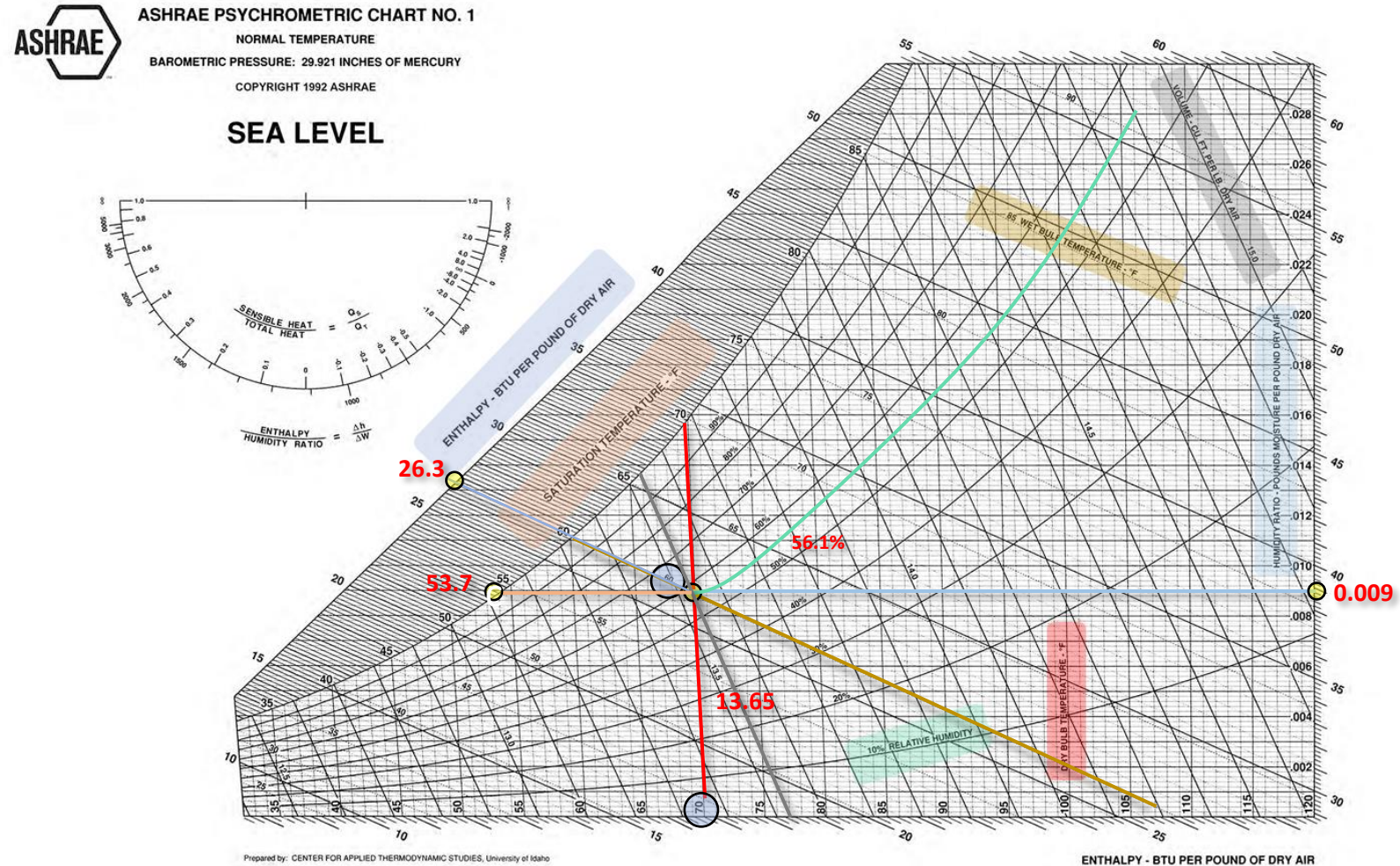
- dry bulb temperature (T_d)=70°F
- wet bulb temperature (T_w)=60°F
- humidity ratio
 $W=0.009$
- relative humidity
 $\phi=56.1\%$
- specific enthalpy
 $h=26.3$ Btu/lb
- specific volume
 $v=13.65$ ft³/lb
- dew point temperature
 $T_{dp}=53.7^\circ\text{F}$



psychrometric chart

The air leaves a cooling coil is at 70°F T_d and 60°F T_w

- dry bulb temperature (T_d)=70°F
- wet bulb temperature (T_w)=60°F
- humidity ratio
 $W=0.009$
- relative humidity
 $\phi=56.1\%$
- specific enthalpy
 $h=26.3$ Btu/lb
- specific volume
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 $T_{dp}=53.7$ °F





ASHRAE PSYCHROMETRIC CHART NO. 1

NORMAL TEMPERATURE

BAROMETRIC PRESSURE: 29.921 INCHES OF MERCURY

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SEA LEVEL

