



47<sup>TH</sup> TURBOMACHINERY & 34<sup>TH</sup> PUMP SYMPOSIA  
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GEORGE R. BROWN CONVENTION CENTER

# *A Case Study of Vibration in Positive Displacement Pump Systems*

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TEXAS A&M ENGINEERING EXPERIMENT STATION

# Experience

## Trenton Cook

- *FEA of machinery piping systems*
- *Field pulsation and vibration assessment of piping systems*
- *Modal and forced response analysis of mechanical systems*

## Sarah Simons

- *Almost 10 years of experience in flow and acoustic analysis of pump systems*
- *Leads research on fluid property testing, and flow and pulsations in combined machinery types*



# Abstract

*Positive displacement pump systems can experience high piping vibrations. System vibration can have many root causes- including underdamped fluid pulsations, mechanical resonance, and poor skid design. This case study shows the mechanical and acoustic assessment of a reciprocating pumping system which had multiple vibration induced failures. Poor support stiffness, coupled with inadequate pulsation dampener performance resulted in high amplitude piping vibration- requiring both mechanical and acoustic analyses. Collected field pulsation and vibration data are presented, along with follow-up acoustic and finite element modeling results to showcase a solution to pulsation induced vibration in this particular pump system.*



# Problem

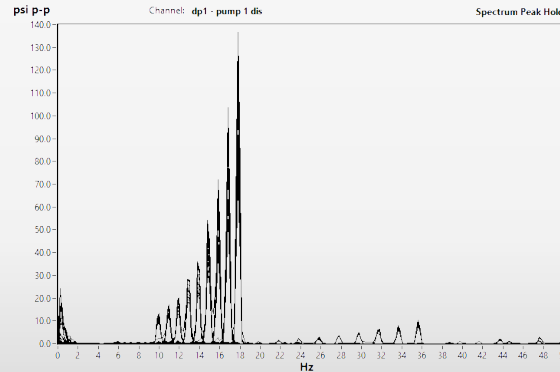
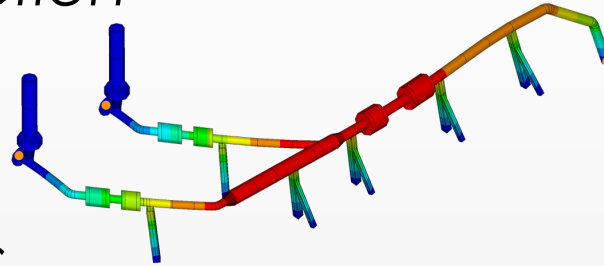
- *Reciprocating pump installation for salt water disposal*
- *Multiple reported piping failures*
- *Failure of pulsation dampener bladders*
- *Reported piping/skid vibration*



*Location of Previous Failure*

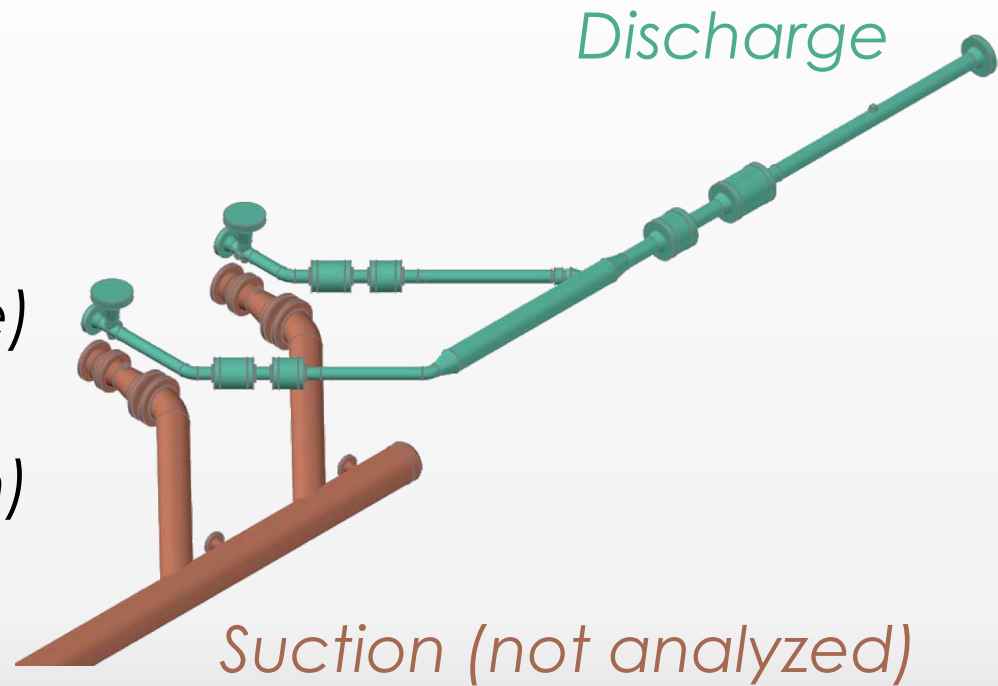
# Solution Overview

- *Field Data Collection*
- *Modal Analysis*
- *FEA Analysis*
- *Pulsation Analysis*
- *Design Changes:*
  - *Mechanical*
  - *Acoustic*



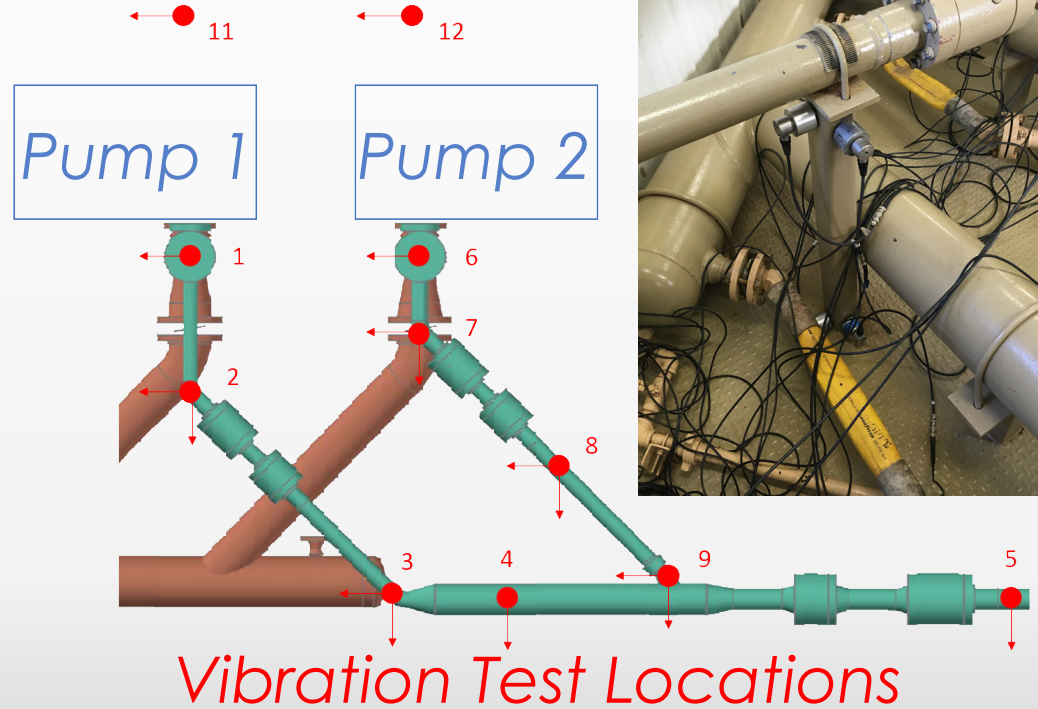
# System Overview

- (2) Triplex pumps, skid mounted on soil pad
- 200-400 RPM
- 800-1400 psi (discharge)
- Pulsation dampeners (discharge and suction)



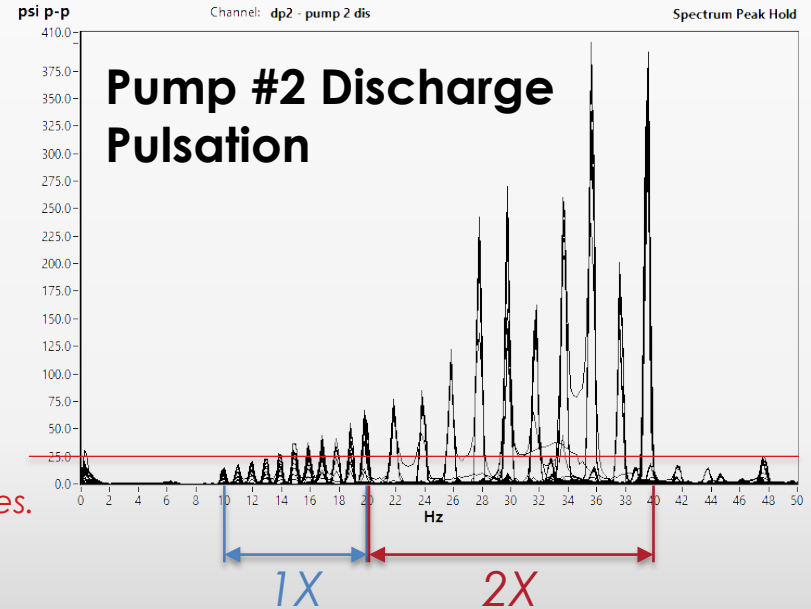
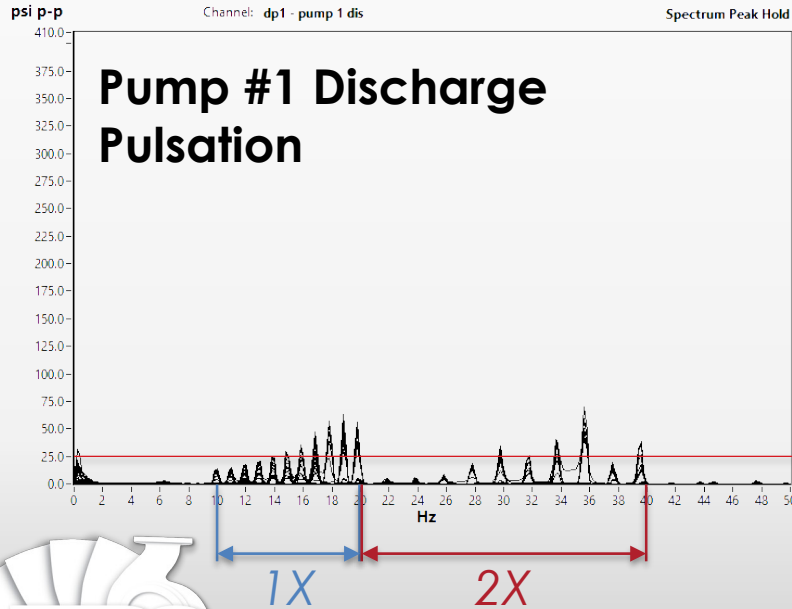
# Field Data Collection

- Independent and parallel operation of Pumps #1 and #2
- Speed sweep (200-400 RPM)
- Multiple pulsation dampeners tested
- Accelerometers
- Pulsation transducers



# Test Case - 1

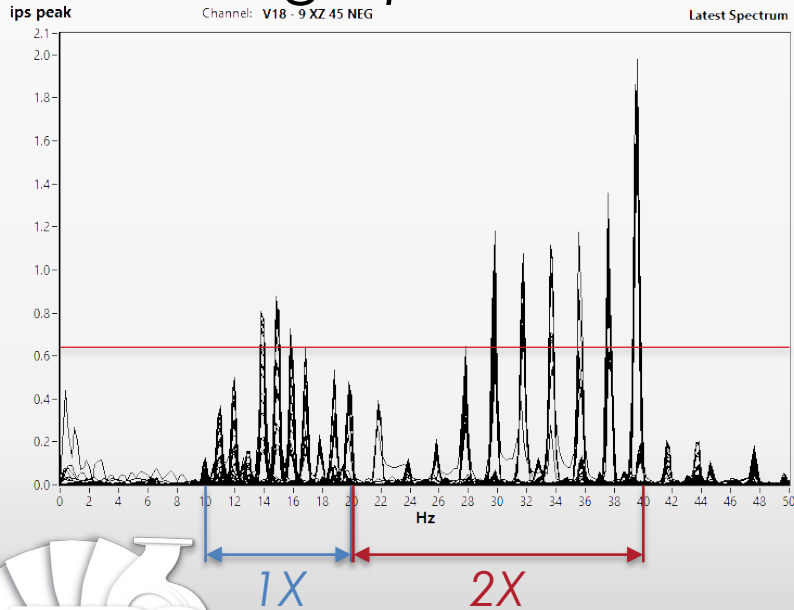
- Parallel operation and speed sweep (200-400 RPM) of Pumps #1 and #2- in original configuration





# Test Case - 1

- *Vibration at wye (location of previous failures)*
- *Vibration frequencies consistent with Pump#2 discharge pulsations*

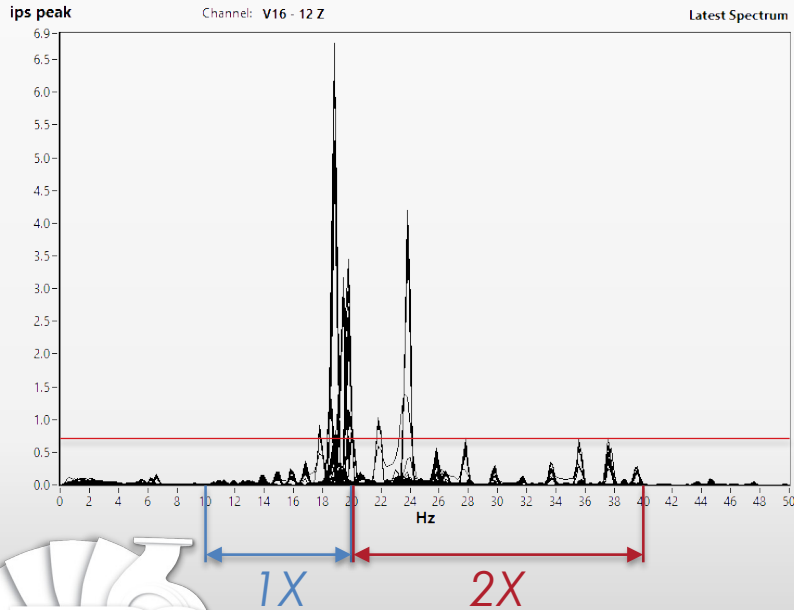


*Previous Failure*



# Test Case - 1

- *Vibration of Pump #2 pulsation dampener (top)*
- *Mechanical resonance near 19 Hz*



*Previous Failure*



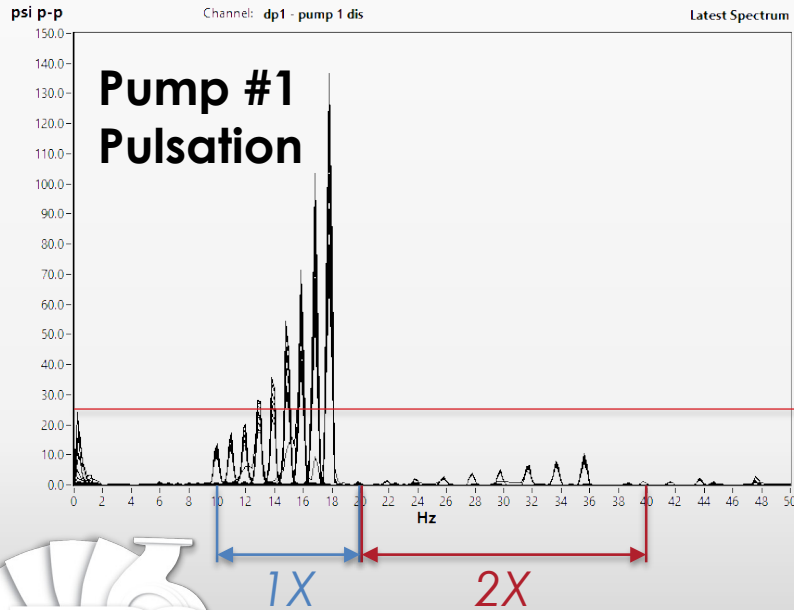
# Test Case - 1

- *Pump #2 pulsation dampener had failed- would not hold charge*
- *High amplitude pulsations (undamped) caused high amplitude piping vibration across operating range*
- *Multiple mechanical piping/support modes excited by high amplitude pulsations*
- *Next Test-- replacing Pump #2 pulsation dampener with alternative model- at site from previous use*

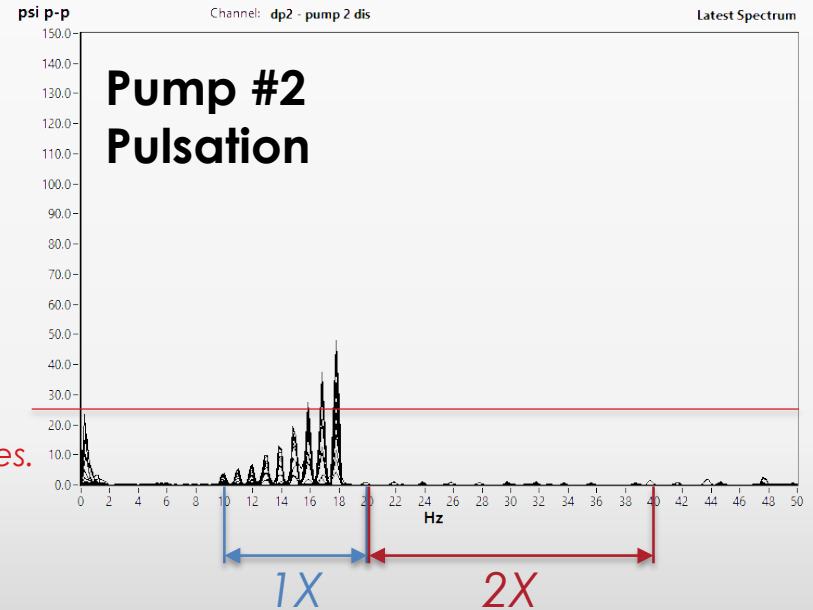


# Test Case - 6

- Parallel operation and speed sweep (200-400 RPM) of Pumps #1 and #2- alternative dampener on Pump #2



2%  
Line Pres.



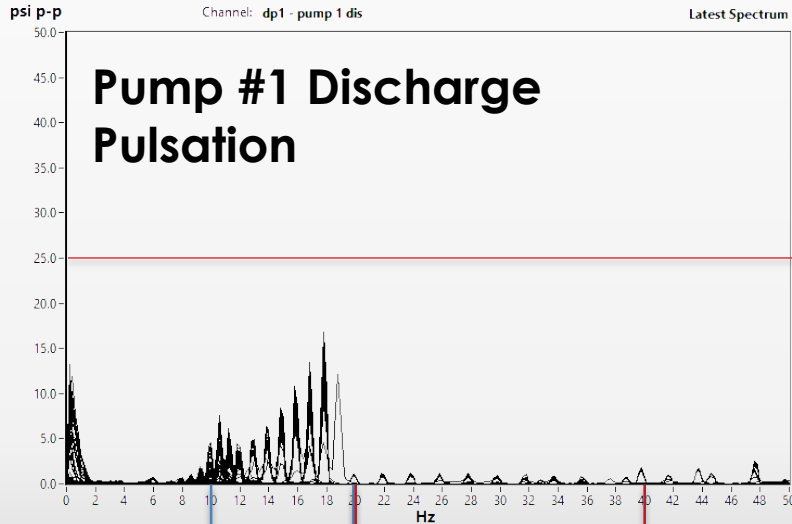
# Test Case - 6

- *Replacing the Pump #2 failed pulsation dampener with alternative dampener significantly reduced pulsation amplitudes in system*
- *Pump #1 still had high amplitude pulsations and vibration still persisted, despite general reduction in amplitudes*
- *Next Test--replacing Pump #1 pulsation dampener with alternative model- to match Pump #2*

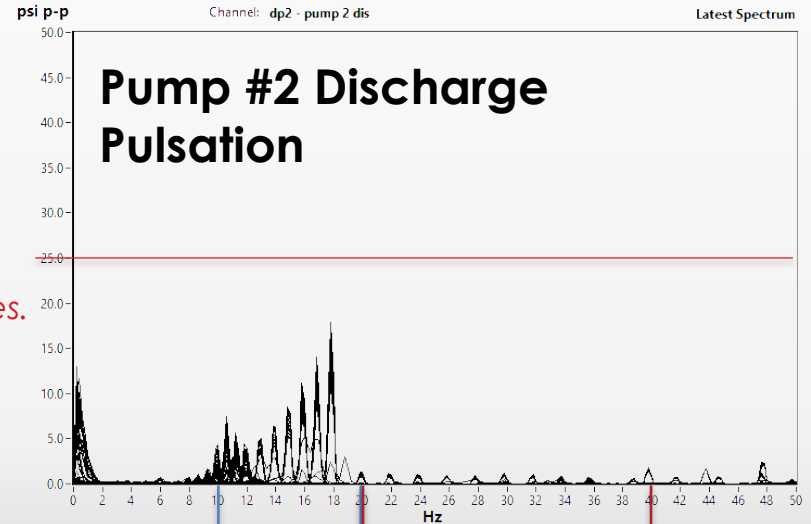


# Test Case - 7

- Parallel operation and speed sweep (200-400 RPM) of Pumps #1 and #2- with alternative dampeners

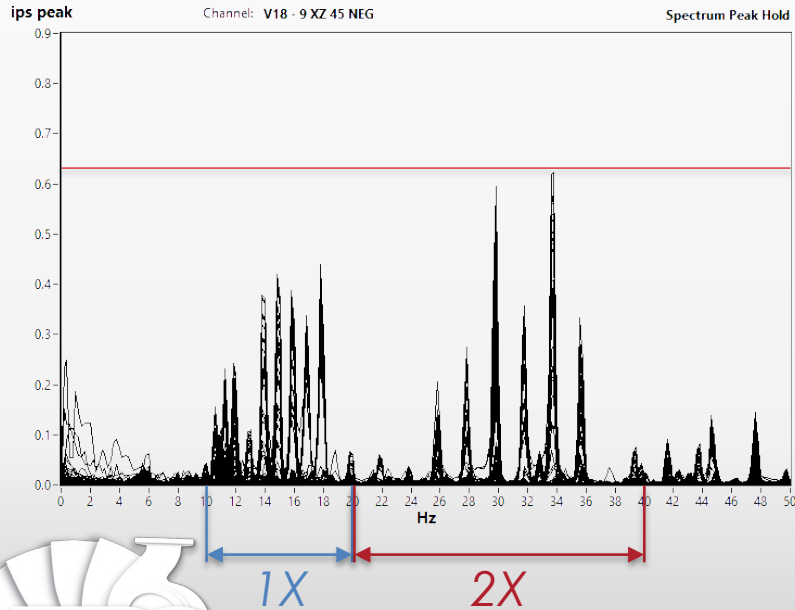


2%  
Line Pres.



# Test Case - 7

- *Vibration at wye significantly reduced*
- *Potential mechanical resonances still at 2X*

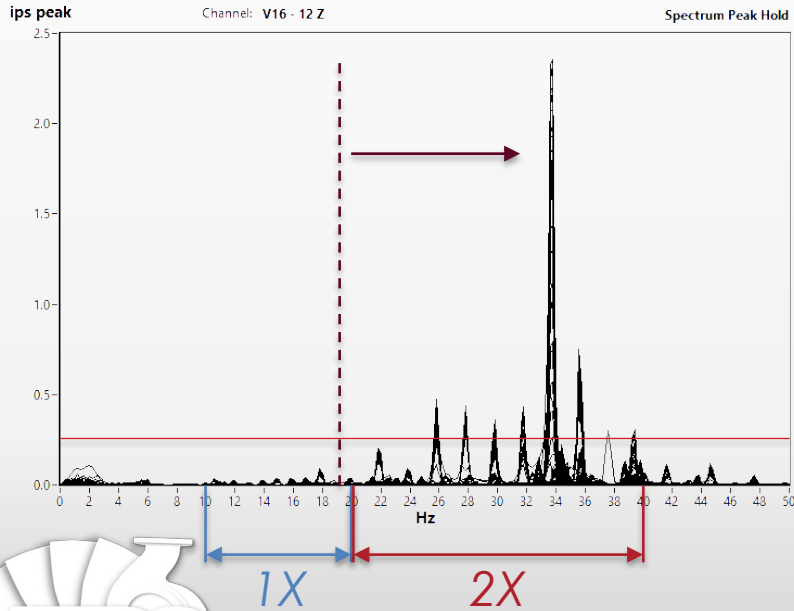


*Previous Failure*



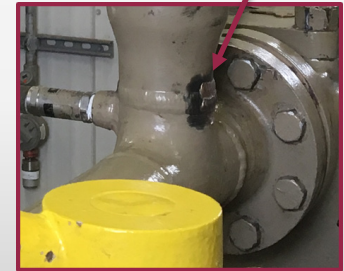
# Test Case - 7

- *Vibration of Pump #2 pulsation dampener (top)*
- *Mechanical resonance near 34 Hz (shifted)*



Alt Dampener

Previous Failure





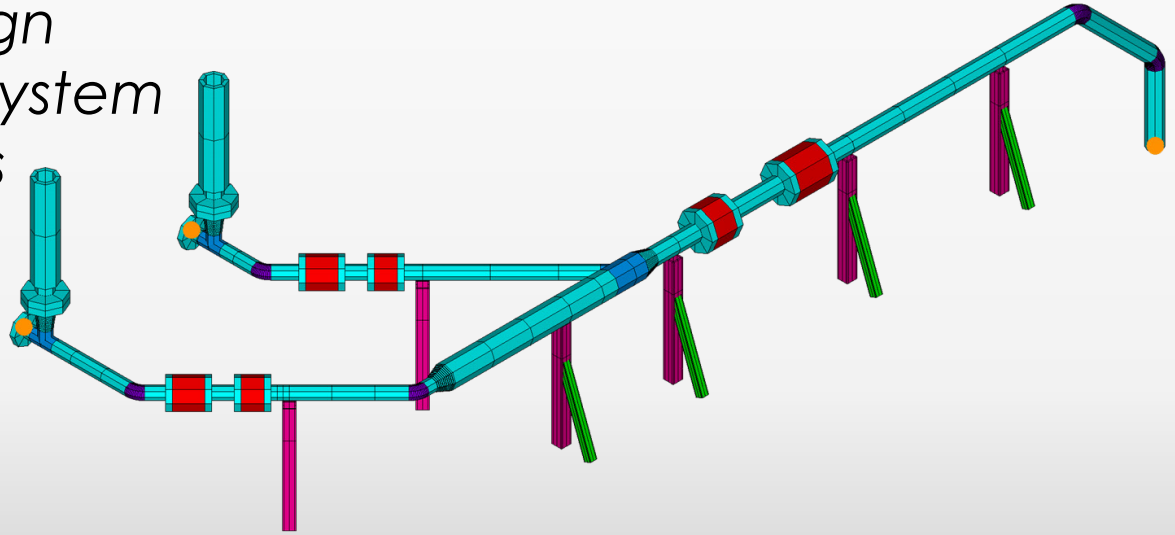
# Test Case - 7

- *Replacing both Pump #1 and #2 dampeners with alternative design reduced the system pulsation amplitudes within acceptable limits*
- *System vibration amplitudes significantly reduced with alternative dampeners*
- *Some high amplitude vibration still persistent- associated with mechanical resonances*
- *Pulsation dampeners recommended to be re-sized*
- *Additional modal analysis and bracing recommended*



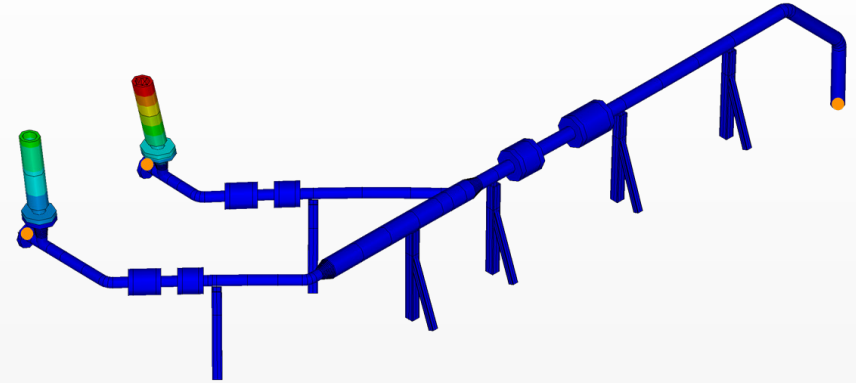
# Modal Analysis

- *Finite Element model built to perform modal analysis of piping system*
- *Results used to design additional support system to place key modes beyond range of operating speed*

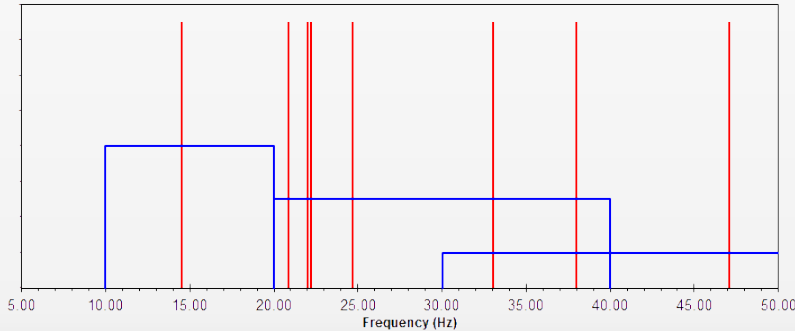


# Modal Analysis

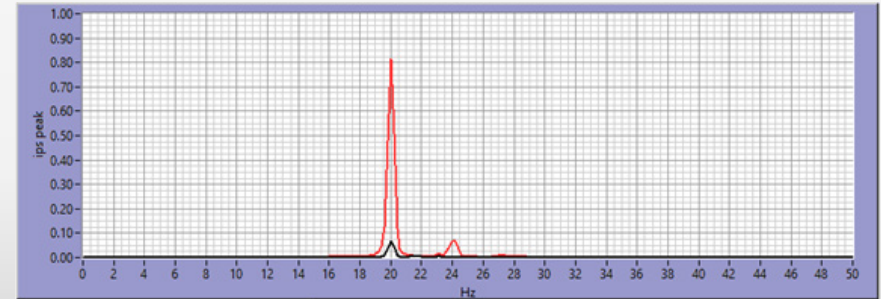
- Impact test data collected in the field
- Modal analysis evaluated with FE model



*FEA Modal Results ~22 Hz*



*Modal Interference Plot*

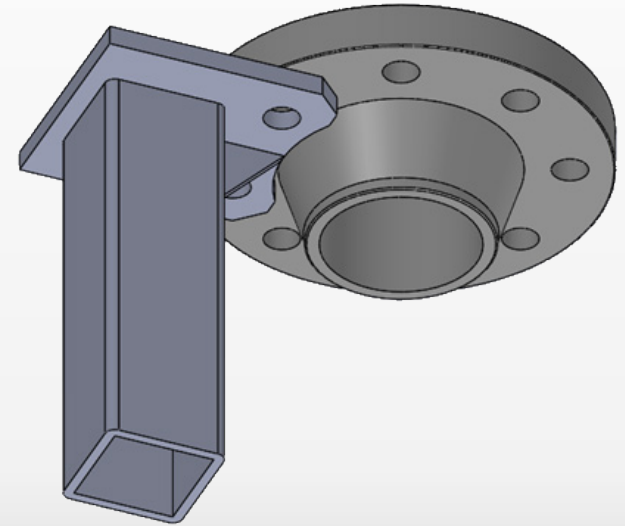
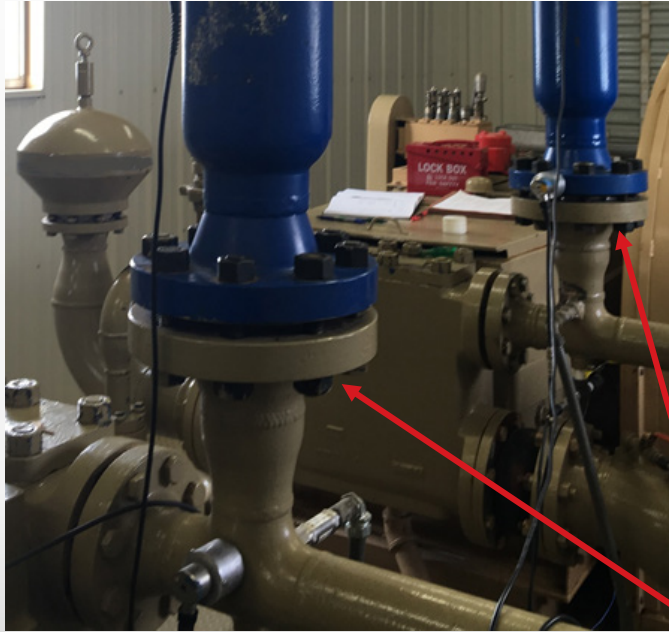


*Field Bump Data ~20 Hz*



# Mechanical Recommendations

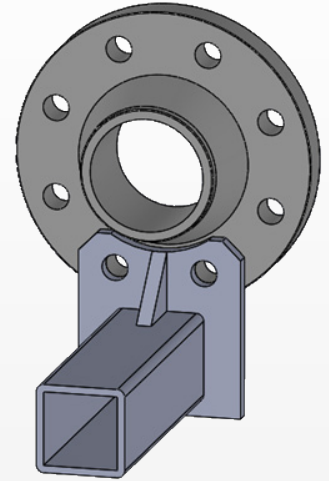
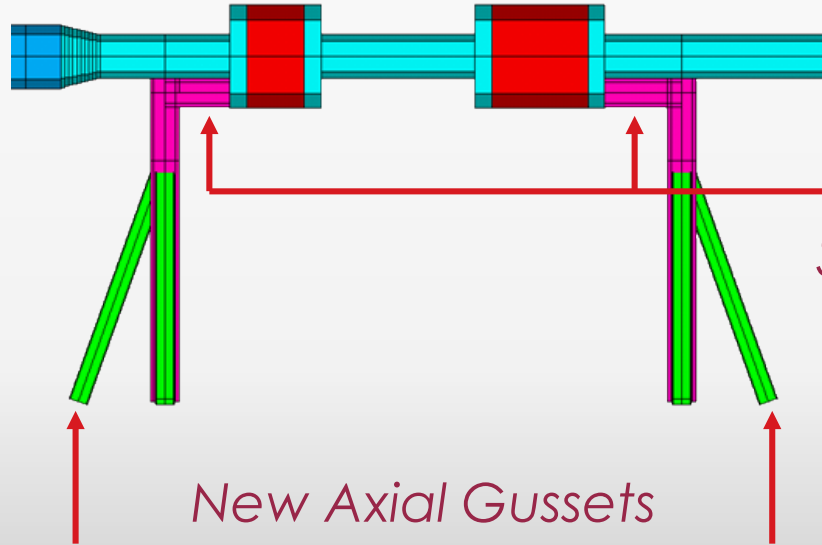
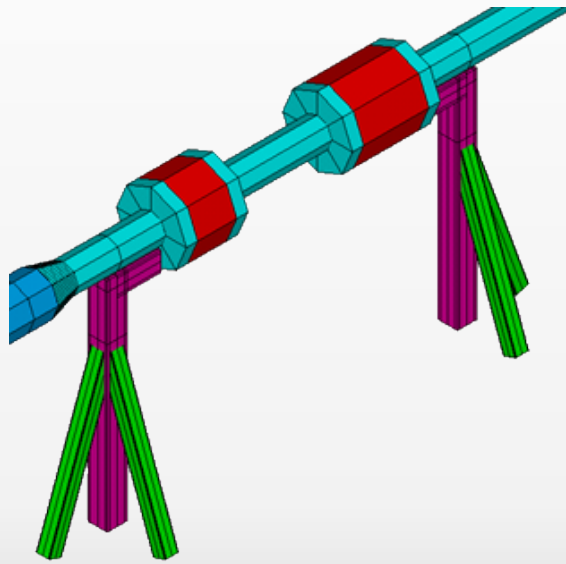
- *Additional bracing of pulsation dampener flange*



*New Flange Support*

# Mechanical Recommendations

- Additional bracing of discharge header

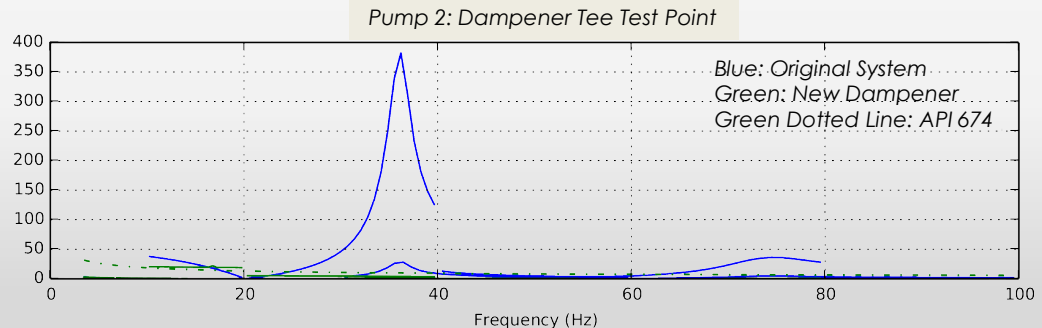


Secure Piping to Restraints

New Axial Gussets

# Pulsation Analysis

- Alternate dampeners in the field not rated for long-term operation at the higher end of discharge pressure
- Pulsation analysis performed to size new dampeners designed for this application
- Initial model results matched well with field data and the model with the new dampener showed sufficient attenuation of the pump excitations, similar to Field Test Case 7 results.

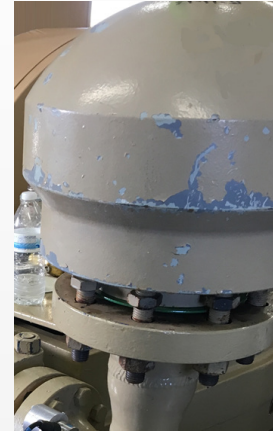


# Dampener Evaluation

- Dampeners need to be charged to 70-90% line pressure
- Avoid tall and heavy dampeners that have low MNFs
- Use bladder material suited for the application



Original



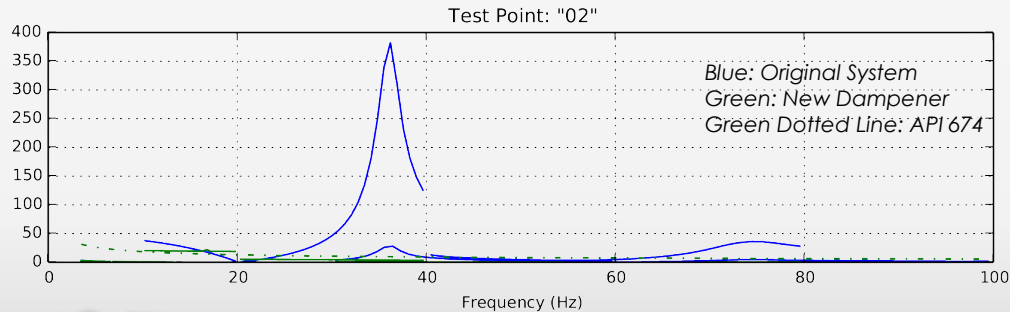
Tested



Final  
Installed

# Solution Results

- System operating without failures or excessive piping/support vibration
- Pulsation dampeners performing without failure





*Questions?*

